## TOCHNOG PROFESSIONAL User's manual

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# 10 User supplied subroutines

# **1** Basic information

# 1.1 How to perform a calculation and how to get started

Create an input file, e.g. **problem.dat**. The default input file is **tochnog.dat**, which will be used if no other input file is specified. Thus the command **tochnog tochnog.dat** yields output on the screen while **tochnog tochnog.dat** > **tochnog.out** redirects the output to a file.

So to get started do, for example, the following:

- cd test/other
- tochnog condif1.dat

Use the **condif1.dat** test to get started.

- Copy condif1.dat to tochnog.dat.
- Use your favorite editor to open the file **tochnog.dat** and study it.
- Change echo to -yes.
- Remove the parentheses (...) surrounding the **control\_print** statement and save the file.
- Run by typing tochnog or tochnog tochnog or tochnog.dat.
- Study the output on the screen.
- Study the **tochnog.log** file.
- Study the **tochnog.dbs** file. It contains the database after the calculation, and is an input file itself!

Read at least once the start of the data part introduction section.

# 1.2 Pre- and postprocessing

You can use **GID** both for preprocessing (mesh generation) and post processing (plotting). **GID** is commercially available at the **www.gidhome.com** Internet page. A free demo version of is available for download.

Alternatively to **GID** you can use **Mecway** for preprocessing **input\_abaqus** and post processing **control print gmsh**. **Mecway** is commercially available at the **mecway.com** 

Internet page. It is very affordable, and also has build in FE calculations (mostly for mechanical engineering). A free demo version of Mecway is available for download.

You can also use **GMSH** both for preprocessing and post processing. **GMSH** is freely available at **www.geuz.org/gmsh**.

Postprocessing files are written for the visualization program **PARAVIEW**. The **PAR-AVIEW** program is freely available at **www.paraview.org**.

Furthermore. postprocessing files are written for the visualization program **TECPLOT**. These **TECPLOT** are less well maintained then the files for other postprocessing programs.

With **GNUPLOT** you can plot files resulting from **control\_print\_history** and **control\_print\_data\_versus\_data**. Also any other x-y plotting program can be used for such files.

# 1.3 Space discretization, time discretization

The computational domain is divided into finite elements. The elements connect at nodes. Either one-dimensional (1D), two-dimensional (2D), three-dimensional (3D) or axi-symmetrical (2D) domains can be used.

Only first order in time equations are solved. Time derivatives are approximated with Euler backward time discretization.

## 1.4 Program capabilities

• Input

Format free input. Words and no 'magic numbers' in rigidly defined columns are used. Boundary conditions can be imposed onto at geometrical entities, as well as onto elements and nodes.

• Output/plotting

Output can be printed over user-specified geometrical objects (points, lines, quadrilaterals,...) as well as at nodes.

The history of each variable, and for functions of variables, can be printed over userspecified geometrical objects as well as at nodes.

Interface files for the GID pre- and post processor.

• Finite elements

1D, 2D and 3D. Tochnog mostly uses isoparametric elements. There are also springs, trusses, beams and contact-springs however.

Linear and quadratic simplex elements (triangles, tetrahedrons). Linear and quadratic prism elements. A full family of first to fourth order bar, quadrilateral and brick elements.

 $\bullet \ {\rm Mesh \ generation/refining/etc.}$ 

Macro regions are automatically divided into finite elements.

Local h-refinement

Global h-refinement (more elements).

Global p-refinement (polynomial refinement).

## • Differential equations (materials)

Convection-diffusion equation:

- Temperature calculations.

Fluids:

- Stokes and Navier-Stokes.

Solids:

- Elasticity (isotropy and transverse isotropy).

- Elasto-Plasticity (Von-Mises, Mohr-Coulomb, Gurson, etc.; plasticity surfaces can be arbitrarily combined).

- Hypo-Plasticity (Von-Wolffersdorff, Masin, cohesion, intergranular strains, pressure dependent initial void ratio).

- Damage.
- Thermal stresses.
- Hypoelasticity.
- Viscoelasticity.
- Viscoplasticity.
- Viscosity.

Ground water flow equation:

- Storage equation. - saturated and non-saturated - multiple phreatic levels - piping and lifting safety - fully coupled consolidation analysis

Wave equation.

• Interaction analysis

Automatic fluid-solid interaction.

Temperature effects on fluids, solids.

- Contact analysis Contact with and without friction. Frictional heat generation.
- Frames of description Lagrangian and Eulerian
- Types of analysis Static, quasi-static and dynamic analysis.

• Parallelization

Full shared memory parallelization node and element loops

• Special features

Automatic time-stepping (large steps for good iteration behavior, small steps for bad iteration behavior).

Automatic distribution of tendon trusses over finite elements (automatic embedment). Restart possibility.

Convection wiggle stabilization (both for low and high order elements).

# 1.5 Files used by Tochnog

- Input file. For example **condif1.dat**. The input file consists of an initialization part (which dof's should be solved, etc.) and a data part (elements, nodes, etc.).
- Runtime input file. For example **condif1.run**. Use it to give Tochnog data records on the fly (while it is running).
- Plot files. For example condif1 flavia.msh and condif1 flavia.res.
- Database file. For example, after the calculation with input file **condif1.dat** the database file **condif1.dbs** will be written. It contains everything (nodes, elements, solutions fields, etc.). On error exit for example **condif1\_error.dbs** will be generated.
- Scratch file tochnog tmp.txt. Don't use this name yourself.
- Log file tochnog.log. Contains log messages of calculations.

# 2 Equations

### 2.1 Convection and diffusion of heat

### 2.1.1 Convection-diffusion equation

$$\rho C(\dot{T} + \beta_i \frac{\partial T}{\partial x_i}) = k_i \frac{\partial^2 T}{\partial x_i^2} - aT + f$$

The primary dof is the condif\_temperature T. Further notation:  $\rho$  group\_condif\_density; C group\_condif\_capacity; x space coordinate;  $\beta_i$  group\_condif\_flow in *i*-direction;  $k_i$  group\_condif\_conductivity in *i*-direction; a group\_condif\_absorption; f condif\_heat\_volume. Typical applications are heat conduction and heat conduction in a flow.

### 2.1.2 Convection to environment

$$q_c = \alpha_c (T - T_c)$$

Here  $q_c$  is the **condif\_convection\_edge\_normal** heat flux,  $\alpha_c$  is the convection coefficient and  $T_c$  is the environmental temperature for convection.

### 2.1.3 Radiation to environment

$$q_r = \alpha_r (T^4 - T_r^4)$$

Here  $q_r$  is the **condif\_radiation\_edge\_normal** heat flux,  $\alpha_r$  is the radiation coefficient and  $T_r$  is the environmental temperature for radiation.

# 2.2 Material deformation and flow

$$\rho \dot{v_i} = \frac{\partial \sigma_{ij}}{\partial x_j} + (1 - \beta T)\rho g_i - dv_i + f_i$$

Notations:  $\rho$  group \_\_materi \_\_density;  $v_i$  materi \_\_velocity in *i*-direction;  $\sigma_{ij}$  materi \_\_stress matrix; x space coordinate;  $\beta$  group \_\_materi \_\_expansion \_\_volume; T (optional) condif \_\_temperature;  $g_i$  force \_\_gravity; d is the group \_\_materi \_\_damping coefficient (see also the dynamics section near the end of this manual);  $f_i$  force \_\_volume. The equation is given for space coordinates following the material velocities  $v_i$ .

TOCHNOG allows you to build your favorite material, by adding separate contributions to the stresses  $\sigma_{ij}$ . In this way you can build solids or fluids or things in between. The separate contributions will be listed below. First two typical examples are given.

Nearly incompressible Navier Stokes:

```
materi_velocity
materi_stress
end_initia
...
mesh -fixed_in_space -fixed_in_space
timestep_predict_velocity 0 -yes
...
group_type 0 -materi
group_materi_elasti_compressibility 0 1.0
group_materi_viscosity 0 1.2
...
```

Linear solid:

```
materi_velocity
materi_velocity_integrated
materi_stress
end_initia
...
group_type 0 -materi
group_materi_elasti_young 0 1.e10
group_materi_elasti_poisson 0 0.2
group_materi_memory 0 -updated_linear
```

### 2.2.1 Memory

#### The -updated Lagrange formulation

Deformations (i.e. the incremental deformation matrix F) refers to the previous time point. TOCHNOG decomposes the incremental deformation tensor with a polar decomposition into F = RU with F the incremental deformation matrix, R the incremental rotation matrix and U the incremental stretch matrix. The incremental stretch matrix U is used to determine the incremental strain matrix  $0.5(U + U^T) - I$  with I the identity tensor. The stresses at a new timepoint are calculated as:

- calculate extra stresses due to incremental strain matrix
- add these extra stresses to the stresses of the previous time point
- rotate these new stresses by the matrix R

### The - updated jaumann Lagrange formulation

Deformations (i.e. the incremental deformation matrix F) refers to the previous time point. The incremental stretch matrix U is used to determine the incremental strain matrix  $0.5(F + F^T) - I$  with I the identity tensor. The incremental rotation matrix R is  $0.5(F - F^T) + I$ . The stresses at a new timepoint are calculated as:

- calculate extra stresses due to incremental strain matrix
- add these extra stresses to the stresses of the previous time point
- rotate these new stresses by the matrix R

#### The - updated linear Lagrange formulation

Deformations (i.e. the incremental deformation matrix F) refers to the previous time point. Any rigid body rotation between the two time points are neglected, so TOCHNOG decomposes the incremental deformation tensor with a polar decomposition into F = U with F the incremental deformation matrix, and U the incremental stretch matrix. The linear engineering strains in the deformed configuration are used as incremental strain matrix  $0.5(F + F^T) - I$ . The stresses at a new timepoint are calculated as:

- calculate extra stresses due to incremental strain matrix
- add these extra stresses to the stresses of the previous time point

#### The -total Lagrange formulation

Deformations (i.e. the total deformation matrix F) refers to the time 0. TOCHNOG decomposes the total deformation tensor with a polar decomposition into F = RU with F

the total deformation matrix, R the total rotation matrix and U the total stretch matrix. The total stretch matrix U is used to determine the total strain matrix  $0.5(U + U^T) - I$  with I the identity tensor. The stresses at a new timepoint are calculated as:

- back-rotate the old stresses at the previous time point to time 0 with the old rotation matrix
- calculate extra stresses due to incremental strain matrix
- add these extra stresses to the back-rotated old stresses of the previous time point
- rotate the added stresses with the new rotation matrix R to the new configuration

### The -total linear Lagrange formulation

Deformations (i.e. the total deformation matrix F) refers to the time 0. TOCHNOG neglects any rigid body rotations and uses linear engineering strains  $0.5(F + F^T) - I$ . The difference in these linear engineering strains between two time points are the incremental strains.

The stresses at a new timepoint are calculated as:

- calculate extra stresses due to incremental strain matrix
- add these extra stresses to the stresses of the previous time point

### See also group materi memory.

### 2.2.2 Elasticity

The elastic stress rate is

# $C_{ijkl} \dot{\epsilon_{kl}}^{elas}$

where  $C_{ijkl}$  is the elastic modulus tensor (which is a doubly symmetric tensor:  $C_{ijkl} = C_{jikl}$ ,  $C_{ijkl} = C_{ijlk}$  and  $C_{ijkl} = C_{jilk}$ ), and  $\epsilon_{kl}^{\text{elas}}$  is the elastic strain rate. See the plasticity section for a definition of the elastic strain rate.

For an isotropic material

$$C_{0000} = C_{1111} = C_{2222} = \frac{E(1-\nu)}{(1+\nu)(1-2\nu)}$$
$$C_{0011} = C_{0022} = C_{1122} = \frac{E\nu}{(1+\nu)(1-2\nu)}$$
$$C_{0101} = C_{0202} = C_{1212} = \frac{E}{1+\nu}$$

with E group materi elasti young modulus and  $\nu$  group materi elasti poisson ratio (the remaining non-zero moduli follow from the double symmetry conditions).

For a transverse isotropic material the material has one unique direction (think of an material with fibers in one direction). Here we take '2' as the unique direction; '1' and '3' are the transverse directions. The material is fully defined by  $E_1$ ,  $E_2$ ,  $\nu_1$ ,  $\nu_2$ , and  $G_2$ . This set of parameters leads directly to a set of elasticity coefficients  $C_{ijkl}$ . The parameters can be given in **group materi elasti transverse isotropy**,

The nonlinear elasticity polynomials is a strain dependent model. In this model, the 'young's stiffness' modulus is made dependent of the size of the strains via a series of polynomials

$$E = E_0 + E_1 \epsilon^1 + E_2 \epsilon^2 + \dots \tag{1}$$

where

$$\epsilon = \sqrt{(\epsilon_{ij}\epsilon_{ij})} \tag{2}$$

with  $\epsilon_{ij}$  the components of the strain matrix. The parameters  $E_0$  etc. need to be specified in the **group materi elasti young polynomial** record.

If the pressure state in a point is positive, we apply  $E = E_0$ .

The <u>power law nonlinear elasticity</u> is a stress dependent model which typically is used to model the elastic behavior of granular materials. It can be combined with plastic models, for example with the di Prisco plasticity model for soils, and with a poisson ratio.

In this model, the 'young's stiffness' modulus is made a function of the average stress state:

$$E = E_0 + E_1 (p/p_1)^{\alpha}$$
(3)

with conditions however:

$$E >= E_2 \tag{4}$$

$$E <= E_3 \tag{5}$$

where p is the pressure. The parameters  $E_0$ ,  $E_1$ ,  $E_2$ ,  $E_3$ ,  $p_1$ , and  $\alpha$  need to be specified in the **group\_materi\_elasti\_young\_power** record.

In the group \_\_materi \_\_elasti \_\_poisson \_\_power model, the poisson ratio is made a function of the average stress state:

$$\nu = \nu_0 + \nu_1 (p/p_1)^{\alpha} \tag{6}$$

with condition however:

$$\nu <= \nu_2 \tag{7}$$

where p is the pressure. The parameters  $\nu_0$ ,  $\nu_1$ ,  $\nu_2$ ,  $p_1$ , and  $\alpha$  need to be specified in the **group materi elasti poisson power** record.

The stiffness matrix  $C_{ijkl}$  for the Borja Tamagnini nonlinear elasticity model is specified in

The model contains  $G_0$ ,  $\alpha$ ,  $\hat{k}$  and  $p_r$  as user specified constants which need to be specified in the **group materi elasti borja tamagnini** record.

The Lade nonlinear elasticity is a stress dependent model which typically is used to model the elastic behavior of granular materials. It can be combined with plastic models, for example with the di Prisco plasticity model for soils.

The stress rates are linked to the strain rates by the equation:

$$\dot{\epsilon_{ij}} = \frac{\partial W^2}{\partial \sigma_{ij} \partial \sigma_{hk}} \ \dot{\sigma_{hk}} \tag{8}$$

where the function W is

$$W = \frac{X^{1-\lambda}}{2B(1-\lambda)}$$

where

$$X = p^2 + R^* \operatorname{abs}(s_{ij} s_{ij})$$

with pressure  $p = (\sigma_{11} + \sigma_{22} + \sigma_{33})/3$  and deviatoric stresses  $s_{ij} = \sigma_{ij} - p\delta_{ij}$ .

The model contains three user specified constants B, R,  $\lambda$  which need to be specified in the **group\_materi\_elasti\_lade** record. B and  $\lambda$  are defined by means of an isotropic unloading test, and R by means of an unloading-standard-triaxial-compression test. For example for a loose sand B = 1028, R = 0.25,  $\lambda = 0.28$ . See [10] for the details.

The model cannot be used in combination with a poisson ratio.

### 2.2.3 Elasto-Plasticity

Plastic strain

In plastic analysis, the **materi\_strain\_elasti** rate follows by subtracting from the **materi\_strain\_total** rate the **materi\_strain\_plasti** rate

$$\dot{\epsilon_{ij}}^{\text{elas}} = \dot{\epsilon_{ij}} - \dot{\epsilon_{ij}}^{\text{plas}}$$

where the materi strain total rate is

$$\dot{\epsilon_{ij}} = 0.5(\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i})$$

The **materi\_strain\_plasti** rate follows from the condition that the stress cannot exceed the yield surface. This condition is specified by a yield function  $f^{\text{yield}}(\sigma_{ij}) = 0$ . The direction of the plastic strain rate is specified by the stress gradient of a flow function  $\frac{\partial f^{\text{flow}}}{\partial \sigma_{ij}}$ . If the yield function and flow function are chosen to be the same, the plasticity is called associative, otherwise it is non-associative.

Von-Mises is typically used for metal plasticity. Mohr-Coulomb and Drucker-Prager are typically used for soils and other frictional materials. The plasticity models can freely be combined; the combination of the plasticity surfaces defines the total plasticity surface.

Typically, if you use Mohr-Coulomb or Drucker-Prager to model shear failure for soils, you should use the tension limiting model to limit tension stresses, preferably **group\_materi\_plasti\_tension\_direc** 

First some stress quantities which are used in most of the plasticity models are listed.

Equivalent Von-Mises stress:

$$\bar{\sigma} = \sqrt{\frac{s_{ij}s_{ij}}{2}}$$

Mean stress:

$$\sigma_m = \frac{\sigma_{11} + \sigma_{22} + \sigma_{33}}{3}$$

Deviatoric stress:

$$s_{ij} = \sigma_{ij} - \sigma_m \delta_{ij}$$

CamClay plasticity model

Here we provide the equations of the Cam Clay model (Roscoe and Burland, 1968, summarized e.g. by Wood, 1990, see [20]). All stresses are effective (geotechnical) stresses, i.e.compression is positive! Definitions of variables:

$$p = (\sigma_1 + \sigma_2 + \sigma_3)/3$$

$$q = \left\{\frac{1}{2}\left[(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2\right]\right\}^{1/2}$$

in the principal stress axes. The CamClay yield rule, which is also the flow rule, reads:

$$f = g = q^2 - M^2[p(p_0 - p)] = 0$$

M is a soil constant and  $p_0$  is a history (hidden) variable which corresponds to the preconsolidation mean pressure. The hardening function, evolution, of  $p_0$  reads:

$$dp_0 = \frac{p_0(1+e)d\varepsilon_v^p}{\lambda - \kappa}$$

in which

$$d\varepsilon_v^p = d\varepsilon_{11}^p + d\varepsilon_{22}^p + d\varepsilon_{33}^p$$

and  $\lambda$  and  $\kappa$  are user specified soil constants. Further *e* is the void ratio with the evolution equation:

$$de = -d\varepsilon_v (1+e)$$

in which

$$d\varepsilon_v = d\varepsilon_{11} + d\varepsilon_{22} + d\varepsilon_{33}$$

Notice that this is a geometrical linear approximation for void ratio changes. The poisson ratio  $\nu$  reads:

$$\nu = \frac{3K - 2G}{2G + 6K}$$

in which the elastic bulk modulus K is given by:

$$K = (1+e)p/\kappa$$

and the Young's modulus E:

$$E = 2. * G * (1 + \nu)$$

in which G is a user specified soil constant, By using this  $\nu$  and E the classical isotropic stress-strain law is used to calculate the stresses.

The soil constants M,  $\kappa$ ,  $\lambda$  need to be specified in **group\_materi\_plasti\_camclay**. The soil constant G, need to be specified in **group\_materi\_elasti\_camclay\_g**. For an alternative see **group\_materi\_elasti\_camclay\_poisson**. The history variables e,  $p_0$  need to be initialized by **materi\_plasti\_camclay\_history** record (and given initial values in **node\_dof** records).

Remark 1: An additional parameter N can be often found in textbooks on the Cam Clay model. We don't include it since it is linked to other model parameters via:

$$1 + e = N - \lambda \ln p_0 + \kappa \ln(p_0/p)$$

Remark 2: If you apply a geometrical linear analysis, see section 8.4, then also the calculation of de void ratio development is linearized, and so will contain some error as compared to the exact void ratio change. Hence for very large deformations, say above 10 percent or so, don't use such geometrical linear analysis.

Remark 3: This camclay law is very sensitive for near-zero stresses. It is best to only use it in combination with application of initial gravity stresses by means of **control reset dof**.

Cap1 plasticity model

This group\_materi\_plasti\_cap1 model is the first cap model that accounts for permanent plastic deformations under high pressures for granular materials. It is intended to be used in combination with shear plasticity models like Drucker-Prager, Mohr-Coulomb, etc.

First the average stress p and the equivalent shear stress q are introduced:

$$p = -(\sigma_{11} + \sigma_{22} + \sigma_{33})/3$$

$$q = \left\{\frac{1}{2}\left[(\sigma_{11} - \sigma_{22})^2 + (\sigma_{22} - \sigma_{33})^2 + (\sigma_{33} - \sigma_{11})^2\right] + 3(\sigma_{12}^2 + \sigma_{23}^2 + \sigma_{31}^2)\right\}^{1/2}$$

These are used to define the cap plastic yield function:

$$f = \frac{q^2}{M^2} + p^*(p^* - p_c^*)$$

where

$$p^* = p + c \cot \phi$$
  $p_c^* = p_c + c \cot \phi$ 

The parameter  $p_c$  is a history variable of this model. The parameter  $\phi$  is the coulomb friction angle in radians, and c is the cohesion. The parameter M denotes the tangent of the Critical State Line in the model, Typically you can use:

$$M = \frac{6\sin\phi}{3-\sin\phi}$$

The history parameter  $p_c$  is assumed to harden with the cap plastic volume strain rate according to the rate form:

$$\dot{\epsilon}^p_{cv} = \frac{\lambda^*/\kappa^* - 1}{K^{ref}} \left(\frac{p^{ref}}{p_c^*}\right)^m \dot{p}_c$$

Here  $\kappa^*$  is the swelling index (e.g. 0.03),  $\lambda^*$  is the compression index (e.g. 0.15),  $K^{ref}$  is the bulk modulus at stress  $p^{ref}$  (typically 100kPa), which typically can be taken as:  $K^{ref} = \frac{E^{ref}}{3(1-2\nu)}$ , and finally m is an exponent (e.g. 0.6).

Initialize **materi\_plasti\_cap1\_history** in the initialization part. The state variable  $p_c$  for this hardening soil model enters the **node\_dof** records. You need to give an initial value for it in the **node\_dof** records. See also [2].

### Cap2 plasticity model

This is the second cap model that accounts for permanent plastic deformations under high pressures for granular materials. It is intended to be used in combination with shear plasticity models like Drucker-Prager, Mohr-Coulomb, etc.

First a deviatoric stress measure t and hydrostatic stress measure p are defined

$$t = \sqrt{3}\bar{\sigma}$$
$$p = -\sigma_m$$

See above for  $\bar{\sigma}$  and  $\sigma_m$ . The yield rule for the **group** materi plasti cap2 model reads:

$$f = \sqrt{(p - p_a)^2 + \left[\frac{Rt}{(1 + \alpha - \frac{\alpha}{\cos\phi}}\right]^2 - R(c + p_a tan\phi)}$$

Here c is the cohesion and  $\phi$  is the friction angle which should be taken equal to the values in the shear flow rule which you use. The parameter  $p_a$  follows from

$$p_a = \frac{p_b - Rc}{1 + R \tan\phi}$$

where the hydrostatic compression yield stress  $p_b$  is to be defined with an table of volumetric plastic strains  $\epsilon_v^p$  versus  $p_b$  with  $\epsilon_v^p = \epsilon_{11}^p + \epsilon_{22}^p + \epsilon_{33}^p$ . As always, positive strain denote extension whereas negative strains denote compression.

Associative flow is used, so the flow rule is taken equal to the yield rule.

Summarizing the **group\_materi\_plasti\_cap2** model needs as input the cohesion c, the friction angle  $\phi$ , the parameter  $\alpha$  (typically 1.  $10^{-2}$  up to 5.  $10^{-2}$ ), and a table  $\epsilon_v^p$  versus  $p_b$ .

Compression limiting plasticity model

This  $group\_materi\_plasti\_compression$  model uses a special definition for the equivalent stress

$$\bar{\sigma} = \sqrt{{\sigma_{min}}^2}$$

where  $\sigma_{min}$  is the largest compressive principal stress. The model now reads

$$\bar{\sigma} - \sigma_y = 0$$

This plasticity surface limits the allowed compressive stresses.

di Prisco plasticity model

The di Prisco model is an non-associative plastic model for soils, which can be typically combined with the 'Lade elastic model'. This di Prisco model is a rather advanced soil model, which is explained in more detail in [3] and [9]. The yield rule reads:

$$f = 3\beta_f(\gamma - 3) \ln\left(\frac{r}{r_c}\right) - \gamma J_{3\eta^*} + \frac{9}{4}(\gamma - 1)J_{2\eta^*}$$

and the flow rule yields:

$$g = 9(\gamma - 3) \ln\left(\frac{r}{r_g}\right) - \gamma J_{3\eta^*} + \frac{9}{4}(\gamma - 1)J_{2\eta^*}$$

This is an anisotropic model in which the first and second invariant of the stress rate  $\eta^*$  are defined relative to the rotation axes  $\chi$ .

$$r = \sigma_{ij}\chi_{ij}$$
$$J_{3\eta^*} = \eta^*_{ij}\eta^*_{jk}\eta^*_{ki}$$
$$J_{2\eta^*} = \eta^*_{ij}\eta^*_{ij}$$
$$\eta^*_{hk} = \sqrt{3}\frac{s^*_{hk}}{r}$$

where  $s^*$  follows from

$$s_{hk}^* = \sigma_{hk}^* - r\chi_{hk}$$

Further  $r_g = 1$ .

The history variables are  $\chi_{ij}$  (rotation axes, 9 values),  $\beta$  (yield surface form factor), and  $r_c$  (preconsolidation mean pressure). The evolution laws for these history variables can be found in the papers listed above. The history variables  $\chi_{ij}$  (9 values),  $\beta$ ,  $r_c$  need to be initialized by the **group\_plasti\_diprisco\_history 11** record (and should be given initial values in **node\_dof** records). In a normally consolidated sand with isotropic initial conditions  $\chi_{ij} = \frac{\delta_{ij}}{\sqrt{3}}$ ,  $\beta = 0.0001$  and  $r_c$  equals  $\sqrt{3}$  times the means pressure.

The total model, yield rule and flow rule and evolution laws for history variables, contains a set of soil specific constants. In **group \_materi \_plasti \_diprisco** you need to specify these constants. These constants are explained in more detail in the papers mentioned above, but here we give a short explanation. The constants  $\hat{\theta}_c$ ,  $\hat{\theta}_e$ ,  $\xi_c$  and  $\xi_e$  are linked to the dilatancy and the stress state during failure (standard triaxial compression and extension test in drained conditions). The constants  $\gamma$ ,  $c_p$ ,  $\beta_f$  and  $\beta_f^0$  are defined by means of the experimental curves (q- $\epsilon_{axial}$ ,  $\epsilon_{vol}$ - $\epsilon_{axial}$ ) obtained by performing a standard compression test in drained conditions. Moreover,  $\beta_f$ ,  $\beta_f^0$  and  $t_p$  can also be determined by means of the effective-stress path obtained by performing a standard triaxial compression test in undrained conditions.

Finally  $b_p$  can determined from an isotropic compression test. For a loose sand  $\hat{\theta}_c = 0.253$ ,  $\hat{\theta}_e = 0.0398$ ,  $\xi_c = -0.2585$ ,  $\xi_e = -0.0394$ ,  $\gamma = 3.7$ ,  $c_p = 18.$ ,  $\beta_f = 0.5$ ,  $\beta_f^0 = 1.1$ ,  $t_p = 10.$ , and  $b_p = 0.0049$ .

di Prisco plasticity model with varying density

This essentially is the same as the normal di Prisco model, but instead of one set of parameters you need to specify two sets of parameters, one of loose soil and one for dense soil. The actual applied parameters will then be interpolated from the loose parameters and dense parameters depending on the actual density of the soil. The parameters need to be specified in **group materi plasti diprisco density**.

The history variables are those of **group\_materi\_plasti\_diprisco** and finally extra the relative density (for example 20 or 40). So there are 12 history variables in total.

Drucker-Prager plasticity model

The group materi plasti druck prag model reads

$$3\alpha\sigma_m + \bar{\sigma} - K = 0$$
$$\alpha = \frac{2\sin(\phi)}{\sqrt{3}(3 - \sin(\phi))}$$
$$K = \frac{6c\cos(\phi)}{\sqrt{3}(3 - \sin(\phi))}$$

Here c is the cohesion, which needs to be specified both for the yield function and the flow rule; by choosing different values non-associative plasticity is obtained.

You should also include tension cut-off, preferably with group\_materi\_plasti\_tension\_direct. Generalised Non Associate CamClay for Bonded Soils plasticity model

It is a modification of the 'Milan' model of Prof. Roberto Nova.

Gurson plasticity model

The group materi plasti gurson model reads

$$\frac{3\bar{\sigma}^2}{\sigma_y^2} + 2q_1 f^* \cosh(q_2 \frac{3\sigma_m}{2\sigma_y}) - (1 + (q_3 f^*)^2) = 0$$

Here  $f^*$  is the volume fraction of voids. The rate equation

$$\dot{f^*} = (1 - f^*) f^* \epsilon_{kk}^{\text{plas}}$$

defines the evolution of  $f^*$  if the start value for  $f^*$  is specified. Furthermore,  $q_1$ ,  $q_2$  and  $q_3$  are model parameters.

Hardening-Soil model

In this section, the principal stresses are ordered such that

$$\sigma_3 > \sigma_2 > \sigma_1$$

so that  $\sigma_1$  is the largest compressive stress. Likewise for the principal plastic strains:

$$\epsilon_3^p > \epsilon_2^p > \epsilon_1^p$$

First the elasticity parameters are defined. The elasticity parameters for the first loading are:

Young's modulus = 
$$E_{50} = E_{50}^{ref} \left( \frac{\sigma_3 + c \, \cot \phi}{\sigma_{50}^{ref} + c \, \cot \phi} \right)^m$$
 and Poisson's ratio =  $\nu_{50}$ 

The elasticity parameters for the elastic unloading and reloading are:

Young's modulus = 
$$E_{ur} = E_{ur}^{ref} \left( \frac{\sigma_3 + c \, \cot \phi}{\sigma_{ur}^{ref} + c \, \cot \phi} \right)^m$$
 and Poisson's ratio =  $\nu_{ur}$ 

The yield function reads:

$$f = \frac{1}{E_{50}} \frac{q}{1 - q/q_a} - \frac{2q}{E_{ur}} - \gamma^p$$

where q is the equivalent shear stress and  $\gamma^p$  is the equivalent plastic shear strain. The equivalent asymptotic shear stress reads

$$q_a = \frac{q_f}{R_f}$$

in which  $q_f$  is the shear failure stress, and  $R_f$  is the failure ratio.

Specify all elasticity parameters in group materi elasti hardsoil. Typically you have:

- $E_{50}^{ref}$  from experiment at stress  $\sigma_{50}^{ref}$
- $\nu_{ur}$  from experiment or the typical undrained value 0.495 or the typical drained value 0.3
- m from experiment or the typical value 0.5
- $E_{ur}^{ref}$  from experiment at stress  $\sigma_{ur}^{ref}$ , or the typical value  $3E_{50}^{ref}$
- $\nu_{ur}$  from experiment or the typical undrained value 0.495 or the typical drained value 0.2

Specify all plasticity parameters in group materi plasti hardsoil.

- $\phi$  from experiment (maximum friction angle)
- c from experiment (cohesion)
- $\psi$  from experiment (maximum dilatancy angle)
- $R_f$  from experiment or the typical value 0.9 (failure ratio)

Initialize materi\_strain\_plasti\_hardsoil in the initialization part. This causes that the node\_dof records will be filled with the shear plastic strains. Also initialize materi plasti hardsoil history.

You can add an initial contribution to the  $\gamma^p$  by setting **control\_materi\_plasti\_hardsoil\_gammap\_initial** to **-yes**. This tells tochnog to create an extra contribution to  $\gamma^p$  exactly such that the yield function is zero-valued. This is convenient to start the calculation with hardsoil with deviatoric stresses which would have been outside the yield surface without this extra contribution. The extra addition to  $\gamma^p$  is saved in the record **element\_intpnt\_materi\_plasti\_hardsoil\_gammap\_in** for each integration point of elements. The creation of this extra initial contribution is done in the first timestep of the timesteps of the corresponding **control\_timestep** record with the same index.

See also [18] for some details. Especially notice that the model is more suited for monotonic loading than for load cycling (since it violates thermodynamics and tends to generate energy).

Mohr-Coulomb plasticity model

The group materi plasti mohr coul model reads

$$0.5(\sigma_1 - \sigma_3) + 0.5(\sigma_1 + \sigma_3)\sin(\phi) - c \cos(\phi) = 0$$

Here c is the cohesion,  $\sigma_1$  is the largest principal stress and  $\sigma_3$  is the smallest principal stress. The angle  $\phi$  needs to be specified for both the yield condition and the flow rule; by choosing different values, non-associative plasticity is obtained.

As an alternative consider using group\_materi\_plasti\_mohr\_coul\_direct, which is more stable and fast.

 $You should also include tension cut-off, preferably with {\it group materi plasti tension direct}.$ 

Mohr-Coulomb hardening-softening plasticity model

The group <u>materi</u> plasti <u>mohr</u> coul <u>hardening</u> softening model is the same as the standard Mohr-Coulomb model. Now, however, the parameters c and  $\phi$  (both for the yield rule and for the flow rule) are softened on the effective plastic strain  $\kappa^{shear}$ .

For example, for the cohesion a linear variation is taken between the initial value  $c_0$  at  $\kappa^{shear} = 0$ , up to  $c_1$  at a specified critical value of  $\kappa^{shear}$ , and constant  $c_1$  for larger values of  $\kappa^{shear}$ . The same is done for  $\phi$  for the yield rule and for the flow rule.

You should also include tension cut-off, preferably with group materi plasti tension direct.

Tension limiting plasticity model

This group\_materi\_plasti\_tension model uses a special definition for the equivalent stress

$$\bar{\sigma} = \sqrt{\sigma_{max}^2}$$

where  $\sigma_{max}$  is the largest principal tension stress.

$$\bar{\sigma} - \sigma_y = 0$$

This plasticity surface limits the allowable tension stresses.

A simple model for concrete can be obtained as follows. Use **group \_ materi \_ plasti \_ tension** to limit the tension strength ft. Use **group \_ materi \_ plasti \_ vonmises** to limit the compressive strength fc. The tension strength could be softened to zero over an effective plastic strain  $\kappa$  of, say, 1 percent. The compressive strength could be softened to zero over an effective plastic strain  $\kappa$  of, say, 10 percent.

### Von-Mises plasticity model

The group materi plasti vonmises model reads

$$\sqrt{3}\ \bar{\sigma} - \sigma_y = 0$$

where without hardening the yield value is fixed  $\sigma_y = \sigma_{y0}$ .

If however the **group\_materi\_plasti\_vonmises\_nadai** hardening law for Von-Mises plasticity is specified then

$$\sigma_y = \sigma_{y0} + C(\kappa_0 + \kappa)^n$$

where C,  $\kappa_0$  and n are parameters for the hardening law, and  $\kappa$  is the isotropic hardening parameter (see later). The parameter  $\sigma_{y0}$  is specified by **group materi plasti vonmises**.

### Isotropic Hardening and softening

The size of the total plastic strains rate is measured by the **materi\_plasti\_kappa** parameter

$$\dot{\kappa} = \sqrt{0.5 \dot{\epsilon}_{ij}^{\rm plas} \dot{\epsilon}_{ij}^{\rm plas}}$$

The size of the shear plastic strains rate is measured by the  ${\bf materi\_plasti\_kappa\_shear}$  parameter

$$\dot{\kappa}^{shear} = \sqrt{0.5 \dot{\epsilon}_{ij}^{\text{shear,plas}} \dot{\epsilon}_{ij}^{\text{shear,plas}}}$$

where the plastic shear strains are defined by

$$\dot{\epsilon}_{ij}^{\rm shear, plas} = \dot{\epsilon}_{ij}^{\rm plas} - \delta_{ij} (\dot{\epsilon}_{11}^{\rm plas} + \dot{\epsilon}_{22}^{\rm plas} + \dot{\epsilon}_{33}^{\rm plas})/3$$

These parameters  $\kappa$  and  $\kappa^{shear}$  can be used for isotropic hardening. Use the **dependency diagram** for this.

### Kinematic Hardening

The **materi\_plasti\_rho** matrix  $\rho_{ij}$ , governs the kinematic hardening in the plasticity models. It is used in the yield rule and flow rule to get a new origin by using the argument  $\sigma_{ij} - \rho_{ij}$ :

$$f^{\text{yield}} = f^{\text{yield}}(\sigma_{ij} - \rho_{ij})$$
$$f^{\text{flow}} = f^{\text{flow}}(\sigma_{ij} - \rho_{ij})$$

where the rate of the matrix  $\rho_{ij}$  is taken to be

$$\dot{\rho_{ii}} = a \epsilon_{ii}^{i}^{\text{plas}}$$

where a is a user specified factor (see group materi plasti kinematic hardening).

Plastic heat generation

The plastic energy loss can be partially turned into heat rate per unit volume q:

$$q = \eta \, \sigma_{ij} \, \epsilon_{ij}^{\text{plas}}$$

where  $\eta$  is a user specified parameter (between 0 and 1) specifying which part of the plastic energy loss is turned into heat (see **group materi plasti heat generation**).

### 2.2.4 Hypo-Plasticity

In hypoplasticity a direct relation is used between strain rates and effective stress rates. Rigid body rotations (objectivity) are treated elsewhere (see the section on memory). The effective stress tensor  $\sigma_{ij}$  follows from the total stress tensor  $\sigma_{ij}$  minus any pore pressures (see groundflow). The Masin law is tuned to clays. The Wolffersdorff law is tuned to sands but can also be used for clays. The Niemunis visco law describes time dependent soil behaviour. If you need a cyclic law, you should use the Wolffersdorff law with intergranular strains and especially specify the correct theta. For many cycles the isa-intergranular strain formulation can be used.

### Masin law

The law proposed by MASIN [11] and [12] is used. This law is formulated in kPa; you need to make the remainder of the input file consistent with that.

The constitutive equation in rate form reads:

$$\overset{\circ}{\mathbf{T}} = \mathcal{L} : \mathbf{D} + f_d \mathbf{N} \| \mathbf{D} \| \tag{9}$$

where  $\mathbf{D}$  is the Euler's stretching tensor,  $\mathbf{T}$  is the Cauchy stress tensor and

$$\mathcal{L} = 3f_s \left( c_1 \mathcal{I} + c_2 a^2 \hat{\mathbf{T}} \otimes \hat{\mathbf{T}} \right) \qquad \mathbf{N} = \mathcal{L} : \left( -Y \frac{\mathbf{m}}{\|\mathbf{m}\|} \right) \qquad \hat{\mathbf{T}} = \frac{\mathbf{T}}{\operatorname{tr} \mathbf{T}}$$
(10)

**1** is the second–order identity tensor and  $\mathcal{I}$  is the fourth–order identity tensor, with components:

$$\left(\mathcal{I}\right)_{ijkl} = \frac{1}{2} \left( 1_{ik} 1_{jl} + 1_{il} 1_{jk} \right) \tag{11}$$

The functions  $f_s(\operatorname{tr} \mathbf{T})$  (barotropy factor) and  $f_d(\operatorname{tr} \mathbf{T}, e)$  (pyknotropy factor) are given by:

$$f_s = -S_i \frac{\operatorname{tr} \mathbf{T}}{\lambda^*} \left(3 + a^2 - 2^\alpha a \sqrt{3}\right)^{-1} f_d \qquad = \left[-\frac{2\operatorname{tr} \mathbf{T}}{3sp_r} \exp\left(\frac{\ln\left(1+e\right) - N}{\lambda^*}\right)\right]^\alpha \tag{12}$$

where  $p_r$  is the reference stress for the parameter N, typically taken as 1 kPa, and the factor  $S_i$  is a function of sensitivity s:

$$S_i = \frac{s - k(s - s_f)}{s} \tag{13}$$

The scalar function Y and the second-order tensor  $\mathbf{m}$  are given, respectively, by:

$$Y = \left(\frac{\sqrt{3}a}{3+a^2} - 1\right) \frac{(I_1 I_2 + 9I_3)\left(1 - \sin^2\varphi_c\right)}{8I_3 \sin^2\varphi_c} + \frac{\sqrt{3}a}{3+a^2}$$
(14)

in which:

$$I_1 = \operatorname{tr} \mathbf{T}$$
  $I_2 = \frac{1}{2} \left[ \mathbf{T} : \mathbf{T} - (I_1)^2 \right]$   $I_3 = \det \mathbf{T}$ 

 $\operatorname{and}$ 

$$\mathbf{m} = -\frac{a}{F} \left[ \hat{\mathbf{T}} + \hat{\mathbf{T}}^* - \frac{\hat{\mathbf{T}}}{3} \left( \frac{6 \hat{\mathbf{T}} : \hat{\mathbf{T}} - 1}{(F/a)^2 + \hat{\mathbf{T}} : \hat{\mathbf{T}}} \right) \right]$$
(15)

in which:

$$\hat{\mathbf{T}}^{*} = \hat{\mathbf{T}} - \frac{1}{3} \qquad F = \sqrt{\frac{1}{8} \tan^{2} \psi + \frac{2 - \tan^{2} \psi}{2 + \sqrt{2} \tan \psi \cos 3\theta} - \frac{1}{2\sqrt{2}} \tan \psi} \qquad (16)$$

$$\tan \psi = \sqrt{3} \| \hat{\mathbf{T}}^* \| \qquad \cos 3\theta = -\sqrt{6} \frac{\operatorname{tr} \left( \mathbf{T}^* \cdot \mathbf{T}^* \cdot \mathbf{T}^* \right)}{\left( \hat{\mathbf{T}}^* : \hat{\mathbf{T}}^* \right)^{3/2}}$$
(17)

Finally, the scalars a,  $\alpha$ ,  $c_1$  and  $c_2$  are given as functions of the material parameters  $\varphi_c$ ,  $\lambda^*$ ,  $\kappa^*$  and r by the following relations:

$$a = \frac{\sqrt{3} \left(3 - \sin \varphi_c\right)}{2\sqrt{2} \sin \varphi_c} \qquad \qquad \alpha = \frac{1}{\ln 2} \ln \left[\frac{\lambda^* - \kappa^* S_i}{\lambda^* + \kappa^* S_i} \left(\frac{3 + a^2}{a\sqrt{3}}\right)\right] \tag{18}$$

$$c_1 = \frac{2\left(3 + a^2 - 2^\alpha a\sqrt{3}\right)}{9rS_i} \qquad c_2 = 1 + (1 - c_1)\frac{3}{a^2} \tag{19}$$

Evolution of the state variables e (void ratio) and s (sensitivity) is governed by

$$\dot{e} = (1+e)\operatorname{tr}\mathbf{D} \tag{20}$$

$$\dot{s} = -\frac{k}{\lambda^*} (s - s_f) \sqrt{(\dot{\epsilon}_v)^2 + \frac{A}{1 - A} (\dot{\epsilon}_s)^2}$$
(21)

where  $\dot{\epsilon}_v = \operatorname{tr} \mathbf{D}$  and  $\dot{\epsilon}_s = \sqrt{2/3} \|\operatorname{dev} \mathbf{D}\|$ .

The basic hypoplastic model requires five constitutive parameters, namely  $\varphi_c$ ,  $\lambda^*$ ,  $\kappa^*$ , N and r, state is characterised by the Cauchy stress **T** and void ratio e.

	$\varphi_c$	$\lambda^*$	$\kappa^*$	N	r	k	A	$s_f$
London clay	$22.6^{\circ}$	0.11	0.016	1.375	0.4	-	-	-
Pisa clay	$21.9^{\circ}$	0.14	0.0075	1.56	0.3	0.4	0.1	1

 Table 1: Typical parameters of the hypoplastic model for clays.

An extended model allows us to take into account the effects of meta-stable structure of natural clays. This extension requires three additional parameters  $(k, A, s_f)$ , and one additional state variable s. A basic model without the structure effects is recovered if  $s = s_f = 1$  and  $A \neq 1$ . The s should be always greater or equal to 1.

The basic law parameters should be specified in **group\_materi\_plasti\_hypo\_masin**. The extended parameters for the structure should be specified in **group\_materi\_plasti\_hypo\_masin\_structu**. The hypoplastic history variables, *e* for this basic model, and *e* and *s* for the extended model, should be initialised with **materi\_plasti\_hypo\_history**. As an alternative to specify the *e* you can specify the OCR at the start of the calculation in **group\_materi\_plasti\_hypo\_masin\_ocr** (which is used to determine the initial *e* via  $e = exp(N - \lambda^* ln(|OCR|) - \lambda^* ln(|p/p_r|)) - 1$ .).

### Masin clay law

This law is an improved version of the Masin law of the previous section. This law is formulated in kPa, so you should use that as units in your input file.

Anisotropy of sedimentary clays is such a significant feature of their mechanical behaviour that it cannot be ignored in finite element calculations. The hypoplastic model for clays with explicit formulation of the asymptotic state boundary surface is combined with an anisotropic form of the stiffness tensor. The resulting model predicts correctly the very small strain stiffness anisotropy. Anisotropy influences the undrained stress path, and this model is able to model such influence. For theoretical backgroup see [13] and [14].

User input can be defined with the following records: control materi plasti hypo masin clay ocr appl

```
group_materi_plasti_hypo_cohesion,
group_materi_plasti_hypo_masin_clay,
group_materi_plasti_hypo_masin_clay_advanced_parameters,
group_materi_plasti_hypo_masin_clay_advanced_direction,
group_materi_plasti_hypo_masin_clay_ocr,
group_materi_plasti_hypo_masin_clay_structure,
group_materi_plasti_hypo_strain_intergranular_masin_clay.
```

A biaxial input file showing all available options is given here below:

echo -no
number\_of\_space\_dimensions 2
materi\_velocity
materi\_displacement
materi\_strain\_intergranular

```
materi_strain_total
materi_plasti_hypo_history
materi_stress
end_initia
node 1 0.00 0.00
node 2 1.00 0.00
node 3 0.00 1.00
node 4 1.00 1.00
element 1 -quad4 1 2 3 4
start_define
  left_edge geometry_line 0
end_define
left_edge 0.00 0.00 0.00 1.00 1.e-5
start_define
  lower_edge geometry_line 1
end_define
lower_edge 0.00 0.00 1.00 0.00 1.e-5
start_define
  upper_edge geometry_line 2
end_define
upper_edge 0.00 1.00 1.00 1.00 1.e-5
start_define
  right_edge geometry_line 3
end_define
right_edge 1.00 0.00 1.00 1.00 1.e-5
start_define
  sig0 -50.
end_define
force_edge_normal_geometry 0 -geometry_line 3
force_edge_normal
                             0 sig0
             0 -left_edge -velx
bounda_dof
            1 -lower_edge -vely
2 -upper_edge -vely
bounda_dof
bounda_dof
               2 -1.
bounda_time
group_type 0 -materi
group_materi_plasti_hypo_masin_clay_ocr 0 1.2
group_materi_plasti_hypo_cohesion 0 0.
group_materi_plasti_hypo_masin_clay 0
                        25 (phic in degrees)
                        0.1 (lambda_star)
                        0.01 (kappa_star)
```

```
1 (N)
                       0.2 (nu_pp)
(
group_materi_plasti_hypo_masin_clay_structure 0
                       0.4 (k)
                       0.1 (A)
                       1.5 (s_f)
group_materi_plasti_hypo_strain_intergranular_masin_clay 0
                       5.e-5 (R)
                       270. (A_g)
                       1.0 (n_g)
                       0.5 (m_rat)
                       0.08 (beta_r)
                       7.0 (chi)
                       7.0 (theta)
group_materi_plasti_hypo_masin_clay_visco 0
                       1.e-6 (D_r)
                       0.1 (I_v)
group_materi_plasti_hypo_masin_clay_ocr 0
                       1.5 (ocr)
group_materi_plasti_hypo_masin_clay_advanced_double 0
                       1.0 (alpha_g)
                       equation 40 (alpha_f)
                       0.3 (a_y)
                       2.0 (o_c)
)
post_point 1 0.5 0.5
control_reset_dof
                             10 -hyhis0
control_reset_value_constant 10 0.6
control_reset_dof
                             20 -sigxx -sigzz
control_reset_value_constant 20 sig0
                             30 -sigyy
control_reset_dof
control_reset_value_constant 30 sig0
                             40 -hyhis4 (structure)
control_reset_dof
control_reset_value_constant 40
                                 2
control_reset_dof
                             50 -epixx
control_reset_value_constant 50 -0.00001
control_reset_dof
                             60 -epiyy
control_reset_value_constant 60 -0.00001
                             70 -epizz
control_reset_dof
control_reset_value_constant 70 -0.00001
control_print_history
                             80 -post_point_dof 1 -sigyy
                             90 1.e-4 0.1
control_timestep
```

```
control_materi_plasti_hypo_masin_clay_ocr_apply 90 -no
control_print 90 -time_current -post_node_rhside_ratio -post_point_dof
control_print_history 90 -post_point_dof 1 -sigyy
end_data
```

A final notice: this law only functions well if the OCR does not become below 1. If Tochnog detects a void ratio violating this condition, it will automatically change the void ratio in such a way that the OCR becomes 1, and the calculation can continue.

### Wolffersdorff law

The law proposed by WOLFFERSDORFF [19] is used.

$$\dot{\sigma}_{ij} = L_{ijkl}\dot{\epsilon}_{ij} + f_d N_{ij}\sqrt{\dot{\epsilon}_{kl}\dot{\epsilon}_{kl}} = L_{ijkl}(d_{kl} - f_d Y m_{kl}||d||)$$

Here the part with  $L_{ijkl}$  gives a linear relation between strain rates and stress rates and the part with  $N_{ij}$  gives a nonlinear relation. The constitutive tensors  $L_{ijkl}$  and  $f_d N_{ij}$  are functions of the effective stress tensor  $\sigma_{ij}$  and void ratio e. In the above d denotes the strain rate tensor  $\epsilon$ , Y denotes the degree of nonlinearity  $Y = ||L^{-1} : N||$  and the flowrule m is defined by  $m = -(L^{-1} : N)^{\rightarrow}$  where a  $\rightarrow$  denotes euclidian normalisation.

$$\begin{split} L_{ijkl} &= f_b f_e \frac{1}{\hat{\sigma}_{mn} \hat{\sigma}_{mn}} L_{ijkl} \\ N_{ij} &= f_b f_e \frac{F a}{\hat{\sigma}_{kl} \hat{\sigma}_{kl}} \left( \hat{\sigma}_{ij} + \hat{\sigma}_{ij}^* \right) \\ \text{and} \quad \hat{\sigma}_{ij} &= \sigma_{ij} / (\sigma_{mn} \delta_{mn}) \quad , \quad \hat{\sigma}_{ij}^* = \hat{\sigma}_{ij} - \frac{1}{3} \delta_{ij} \quad , \quad I_{ijkl} = \delta_{ik} \delta_{jl} \; , \\ a &= \frac{\sqrt{3}(3 - \sin \varphi_c)}{2\sqrt{2} \sin \varphi_c} \\ F &= \sqrt{\frac{1}{8} \tan^2 \psi} + \frac{2 - \tan^2 \psi}{2 + \sqrt{2} \tan \psi \cos 3\theta} - \frac{1}{2\sqrt{2}} \tan \psi \; , \\ \tan \psi &= \sqrt{3} \sqrt{\hat{\sigma}_{ij}^* \hat{\sigma}_{ij}^*} \quad , \quad \cos 3\theta = -\sqrt{6} \; \frac{\hat{\sigma}_{ij}^* \hat{\sigma}_{jk}^* \hat{\sigma}_{ki}^*}{\left[\hat{\sigma}_{mn}^* \hat{\sigma}_{mn}^*\right]^{3/2}} \quad . \end{split}$$

For the  $\hat{L_{ijkl}}$  above we have:

$$\hat{L}_{ijkl} = \left(F^2 I_{ijkl} + a^2 \,\hat{\sigma}_{ij} \hat{\sigma}_{kl}\right)$$

For  $\hat{\sigma}_{ij}^* = 0$  is F = 1.

The scalar factors  $f_b$ ,  $f_e$  and  $f_d$  take into account the influence of mean pressure and density:

$$\begin{aligned} f_b &= \frac{h_s}{n} \left(\frac{e_{i0}}{e_{c0}}\right)^{\beta} \frac{1+e_i}{e_i} \left(-\frac{\sigma_{ij}\delta_{ij}}{h_s}\right)^{1-n} \left[3+a^2-a\sqrt{3}\left(\frac{e_{i0}-e_{d0}}{e_{c0}-e_{d0}}\right)^{\alpha}\right]^{-1} &, \\ f_d &= \left(\frac{e-e_d}{e_c-e_d}\right)^{\alpha} &. \end{aligned}$$

and  $f_e = \left(\frac{e_c}{e}\right)^{\beta}$ .

Three characteristic void ratios  $-e_i$  (during isotropic compression at the minimum density),  $e_c$  (critical void ratio) and  $e_d$  (maximum density) – decrease with mean stress:

$$\frac{e_i}{e_{i0}} = \frac{e_c}{e_{c0}} = \frac{e_d}{e_{d0}} = \exp\left[-\left(-\frac{\sigma_{ij}\delta_{ij}}{h_s}\right)^n\right]$$

The range of admissible void ratios is limited by  $e_i$  and  $e_d$ . The model parameters can be found in Tab. 2. They correspond to Hochstetten sand from the vicinity of Karlsruhe, Germany [19].

$\varphi$ [°]	$h_s$ [MPa]	n	$e_{c0}$	$e_{d0}$	$e_{i0}$	$\alpha$	β
33	1000	0.25	0.95	0.55	1.05	0.25	1.0

Table 2: Basic hypoplastic parameters of Hochstetten sand.

The basic law parameters should be specified in **group \_\_materi \_\_plasti \_\_hypo \_\_wolffersdorff**. The hypoplastic history variables should be initialised with **materi \_\_plasti \_\_hypo \_\_history**.

### Wolffersdorff law - extended Niemunis version

An extended version of this hypoplastic law is given in [16]. An extra term is added to  $\hat{L}_{ijkl}$ , in index notation:

$$b^2(\delta_{ik}\delta_{jl} - \frac{1}{3}\delta_{ij}\delta_{kl})$$

Here  $b^2$  is given in [16], depending on the input parameter  $\nu$ , which can be seen as the classical Poisson ratio, typically 0.2 or 0.3. The first part of  $f_d$  is determined:

if 
$$e \ge e_d$$
 then  $f_d = \left(\frac{e - e_d}{e_c - e_d}\right)^{\alpha}$   
if  $e < e_d$  then  $f_d = -\left(\frac{e_d - e}{e_c - e_d}\right)^{\alpha}$ 

Then  $\overline{f}_d$  is determined:

$$f_d$$
 equation 4.221 in [16]

where

$$M_e^{(d)}$$
 and  $M_T^{(d)}$  equation 4.211 in [16]

Finally  $f_d$  is checked

if 
$$(f_d < 1)$$
 then  $f_d := f_d + (1 - f_d)^z \overline{f}_d$ 

where z is an input parameter specified, typically 5.

You need to specify the parameters  $\nu$  and z in group materi plasti hypo wolffersdorff niemunis.

Wolffersdorff pressure dependent initial void ratio extension

You can correct the initial void ratio  $e_0$ , as specified in the initial value for the history variable in the **node\_dof** records, for the initial pressure to obtain a corrected initial void ratio  $e_0$ .

$$\frac{e}{e_0} = \exp\left[-\left(-\frac{\sigma_{ij}\delta_{ij}}{h_s}\right)^n\right]$$

See the basic law description for the parameters  $h_s$  and n. The  $\sigma_{ij}$  denotes the effective stress tensor (total stresses minus any groundflow pressure). This pressure dependent initial void ratio correction can be activated by **control\_materi\_plasti\_hypo\_pressure\_dependent\_void\_ratio**. After the initial void ratio has been established, the development of the void ratio is governed by volumetric compression or extension of the granular skeleton.

Niemunis visco law

For visco hypoplasticity with intergranular strains the stress rate reads:

$$\dot{\sigma}_{ij} = M_{ijkl} \dot{\epsilon}_{kl} - L_{ijkl} \dot{\epsilon}_{kl}^{vis}$$

For visco hypoplasticity the  $L_{ijkl}$  reads:

$$L_{ijkl} = f_b \hat{L}_{ijkl}$$

where

$$f_b = \frac{-\sigma_{kk}}{(1+a^2/3)\kappa}$$

where  $\kappa$  is a user specified material constant  $\kappa$  (= Butterfield's swelling index upon isotropic unloading), and *a* relates to the user specified residual (=critical) friction angle  $\varphi_c$  as:

$$a = \frac{\sqrt{3}(3 - \sin\varphi_c)}{2\sqrt{2}\sin\varphi_c}$$

The pressure normalised stiffness is:

$$\hat{L}_{ijkl} = F^2 I_{ijkl} + a^2 \hat{\sigma}_{ij} \hat{\sigma}_{kl} + b^2 (I_{ijkl} - \frac{1}{3} I_{ikjl})$$

where

$$b^2 = \frac{(1 + \frac{1}{3}a^2)(1 - 2\nu)}{1 + \nu} - 1$$

Notice that the equation for b only holds true for non-negative right-hand-side, so that puts limits on the allowed values for  $\varphi_c$  and  $\nu$ .

For visco hypoplasticity the  $M_{ijkl}$  reads:

$$M_{ijkl} = [\rho^{\chi}m_T + (1 - \rho^{\chi})m_R]L_{ijkl} + + \begin{cases} \rho^{\chi}(1 - m_T)L_{ijmn}\hat{S}_{mn}\hat{S}_{kl} & \text{for } \hat{S}_{ij}\dot{\epsilon}_{ij} > 0\\ \rho^{\chi}(m_R - m_T)L_{ijmn}\hat{S}_{mn}\hat{S}_{kl} & \text{for } \hat{S}_{ij}\dot{\epsilon}_{ij} \le 0 \end{cases}$$

where  $\hat{S}$  intergranular strains are the same as in the formulation without viscosity.

The viscosity strain rate is assumed to be:

$$\dot{\epsilon}_{ij}^{vis} = D_r \hat{m}_{ij} (\frac{1}{OCR})^{\frac{1}{I_v}}$$

where the normalised flow rule  $\hat{m}_{ij}$  is

$$\hat{m}_{ij} = \frac{m_{ij}}{\sqrt{m_{ij}m_{ij}}}$$

with

$$m_{ij} = -\left[\frac{F^2}{a^2}(\hat{\sigma}_{ij} + \hat{\sigma}^*_{ij}) + \hat{\sigma}_{kl}\hat{\sigma}_{kl}\hat{\sigma}^*_{ij} - \hat{\sigma}_{ij}\hat{\sigma}_{kl}\hat{\sigma}^*_{kl}\right]$$

The over-consolidation ratio OCR appearing in the expression for the viscous creep rate is a function of the effective stress  $\sigma_{ij}$  and of the void ratio e

$$\mathrm{OCR} = \frac{p_e}{{p_e}^+}$$

wherein the void ratio is hidden in the equivalent pressure  $p_e$  and  $p_e^+$  is a special stress invariant.

The equivalent pressure  $p_e$  is calculated from

$$\ln\left(\frac{1+e_{e0}}{1+e}\right) = \lambda \ln\left(\frac{p_e}{p_{e0}}\right)$$

with a user specified material constant  $\lambda$  (= Butterfield's first compression index) and also user-specified reference parameters  $e_{e0}$ ,  $p_{e0}$  which describe any pair of the void ratio and the effective pressure registered upon an isotropic  $D_r$ -isotach, i.e. during an isotropic first (= virgin) compression test with a constant volumetric rate of deformation equal to  $-\sqrt{3}D_r\frac{\lambda}{\lambda-\kappa}$ .

The stress invariant  $p_e^+$  is calculated using

$$p_e^{+} = \begin{cases} \frac{p}{\beta_R - 1} \left[ \beta_R \sqrt{1 + \eta^2 (\beta_R^2 - 1)} - 1 \right] & \text{if } \eta < 1\\ p(1 + \eta^2) \frac{1 + \beta_R}{2} & \text{otherwise} \end{cases}$$

wherein

$$\eta = q/(Mp) \ \, {\rm and} \ \ \, M = \frac{6F\sin\varphi_c}{3-\sin\varphi_c}$$

where  $p = -\sigma_{kk}/3$  and  $q = \sqrt{\frac{3}{2}\sigma_{kl}^*\sigma_{kl}^*}$  are the popular Roscoe's stress invariants. and  $\beta_R$  (= flattening factor for the Rendulic's cap) are the user supplied material constants.

You can specify an initial value of the void ration  $e_0$  in **-hyhis0** with **control\_reset\_dof**. Then the OCR can be calculated with the above equations. As an alternative you can specify the OCR at the start of the calculation in **group\_materi\_plasti\_hypo\_niemunis\_visco\_ocr**; then the initial void ratio will be calculated as follows:  $p_e^+$  will be determined from the equation above, then  $p_e$  is determined from  $p_e = OCRp_e^+$  and then the initial void ratio  $e_0$  is determined from  $e_0 = (1 + e_e 0) * (p_e/p_{e0})^{-\lambda} - 1$ . (reference: Niemunis communications). Application of the specified OCR is triggered by **control materi plasti hypo niemunis visco ocr apply**.

User parameters should be specified in group materi plasti hypo niemunis visco.

### Cohesion extension

A simplistic approach to include cohesion is used here. Instead of feeding the real effective stress state  $\sigma_{ij}$  into the hypoplastic law, an alternative effective stress state  $\sigma_{ij}^c$  is used. Cohesion is modeled by subtracting in each of the normal stress components a value c representing cohesion:  $\sigma_{11}^c = \sigma_{11} - c$ ,  $\sigma_{22}^c = \sigma_{22} - c$  and  $\sigma_{33}^c = \sigma_{33} - c$ . The shear stresses are not altered:  $\sigma_{12}^c = \sigma_{12}$ , etc.

The cohesion value should be specified in group materi plasti hypo cohesion.

#### Intergranular strains extension

In order to take into account the recent deformation history, an additional tensorial state variable  $S_{ij}^{1}$  is introduced.

Denoting the normalized magnitude of  $S_{ij}$ 

$$\rho = \frac{\sqrt{S_{ij}S_{ij}}}{R}$$

(R is a material parameter) and the direction of  $S_{ij}$ 

$$\hat{S}_{ij} = \frac{S_{ij}}{\sqrt{S_{kl}S_{kl}}}$$

 $(\hat{S}_{ij} = 0 \text{ for } S_{ij} = 0)$ , the evolution equation for the intergranular strain tensor reads:

$$\dot{S}_{ij} = \begin{cases} (I_{ijkl} - \rho^{\beta_r} \hat{S}_{ij} \hat{S}_{kl}) \dot{\epsilon}_{kl} & \text{for} \quad \hat{S}_{ij} \dot{\epsilon}_{ij} > 0\\ \dot{\epsilon}_{ij} & \text{for} \quad \hat{S}_{ij} \dot{\epsilon}_{ij} \le 0 \end{cases}$$

where  $\dot{S}_{ij}$  is the objective rate of intergranular strain. Rigid body rotations are treated elsewhere (see the section on memory). From the evolution equation (2.2.4) it follows that  $\rho$  must remain between 0 and 1.

 $<sup>{}^{1}</sup>S_{ij}$  is denoted  $\delta_{ij}$  in the paper [15]. However, in order to avoid confusion with Kronecker delta, another symbol is used here.

The general stress-strain relation is now written as

$$\dot{\sigma}_{ij} = M_{ijkl} \dot{\epsilon}_{kl}$$
 .

The fourth order tensor  $M_{ijkl}$  represents the incremental stiffness and is calculated from the hypoplastic tensors  $L_{ijkl}$  and  $N_{ij}$  which may be modified by scalar multipliers  $m_T$  and  $m_R$ , depending on  $\rho$  and on the product  $\hat{S}_{ij}\dot{\epsilon}_{ij}$ :

$$\begin{split} M_{ijkl} &= [\rho^{\chi} m_{T} + (1 - \rho^{\chi}) m_{R}] L_{ijkl} + \\ &+ \begin{cases} \rho^{\chi} (1 - m_{T}) L_{ijmn} \hat{S}_{mn} \hat{S}_{kl} + \rho^{\theta} f_{d} N_{ij} \hat{S}_{kl} & \text{for} \quad \hat{S}_{ij} \dot{\epsilon}_{ij} > 0 \\ \rho^{\chi} (m_{R} - m_{T}) L_{ijmn} \hat{S}_{mn} \hat{S}_{kl} & \text{for} \quad \hat{S}_{ij} \dot{\epsilon}_{ij} \le 0 \end{cases} \end{split}$$

 $\chi$  and  $\theta$  are additional material parameters.

An example intergranular parameters can be found in Tab. 3.

R	$m_R$	$m_T$	$\beta_r$	$\chi$	$\theta$
$1 \cdot 10^{-4}$	5.0	2.0	0.50	6.0	6.0

 Table 3: Example of Intergranular hypoplastic parameters.

The intergranular parameters should be specified in **group \_materi \_plasti \_hypo \_strain \_intergranular**. Additionally you need to include **materi strain intergranular** in the initialisation part.

The additional parameter theta is very important only for the accumulation of permanent displacements or pore pressures in cyclic or dynamic analysis with small strains. For monotonic loading or higher strains theta is not very important. And thus for such monotonic loading or higher strains you should take  $\theta = \chi$ .

### ISA-Intergranular strains extension

Author: William Fuentes

The ISA plasticity [6] stands from Intergranular Strain Anisotropy and refers to a mathematical platform which allows to formulate new constitutive models or to extend existing hypoplastic models for soils. This family of models enables the simulation of small strain effects, such as the increase of stiffness and the reduction of the plastic accumulation under repetitive loops. Similar to the original intergranular strain concept by Niemunis and Herle (1996), it allows to couple existing hypoplastic relations to extend their capabilities for cyclic loading. However, the ISA plasticity contrasts with the previous formulation with the incorporation of an elastic locus related to a specific strain amplitude. This new feature makes its formulation elastoplastic, but the yield surface is now defined within the intergranular strain space. The general stress-strain relation is now written according to:

$$\dot{\sigma_{ij}} = M_{ijkl} \dot{\epsilon}_{kl}$$

whereby  $M_{ijkl}$  is the stiffness tensor. The yield surface F of the model is defined within the space of the intergranular strain  $S_{ij}$  and distinguishes elastic conditions F < 0 from plastic conditions F = 0. The formulation of  $M_{ijkl}$  reads:

$$M_{ijkl} = \begin{cases} m(L_{ijkl} + \rho^x f_d N_{ij} \hat{n}_{kl}) & \text{for } F \ge 0\\ m_R L_{ijkl} & \text{for } F = 0 \end{cases}$$

whereby m,  $\rho^x$ ,  $f_d$  and  $m_R$  are scalar factors,  $N_{ij}$  is the non-linear stiffness tensor,  $L_{ijkl}$  is the linear stiffness tensor and  $\hat{n}_{kl}$  is the intergranular strain model flow rule. The ISA model proposes an evolution equation for the intergranular strain  $S_{ij}$  with an elastoplastic formulation:

$$\dot{S}_{ij} = \dot{\varepsilon}_{ij} - \dot{\lambda}_H \hat{n}_{ij}$$

whereby  $\dot{\lambda}_H$  is a plastic multiplier ( $\dot{\lambda}_H = 0$  for F < 0). The model may be enhanced to capture the plastic accumulation for larger number of cycles with an additional state variable  $\varepsilon_{\text{acc}}$ . This variable is able to distinguish between consecutive cycles or non-consecutive cycles [7]:

$$\varepsilon_{\rm acc} = \varepsilon_{\rm acc} + \frac{C_a}{R} (1 - y_h - \varepsilon_{\rm acc}) \sqrt{\Delta \varepsilon_{ij} \Delta \varepsilon_{ij}}$$

The parameter  $C_a$  controls the rate at which the plastic accumulation reduces upon consecutive cycles. The exponent  $\chi$  is increased to  $\chi_{\max}$  to produce this effect on the constitutive equation.

Typical parameters are  $\chi_{\text{max}} = 20$  and  $C_a = 0.017$ . See also **group\_materi\_plasti\_hypo\_strain\_isa**.

### 2.2.5 Damage

In the presence of materi damage d, the materi stress follows:

$$\sigma_{ij}^{\text{damaged}} = (1 - d)\sigma_{ij}^{\text{undamaged}}$$

For the damage, the group materi damage mazars model is available:

$$d = d_t \alpha^{\beta} + d_c (1 - \alpha)^{\beta}$$

where

$$d_t = 1. - (1 - a_t) \frac{\epsilon^0}{\epsilon^{\mathrm{eq}}} - a_t \ e^{-b_t(\epsilon^{\mathrm{eq}} - \epsilon^0)}$$

 $\operatorname{and}$ 

$$d_c = 1. - (1 - a_c) \frac{\epsilon^0}{\epsilon^{eq}} - a_c e^{-b_t(\epsilon^{eq} - \epsilon^0)}$$

Here  $\epsilon^{\text{eq}}$  contains the positive principal strains. The parameter  $\alpha$  is given by the ratio  $\frac{\epsilon^{\text{eq}}}{\epsilon}$ , where  $\epsilon$  contains the total strains (both negative and positive). The parameter  $\epsilon^0$  is the strain threshold for damage; other material parameters are  $\beta$ ,  $a_t$ ,  $b_t$ ,  $a_c$ ,  $b_c$ . Typically for concrete:

 $1.e-4 < \epsilon^0 < 3.e-4 \ ; \ \beta = 1. \ ; \ 1 < a_t < 1.5 \ ; \ 500 < b_c < 2000 \ ; \ 0.7 < a_c < 1.2 \ ; \ e^4 < b_t < 5e^4$ 

You can combine damage freely with plasticity models or other material behavior.

### 2.2.6 Average stress (hydrostatic compressibility)

An extra average stress contribution on each of  $\sigma_{11}$ ,  $\sigma_{22}$  and  $\sigma_{33}$  is

$$\frac{1}{co}\frac{\partial v_i}{\partial x_i}$$

where *co* is the **group\_materi\_elasti\_compressibility**, which should not be 0. This pressure term can e.g. be used to model nearly incompressible fluids. The compressibility contribution should be combined with a contribution for the deviatoric stresses (e.g. **group materi viscosity**).

### 2.2.7 Undrained groundflow analysis

In case you want to perform an undrained groundflow analysis, but do not want to have both the material velocity and groundflow equations at the same time in system matrix, you can use **group\_materi\_undrained\_capacity**. Then the following equation will be used to determine the total groundwater pressure changes in an element:

$$C \ \dot{p}_{total} = \frac{\partial v_i}{\partial x_i}$$

which actually is the groundflow storage equation without permeability. The above equation can be solved on an element-by-element level, so that the groundflow hydraulic pressure head and the storage equation do not need to be added to the complete system matrix. The capacity C should be specified in **group\_materi\_undrained\_capacity**. Results for the pressure in a element will be written to **element\_intpnt\_materi\_undrained\_pressure**. Application of this undrained analysis can be switched off and on with **control\_materi\_undrained\_apply**.

This option is convenient to prevent the need for large, and ill-conditioned, system matrices in coupled soil - groundwater analysis. Typically the computational strategy may be like this:

```
...
(include capacity for undrained analysis in relevant groups)
group_materi_undrained_capacity ...
...
(set the hydraulic pressure heads, and fix them for the remainder of the calculation)
control_reset_dof ...-pres
bounda_dof ...-tpres
...
(solve material displacements in the remainder of the calculation)
control_timesteps ...
control_materi_undrained_apply ... -yes
...
```

The advantage of the above computational strategy is that never a system matrix with both material velocities and groundflow pressures needs to be solved. When solving the remainder of the calculation Tochnog uses the fixed total pressure from the hydraulic pressure heads plus the excessive undrained pressure of the remainder of the calculation as the full total pressure (when determining total stresses from effective stresses plus full total pressure). Alternatively to setting the hydraulic pressure head at the start with the **control\_rest\_dof**, you can also solve the gravity state for hydraulic pressure heads and material displacements (at the expense of a system matrix with both material velocities and groundflow hydraulic pressures in this gravity calculation; but in the gravity calculation only and the remainder of the calculation).

### 2.2.8 Thermal stresses

Temperature rates cause fictitious thermal strain rates

 $-\alpha \dot{T} \delta_{ij}$  where  $\delta_{ij} = 1$  if i = j else  $\delta_{ij} = 0$ 

where  $\alpha$  is the **group \_\_materi \_\_expansion \_\_linear** coefficient and  $\dot{T}$  is the **condif \_\_temperature**. These fictitious thermal strain rates in turn lead to stress rates.

### 2.2.9 Hyper elasticity

Hyper elasticity is used to model rubbers. It should be combined with a total Lagrange formulation for the memory of the material (so use **-total** for **group materi memory**).

The stresses follow from a strain energy function (with  $C_{ij}$  components of the matrix C, and where F is the deformation tensor and U is the stretch tensor following from the polar decomposition of the deformation tensor)

$$2\frac{\partial W}{\partial C_{ij}}$$
$$C = F^T F = U^T U$$

**Deviatoric** contributions

To obtain a purely deviatoric function, the following strain measures are used (with  $I_1$ ,  $I_2$  and  $I_3$  the first, second and third invariant of the elastic strain matrix C respectively)

$$J_1 = I_1 I_3^{\frac{-1}{3}} \quad J_2 = I_2 I_3^{\frac{-2}{3}}$$

The group\_materi\_hyper\_besseling function reads ( with  $K_1$ ,  $K_2$  and  $\alpha$  user defined constants)

$$W = K_1(J_1 - 3)^{\alpha} + K_2(J_2 - 3)$$

The group\_materi\_hyper\_mooney\_blatz\_ko function reads (with G and  $\beta$  user defined constants)

$$W = G * 0.5 * (I_1 - 3.0 + (2.0/\beta)(J^{-\beta} - 1.));$$

This Blatz-Ko hyperelastic material hardens in compression, and softens slightly in tension; it models a foamlike rubber.

The group materi hyper mooney rivlin function reads (with  $K_1$  and  $K_2$  user defined constants)

$$W = K_1(J_1 - 3) + K_2(J_2 - 3)$$

The group\_materi\_hyper\_neohookean function reads (with  $K_1$  a user defined constant)

$$W = K_1(J_1 - 3)$$

The group materi hyper reduced polynomial function reads (with  $K_i$  user defined constants)

$$W = K_i (J_1 - 3)^i$$

where a summation over  $i = 1, 2, \ldots$  is applied.

### Volumetric contributions

First we define  $J = \sqrt{I_3}$ . Then a volumetric part can be added to the strain energy. The group materi hyper volumetric linear contribution reads:

$$W = \frac{K}{2}(J-1)^2$$

The group materi hyper volumetric murnaghan contribution reads:

$$W = \frac{K}{\beta} \left(\frac{1}{\beta - 1}J^{-\beta} + 1\right)J$$

The group materi hyper volumetric polynomial contribution reads:

$$W = \frac{K_i}{2}(J-1)^{2i}$$

for i = 0, 1, ...

The group materi hyper volumetric simo taylor contribution reads:

$$W = \frac{K}{2}((J-1)^2 + (lnJ)^2)$$

The group materi hyper volumetric ogden contribution reads:

$$W = \frac{K}{\beta} \left(\frac{1}{\beta} (J^{-\beta} - 1) + lnJ\right)$$

As an example, you can combine the **group\_materi\_hyper\_mooney\_rivlin** energy function with the **group\_materi\_hyper\_volumetric\_linear** so that the total strain energy function becomes:

$$W = K_1(J_1 - 3) + K_2(J_2 - 3) + \frac{K}{2}(J - 1)^2$$

Here the initial shear modulus and bulk modulus are included as:

initial shear modulus 
$$= 2(K_1 + K_2)$$

### initial bulk modulus = K

respectively.

### 2.2.10 Viscoelasticity

Viscoelasticity is modeled with n parallel **group\_materi\_maxwell\_chain**. Each of the chains contains a spring with stiffness  $E^m$  in line with a dash pot with relaxation time  $t^m$  (m indicates the m-th maxwell chain). The viscoelastic stress rate is given by (with  $C^m_{ijkl}$  is the elastic tensor modulus of the m-th maxwell chain (depending on  $E^m$  and the poisson ratio))

$$\sum_{m=0}^{m=n-1} (C_{ijkl}{}^m \dot{\epsilon_{kl}}^{\text{elas}} - \frac{\sigma_{ij}{}^m}{t^m})$$

### 2.2.11 Viscoplasticity

Viscoplasticity is a model for rate-dependent plasticity. Rate dependent plasticity is important for (high-speed) transient plasticity calculations. It should be used in combination with a plasticity law. Viscoplasticity influences the stresses via the plastic strains.

The group materi plasti visco exponential model reads

$$\dot{\epsilon_{kl}}^{\text{plas}} = \gamma \ p \ e^{\alpha f} \ \frac{\partial f^{\text{flow}}}{\partial \sigma_{kl}}$$

where  $\gamma$  and  $\alpha$  are material fluidity constants and p is the pressure. In case the  $\alpha f$  becomes larger than a limit, it is substituted by the limit to prevent the exponent from becoming excessive large. You can set the limit with the **group\_materi\_plasti\_visco\_exponential\_limit** record. This model was first developed for visco-plastic soil behavior.

The group materi plasti visco power model reads

$$\dot{\epsilon_{kl}}^{\text{plas}} = \eta(f)^p \ \frac{\partial f^{\text{flow}}}{\partial \sigma_{kl}}$$

where  $\eta$  (fluidity constant), and p (power) are user specified parameters.

 $\operatorname{and}$ 

### 2.2.12 Viscosity

The viscous contribution to the total stress is

 $2\nu D_{ij}$ 

where

$$D_{ij} = 0.5\left(\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i}\right)$$

and divergence is neglected since we only model slightly compressible flows. Viscous heat generation

The viscous energy loss is turned into heat rate per unit volume q:

$$q = 2\nu D_{ij} D_{ij}$$

See group materi viscosity heatgeneration.

### 2.3 Contact analysis

### 2.3.1 Penalty formulation

Attention: the contact algorithm is experimental and needs more testing.

In contact analysis, normal forces  $F_n$  follow from the condition that bodies cannot penetrate each other. Since we use a penalty formulation, the normal force is given by

$$F_n = \lambda u_n$$

where u is the penetration and  $\lambda$  is called the **contact\_penalty\_velocity** because its generates forces on the velocity dof's. You can also impose **groundflow\_pressure** and **condif\_temperature** contact conditions by specifying the penalty factors **contact\_penalty\_pressure** and **contact\_penalty\_temperature**.

The contact algorithm is presently under development, and not guruanteed to work for all calculations. You must check your results.

### 2.3.2 Friction and frictional heat generation

This normal force leads to a friction force  $F_f$  which equals

$$F_f = \nu F_n$$

where  $\nu$  is the *friction* coefficient (see **contact\_plasti\_friction**. The friction force causes heat generation rate Q:

$$Q = \eta F_f v_f$$

where  $v_f$  is the slip velocity, and the factor  $\eta$  is a user specified factor which determines which part of the frictional energy loss is transformed into heat ( $\eta$  is between 0 and 1; see **contact heat generation**).

### 2.4 Ground water flow

### 2.4.1 Storage equation for fully saturated analysis

The equations in this section come from [22]. The hydraulic pressure head h follows from the storage equation (with infinite stiffness solid particles):

$$c \dot{h} = (k_1^p \frac{\partial^2 h}{\partial x_1^2} + k_2^p \frac{\partial^2 h}{\partial x_2^2} + k_3^p \frac{\partial^2 h}{\partial x_3^2}) + \frac{\partial v_i}{\partial x_i} - \alpha \dot{T} + f$$

Primary dof is the hydraulic pressure head h, which is initialised with **groundflow\_pressure** in the initialisation part, and which gets a label **-pres** in the **node\_dof** records. Further notation: c **group\_groundflow\_capacity** (storativity);  $k_i^p$  **group\_groundflow\_permeability** in *i*-direction (permeability);  $x_i$  space coordinate;  $v_i$  material velocity (if present);  $\alpha$  **group\_groundflow\_expans** is the expansion coefficient of the groundwater for temperature changes. The equation is given for space coordinates following material velocities  $v_i$  (if present). The water mass acceleration (water newton inertia) is neglected in the above equation. See the naming conventions later how to determine the permeability from experimental data.

### Groundflow velocities

The groundflow velocities, after initializing groundflow velocity, follow from:

$$v_i{}^{\mathrm{g}} = k_i^p \frac{\partial h}{\partial x_i}$$

#### Total groundwater pressure

The total groundwater pressure, or pore-pressure, is for example needed to calculate the total stresses in soils. The total groundwater pressure follows from:

$$p_{\text{total}} = h - \rho g z$$

where g is the gravitational acceleration, and  $\rho$  is the **groundflow\_density** (Please notice that g and z typically are negative numbers).

Tochnog considers pressure a pore pressure of p = 0, or positive, as indication that there is in fact no water pressure, so the porous soil skeleton is filled with air. In this case, the total soil stress is only composed by the effective stress of the soil skeleton.

The total stress in saturated soils follows from: total soil stress = effective soil stress + total groundwater pressure (pore pressure). This will only be done for isoparametric finite elements which have groundflow data specified.

For non-saturated soild you optionally can specify **groundflow** pressure factor such that the total groundwater pressure will be added only with this specified factor.

### Static groundwater pressure

The static pressure due to gravity is:

$$p_{\text{static}} = \rho g \Delta z$$

where the  $\Delta z$  is the distance to the groundwater level, the phreatic level. The phreatic level needs to be specified with the groundflow\_phreatic\_level record. Alternatively you can specify post\_calcul\_static\_pressure\_height or post\_calcul\_static\_pressure. If all of these are not specified, the static pressure cannot be determined, so it remains zero.

### Dynamic groundwater pressure

The dynamic groundwater pressure follows from

 $p_{\rm dynamic} = p_{\rm total} - p_{\rm static}$ 

### **Boundary conditions**

If the groundwater velocity is 0 normal to an edge (say at the interface with a rock layer it is zero), then you should prescribe nothing on that edge (Tochnog will then take care of that boundary conditions for you).

At the phreatic level where the groundflow meets free air the hydraulic pressure head should become  $\rho gz$ . You can either set this yourself by using **bounda\_dof** combined with **bounda\_time** or else demand that Tochnog automatically does it for you by activating the option **groundflow phreatic bounda**.

At edges where you have some other hydraulic pressure head you need to specify that head yourself with **bounda\_dof** and **bounda\_time** records. If you want to set directly the hydraulic head you need to use **- pres** in **bounda\_dof**, however if you want to set the total pressure (pore pressure) you need to use **- topres** in **bounda\_dof**.

If gravity is not of importance, e.g. in biomechanics where the storage equation is used to model fluid transport in soft tissues, the hydraulic pressure head h is equal to the total pressure, and thus is zero at edges where the water meets the free air. In this case, set h to zero by using **bounda dof** combined with **bounda time**.

### Postprocessing

For all printing, plotting etc. you normally get the hydraulic pressure head h since it is the primary dof solved in the storage equation. The total pressure, static pressure and dynamic pressure are obtained using the **post** calcul option.

### Capacity and permeability explanation

Following conventional naming, we remind the user that the capacity depends on the porosity n and water compressibility  $\beta$ :

 $c = n \beta$ 

and the permeability  $k^p$  used in the storage equation above you can determine from an experiment with

$$k^p = \frac{k_{experiment}}{\rho |g|}$$

where  $k_{experiment}$  is the experimental value in units  $\frac{length}{time}$  (by example  $\frac{m}{s}$  in SI units).

### 2.4.2 Non-saturated analysis

### with diagrams

You can perform a non-saturated analysis by making the permeability dependent on the groundwater total pressure (= pore pressure) by a dependency diagram. The diagram accounts for high permeability at saturation and low permeability at non-saturation. For example, do something like:

```
dependency_item 10 -group_groundflow_permeability 0 -to_pres 4
dependency_diagram 10 -100. 0.0 0.05 100.
1.e-2 1.e-2 1.e-8 1.e-8
1.e-2 1.e-2 1.e-8 1.e-8
....
```

The atmospheric air pressure is 0, so that is where the permeability starts changing it's value in the table. You can also specify a table for **group\_groundflow\_capacity** to model non-saturated capacity.

#### van Genuchten

As an alternative to specifying diagrams you can use the specific van-Genuchten model for non-saturated ground water flow. The pore-pressure head is defined by

$$\phi_p = -\frac{p}{\rho g}$$

with p the pore pressure (= total pressure),  $\rho$  the ground water density and g is the absolute value of the gravity acceleration (typically 9.81). De degree of saturation is a function of the pore-pressure head

 $S = S(\phi_p)$ 

The total capacity is the sum of the saturated capacity and a non-saturated part:

$$c = c_{\rm sat} + n \frac{dS(\phi_p)}{d\phi_p}$$

where where  $c_{\text{sat}}$  is the saturated groundflow capacity as specified by **group\_groundflow\_capacity** and *n* is the porosity specified by **group\_porosity**. The total permeabilities  $k_i$  are written as a relative factor of the saturated permeabilities

$$k_i = k_{\rm rel}(S)k_{sat,i}$$

where  $k_i$  is the total permeability in direction i,  $k_{rel}(S)$  is a factor dependent on the saturation S and  $k_{sat,i}$  is the saturated permeability specified by **group groundflow permeability**.

Now for the van-Genuchten model we have

$$S(\phi_p) = S_{\text{residu}} + (S_{\text{sat}} - S_{\text{residu}}) (1 + (g_a |\phi_p|)^{g_n})^{(1-g_n)/g_n}$$

which has the following model parameters:  $S_{\text{residu}}$  is the residual saturation,  $S_{\text{sat}}$  normally is 1.0 but may be less than 1.0 if in case of trapped air, and  $g_a$  and  $g_n$  are constants to be determined experimentally. The derivative of this law defines the additional non-saturated capacity as defined above. After definition of the effective saturation  $S_e$ 

$$S_e = \frac{S - S_{\text{residu}}}{S_{\text{sat}} - S_{\text{residu}}}$$

the relative permeability factor is defined as

$$k_{\rm rel}(S) = (S_e)^{g_l} \left( 1 - (1 - S_e^{g_n/(g_n - 1)})^{(g_n - 1)/g_n} \right)^2$$

To use the model you need to specify the saturated parameters group \_groundflow \_capacity and group \_groundflow \_permeability as usual, specify the porosity in group \_porosity, specify specific van-Genuchten parameters in group \_groundflow \_nonsaturated \_vangenuchten and initialise groundflow \_saturation in the initialisation part.

### 2.4.3 Consolidation analysis

Look in the 'Consolidation' section of the 'Interaction analyzes and advanced analyzes' chapter in the end of this manual on how to perform consolidation analyzes (combined groundwater flow with soil stress analyzes).

In case you have **groundflow\_total\_pressure\_limit** set to 0 and the total pressure is 0, then Tochnog assumes that there is no water so the consolidation part in the equations will also be skipped. In case you have **groundflow\_total\_pressure\_limit** set to a high positive value this will not be done, so the consolidation part will also be used in case the total pressure is 0 (or positive).

# 2.5 Wave equation

$$\frac{\partial \dot{s}}{\partial t} = c^2 \left(\frac{\partial^2 s}{\partial x_1^2} + \frac{\partial^2 s}{\partial x_2^2} + \frac{\partial^2 s}{\partial x_3^2}\right)$$
$$\frac{\partial s}{\partial t} = \dot{s}$$

The primary dof's are the **wave\_scalar** s and its first time derivative **wave\_fscalar**  $\dot{s}$  (as TOCHNOG only solves first order in time equations, the first time derivative of s also becomes primary dof in order to turn this second order in time equation into a set of first order in time equations). Further notation: x space coordinate, t time and c speed of sound.

### 2.6 Probabilistic distributions

The section summarises mathematical formulation of the so-called random finite element method, as described, e.g. in [8].

Distribution of a random variable (e.g., C) is controlled by these basic parameters: parameters of the statistical distribution (typically mean value  $\mu_C$  and standard deviation  $\sigma_C$ ) and so-called correlation length  $\theta_C$  that controls spatial variability of variable C.

Two probabilistic distributions are available in Tochnog: normal distribution and log-normal distribution. Probability function P(C) of normal distribution is defined as:

$$P(C) = \frac{1}{\sigma_C \sqrt{2\pi}} \exp\left[-\frac{\left(C - \mu_C\right)^2}{2\sigma_C^2}\right]$$
(22)

where  $\mu_C$  is a mean value and  $\sigma_C$  is standard deviation. Probability function P(C) of log-normal distribution is defined as:

$$P(C) = \frac{1}{C\sigma_{\ln C}\sqrt{2\pi}} \exp\left[-\frac{\left(\ln C - \mu_{\ln C}\right)^2}{2\sigma_{\ln C}^2}\right]$$
(23)

Quantities  $\mu_{\ln C}$  and  $\sigma_{\ln C}$  may be calculated from  $\mu_C$  and  $\sigma_C$  using

$$\sigma_{\ln C} = \sqrt{\ln\left[1 + \left(\frac{\sigma_C}{\mu_C}\right)^2\right]} \qquad \qquad \mu_{\ln C} = \ln\mu_C - \frac{1}{2}\sigma_{\ln C}^2 \qquad (24)$$

### 2.6.1 Generation of random field

A number of different techniques to generate random fields is available (see, e.g., [5]). In this following, the most simple method based on Cholesky decomposition of the correlation matrix.

First, vector **X** of statistically independent random numbers  $x_1, x_2, \ldots, x_n$  (where *n* is number of elements in the FE mesh) with a standard normal distribution (i.e., with probability function of Eq. (22) with  $\mu_C = 0$  and  $\sigma_C = 1$ ) is generated.

A correlation matrix  $\mathbf{K}$ , which represents the correlation coefficient between each of the element used in the finite element analysis, is assembled. The correlation matrix  $\mathbf{K}$  has the following form:

$$\mathbf{K} = \begin{bmatrix} 1 & \rho_{12} & \dots & \rho_{1n} \\ \rho_{21} & 1 & \dots & \rho_{2n} \\ \vdots & \vdots & \ddots & \vdots \\ \rho_{n1} & \rho_{n2} & \dots & 1 \end{bmatrix}$$
(25)

where  $\rho_{ij}$  is the correlation coefficient between elements *i* and *j*, calculated using Markov function:

$$\rho_{ij} = \exp\left[-\frac{2x_{ij}}{\theta_C}\right] \tag{26}$$

where  $x_{ij}$  is absolute distance between elements *i* and *j* (distance between centers of gravity of elements *i* and *j*). For anisotropic case Eq. (26) reads

$$\rho_{ij} = \exp\left[-2\sqrt{\left(\frac{\tau_{xij}}{\theta_{Cx}}\right)^2 + \left(\frac{\tau_{yij}}{\theta_{Cy}}\right)^2 + \left(\frac{\tau_{zij}}{\theta_{Cz}}\right)^2}\right]$$
(27)

where  $\theta_{Cx}$  is a correlation coefficient in direction of x-axis and  $\tau_{xij}$  is a distance between two elements *i* and *j* in *x* direction. The same notation applies for *y* and *z* directions.

The matrix **K** is positive definite and hence, the standard Cholesky decomposition algorithm can be used to factor the matrix into upper and lower triangular forms, **S** and **S**<sup>T</sup>, respectively:

$$\mathbf{S}^T \mathbf{S} = \mathbf{K} \tag{28}$$

The vector of correlated random variables **G** (i.e.,  $G_1, G_2, \ldots, G_n$ , where  $G_i$  specifies the random component of variable C in element i) is calculated by

$$\mathbf{G} = \mathbf{S}^T \mathbf{X} \tag{29}$$

Vector  $\mathbf{X}$  is generated as described above.

Finally, value of the variable C is assigned to each element  $(C_i)$  by the following transformation:

• for normally distributed variable C:

$$C_i = \mu_C + \sigma_{CA} G_i \tag{30}$$

where  $\sigma_{CA}$  is calculated from  $\sigma_{C}$  as described in the following section.

• for log-normally distributed variable C:

$$C_i = \exp\left(\mu_{\ln C} + \sigma_{\ln CA}G_i\right) \tag{31}$$

where  $\mu_{\ln C}$  is calculated by Eq. (24)b using  $\sigma_{\ln CA}$  instead of  $\sigma_{\ln C}$ ;  $\sigma_{\ln CA}$  is calculated from  $\sigma_{\ln C}$  as described in the following section.

### 2.6.2 Local averaging

The input parameters of C that relate to the mean, standard deviation and spatial correlation length are assumed to be defined at the point level. Due to the finite size of each finite element, point statistical distribution must be averaged over the element. This results in reduced  $\sigma_{\ln C}$  in the case of log-normal distribution and reduced  $\sigma_C$  in the case of normal distribution.  $\mu_{\ln C}$  in the first case and  $\mu_C$  in the second case remain unaffected.

The locally-averaged standard deviations ( $\sigma_{\ln CA}$ ,  $\sigma_{CA}$ ), which are used in Eqns. (30, 31), are calculated from their point values using

$$\sigma_{\ln CA}^2 = \gamma \ \sigma_{\ln C}^2 \qquad \qquad \sigma_{CA}^2 = \gamma \ \sigma_{C}^2 \tag{32}$$

where  $\gamma$  is the variance reduction factor calculated by integration of the Markov function (26). In 1D for a finite element of side length  $\alpha \theta_C$ 

$$\gamma = \frac{2}{\left(\alpha\theta_C\right)^2} \int_0^{\alpha\theta_C} \exp\left(-\frac{2}{\theta_C}\sqrt{x^2}\right) (\alpha\theta_C - x) dx \tag{33}$$

In 2D for square finite element of side length  $\alpha \theta_C$ 

$$\gamma = \frac{4}{(\alpha\theta_C)^4} \int_0^{\alpha\theta_C} \int_0^{\alpha\theta_C} \exp\left(-\frac{2}{\theta_C}\sqrt{x^2 + y^2}\right) (\alpha\theta_C - x)(\alpha\theta_C - y)dxdy \tag{34}$$

In 3D for hexahedral finite element of side length  $\alpha \theta_C$ 

$$\gamma = \frac{8}{(\alpha\theta_C)^6} \int_0^{\alpha\theta_C} \int_0^{\alpha\theta_C} \int_0^{\alpha\theta_C} \exp\left(-\frac{2}{\theta_C}\sqrt{x^2 + y^2 + z^2}\right) (\alpha\theta_C - x)(\alpha\theta_C - y)(\alpha\theta_C - z)dxdydz$$
(35)

For the anisotropic case in 2D:

$$\gamma = \frac{4}{l^4} \int_0^l \int_0^l \exp\left[-2\sqrt{\left(\frac{x}{\theta_{Cx}}\right)^2 + \left(\frac{y}{\theta_{Cy}}\right)^2}\right] (l-x)(l-y)dxdy \tag{36}$$

and for the anisotropic case in 3D:

$$\gamma = \frac{8}{l^6} \int_0^l \int_0^l \int_0^l \exp\left[-2\sqrt{\left(\frac{x}{\theta_{Cx}}\right)^2 + \left(\frac{y}{\theta_{Cy}}\right)^2 + \left(\frac{z}{\theta_{Cz}}\right)^2}\right] (l-x)(l-y)(l-z)dxdydz$$
(37)

In order to calculate the variance reduction due to local averaging correctly, all elements in the mesh should be of the same size and all elements should be regular squares. If irregular elements are used, exact value of  $\gamma$  is in Tochnog approximated by calculation of  $\gamma$  for an equivalent square element using Eq. (35) with area equal to an average area of all elements in the mesh.

The approximate value of  $\gamma$  requires that you use as much as possible elements of the same size and shape in the complete calculation domain.

# 2.6.3 Monte Carlo simulations

The most simple but very powerful technique to solve the probabilistic problem is a Monte Carlo technique. The same problem is solved many times, each time with different fields of random variables generated according to prescribed parameters.

The whole problem is solved in the following steps:

1. Generate random fields according to Sec. 2.6.1 using **control\_distribute** command as many times as many variables are treated as random. In principle, any variable can be related as random. For example material parameters, dof's (e.g., history variables), etc.

- 2. Solve the problem using finite element method. Collect required results of each Monte Carlo realisation into an output file. The user can prescribe any result to be collected into an output file using **control\_repeat\_save** command (e.g., **time\_current**, final displacement of a selected point, etc.).
- 3. Repeat items 1. and 2. m-times, where m is a prescribed number of Monte Carlo realisations. m value is specified in Tochnog input file using **control** repeat command.
- 4. Evaluate results statistically. More complex statistical evaluation is done by the user, calculation of mean value and standard deviation can be done in Tochnog using **control repeat save calculate** command.

### 2.6.4 Input data records

A typical piece of input file could be like this:

```
print group data ... (print in the gid files distributed group data so that
you get a plot of it)
control distribute 10 ... (distribute something with correlation in space)
control distribute parameters 10 ...
control distribute correlation length 10 ...
control distribute 20 ... (distribute something else without correlation in
space)
control distribute parameters 20 ...
. . .
control timestep 30 ... (do timesteps)
control timestep iterations automatic 30 ... (with automatic timestep-
ping)
 {\bf control \ timestep \ iterations \ automatic \ stop \ 30 \ -continue \ (don't \ abort
the calculation if the minimum step size is reached, e.g. in a stability calcula-
tion)
. . .
control print data versus data 40 ... (save data for repeats in a dvd
file)
control repeat 50 100 10 (jump 100 times back to control index 10)
control repeat save 50 ... (select results to be saved for each repeat)
control repeat save calculate 50 -yes (perform statistical analysis on
saved results)
. . .
control_print gid 100 -yes
control print 100-repeat save result-repeat save calculate result
. . .
```

# 3 Input file, general remarks

The input is free format. Comments are enclosed between ( ), e.g. (this is comment only); a comma , is not allowed inside comments. The input should consist of an initialization part and a data part, separated by end initia and ended by end data

initialization
...
initialization
end\_initia
data\_item index data\_values
...
data\_item index data\_values
end\_data

**Bold** printed data in this manual can be used literally. *Italic* printed data is only symbolic (it represents a number or a word).

# 4 Input file, initialization part

The initialization part contains initialization records and an end initia record

initialization .... initialization end initia

The example below initializes a solid material

```
echo -yes
number_of_space_dimensions 2
materi_velocity
materi_strain_total
materi_stress
end initia
```

The echo (always the first record), number of space dimensions (always the second record), and end initia record should be supplied always. Use echo -yes to echo the input and echo -no to not echo the input. Use number of space dimensions 1 for 1D problems, etc.. The records materi velocity, materi strain total and materi stress create a velocity, strain and stress field in the entire domain. In the following sections, all possible *initialization* records are discussed. Most of these records create an doffield, a physical field like a temperature field or a strain field, over the computational domain.

# 4.1 echo switch (first record of initialization part)

If *switch* is **-yes** the input will be echoed. If *switch* is **-no** the input will not be echoed. This needs to be the first record.

# 4.2 number\_of\_space\_dimensions number\_of\_space\_dimensions (second record of initialization part)

Set number of space dimensions to 1 in 1D, etc.. This needs to be the second record.

### 4.3 derivatives (third record of initialization part, if specified)

If this record is included, the time derivative and the space derivatives will be stored in the **node\_dof** records. This is only required for a limited number of models. The model description will specify if this **derivatives** initialization is needed.

# 4.4 beam rotation

The beam rotations  $\phi_x$ ,  $\phi_y$  and  $\phi_z$  are added to the **node\_dof** records.

Please notice that always all three rotations are included. Typically for a 2D calculation you may want to fix the  $\phi_x$  and  $\phi_y$  to 0, by using a **bounda** dof record.

# 4.5 condif temperature

The temperature T is added to the **node** dof records.

# 4.6 groundflow pressure

The pressure p is added to the **node** dof records.

# 4.7 groundflow pressure gradient

The gradient of the hydraulic pressure  $\frac{dh}{dx}\frac{dh}{dy}\frac{dh}{dz}$  is added to the **node\_dof** records.

# 4.8 groundflow saturation

The groundflow saturation S is added to the **node** dof records.

# 4.9 groundflow velocity

The ground water flow velocity  $v_i^{g}$  is added to the **node** dof records.

# 4.10 materi damage

The damage *d* is added to the **node\_dof** records. Also **materi\_velocity** and **materi strain total** should be initialized.

# 4.11 materi acceleration

The accelerations  $a_i$  are added to the **node** dof records.

# 4.12 materi displacement

The displacements u, v, w are added to the **node\_dof** records. If **materi\_displacement** is initialized, then equations like the convection and diffusion of heat equation or the ground water flow equation are evaluated over the deformed volume, which is the sum of the nodal coordinates plus its displacements. Also if **materi\_displacement** is initialized, the total Lagrange model will be used in stress analysis.

Condition: also **materi**\_**velocity** should be initialized because the displacement follows from integration of the velocity.

# 4.13 materi displacement relative

Displacement relative to a previous point in the calculation. These are the current displacements minus the displacements before these were changed with timesteps in **control timestep** or a displacement reset in **control reset dof**.

For example, this option comes handy when you want to understand the extra displacements caused by the last timesteps.

# 4.14 materi history variable number\_of\_variables

Generic history variables which can e.g. be used in some user supplied routines or otherwise.

# 4.15 materi maxwell stress number\_of\_chains

Maxwell stress  $\sigma_{11}^{m} \sigma_{12}^{m} \sigma_{13}^{m} \sigma_{22}^{m} \sigma_{33}^{m} \sigma_{33}^{m}$  is added to the **node\_dof** records. The parameter *number\_of\_chains* should match data item **group\_materi\_maxwell\_chain**. The number of maxwell stresses is 6 \* *number\_of\_chains*.

# 4.16 materi\_plasti\_camclay\_history

The history variables  $e_0$  and  $p_0$  for the camclay plasticity models are added to the **node** dof records.

# 4.17 materi plasti cap1 history

The history variable  $p_c$  for the cap1 plasticity models is added to the **node** dof records.

# 4.18 materi plasti diprisco history number\_of\_history\_variables

The history variable di Prisco plasticity models are added to the **node \_dof** records. For the **group \_materi \_plasti \_diprisco** model you need to set *number\_of\_history\_variables* to 11. For the **group \_materi \_plasti \_diprisco \_density** model you need to set *number of history variables* to 12.

# 4.19 materi plasti f

The plastic yield rule f is added to the **node** dof records. This should only be used for elasto-plastic calculations, and not for visco-plastic calculations.

# 4.20 materi plasti f nonlocal

The nonlocal plastic yield rule fn is added to the **node** dof records. See also: **nonlocal**.

# 4.21 materi plasti generalised non associate cam clay for bonded soils his

The history variables for the Generalised Non Associate Cam Clay for Bonded Soils plasticity model are added to the **node dof** records.

# 4.22 materi\_plasti\_hardsoil\_history

The history variable abs(p) for the hardsoil plasticity model is added to the **node\_dof** records. It contains the maximum pressure history.

# 4.23 materi plasti hypo history

Eight history variables for the hypo-plasticity models are added to the **node** dof records.

The first history variable -hyhis0 contains the void ratio, and should be initialized by initially specifying node\_dof records, or optionally specifying an OCR value with a group\_materi\_plasti\_hypo.. record. The second history variable -hyhis1 will be filled with the time step size of the hypoplastic substepping scheme; this is meant for postprocessing only.

The third history variable -hyhis 2 will be filled with the mobilized friction angle in degrees; this is meant for postprocessing only.

The fourth history variable - hyhis3 will be filled with the a measure of the effective stiffness following from the hypoplasticity law ( $\sqrt{M_{ijkl}M_{ijkl}}$ ); this is meant for postprocessing only.

The fifth history variable -hyhis4 for all laws with the structure option, will be filled with the structure variable s and should be initialized by initially specifying **node** dof records.

The sixth history variable - hyhis5, will be filled with the OCR value; this is meant for postprocessing only. This is only available for the group \_materi \_plasti \_hypo \_masin \_clay and group \_materi \_plasti \_hypo \_niemunis \_visco laws.

The seventh history variable -hyhis6 for the group materi plasti hypo wolffersdorff, will be filled with the density index  $I_d = \frac{ec-e}{ec-ed}$ ; this is meant for postprocessing only.

The eight history variable -hyhis7 for hypoplasticity laws with intergranular strains, will be filled with the intergranular strain variable  $\rho$ ; this is meant for postprocessing only.

## 4.24 materi plasti kappa

The size of the plastic strain  $\kappa$  is added to the **node** dof records. See the theory section.

## 4.25 materi plasti kappa shear

The size of the shear part of the plastic strain  $\kappa^{shear}$  is added to the **node\_dof** records. See the theory section.

## 4.26 materi plasti phimob

The mobilized friction angle  $\phi_{mob}$  is added to the **node\_dof** records. It is defined as the angle, in radians, for which the yield function

$$f = 0.5(\sigma_2 - \sigma_0) + 0.5(\sigma_2 + \sigma_0) * sin(\phi_{mob}) - ccos(\phi_{mob})$$

becomes zero. This is available for mohr-coulomb plasticity only. Please realise that in regions with substantial cohesion the mobilized friction angle  $\phi_{mob}$  can exceed the friction angle  $\phi$  from the plasticity law. In case of zero cohesion, or cohesion small relative to the stresses, yield is reached if the  $\phi_{mob}$  reaches the friction angle  $\phi$ . The definition above can give either negative or positive values for  $\phi_{mob}$ ; negative values simply indicate that the stress state is far away from yielding. The phimob is calculated in degrees.

#### 4.27 materi plasti rho

The plastic kinematic hardening vector  $\rho_{11} \rho_{12} \rho_{13} \rho_{22} \rho_{23} \rho_{33}$  is added to the **node\_dof** records. See also **group materi plasti kinematic hardening**.

## 4.28 materi strain energy

The material strain energy  $0.5\sigma_{ij}\epsilon_{ij}^{\text{elas}}$  is added to the **node\_dof** records. You can print or plot it to see where energy is stored after loading. Also **materi\_stress** and **materi strain elasti** should be initialised.

## 4.29 materi strain elasti

The elastic strain  $\epsilon_{kl}^{\text{elas}}$  is added to the **node** dof records. See also: materi strain total.

## 4.30 materi strain intergranular

The intergranular strain  $S_{ij}$  is added to the **node\_dof** records. This can be used by hypoplasticity laws, see the theory section.

### 4.31 materi strain isa c

The ISA intergranular back-strain  $C_{ij}$  is added to the **node\_dof** records. This can be used by hypoplasticity laws with ISA-intergranular strains, see the theory section.

## 4.32 materi strain isa eacc

The ISA intergranular accumulated-strain  $\epsilon_{acc}$  is added to the **node \_dof** records. This can be used by hypoplasticity laws with ISA-intergranular strains, see the theory section.

## 4.33 materi strain plasti

The plastic strain  $\epsilon_{kl}$  plas is added to the **node** dof records. See also: **materi** strain total.

#### 4.34 materi strain plasti camclay

The plastic strain  $\epsilon_{kl}$ <sup>plas</sup> specifically for the camclay model is added to the **node\_dof** records. See also: **materi strain plasti**.

## 4.35 materi\_strain\_plasti\_cap

The plastic strain  $\epsilon_{kl}$  plas specifically for cap models is added to the **node\_dof** records. See also: **materi strain plasti**.

## 4.36 materi strain plasti compression

The plastic strain  $\epsilon_{kl}$ <sup>plas</sup> specifically for the compression model is added to the **node\_dof** records. See also: **materi strain plasti**.

## 4.37 materi strain plasti diprisco

The plastic strain  $\epsilon_{kl}$ <sup>plas</sup> specifically for the diprisco model is added to the **node\_dof** records. See also: **materi strain plasti**.

#### 4.38 materi strain plasti generalised non associate cam clay for bonded s

The plastic strain  $\epsilon_{kl}^{\text{plas}}$  specifically for the generalised non associate cam clay for bonded soils model is added to the **node dof** records. See also: **materi strain plasti**.

## 4.39 materi strain plasti druckprag

The plastic strain  $\epsilon_{kl}^{\text{plas}}$  specifically for the drucker-prager model is added to the **node\_dof** records. See also: **materi strain plasti**.

## 4.40 materi strain plasti hardsoil

The plastic strain  $\epsilon_{kl}^{\text{plas}}$  specifically for the hardsoil model is added to the **node\_dof** records. See also: **materi strain plasti**.

## 4.41 materi strain plasti mohr coul

The plastic strain  $\epsilon_{kl}$  plas specifically for the mohr\_coulomb models is added to the **node\_dof** records. See also: **materi strain plasti**.

## 4.42 materi strain plasti tension

The plastic strain  $\epsilon_{kl}$  plas specifically for the tension model is added to the **node\_dof** records. See also: **materi\_strain\_plasti**.

## 4.43 materi strain plasti vonmises

The plastic strain  $\epsilon_{kl}$  specifically for the von-mises model is added to the **node\_dof** records. See also: **materi strain plasti**.

## 4.44 materi strain total

The total strain  $\epsilon_{kl}$  is added to the **node\_dof** records. All strains are time integrals of the strain rates for a specific material particle which happens to be present in the node.

## 4.45 materi\_strain\_total\_kappa

The maximum strain size is added to the **node dof** records.

## 4.46 materi\_strain\_total\_compression\_kappa

The maximum principal compression total strain as occurred in history is added to the **node dof** records.

## 4.47 materi\_strain\_total\_shear\_kappa

The maximum size of the deviatoric part of the total strain as occurred in history is added to the **node dof** records.

#### 4.48 materi strain total tension kappa

The maximum principal tensional total strain as occurred in history is added to the **node\_dof** records.

### 4.49 materi stress

The stresses  $\sigma_{11} \sigma_{12} \sigma_{13} \sigma_{22} \sigma_{23} \sigma_{33}$  are added to the **node** dof records.

## 4.50 materi stress pressure history

The maximum of the absolute value of the pressure which occurs over time is added to the **node\_dof** records. See also **group\_materi\_elasti\_stress\_pressure\_history\_factor** in the data part.

## 4.51 materi velocity

The velocities  $v_i$  are added to the **node** dof records.

## 4.52 materi velocity integrated

The integrated velocities  $vi_i$  are added to the **node\_dof** records. The integration of nodal velocities in fact results in displacements. But asking for these integrated velocities doesn't activate automatically that the calculation is done over the total deformed volume (as is the case when you initialize **materi\_displacement**), and not automatically a total Lagrange model is used in stress analysis.

#### 4.53 materi void fraction

The material void fraction  $f^*$  is added to the **node\_dof** records. This is required for the **group materi plasti gurson** model.

#### 4.54 materi work

The material second order work  $\dot{\sigma}_{ij}\dot{\epsilon}_{ij}$  is added to the **node\_dof** records. You can print or plot it to see where material instabilities are present.

## 4.55 mrange maximum\_range\_length

Sets the maximum length of ranges -**ra** ... -**ra**.

#### 4.56 mstring maximum\_number\_of\_strings

Sets the maximum number of strings in a define block.

## 4.57 wave scalar

Scalar in wave equation is **node\_dof** records. Condition: also **wave\_fscalar** should be initialized.

## 4.58 wave fscalar

The first time derivative in the wave equation is added to the **node\_dof** records. Condition: also **wave scalar** should be initialized.

## 4.59 end initia (last record of initialization part)

## 5 Input file, data part, introduction

Data items in the data part are used to control the calculation, select required output, give dof's initial values, etc.. Note that an **end data** record is needed.

data\_item index data\_values ... data\_item index data\_values end data

Consider the following example

```
element 0 -tria3 0 1 2
element 1 -tria3 1 2 3
node 0 0. 0.
node 1 1. 0.
node 2 0. 1.
node 3 1. 1.
...
end data
```

Note that the data items **element** and **node** are indexed. In fact most data items need to be indexed. Indexing starts at 0 (all numbering in TOCHNOG starts at 0). Indices need not strictly be sequential (e.g. only the indices 1,2 and 5 of a data item may be specified).

The following sections first treat some extras that can be used in the data part. After that, all possible data items are specified.

#### Arithmetic blocks

You also can use the arithmetic expressions **plus**, **minus**, **multiply** and **divide**. We show some examples:

(make A equal to 4.1) start\_arithmetic A 1.1 plus 3 end\_arithmetic ... (make B equal to 3.2) start\_arithmetic B 3.2 end\_arithmetic .... (make C equal to 7.3) start\_arithmetic C A plus B end\_arithmetic .... (make D equal to 14.6) start\_arithmetic D A plus B multiply 2. end\_arithmetic

Expressions will be evaluated from left to right. Words from define blocks will *not* be recognized in arithmetic blocks.

Automatic counting: the counters

The words **counter\_a**, **counter\_b**, **counter\_c** and **counter\_d** are reserved words in the data part. If they are found, they will be substituted by their integer value. After its value is substituted, the counter will be incremented by 1. Initially the value for counters is 0. The example below shows a typical application.

start\_define left \_edge geometry\_line counter\_a end\_define start\_define right\_edge geometry\_line counter\_a end\_define ... left\_edge 0. 0. 0. 10. 1.e-4 right\_edge 2. 0. 2. 10. 1.e-4 ... bounda\_dof 1 -left\_edge -velx bounda\_time 1 0. bounda\_dof 2 -right\_edge -velx bounda\_time 2 1.3 ...

Notice that we automatically give the geometry lines a unique number in this way; the unique number is not really of interest in the remainder of the input file, so the application of a counter is convenient.

Finally, also the words counter a apply, counter b apply, counter c apply and

**counter\_d\_apply** are available. They will be substituted by the current value of the counters, without that the counters are incremented.

#### Conditional blocks

Parts of the input file can be processed conditionally within start\_if ...end\_if blocks. This is illustrated below with an example:

Example:

start\_define
do\_complete\_calculation true
end\_define
...
start\_if do\_complete\_calculation
...
end\_if
...

The part in the start\_if ... end\_if block is only done if do\_complete\_calculation is set to true, like in the example. If do\_complete\_calculation is set to false that part will be skipped. You also can use start\_if\_not ... end\_if\_not blocks, so that actions are NOT taken if the defined variable is set to true.

Control indices

All possible data items are defined in the following sections. It only makes sense to specify some of the data items before the calculation; the other data items are only meant to be printed after the calculation. The example below specifies a 1D temperature calculation.

echo - no number\_of\_space\_dimensions 1 condif\_temperature end\_initia node 1 0 node 2 1 node 3 2 element 1 - bar2 1 2 element 2 - bar2 2 3

```
bounda _ dof 0 1 -temp
bounda _ time 0 0.0 0. 1. 1. 100. 1.
bounda _ dof 1 3 -temp
bounda _ time 1 0.0 0.0 100.0 0.
group _ type 0 -condif
group _ condif _ density 0 1.0
group _ condif _ capacity 0 0.1
group _ condif _ conductivity 0 0.1
group _ condif _ flow 0 0.
control _ timestep 0 0.1 10.0
control _ print 0 -time _ current -node _ dof
control _ print _ database 1 -separate _ index
control _ timestep 2 0.2 10.0
control _ print 2 -time _ current -node _ dof
```

end\_data

Note how the indices of control items like **control\_timestep** and **control\_print** are used to control the sequence of events. First, (index=0) time steps of size 0.1 are taken and for each time step results are printed. Then (index=1) the database is printed which can serve as a point of restart. Finally (index=2) time steps of size 0.2 are taken and for each time step results are printed.

<u>Define blocks</u>

You can define a word to represent a set of strings. For each word defined, you need to specify a **start\_define** ... **end\_define** block. Within the block, you first specify the word, and then you specify the set of strings. Later in the data part, you can use the defined words as the replacement of the set of strings.

Example:

```
start_define
velocity 1.34
end_define
start_define
left_edge geometry_line 1
end_define
...
left_edge 0. 0. 0. 10. 1.e-4
...
bounda_dof 1 -left_edge -velx
bounda_time 1 0. 0. 100. velocity
```

. . .

The words **plus**, **minus**, **multiply** and **divide** as used in arithmetic blocks are prohibited in define blocks.

Include files

. . .

You can use **include** *filename* in the data part, to request that the file with name *filename* is included. This is handy to include often used data parts, or include a mesh generated by a pre-processor, etc.

The included file itself is not allowed to have an **include** in the data part.

The included file should not contain comments ( ... ). The included file needs to be ended with an **end\_data**. On some MS windows computers two **end\_data** records are needed, so try that in case of trouble. On MS windows 32 bit computers include is not available.

Numbering of values in records

The numbering of values in records in illustrated by **node\_dof** records. Look at the following piece of input file

number \_ of \_ space \_ dimensions 2
materi \_ velocity
materi \_ stress
end \_ initia
...
node \_ dof 1 0.0 0.0 -1.0 0.0 0.0 -1.0 0.0 -1.0
node \_ dof 2 0.0 0.0 -1.0 0.0 0.0 -1.0 0.0 -1.0
...
end \_ data

Here **node\_dof** records 1 and 2 are initialized. The initial velocities are 0, and for the initial stresses we use  $\sigma_x x = -1$ ,  $\sigma_y y = -1$  and  $\sigma_z z = -1$ . The total list of dof's in the **node\_dof** record is -vely, -vely, -sigxx, -sigxy, -sigyz, -sigyz, and -sigzz.

We refer to -velx as the 0'th value in the **node\_dof** record, -vely as the 1'th value, etc. So printing the history of the -sigxx stress of **node** dof record 1 is obtained by this:

• • •

```
control_timestep 10 ...
control_print_history 10 -node_dof 1 2
...
end data
```

where the number 2 refers to the -sigxx stress. See also the definition of the control\_print\_history record for this. As an alternative, sometimes you can use names instead of numbers, for example here:

```
control_timestep 10 ...
control_print_history 10 -node_dof 1 -sigxx
...
end data
```

#### Ranges

Ranges are general input formats used for indices and data values. Possible ranges are illustrated by the following examples

-all -ra 12 32 44 -ra -ra -from 5 -to 16 -ra -ra -from 5 -to 25 -step 2 -ra

The -all range is not available for indices.

The data values for a data item can be specified as a range if this is allowed for in the description of the data item. All words in the data part (or part of an index) need to be preceeded with a 'tic' (-). In the example the **node\_dof** records 1 to 100 are initialized

node dof -ra -from 1 -to 100 -ra 1. 0. 0.

Types of dof's

Some of the dof's are <u>principal</u> dof's: these are **materi\_velocity**, **condif\_temperature**, **groundflow\_pressure**, **wave\_fscalar**. These are the dof's which are solved by the equilibrium equations (conservation laws).

The other dof's, like **materi\_stress** and so, follow from these principal dof's (strains follow from displacement derivatives, stresses follow from strains by material laws, etc.).

Furthermore, for all the dof's we have <u>primary</u> values, which are the dof's themselves, and <u>derived</u> dof's, which are the space and time derivatives of the primary dof's.

## 6 Input file, data part, data records

## 6.1 area\_element\_group index geometry\_entity\_item geometry\_entity\_index element\_group

This record is used to generate **element\_group** records. Each element, all of whose nodes are part of the *geometry\_item*, will get an **element\_group** record with value *element\_group*. Please realise that the geometry entity can be a two-dimensional area, a volume, etc.

This option comes handy whenever a part of the domain gets some specific element data. For example, this would be the case if different areas in the structure have different material properties like stiffness, etc.

Beware: any directly specified **element\_group** records will be overwritten. Attention: default this **area\_element\_group** will only evaluated at the very start of the calculation. By setting *switch* in **area\_element\_group\_time** to **-yes** however you can require that this record is evaluated at all times.

We show here two ways to get different element data in different regions: Both ways give elements with young 1.2 from x=0 to x=1, and elements with young 3.3 from x=1 to x=2.

First way:

```
..
node 1 0.
node 2 1.
node 3 2.
element 1 - bar2 1 2
element 2 - bar2 2 3
element _group 1 0
element _group 2 1
..
group _type 0 - materi
group _materi _elasti _young 0 1.2
group _type 1 - materi
group _materi _elasti _young 1 3.3
..
control _mesh _refine _globally 10 -h _refinement
..
```

Second way:

```
..
node 1 0.
node 2 1.
node 3 2.
element 1 -bar2 1 2
element 2 -bar2 2 3
..
group_type 0 -materi
group_materi_elasti_young 0 1.2
group_type 1 -materi
group_materi_elasti_young 1 3.3
..
geometry_line 1 0. 1. 1.e-4
geometry_line 2 1. 2. 1.e-4
area_element_group 1 -geometry_line 1 0
area_element_group 2 -geometry_line 2 1
..
```

 $See also \ \ area\_element\_group\_method, area\_element\_group\_sequence\_element\_group\_etc.$ 

#### 6.2 area element group element index name

With **area\_element\_group\_element** you select the name of the elements for which the **area\_element\_group** will be used; if this **area\_element\_group\_element** is not specified then all elements will be used.

## 6.3 area element group interface index switch

If *switch* is set to **-yes** the **area\_element\_group** record with the same index will also be used for interface elements. If *switch* is set to **-no** the **area\_element\_group** record with the same index will not be used for interface elements. Presently the *switch* can be only set to **-no**.

## 6.4 area element group method index method

Set *method* to **-all** or **-any**. If *method* is set to **-all**, then the corresponding **area\_element\_group** is applied to elements for which all nodes are inside the specified geometry. If *method* is set to **-any**, then the corresponding **area\_element\_group** is applied to elements for which any of the nodes is inside the specified geometry. If *method* is set to **-any\_but\_not\_all**, then the corresponding **area\_element\_group** is applied to elements for which any of the nodes are inside the specified geometry. If *method* is set to **-any\_but\_not\_all**, then the corresponding **area\_element\_group** is applied to elements for which any but not all of the nodes are inside the specified geometry. If *method* is set to a positive integer

number then the corresponding **area\_element\_group** is applied to elements for which at least such amount of the nodes is inside the specified geometry. Default *method* is **-all**.

## 6.5 area element group node index node\_0 node\_1 ... element\_group

Similar to **area\_element\_group**. Now, however, directly the global node numbers are specified.

## 6.6 area element group time index switch

If *switch* is set to **-yes** the corresponding **area**\_**element**\_**group** is evaluated at all times (as opposed to only at the start of the calculation).

6.7 area element group sequence index element\_0 element\_1 ...

 $See \ {\bf area\_element\_group\_sequence\_element\_group}.$ 

## 6.8 area element group sequence element index name

 $See \ area\_element\_group\_sequence\_element\_group.$ 

# **6.9** area\_element\_group\_sequence\_element\_group index group\_0 group\_1 ...

#### General description

This option works more or less the same as the **area\_element\_group** option. Read that description first.

With this option however, you can specify what the element group numbers of an area (geometry), or set of element numbers, will be in time. This allows for an easy modeling of change of material models.

This option works in combination with the **area\_element\_group\_sequence\_\*** records (with the same index).

See also control mesh change element group.

Selection of elements for which the element group changes over time

With **area** <u>element</u> <u>group</u> <u>sequence</u> <u>geometry</u> you select the area (geometry) for which the time sequence of group numbers should be used.

With **area\_element\_group\_sequence** you select the elements for which the time sequence of group numbers should be used.

You can use both **area\_element\_group\_sequence\_geometry** and **area\_element\_group\_sequence** to select a combination of elements in a geometry and directly specified element numbers. As a completely separate option do not use any of **area\_element\_group\_sequence\_geometry** and **area\_element\_group\_sequence** at all. Then at a time point *time\_i* the elements which have group number *group\_(i-1)* will get new group number *group\_i*. So the previous group number of elements is used to set the current group number of elements (and geometries are not used to change the group numbers).

With **area\_element\_group\_sequence\_element** you select the name of the elements for which the sequence of time versus group will be used; if this **area\_element\_group\_sequence\_element** is not specified then all elements will be used.

Specification of new element group numbers in time

With area\_element\_group\_sequence\_time and area\_element\_group\_sequence\_element\_group you select time points at which groups should become active; for example, group\_0 becomes active at time 0 etc.

Remarks

Remark 1: If you want the stresses, strains, etc. to be reset to 0. when the element group changes, then use a **control reset geometry** record for that.

Remark 2: It is more convenient and clear to use the **start\_define end\_define** option to define the geometries.

Examples

Example:

```
area_element_group_sequence_geometry 0 -geometry_brick 1
area_element_group_sequence_element 0 -hex8
area_element_group_sequence_time 0 0. 2. 3.
area_element_group_sequence_element_group 0 1 5 4
group_type 1 ...
group_type 5 ...
group_type 4 ...
control reset geometry 10 -geometry brick 1
```

In the selected geometry element group 1 will be used starting from time 0 for elements -hex8. Starting from time 2 element group 5 will be used, etc. Same example, now with defines however:

```
start_define
soil_empty_wall geometry_brick 1
end_define
...
area_element_group_sequence_geometry 0 -soil_empty_wall
area_element_group_sequence_element 0 -hex8
area_element_group_sequence_time 0 0. 2. 3.
area_element_group_sequence_element_group 0 1 5 4
group_type 1 ...
...
group_type 5 ...
...
group_type 4 ...
...
control_reset_geometry 10 -soil_empty_wall
...
```

Now an example of the separate option:

. . .

```
area_element_group_sequence_time 0 0. 2. 3.
area_element_group_sequence_element_group 0 1 5 4
element_group 77 1
element_group 78 1
group_type 1 ...
...
group_type 5 ...
...
group_type 4 ...
```

At time 0. elements 77 and 78 have group number 1. At time 2. the elements with group number 1 get group number 5. At time 3. the elements with group number 5 get group number 4.

## 6.10 area\_element\_group\_sequence\_geometry index geometry\_entity\_item geometry\_entity\_index

See area element group sequence element group.

## 6.11 area element group sequence geometry method index method

Set *method* to **-all** or **-any**. If *method* is set to **-all**, then the corresponding **area\_element\_group\_sequence\_ge** is applied to elements for which all nodes are inside the specified geometry. If *method* is set to **-any**, then the corresponding **area\_element\_group\_sequence\_geometry** is applied to elements for which any of the nodes is inside the specified geometry. Default *method* is **-all**.

## 6.12 area element group sequence interface index switch

If *switch* is set to **-yes** the **area\_element\_group\_sequence\_\*** will be used for interface elements also. If *switch* is set to **-no** the **area\_element\_group\_sequence\_\*** will not be used for interface elements. Default *switch* is set to **-no**.

#### 6.13 area element group sequence time index time\_0 time\_1 ...

See area element group sequence element group.

## 6.14 area\_node\_dataitem index geometry\_entity\_item geometry\_entity\_index data\_item\_name

This record is used to generate *data\_item\_name* records on all nodes located on the specified geometrical entity. The values for the *data\_item\_name* should be specified in the **area\_node\_dataitem\_double** record for real precision values, or in the **area\_node\_dataitem\_integer** record for integer values (or words).

## 6.15 area node dataitem double index value\_0 value\_1 ...

See area node dataitem.

6.16 area node dataitem integer index value\_0 value\_1...

See area node dataitem.

#### 6.17 axisymmetric switch

If *switch* is set to **-yes**, the calculation becomes axi-symmetrical. If *switch* is set to **-no**, the calculation becomes not axi-symmetrical. In case a **group\_axisymmetric** is specified for some group, that overrules this **axisymmetric** record.

## 6.18 bounda alternate index bounda\_index\_0 bounda\_index\_1 ...

This option takes care that between successive iterations only one of the specified **bounda\_dof** is not used. For example if **bounda\_dof** records with index 10, 20 and 30 are present in the input file, and you use **bounda\_alternate 10 20 30** then in subsequent iterations the following index is not used: 10, 20, 30, 10, 20, 30, 10, ... etc.

This option comes handy to allow for very large calculations on a computer with limited memory. By putting alternating bounday conditions on velocities, pressures or temperatures the system of active equations to be solved in each iterations is only of a limited size. And then using enough iterations the solutions for all dof's can slowly converge to the actual coupled solution.

As example consider a large 3d calculation where displacements and hydraulic pressure heads need to be solved:

solver \_\_matrix \_\_symmetric -yes .... bounda \_\_alternate 10 20 30 40 bounda \_\_dof 10 -all -velx bounda \_\_dof 20 -all -vely bounda \_\_dof 30 -all -velz bounda \_\_dof 40 -all -pres .... control \_\_timestep 100 .. control \_\_timestep iterations 100 20

The above **bounda\_dof** records are additional to the normally present records, like fixing displacements at sides of the domain, boundary conditions on hydraulic pressure, etc. The **bounda\_alternate** record instructs tochnog to subsequently neglect the record 10, 20, 30, 40, 10, ..., etc. When a record is neglected the corresponding solution field can be solved. For example in the first iteration the solution field for the x-displacement can be solved, while the y-displacement and z-displacement and hydraulic pressure head are kept fixed. And thus the total system of equations is much smaller, approximately 4 times less dof's need to be solved by the pardiso solver, which in fact is the bottleneck in computer memory usage for very large calculations. Notice that we asked tochnog to use the symmetric equation

solver, since the pressures and displacements are not used simultaneously, so we don't have the disadvantage of a non-symmetric matrix with displacement and pressure contributions.

As another example we use a classical staggered solution for displacements and water pressures:

solver\_matrix\_symmetric -yes ... bounda\_alternate 10 20 bounda\_dof 10 -all -velx -vely -velz bounda\_dof 20 -all -pres ... control\_timestep 100 .. control\_timestep\_iterations 100 20

You should not specify **bounda\_time** records i.c.w. **bounda\_dof** records which are used in **bounda\_alternate**. The **bounda\_time** records will not be used.

## 6.19 bounda baseline correction time\_start time\_end

If this record is specified baseline correction is performed after one of:

- reading SMC files with uncorrected accelerations in **bounda dof** i.c.w. **bounda time smc**.
- direct specification of acceleration in **bounda** dof i.c.w. bounda time.

Such baseline correction is needed to suppress artificial drift in velocity signals following from the acceleration signal.

The correction actually is done by adding a parabolic acceleration signal to the specified accelerations, thus giving a corrected acceleration in time. The parabolic (second order) signal contains three constant coefficients. These are determined by demanding that the corrected acceleration signal leads to a minimal sum of squared velocities over the considered time interval.

This correction is done over the time interval from *time\_start* up to *time\_end*. Typically *time\_start time\_end* are the start time and the end time of the time interval in which you apply base excitation. You need to specify these times in units that you actually use in your Tochnog calculation (so not in the units of the SMC file).

If this **bounda\_baseline\_correction** is not specified the data will be used directly without a correction.

See also bounda baseline correction parameters.

#### 6.20 bounda baseline correction parameters index ...

The parameters for the parabolic baseline correction are written in this record. In future calculations you can use the parameters yourself by setting this record in the input file; then the parameters will not be determined again by the baseline correction algorithm; the parameters in the specified record will be used instead.

## 6.21 bounda constant index switch

This record can be used i.s.o. the **bounda\_time** record. If *switch* is set to **-yes** the prescribed dofis kept constant. This is only available for velocities, pressures and temperatures. This is not available for time derivatives **ttemp**, **tpres** and **ttotal pres**.

## 6.22 bounda dof index node\_range dof\_0 dof\_1 ...

States which dof's in which nodes get prescribed values by adjustment of the **node\_dof** records. The item *node\_range* represents a range of node numbers. In stead of a node range also, by example, **-geometry\_line 1** can be used, indicating that the nodes on line 1 get the prescribed boundary values. The items  $dof_0$  etc. are one of the primary dof's listed at **dof\_label**.

For a specific *index*, only one of **bounda\_force** or **bounda\_dof** can be specified (thus either Neumann conditions or Dirichlet conditions).

Example for discrete node forces in y-direction on the nodes on a line:

bounda\_dof 0 -geometry\_line 1 -vely bounda\_time 0 0. 0. 1. 1. 100. 1.

Normally you only should specify boundary conditions on principal dof's (like velocity, temperature, etc.) and not on strain, stresses, etc.!

Specially for velocity (displacement) dof's, you can prescribe that nodes should not move in a direction normal to a plane. For this, specify -veln for  $dof_0$  to indicate that the normal velocity to a plane is 0. The normal direction should be given with **bounda\_normal**; if however a geometrical entity is used to specify the nodes, you do not necessarily need to specify the **bounda\_normal**, thus the normal from the geometrical entity is then used instead. The **bounda\_time** record should not be specified (it is irrelevant). Internally in Tochnog a multi-point-constraint will be generated to accomplish this condition of zero velocity in normal direction.

Specially for velocity (displacement) dof's, you can prescribe a rotation around either the x-axis, y-axis or z-axis. In 1D you cannot use this record. In 2D you can only specify a

rotation around the z-axis. In 3D you can specify each of the three axis. Example of an x-axis rotation of node 12 with angular velocity of 0.33 [degrees per unit time]:

bounda\_dof 0 12 -rotation\_x\_axis bounda\_time 0 0.33

For the rotation 0.33 the rotation vector points in the positive x-axis direction.

Specially for the groundflow phreatic head h, you can prescribe the physical pore pressure **-total\_pressure** and Tochnog will automatically calculate the corresponding hydraulic pressure head h. Also specially for the groundflow phreatic head h, you can prescribe the time rate of the physical pore pressure **-ttotal\_pressure** and Tochnog will automatically calculate the corresponding hydraulic pressure head h. Also specially for the groundflow phreatic head h, you can prescribe the time rate of the hydraulic pressure head h. Also specially for the groundflow phreatic head h, you can prescribe the time rate of the hydraulic pressure head **-tpres**. Specially for the temperature you can prescribe the time rate of the temperature **-ttemp**.

As a special option you can specify also, for example, **-element\_group 1** in stead of a node range. Then nodes of elements which have **element\_group** set to 1 will get the prescribed boundary values.

As a special option you can specify also, for example, -element\_geometry 1 in stead of a node range. Then nodes of elements which have element\_geometry set to 1 will get the prescribed boundary values.

As a special option you can specify also, for example, -geometry\_set 1 in stead of a node range. Then nodes of elements which have any of the elements belonging to geometry\_set 1 will get the prescribed boundary values.

Notice: if several **bounda\_dof** records act on a node, only the record with the highest index will be used.

See also: bounda\_time, bounda\_sine, bounda\_constant, bounda\_dof\_radial, bounda\_dof\_cylindrical, force\_edge and force\_volume.

# **6.23 bounda\_dof\_cylindrical** index $x_{first} y_{first} z_{first} x_{second} y_{second}$

Specially for velocity (displacement) dof's, you can prescribe velocities cylindrical to a line specified with the point  $x_{first}$ ,  $y_{first}$ ,  $z_{first}$  and  $x_{second}$ ,  $y_{second}$ ,  $z_{second}$ ; in 1D only x values should be specified, and in 2D only x, y values should be specified. Example:

bounda dof 10 -ra ... -ra -velx -vely -velz

bounda\_dof\_cylindrical 10 1.23 3.43 5.12 1.23 3.43 15.12 bounda\_time 10 0. 0. 1. 1. 100. 1.

The velocity increases linearly in size away from the specified line (at unit distance away from the line the velocity has size 1; you can scale it by the **bounda time** record).

## 6.24 bounda dof radial index x y z

Specially for velocity (displacement) dof's, you can prescribe velocities radial to a specified point x, y, z; in 1D only x should be specified, and in 2D only x, y should be specified. Example:

bounda\_dof 10 -ra -ldots -ra -velx -vely -velz bounda\_dof\_radial 10 1.23 3.43 5.12 bounda\_time 10 0. 0. 1. 1. 100. 1.

A radial velocity is prescribed on nodes in a specified range, relative to point 1.23, 3.43, 5.12 and with the time table given by **bounda\_time**. The velocity increases linearly in size away from the specified point x, y, z (at unit distance away from the specified point x, y, z the velocity has size 1; you can scale it by the **bounda\_time** record).

#### **6.25** bounda factor index $a_0 a_1 \ldots a_n$

This data item defines a polynomial in space. This polynomial gives a factor which is used as a multiplication factor for **bounda\_time** records (with the same index). In this way, you can obtain coordinate dependent boundary conditions.

In 1D the polynomial is  $a_0 + a_1x$  (specify 2 values). In 2D the polynomial is  $a_0 + a_1x + a_2y$  (specify 3 values). In 3D the polynomial is  $a_0 + a_1x + a_2y + a_3z$  (specify 4 values).

## **6.26** bounda factor parabolic x index $a_0 a_1 a_2$

This data item defines a quadratic polynomial in x-direction.

The polynomial is  $a_0 + a_1x + a_2x^2$  (specify 3 values).

#### 6.27 bounda force index node\_range dof\_0 dof\_1 ...

States which ones from the list of dof's in which nodes get prescribed nodal forces. The item *node\_range* represents a range of node numbers. In stead of a node range also, for example, **-geometry\_line 1** can be used, indicating that the nodes on line 1 get the prescribed nodal forces. The items *dof\_0* etc. can be one of the items listed at **dof\_label**. However, neither **-dis** and **-scal** can be used.

For a specific *index*, only one of **bounda\_force** and **bounda\_dof** can be specified; thus, either Neumann conditions or Dirichlet conditions can be applied to a particular node, but nor both.

<u>Attention</u>: with this option you get the same nodal force on all the specified nodes. If you want to apply a distributed force on a edge, however, you should use **force\_edge**. That option gives forces consistent with the displacement field, so not necessarily the same for all nodes. For example the nodes on the side of linear elements on a edge get only half the force.

As a special option you can specify also, for example, **element\_geometry 1** in stead of a node range. Then nodes of elements which have **element\_group** set to 1 will get the prescribed nodal forces.

Notice: if several **bounda** force records act on a node, the imposed forces are summed.

See also: bounda time, bounda sine and force edge.

#### 6.28 bounda found index found

This record is meant for printing only. A value of **-yes** indicates that the corresponding **bounda** \* records are indeed used at some nodes. A value of **-no** indicates that the corresponding **bounda** \* records are not used at all at some nodes.

## 6.29 bounda geometry method index node\_type

If boundary conditions are imposed on a geometry, you can set with this record which node type should be used. If *node\_type* is set to **-node\_start\_refined** the values of **-node\_start\_refined** are used to determine if nodes are located on the geometry. If *node\_type* is set to **-node** the values of **-node** are used to determine if nodes are located on the geometry. If *node\_type* is set to **-node** the values of **-node** are used to determine if nodes are located on the geometry. If *node\_type* is set to **-plus\_displacement** the values of **-node** plus nodal displacements are used to determine if nodes are located on the geometry.

#### **6.30** bounda normal index normal\_x normal\_y normal\_z

This record specifies the components of a normal vector to a plane on which nodes should slide (the nodes are not allowed to move normal to the plane). In 3D you need to specify all of *normal\_x normal\_y normal\_z*. In 2D you need to specify only *normal\_x normal\_y*. In 1D you need to specify only *normal\_x*.

See also **bounda** dof.

## 6.31 bounda print mesh dof dof 0 dof 1 ...

See print mesh dof.

6.32 bounda\_print\_mesh\_dof\_geometry\_geometry\_item\_name\_geometry\_item\_index

See print mesh dof.

#### 6.33 bounda print mesh dof values value dof 0 value dof 1 ...

See print mesh dof.

# **6.34 bounda\_sine** index start\_time end\_time freq\_0 amp\_0 freq\_1 amp\_1

The **bounda\_dof** or **bounda\_force** record with the same *index* is imposed with the sum of the sine functions; the first sine function has frequency  $freq_0$  and amplitude  $amp_0$ , the second sine function has frequency  $freq_1$  and amplitude  $amp_1$ , etc.. More general behavior in time can be imposed by using **bounda\_time** records. For a specific *index* only one of **bounda\_time** and **bounda\_sine** can be specified.

As a typical application the response due to the excitation with a frequency spectrum can be analyzed. Just print the relevant response by **control\_print\_history** and extract the frequency spectrum of that response signal.

The sine loads will be only imposed after *start\_time*, and will not be imposed anymore after *end\_time*. The sine functions start at time *start\_time* (then they have value 0).

As a special option setting a frequency to 0 enforces tochnog to use a constant static value of the specified amplitude.

#### 6.35 bounda time index time load time load ....

This record specifies a multi linear time-load diagram for the **bounda\_dof** or **bounda\_force** record with the same *index*. Between two time points in the diagram, the load is interpolated linearly (ramp function between the two points).

At all times that an dofis not prescribed in such way, it is free and determined with the governing differential equations. For a specific *index* only one of **bounda\_time**, **bounda\_sine** and **bounda\_time\_user** can be specified.

As a special option, you can specify only one value in the **bounda\_time** record if the load is constant over time (so not time-load sets but directly the constant load value).

As a further special option, you can specify no **bounda\_time** and no **bounda\_sine** at all; then a 0 value is assumed.

### 6.36 bounda time factor index factor

With this record you can specify an multiplication factor to be used for loads specified by **bounda\_time**. This option comes handy when you import a time-load table from some external data source, which uses some other definition of the load as you do in the tochnog input file. For example, if you specify accelerations in metric units but the external source specifies the accelerations as part of the gravity acceleration, you can convert the load in the time-load table with this factor.

Default, if **bounda** time factor is not specified, the factor is set to 1.

#### 6.37 bounda time offset index time\_offset

With this record you can specify an offset to be used for times specified by **bounda\_time**. The actual times will become time offset added to the specified times in **bounda\_time**. This option comes handy when you import a time-load table from some external data source, but would like to apply the table at a different moment in time in the calculation. You need to specify *time\_offset* in the units that you actually use in your calculation.

## 6.38 bounda time increment index time\_increment

With this record you can specify that the data as specified in **bounda\_time** is only the load data, so not time points anymore. The time points are automatically calculated from a fixed time increments (and optionally an initial offset as specified in **bounda\_time\_offset**). For example:

• • •

bounda\_dof 10 -geometry\_line -accx
bounda\_time 10 0.2 0.78 1.33 ... (acceleration data only)
bounda\_time\_offset 10 1. (the accelerations start at time 1)
bounda\_time\_increment 10 0.05 (the increments in time are 0.05)
(thus the acceleration is specified at times 1.0, 1.05., 1.10, 1.15, etc.)
...

In this example the acceleration is 0.2 at time 1, it is 0.78 at time 1.05, etc.

## 6.39 bounda time units factor\_time factor\_length

The specified times and data in **bounda**\_**time** may have other units then you actually apply in your calculation. With *factor\_time* you correct the time in **bounda\_time** to get times consistent with your calculation. With *factor\_length* you can correct the data **in bounda\_time** to get data consistent with your calculation. For example, if **bounda\_time** contains [sec] and [cm] and if your actual calculation uses [hour] and [m] then set *factor\_time* to 3600. and set *factor\_length* to 100. This option is presently only available for prescribed accelerations.

#### 6.40 bounda time until data data\_item\_name data\_item\_index date\_item\_number

With this option you can set how much the **bounda time** record with the same index should be applied. Tochnog will look at the value of *data\_item\_name data\_item\_index date\_item\_number*. If that value is lower than *start* Tochnog will start to reduce the applied value in **bounda\_time**. This reduced value will become 0 at the moment that the value of *data item\_name data item\_index date\_item\_number* reaches *wanted*.

A typical application is to slowly reduce prescribed velocities/displacements on a structure so that a mimimum force response of the structure is obtained. This would look like:

bounda \_ dof 210 ... -vely ( apply velocity on structure)
bounda \_ time 210 ...
bounda \_ time \_ until \_ data 210 -post \_ node \_ result 10 -vely ( monitor
structure force response )
( reduce prescribed displacement when force response
is below 30 percent of initial value )
( the reduced velocity will become zero when the force response
reaches 10 percent of initial value )
bounda \_ time \_ until \_ value \_ minimum 210 0.1 0.3
...

#### 6.41 bounda time until value minimum wanted start

See bounda time until data.

#### 6.42 bounda time smc index switch

If *switch* is set to **-yes** the SMC file *index.*smc will be read. Such Strong Motion CD file (SMC file) contains base acceleration time data. This option can be used to read SMC files strictly following the definition from **http://nsmp.wr.usgs.gov/smcfmt.html**. A typical input example for a SMC file looks like:

```
. . .
materi velocity
materi stress
. .
end initia
bounda baseline correction 1. 1.1 (correct acceleration for time 1
to 1.1
bounda dof 10 - geometry line - accx
bounda time smc 10 -yes
bounda time smc offset 10 1. (the base excitation starts at time 1)
bounda time smc units 10 3600. 100. (we use hours and meters)
. . .
control timestep 10 1.e-2 1. (gravity from time 0 to 1)
. . .
control timestep 20 1.e-6 0.1 (base excitation from time 1 to 1.1)
. . .
```

In case the SMC file does not strictly follow the definition from http://nsmp.wr.usgs.gov/smcfmt.html, the option bounda\_time\_smc cannot be used. In such case you can use the actual data lines in a bounda\_time record as follows:

materi\_velocity
materi\_stress
...
end\_initia
...
bounda\_baseline\_correction 1. 1.1 (correct acceleration for time 1
to 1.1

bounda\_dof 10 -geometry\_line -accx include acceleration.dat (include file containing bounda\_time 10 ..., the dots ... represent acceleration data) bounda\_time\_offset 10 1. (the base excitation starts at time 1) bounda\_time\_units 10 3600. 100. (we use hours and meters) .... control\_timestep 10 1.e-2 1. (gravity from time 0 to 1) .... control\_timestep 20 1.e-6 0.1 (base excitation from time 1 to 1.1) ....

Be sure that you take sufficient small time increments while performing the base acceleration steps. See also http://nsmp.wr.usgs.gov/.

## 6.43 bounda time smc offset index time\_offset

The times of the SMC file are incremented with  $time\_offset$ , such that you can use the acceleration data starting from any time point in a calculation. If this record is not specified then  $time\_offset$  is set to 0.

## 6.44 bounda time smc units factor\_time factor\_length

The SMC files have units [cm] for length and [sec] for time. You input file may have other units however. With *factor\_time* you correct the time read from the SMC file to get times consistent with your input file. With *factor\_length* you can correct the data (acceleration, velocity or displacement) read from the SMC file to get data consistent with your input file. For example, if you use [hour] and [m] in your calculation then set *factor\_time* to 3600. and set *factor\_length* to 100.

#### 6.45 bounda time user index switch

If *switch* is set to **-yes** a user supplied routine for the time-load diagram will be used.

See also the file **user.cpp** in the distribution.

## 6.46 bounda water index switch

If *switch* is set to **-yes**, and you specify the pore pressure **-total\_pressure** as dof, the pore pressure is actually determined from the height of the water column between the node and

the phreatic level. In fact the pore pressure is set to density\_water g  $\Delta z$  where g is the gravitational acceleration, and  $\Delta z$  is the distance to the phreatic level.

The water density is given by **groundflow\_density**. The gravity acceleration is given by the vertical component of **force\_gravity**. The water height is relative to the water height is given by **groundflow phreatic level**.

In this case the record **bounda\_time** does not contain the actual value of the pore pressure, but instead it only contains a multiplication factor for the static water pressure as calculated above.

This **bounda\_water** is convenient when the phreatic level is located above the FE mesh. Then this option allows you to impose a pressure boundary condition for the nodes in the FE mesh at the top boundary of the mesh, automatically using a specified phreatic level record.

## 6.47 change\_dataitem index data\_item\_name data\_item\_index data\_item\_number\_0 data\_item\_number\_1 ... operat

With this record you can specify a data item which should be changed over time. The time table should be given in the **change\_dataitem\_time** table as time-value sets; at least two sets should be specified.

The *operat* determines how the time-value sets are used. If *operat* is set to **-use**, then the value of the time-value sets is directly used. If *operat* is set to **-add**, then the value of the time-value sets is interpreted as a rate of change, so that the value is multiplied with the time step and then added to the old value.

Notice that you can change multiple numbers at once.

As a typical example you can use this to prescribe the displacement of a contact geometry over time. Below the y-coordinates of a geometry line which is used in the contact algorithm is changed over time:

```
contact_target_geometry 0 -geometry_line 1
....
geometry_line 1 0. 10. 2. 10.
....
change_dataitem 0 -geometry_line 1 1 -use
change_dataitem_time 0 0. 10. 100. 0.
change_dataitem 1 -geometry_line 1 3 -use
change_dataitem_time 1 0. 10. 100. 0.
....
```

The specified values are evaluated at the start of each timestep.

## 6.48 change\_dataitem\_geometry index geometry\_entity\_name geometry\_entity\_index

For element group data group\_\* you can restrict the application for the change\_dataitem to only those elements which are part of the geometry specified by geometry\_entity\_name geometry\_entity\_index.

#### 6.49 change dataitem time index time value ....

See change dataitem and change dataitem time user.

## 6.50 change dataitem time discrete index switch

If *switch* is set to **-yes** then the changes applied by the **change\_dataitem** and **change\_dataitem\_time** records (with the same index), will be applied at the discrete time points given in **change\_dataitem\_time**. Between those time points, no interpolation is used.

More precise, the change of the data item will be applied directly <u>after</u> the time point has passed.

If you don't specify this **change\_dataitem\_time\_discrete** record then interpolation is used.

## 6.51 change dataitem time method index method

With this record you can require that the cosinus, sinus or tangent of a data value will be changed (in stead of the data value directly itself). The *method* can be set to either **-cosinus**, **-sinus** or **-tangent**. This is typically convenient for geotechnical safety factor calculations where you want that for a mohr coulomb law the cohesion and tangent of the friction angle are decreased at the same ratio in time.

Example:

group\_materi\_plasti\_mohr\_coul\_direct 10 ...

```
(tangent of friction angle reduction)
change_dataitem 10 -group_materi_plasti_mohr_coul_direct 10 0
-use
change_dataitem_time 10 ... (specify tangent values here)
change_dataitem_time_method 10 -tangent
...
(cohesion reduction)
change_dataitem 20 -group_materi_plasti_mohr_coul_direct 10 1
-use
change_dataitem_time 20 ...
...
```

As an extra remark on such 'phi - c' reduction: if you want to calculate the safety factor of the initial geometry you should <u>not</u> use the **mesh -follow material** ... option, since with that the soil would be able to deform to new stable configurations.

## 6.52 change\_dataitem\_time\_user index switch

If *switch* is set to **-yes** a user supplied subroutine is used instead of the **change**\_**dataitem**\_**time** table.

See also the **user.cpp** routine included in the distribution.

## 6.53 check data switch

If *switch* is set to **-yes** the in-core database is checked at some moments during the calculation. You can try this option in case you experience unexpected behavior.

#### 6.54 check error switch

Tochnog will does some error checking which you can suppress by setting *switch* to **-no**.

## 6.55 check element node index switch

Tochnog will check that elements do not have duplicate nodes. If you want to have duplicate nodes on purpose however, you can set *switch* to **-no** so that this checking is suppressed.

#### 6.56 check element shape index factor

Isoparametric elements are mapped from the isoparametric space to the real coordinate space with shape functions. The determinant of the Jacobian of the mapping will have the same value in each integration point if elements are not distorted by the mapping. Thus the relative difference  $\frac{det_{ip}-det_{average}}{det_{average}}$  in each integration point of an element measures the distortion.

Tochnog determines the average of the relative difference for all the integration points in an element.

If this average is larger then *factor* a warning message will be printed. Furthermore, if **check\_element\_shape** is specified the average will be stored in a record **element\_shape** in the database dbs file; the average will be plotted in the GID post-processing files so that you can visually inspect where the elements are most distorted.

Perfectly non-distorted isoparametric elements have average 0.

Severely distorted elements have a high average, e.g. larger than 0.25.

#### 6.57 check memory index switch

If *switch* is set to **-yes**, Tochnog checks memory usage of the calculation. If *switch* is set to **-no**, Tochnog does not check memory usage of the calculation.

When checking memory usage Tochnog checks that the calculation fits in the computer RAM memory. Furthermore, on 32 bit systems Tochnog checks that array sizes do not exceed 2Gb.

Default, if **check** memory is not specified, the *switch* is set to **-no**.

## 6.58 check memory usage index switch

If *switch* is set to **-yes** Tochnog keeps record of the highest memory used by the calculation. It will put that highest usage, expressed in GB, in the record **check \_memory \_usage \_result**. This option comes convenient to keep an eye on the memory usage of a calculation, in case you are reaching the limit on your computer. You need to prevent that memory usage exceeds the amount of RAM memory, since swapping to disk is extremely slow.

This option is only available on 64 bit linux. Default, if **check\_memory\_usage** is not specified, then *switch* is set to **-yes**.

#### 6.59 check memory usage result index memory

See check memory usage.

#### 6.60 check nan switch

If *switch* is set to **-yes** some internal result (stresses, etc.) are check for being NAN. NAN represents Not A Number , meaning that the computer cannot represent the result by a number. This means that something is wrong: the solution may have diverged, or you may have a programming error in a user supplied routine, or etc.

## 6.61 check solver eps

If this record is set the solver checks if diagonal terms are smaller than *eps*. That normally indicates some problem in your input file if *eps* is very small.

## 6.62 check target switch

If you set switch to **-no**, any **target** \* records will be neglected. THis allows to run the input file without getting error messages in the log file, for example when testing variations of the input file.

#### 6.63 check used switch

If *switch* is set to **-yes**, Tochnog will check if input data is really used. It will give a message if some input data is not used. In fact, this option will only check if any index of a certain data item is used; thus if any index of a certain data item in the input file has been used in the calculation, all indices of that data item are considered to be used, and no message will be given.

This option comes handy when looking for errors in input files.

Default, when **check** used is not specified, *switch* is set to -no.

## 6.64 check warning switch

Tochnog will does some warning checking which you can suppress by setting *switch* to **-no**.

#### 6.65 condif convection edge normal index $\alpha_c T_r$

Convection coefficient and convection environmental temperature. Also the record **condif convection edge normal geometry** should be specified.

Attention: this option is only available for linear and quadratic isoparametric elements.

**6.66 condif\_convection\_edge\_normal\_element** *index element\_0 element\_1 ...* 

Selects the elements for which the **condif\_convection\_edge\_normal** record with the same *index* should be applied.

**6.67 condif\_convection\_edge\_normal\_element\_group** *index element\_group\_0 element\_group\_1* ...

Selects the element groups for which the **condif\_convection\_edge\_normal** record with the same *index* should be applied.

**6.68 condif\_convection\_edge\_normal\_element\_node** index element node\_0 node\_1 ...

Selects the element and local node numbers for which the **condif\_convection\_edge\_normal** record with the same *index* should be applied.

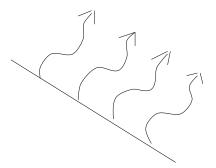
**6.69 condif\_convection\_edge\_normal\_element\_side** *index element\_0 element\_1 ... side* 

Selects the elements and local side number for which the **condif\_convection\_edge\_normal** record with the same *index* should be applied.

6.70 condif\_convection\_edge\_normal\_geometry index geometry\_entity\_name geometry\_entity\_index

Selects the area for which the **condif\_convection\_edge\_normal** record with the same *index* should be applied.

Instead of a number of nodes also, for example, -geometry\_line 1 can be used in 2D, indicating that the nodes on line 1 start to convect. The total edge of an element must be inside



the geometry for the force to become active. For 2D elements the border lines are edges. For 3D elements the border surfaces are edges. See also: **condif convection edge normal**.

#### 6.71 condif convection edge normal node index node\_0 node\_1...

Selects the nodes for which the **condif\_convection\_edge\_normal** record with the same *index* should be applied. The *node\_0* etc. specifies the global node numbers.

#### 6.72 condif heat edge normal index heat

Distributed prescribed heat flux normal normal to the edge of a element. This distributed heat is translated into equivalent nodal heat on the edges of elements. Also the record **condif\_heat\_edge\_normal\_geometry** should be specified, and optionally the record **condif\_heat\_edge\_normal\_time** can be specified.

Attention: this option is only available for linear and quadratic isoparametric elements.

<u>Attention</u>: if this option is used INSIDE a FE mesh, then the elements on each side will get the distributed heat. So the total heat flux will normally become zero since the normals of the elements at the side of the geometry are opposite.

### **6.73 condif\_heat\_edge\_normal\_element** *index element\_0 element\_1*

Restricts the elements to which the **condif\_heat\_edge\_normal** record with the same *index* should be applied.

# **6.74 condif\_heat\_edge\_normal\_element\_group** *index element\_group\_0 element\_group\_1* ...

Restricts the element groups to which the **condif\_heat\_edge\_normal** record with the same *index* should be applied.

#### **6.75 condif\_heat\_edge\_normal\_element\_node** *index element node\_0 node\_1*...

Selects the element and local node numbers for which the **condif\_heat\_edge\_normal** record with the same *index* should be applied.

#### 6.76 condif\_heat\_edge\_normal\_element\_node\_factor index $factor_0$ $factor_1 \dots$

Nodal multiplication factors with which the **condif\_heat\_edge\_normal** will be applied to the element of **condif\_heat\_edge\_normal\_element\_node**. You need to specify a factor for each node on the side. Here  $factor_0$  is the multiplication factor for the first node on the side, etc.

# 6.77 condif\_heat\_edge\_normal\_element\_side index element\_0 element\_1 ... side

Selects the elements and local side number for which the **condif\_heat\_edge\_normal** record with the same *index* should be applied.

#### 6.78 condif heat edge normal factor index $a_0 a_1 \ldots a_n$

This data item defines a polynomial in space. This polynomial gives a factor which is used as a multiplication factor for **condif\_heat\_edge\_normal** records (with the same index). In this way, you can obtain coordinate dependent heat fluxes.

In 1D the polynomial is  $a_0 + a_1x$  (specify 2 values). In 2D the polynomial is  $a_0 + a_1x + a_2y$  (specify 3 values). In 3D the polynomial is  $a_0 + a_1x + a_2y + a_3z$  (specify 4 values).

#### 6.79 condif\_heat\_edge\_normal\_geometry index geometry\_entity\_name geometry\_entity\_index

Selects the area for which the **condif\_heat\_edge\_normal** record with the same *index* should be applied. For example, **-geometry\_line 1** can be used in 2D, indicating that the nodes on line 1 get the distributed heat. The total edge of an element must be inside the geometry for the force to become active. For 2D elements the border lines are edges. For 3D elements the border surfaces are edges.

### 6.80 condif\_heat\_edge\_normal\_node index node\_0 node\_1 node\_2

Selects the nodes for which the **condif\_heat\_edge\_normal** record with the same *index* should be applied. The *node\_0* etc. specify global node numbers.

#### **6.81** condif\_heat\_edge\_normal\_sine index start\_time end\_time freq\_0 amp\_0 freq\_1 amp\_1 ...

Similar to force edge sine, now for heat flux however.

#### 6.82 condif heat edge normal time index time load time load ...

This record specifies a diagram which contains the factors with which the **condif\_heat\_edge\_normal** record with the same index is applied. Linear interpolation is used to extend the *time load* values to the intervals between these pairs. Outside the specified time range a factor 0 is used.

If this record is not specified, the heat flux is applied at all times with a factor 1.

#### 6.83 condif heat volume index heat

Distributed volume heat source. Here *heat* is the distributed heat source value.

See also  $condif\_heat\_volume\_factor$ ,  $condif\_heat\_volume\_geometry$ , and  $condif\_heat\_volume\_time$ .

#### 6.84 condif heat volume element index element\_0 element\_1 ...

Specifies the elements for which the **condif\_heat\_volume** record with the same *index* should be applied.

#### 6.85 condif heat volume element group index element\_group

Specifies the element group for which the **condif\_heat\_volume** record with the same *index* should be applied.

#### **6.86 condif heat volume factor** index $a_0 a_1 \ldots a_n$

This polynomial gives a factor which is used as a multiplication factor for **condif\_heat\_volume** records (with the same index). In this way, you can obtain coordinate dependent forces.

In 1D the polynomial is  $a_0 + a_1x$  (specify 2 values). In 2D the polynomial is  $a_0 + a_1x + a_2y$  (specify 3 values). In 3D the polynomial is  $a_0 + a_1x + a_2y + a_3z$  (specify 4 values).

#### 6.87 condif heat volume geometry index geometry\_name geometry\_index

Specifies the geometry for which the **condif\_heat\_volume** record with the same *index* should be applied.

# **6.88 condif\_heat\_volume\_sine** *index start\_time end\_time freq\_0 amp\_0 freq\_1 amp\_1...*

Similar to force edge sine, now for volume heat source however.

#### 6.89 condif heat volume time index time load time load ....

This record specifies a multi-linear diagram which contains the factors with which the **con-dif heat volume** record with the same index is applied.

If this record is not specified, the heat source is applied at all times with a factor 1.

#### 6.90 condif heat volume user index switch

Set *switch* to **-yes** if you want to call the user supplied routine for heat.

#### 6.91 condif heat volume user parameters index ...

Specify the parameters for the user supplied routine for heat.

#### 6.92 condif radiation edge normal index $\alpha_r T_r$

Radiation coefficient and radiation environmental temperature. Also the record **condif\_radiation\_edge\_norma** should be specified.

Attention: this option is only available for linear and quadratic isoparametric elements.

**6.93 condif\_radiation\_edge\_normal\_element** *index element\_0 element\_1* ...

Selects the elements for which the **condif\_radiation\_normal\_edge** record with the same *index* should be applied.

**6.94 condif\_radiation\_edge\_normal\_element\_node** *index element node\_0 node\_1 ...* 

Selects the element and local node numbers for which the **condif\_radiation\_edge\_normal** record with the same *index* should be applied.

**6.95 condif\_radiation\_edge\_normal\_element\_group** *index element\_group\_0 element\_group\_1* ...

Selects the element groups for which the **condif\_radiation\_normal\_edge** record with the same *index* should be applied.

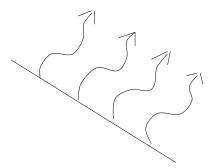
**6.96 condif\_radiation\_edge\_normal\_element\_side** *index element\_0 element\_1...side* 

Selects the elements and side number for which the **condif\_radiation\_edge\_normal** record with the same *index* should be applied.

**6.97 condif\_radiation\_edge\_normal\_geometry** *index geometry\_entity\_name geometry\_entity\_index* 

Selects the area for which the **condif\_radiation\_edge\_normal** record with the same *index* should be applied.

In stead of a number of nodes also, for example, **-geometry\_line 1** can be used in 2D, indicating that the nodes on line 1 radiate heat. The total edge of an element must be inside



the geometry for the force to become active. For 2D elements the border lines are edges. For 3D elements the border surfaces are edges. See also: **condif radiation edge normal**.

#### 6.98 condif radiation edge normal node index node\_0 node\_1...

Selects the nodes for which the **condif\_radiation\_edge\_normal** record with the same *index* should be applied. This is only available for linear elements. The *node\_0* etc. specifies the global node numbers.

#### 6.99 contact apply index switch

If *switch* is set to **-yes**, the contact algorithm is used. If *switch* is set to **-no**, the contact algorithm is not used. This is done for all timestep records.

See also control contact apply.

#### 6.100 contact heat generation factor

This *factor* specifies how much of the frictional energy is transformed into heat (this only makes sense if *friction* in **contact\_plasti\_friction** is not zero, and if **condif\_temperature** is initialized). The *factor* should be between 0 and 1. See also **contact\_target\_geometry**.

#### 6.101 contact penalty pressure pressure\_penalty

The *pressure\_penalty* should be given some high value if the pressure is freely linked at the surfaces of contactor and target. See also **contact target geometry**.

#### 6.102 contact penalty temperature temperature penalty

The *temperature\_penalty* should be given some high value if free heat exchange between contactor and target is possible. See also **contact target geometry**.

#### 6.103 contact penalty velocity velocity\_penalty

The velocity\_penalty essentially puts a spring between the contactor and the target if penetration occurs. Iterations (see control\_timestep\_iterations) are needed; more iterations are needed if the penalty factor is higher. See also contact target geometry.

#### 6.104 contact plasti friction friction

See contact target geometry.

### 6.105 contact\_target\_element\_group\_element\_group\_0 element\_group\_1

This records defines the element groups for which the elements function as target in a contact analysis. It is advised to use different element groups for the contacting and target elements, so that the contact algorithm can distinguish between both. The target element group should consist of more than 1 layer of elements in contact direction (so only one layer of target elements is not allowed). The contacter should be smaller of size than the target.

See also contact target geometry.

#### 6.106 contact target geometry index geometry\_entity\_item geometry\_entity\_index

Attention: the contact algorith is experimental up to now, and may not work for all calculations.

This record specifies a contact geometry. Contacting nodes are forced to stay at the outward normal side of the contact geometry.

The allowed geometries and their material outward normals are listed below

- If a geometry point is used in 1D, the normal is in positive x-direction.
- If a **geometry** <u>line</u> is used in 2D, the normal is the outer product of 3-direction and the line direction (from point 0 to point 1).
- If a geometry circle is used in 2D, the normal is the outward direction at the circle.

- If a **geometry\_circle** is used in 3D, the normal is the outward direction on the circle surface.
- If a **geometry\_ellipse** is used in 2D, the normal is the outward direction at the ellipse.
- If a **geometry\_sphere** is used in 3D, the normal is the outward direction at the sphere.
- If a geometry polynomial is used in 2D, the normal is in positive y-direction.
- If a geometry polynomial is used in 3D, the normal is in positive z-direction.
- If a geometry\_triangle is used in 3D, the normal is in direction of the outer product v01 \* v02 where v01 is the vector from node 0 to node 1 and v02 is the vector from node 0 to node 2.
- If a geometry\_quadrilateral is used in 3D, the normal is in direction of the outer product v01 \* v02 where v1 is the vector from node 0 to node 1 and v02 is the vector from node 0 to node 2. Only non-distorted quadrilaterals should be used.

This normal can be switched sign by setting the **contact\_target\_geometry\_switch** with the same *index* to **-yes**.

In stead of geometries, also contact with target elements will be checked. Only contact with the elements -bar2, -quad4, and -hex8 can be detected. Specify contact\_target\_element\_group for this.

The time steps should be such small, that contacting nodes penetrate the other elements only in small steps.

If a **contact\_target\_geometry** is used, then the contacting node should also be within the tolerance of the geometrical entity to be noticed!

If contact is detected, normal contact forces of size *contact\_penalty\_velocity* \* penetration are generated between the contacting node and the other element. Moreover, also a frictional force of size *friction* \* normal force is generated (see **contact\_plasti\_friction**).

With contact you need more iterations the normal, say 5 or more. See **control\_timestep\_iterations** how to define the number of iterations.

#### 6.107 contact\_target\_geometry\_switch index switch

See contact target geometry.

#### 6.108 control bounda relax index switch

With this **control\_bounda\_relax** you can require Tochnog to store the nodal righthand-sides; for example external nodal forces for nodes with prescribed velocities. These stored nodal right-hand-sides can later be used to relax prescribed boundary conditions; for example a prescribed velocity is removed and substituted by the stored external righthand-side (external force) and slowly set to zero by multiplication with a time function as specified with **bounda\_force** in combination with **bounda\_time**. With the **control\_bounda\_relax\_geometry** record with the same index you can select a specific geometry for which the storing will be done.

A typical example can be found in the **relax1.dat** file in your distribution.

#### 6.109 control\_bounda\_relax\_geometry\_index geometry\_item\_name geometry\_item\_index

See control bounda relax.

#### 6.110 control change dataitem apply index switch

If *switch* is set to **-no**, any **change**\_**dataitem**<sup>\*</sup> data in the input file will be ignored. This is done for timestep records with the same index.

#### 6.111 control check data index switch

If *switch* is set to **-yes** the in-core database is checked at some moments during the calculation, for the specified control index. You can try this option in case you experience unexpected behavior.

#### 6.112 control contact apply index switch

If *switch* is set to **-yes**, the contact algorithm is used. If *switch* is set to **-no**, the contact algorithm is not used. This is done for timestep records with the same index.

Default *switch* is set to **-yes**. See also **contact apply**.

#### 6.113 control convection apply index switch

If *switch* is set to  $-\mathbf{yes}$ , the convection of a material with respect to the mesh is allowed. If *switch* is set to  $-\mathbf{no}$ , the convection of a material with respect to the mesh is not allowed. This is done for timestep records with the same index. See also **convection apply**.

#### 6.114 control\_data\_activate index data\_item\_name\_0 data\_item\_name\_1 ... switch

With this record you can set data items to become activated if *switch* is set to **-yes** or de-activated if *switch* is set to **-no**. The *data item name* specifies a data record name.

# 6.115 control\_data\_arithmetic index data\_item\_name data\_item\_index data\_item\_number operat

This record allows you to change a data item. With *data\_item\_name data\_item\_index data\_item\_number* you select which data item to change. It will be changed with value *val* as specified in the corresponding **control\_data\_arithmetic\_double** record. With *operat* you select how to change the data item; possibilities are **-plus**, **-minus**, **-multiply** and **-divide**.

In stead of a specific index *data\_item\_index* you can also specify a range -ra ... -ra.

In case you specify **-all** for *data\_item\_number* the specified value will be used for all numbers of the record.

#### 6.116 control data arithmetic double index val

See control\_data\_arithmetic.

#### 6.117 control data copy index data\_item\_from data\_item\_to

Copy data item *data\_item\_from* to *data\_item\_to*. The user is responsible to apply only logic copy actions.

Normally the *data\_item\_from* and *data\_item\_to* should have the same length. As a special option however, you can copy **node\_inertia** to **node\_force** records, while using a **control\_data\_copy\_factor** of -1. This allows you to substitute material mass inertia by static nodal forces, for the remainder of the calculation. This in fact is the d'alembert principle.

#### 6.118 control data copy factor index factor

Multiplication factor for control data copy.

### **6.119 control\_data\_copy\_index** *index data\_item\_from index\_from data\_item\_to index\_to*

Copy data item *data\_item\_from* with index *index\_from* to *data\_item\_to* with *index\_to*. The user is responsible to apply only logic copy actions.

#### 6.120 control data copy index factor index factor

Multiplication factor for control data copy index.

#### 6.121 control data delete index data\_item\_name index\_range

Delete one or more data items. The *index\_range* is a number (e.g. 3) or a range (-ra ... -ra, or -all).

If *index data\_item\_name* is a nodal item (for example **node** or **node\_dof**) then *index\_range* can also be a geometrical entity (for example **-geometry\_line 1** or so), and the item will be deleted for nodes located on the geometrical entity.

If *index data\_item\_name* is a element item then *index\_range* can also be a geometrical entity (for example -geometry\_line 1 or so), and the item will be deleted for elements with all nodes located on the geometrical entity.

In the example below element 1-10 and nodes 1-100 are deleted after some time in the calculation; this simulates dismantling a part of a structure somewhere in its lifetime. First, time steps with the total structure are taken; then a part of the structure is dismantled; then time steps with the remaining part of the structure are taken.

control\_timestep 10 ... control\_data\_delete 20 -element -ra -from 1 -to 10 -ra control\_data\_delete 21 -node -ra -from 1 -to 100 -ra ... control\_timestep 30 ...

If an element or node is deleted, then also the corresponding records will be deleted. See also **control data put**.

#### 6.122 control\_data\_put index data\_item\_name index\_range number\_0 number\_1 ...

Puts one or more data items.

The *index\_range* is a number (e.g. 3) or a range (**-ra** ... **-ra**, or **-all**). The **-all** option for *index\_range* is only available for nodal data items (like **node** or **node\_dof**). If *data\_item\_name* is a nodal item then *index\_range* can also be a geometrical entity (for example **-geometry\_line 1** or so), and the item will be put for nodes located on the geometrical entity. If *data\_item\_name* is a element item then *index\_range* can also be a geometrical entity (for example **-geometry\_line 1** or so), and the item then *index\_range* can also be a geometrical entity (for example **-geometry\_line 1** or so), and the item then *index\_range* can also be a geometrical entity (for example **-geometry\_line 1** or so), and the item will be put for elements with all nodes located on the geometrical entity.

With *number\_0 number\_1* etc. you can set which value should be put. For example only using 3 for *number\_0* then you only want to set the fourth value for the data item (remember that numbering starts at 0). To specify the numbers for dof's you can also specify names like -velx, -sigxx, etc. In case you specify -all, then all values should be given in control data put double or control data put integer.

The values to be put should be specified in a **control\_data\_put\_double** record for real data or in a **control\_data\_put\_integer** record otherwise. You should specify a value for each and every specified number.

If the data item already exists it is overwritten; else a new record will be generated.

See also control data delete.

#### 6.123 control data put double index ....

See control data put.

#### 6.124 control data put integer index ...

See control\_data\_put.

#### 6.125 control data save index switch

If *switch* is set to **-yes** save the status of strains, stresses, displacements, etc. At a later point in the calculation you can plot with gid data relative to these saved data with **control print gid save difference**.

• • •

```
control_timestep 10 ...
control_data_save 20 -yes
...
control_timestep 30 ...
control_print_gid 40 -separate_sequential
control_print_gid_save_difference 40 -yes
...
```

#### 6.126 control\_dependency\_apply index switch

If *switch* is set to **-yes**, dependencies as specified with **dependency\_diagram** and **dependency\_item** are included in the calculation. If *switch* is set to **-no**, these dependencies are not included. This is done for timestep records with the same index.

Default, if **control\_dependency\_apply** is not specified, then **dependency\_apply** will be used.

## 6.127 **control\_distribute** index distribution\_type data\_item\_name data\_item\_index data\_item\_number

Apply a random number, based on a **-lognormal** or **-normal** distribution, to the *data\_item\_name* records. This is done for the index *data\_item\_index* and the *data\_item\_number* value in those records (0 for the first value, 1 for the second value, etc.). The *data\_item\_index* can optionally be set to **-all** in stead of a specific index, so that the distribution will be applied to all existing indices.

The *distribution\_type* should be set to **-lognormal** or **-normal**. Use the **control\_distribute\_parameters** record to set the mean value and standard deviation.

The *data\_item\_name* can be one of **group\_\*** or **node\_\***. If you specify a **group\_\***, for example **group\_materi\_elasti\_young** or so, then not the group item record self will be changed, but the item will be changed for the elements which use this record; in this way you can give a random distribution to group data like Young modulus, plastic properties, etc.

It is optionally possible to require a distribution that is correlated in space. To obtain such a correlated distribution, you need to specify the **control\_distribute\_correlation\_length** record. If the specified correlation length is larger than 1.e12 then Tochnog uses a constant G (all components have the same value). As a special option, you can specify a different distribution length in each space direction (in 2D specify 2 values, and in 3D specify 3 values).

With control distribute correlation distance you can set the maximum distance

below which data will be correlated. Above that distance tochnog will not correlate the data. Default, if **control\_distribute\_correlation\_distance** is not specified it will be taken to be 4 times the correlation length.

With **control\_distribute\_minimum\_maximum** you can set the minimum and maximum value which the random numbers are allowed to take. Numbers outside that range will be cutoff to the minimum or maximum value. A typical application would be limiting the void ratio to a range which is needed by a hypoplasticity law.

In the first example, an lognormal distribution with average 100 and standard deviation 1.2 is used to the nodal temperatures:

materi\_velocity condif\_temperature ... control\_distribute 10 -lognormal -node\_dof -all -temp control\_distribute\_parameters 10 100. 1.2 ...

In the second example, a normal distribution with average 1 and standard deviation 1.e-3 is used to the y coordinate of the nodes:

control\_distribute 10 -normal -node -all 1 control\_distribute\_parameters 10 1. 1.e-3

. . .

In the third example, a normal distribution with average 10 and standard deviation 1. is used to the young's modulus of group 7:

control\_distribute 10 -normal -group\_materi\_elasti\_young 7 0 control\_distribute\_parameters 10 10. 1.

This **control\_distribute\_\*** is presently only available on linux computers. The **control\_distribute\_\*** should be before all **control\_reset\_dof** (thus have a higher index).

#### 6.128 control distribute correlation distance index maximum\_distance

6.129 control\_distribute\_correlation\_length index correlation\_length

See control distribute.

6.130 control\_distribute\_minimum\_maximum index minimum maximum

See control distribute.

#### 6.131 control distribute parameters index mean\_value standard\_deviation

See control distribute.

#### 6.132 control distribute seed index seed

For experts only. With this record you can specify the seed which will be used to start the random series of numbers. Use a positive integer value.

As a special option you can set *seed* to **-new** then Tochnog will self choose a seed. As a special option you can set *seed* to **-old** then Tochnog will use the previous seed.

#### 6.133 control element group index switch

If *switch* is set to **-yes** the records **area element group** with be evaluated for the current control index. This option should not be used if the control index is also used for timesteps.

#### 6.134 control element group apply index number

See element group apply.

#### 6.135 control geometry moving *index* -initialise

Initialise all **geometry\_moving** records. That is, determine for all elements at which time they will be excavated by the **geometry\_moving** entities in the remainder of the calculation.

#### 6.136 control groundflow consolidation apply index switch

If switch is set to -no, then the material divergence part in the groundflow equation is skipped.

Attention: If you want consolidation in geotechnics then set the *switch* to **-yes**. If you do not want consolidation in geotechnics then set the *switch* to **-no**.

This is done for timestep records with the same index.

Default, if control\_groundflow\_consolidation\_apply is not specified, then ground-flow consolidation apply will be used.

#### 6.137 control groundflow nonsaturated apply index switch

If *switch* is set to **-no**, then nonsaturated groundflow data (eg van Genuchten) will not be applied; only saturated data will be used.

Default, if control\_groundflow\_nonsaturated\_apply is not specified, then groundflow nonsaturated apply will be used.

6.138 control inertia apply index switch\_0 switch\_1 ...

If  $switch_0$  is set to -yes, the corresponding inertia term is included (material mass, heat capacity, ...). The same for the other switches. A switch should be specified for each of the principal dof's. See the 'input file - data part - introduction - types of dof's' section for an explanation about principal dof's. The sequence of the principal dof's is in the order as initialised in the **initia** ... **end initia** part.

As a special option you can specify only one switch, and then the specified value will automatically be used for all principal dof's.

This **control** inertia apply record is applied for timestep records with the same index.

Default, if **control inertia apply** is not specified, then **inertia apply** will be used.

#### 6.139 control input index switch

If *switch* is set to **-yes** Tochnog reads an extra piece of input from the file *index.***dat**. The piece of input needs to be closed by two **end\_data** statements. Comments ( ... ) are not allowed. All defines and arithmetics cannot be used.

#### 6.140 control materi damage apply index switch

If *switch* is set to **-no**, any damage data in the input file will be ignored. This is done for timestep records with the same index.

This option is convenient for testing your input file just linear, without the need to outcomment each and every part with damage data. See also **materi damage apply**.

#### 6.141 control materi dynamic index factor

Same as materi dynamic but now only for timesteps with the same control index.

#### 6.142 control materi elasti k0 index switch

See group materi elasti k0.

#### 6.143 control materi failure apply index switch

If *switch* is set to **-no**, any damage data in the input file will be ignored. This is done for timestep records with the same index.

This option is convenient for testing your input file just linear, without the need to outcomment each and every part with failure data. See also **materi** failure apply.

#### 6.144 control materi plasti hypo masin ocr apply index switch

If *switch* is set to -yes the OCR will be applied. If *switch* is set to -no the OCR will not be applied.

Default *switch* is **-no**.

### 6.145 control\_materi\_plasti\_hypo\_masin\_clay\_ocr\_apply index switch

If switch is set to **-yes** the OCR will be applied. If switch is set to **-no** the OCR will not be applied.

Default switch is -no.

#### 6.146 control materi plasti hardsoil gammap initial index switch

See theory section on hardsoil.

#### 

If *switch* is set to **-yes** the initial void ratio is corrected for pressure dependency; see the theory section. This is done for the first timestep in the corresponding **control\_timestep** record with the same *index*. Default *switch* is set to **-no**.

#### 6.148 control\_materi\_plasti\_hypo\_niemunis\_visco\_ocr\_apply index switch

If *switch* is set to **-yes** the OCR will be applied. If *switch* is set to **-no** the OCR will not be applied.

Default *switch* is **-no**.

#### 6.149 control materi plasti hypo substepping index switch

If *switch* is set to **-yes** substepping will be applied in hypoplasticity routines. If *switch* is set to **-no** substepping will not be applied in hypoplasticity routines.

If this record is not specified the record materi plasti hypo substepping will be used.

#### 6.150 control materi plasti tension apply index switch

If switch is set to -no, any tension-plasticity data in the input file will be ignored. This is done for timestep records with the same index.

See also materi plasti tension apply.

#### 6.151 control materi plasti visco apply index switch

If *switch* is set to **-no**, any visco-plasticity data in the input file will be ignored. This is done for timestep records with the same index. See also **materi plasti visco apply**.

#### 6.152 control materi updated apply index switch

If *switch* is set to **-no**, any **-updated** material memory will be set to **-updated\_linear**. If *switch* is set to **-yes**, any non-specified material memory will be set to **-updated**. This is done for timestep records with the same index.

#### 6.153 control materi undrained apply index switch

See group <u>materi</u> <u>undrained</u> <u>capacity</u>. Default, if <u>control</u> <u>materi</u> <u>undrained</u> <u>apply</u> is not specified, *switch* is set to **-yes**.

#### 6.154 control materi viscosity apply index switch

If *switch* is set to **-no**, any viscosity in the input file will be ignored. This is done for timestep records with the same index.

### 6.155 control\_mesh\_activate\_gravity\_apply index index\_0 index\_1

With this record you can specify which of the **mesh\_activate\_gravity\_\*** records should be applied, by specifying the indices of the records that should be applied. In case this **control\_mesh\_activate\_gravity\_apply** is not given, all **mesh\_activate\_gravity\_\*** records will be applied. As a special option you can use **-all** indicating that all of the **mesh\_activate\_gravity\_\*** records should be applied (this is the same as not specifying the **control\_mesh\_activate\_gravity\_apply** record at all). As another special option you can use **-none** indicating that none of the **mesh\_activate\_gravity\_\*** records should be applied.

See also mesh activate gravity time.

#### 6.156 control\_mesh\_adjust\_geometry index geometry\_entity\_item\_0 geometry\_entity\_index\_0 geometry\_entity\_item\_1 geometry\_entity\_index\_1

The nodes of the geometry entity 0 are replaced such that they neatly follow the boundary of geometry 1. In this way, it is easy to make a mesh with elements precisely in specific regions, if this is required to give separate **element\_group** data (e.g. materials) to the geometry and it is too difficult to make the mesh at once OK for this.

The created mesh may be quite distorted.

#### 6.157 **control\_mesh\_change\_element\_group** index element\_group\_0 element\_group\_1

Change the group number **element\_group** of elements from *element\_group\_0* to *element\_group\_1*. The index should not be equal to a timestep index.

#### 6.158 control mesh convert index switch

If *switch* is set to **-yes**, tochnog will automatically convert elements:

- -bar2 in 2D to -quad4 if the element is an interface
- -bar3 in 2D to -quad6 if the element is an interface
- -tria3 in 3D to -prism6 if the element is an interface
- -tria6 in 3D to -prism12 if the element is an interface
- $\bullet$  -quad4 in 3D to -hex8 if the element is an interface
- -quad8 in 2D to -quad6 if the element is an interface
- -quad8 in 3D to -hex18 if the element is an interface
- -quad9 in 2D to -quad6 if the element is an interface
- -quad9 in 3D to -hex18 if the element is an interface
- -hex20 in 3D to -hex18 if the element is an interface
- -hex20 in 3D to -hex27 if the element is not an interface
- -prism15 in 3D to -prism12 if the element is an interface
- -prism15 in 3D to -prism18 if the element is not an interface

For an interface you need to specify interface data in the **group \_\_interface...** For example the **-bar2** is connected to two nodes, whereas the converted **-quad4** is connected to four nodes. In a similar manner all other converted elements also get extra nodes. This options makes it easy to obtain a mesh with interface elements. For example generate with GID in a 2d mesh bar elements, insert group data, and use **control\_mesh\_convert** to generate the interface elements. This generation of interfaces only works properly if certain conditions are satisfied:

• Each interface needs to have only isoparametric neighbours which have a total side in common with the interface. For example a **-hex8** interface should only have **-hex8** neighbours.

• Surfaces with interface elements should not intersect with another surface with interface elements.

The new generated nodes will be connected to existing neighbouring element at the interfaces. The **control\_mesh\_convert** tries to do that automatically correct. You can help however by specifying in the record **control\_mesh\_convert\_element\_group** element groups which are located at one side of the interfaces (for example the groups of a pile in soil when an interface is generated between pile and soil).

Example in which a -bar2 interface becomes a -hex8 interface:

number\_of\_space\_dimension 3
...
end\_initia
...
element 1 -bar2 101 102
element\_group 1 10
...
group\_interface 10 -yes
...
control\_mesh\_extrude 100 ...
...
control\_mesh\_convert 110 -yes
...

If switch is set to -no, tochnog will not convert elements.

# **6.159 control\_mesh\_convert\_element\_group index** *element\_group\_0 element\_group\_1* ...

See control mesh convert.

#### 6.160 control mesh convert quad9 quad6 index dir

Convert **quad9** into **quad6** is a 2D calculation. With dir you can decide in which isoparametric direction of the **quad9** nodes should be deleted (so that becomes the linear direction in the **quad6** element). Set dir either to  $-\mathbf{x}$  or to  $-\mathbf{y}$ .

#### 6.161 control mesh convert tria6 tria3 index switch

Convert tria6 into tria3 is a 2D calculation. This is done if *switch* is set to -yes.

#### 6.162 control mesh copy index move\_x move\_y move\_z

This option copies the mesh. Thus you get twice as much elements and nodes. Each new node is moved *move\_x move\_y move\_z* relative to the corresponding old node. In 1d you only should specify *move\_x*. In 2d you only should specify *move\_x move\_y*.

#### 6.163 **control\_mesh\_cut\_geometry** index geometry\_item\_name geometry\_item\_index

This command cuts away a part of the mesh, as defined by geometry\_item\_name geometry\_item\_index. The cut away mesh can be substituted by its nodal forces. Actually, with control\_mesh\_cut\_force you can set for each direction if the nodal force of the cut away mesh should be applied to the remaining mesh. This will be done if you set the corresponding *switch* in control\_mesh\_cut\_force to -yes. In 2D you should set two switches, and in 3D you should set 3 switches.

See mesh\_cut\_1.dat and mesh\_cut\_2.dat in the test directory for examples. See earthquake 2.dat how this control\_mesh\_cut\_geometry command can save you computing time in dynamic calculations with many timesteps.

#### 6.164 control mesh cut force index switch\_0 switch\_1 switch\_2

See control mesh cut geometry.

#### 6.165 control mesh delete element index number 0 number 1 ...

The elements with numbers *number\_0 number\_1* will be deleted. Otherwise the same as **control mesh delete geometry**.

#### 6.166 **control\_mesh\_delete\_geometry** index geometry\_entity\_item geometry\_entity\_index

All elements which are part of the geometry item are deleted. In this way, it is easy to make a mesh with holes, tunneling systems in ground, etc. Remaining nodes in the geometry, are moved onto the edge of the geometry if the corresponding **control\_mesh\_delete\_geometry\_move\_node** 

record with the same index is set to **-yes**; (otherwise, the remaining nodes are left inside the geometry).

For a **geometry\_point**, elements inside the tolerance distance of the point will be deleted. For a **geometry\_circle**, elements in the total inner area of the circle radius plus its tolerance will be deleted. Likewise for other geometries.

If you combine this record with a **control\_timestep** record, then the element will be slowly deleted, starting from a complete element at the start of the timestep up to no element at the end of the timestep. This is accomplished by reducing the nodal forces of the elements slowly to zero; at the end of the timestep, the element is deleted completely. This might be useful for a better convergence behavior of the iterative process.

If an element is being deleted, **element** <u>empty</u> is automatically set to **-empty**, even if the element is not completely deleted yet. This allows you to look with GID 'behind elements that are being deleted' (see also **element empty** and **control print gid empty**).

 $See also \ {\bf control\_mesh\_delete\_geometry\_move\_node, \ {\bf control\_mesh\_delete\_geometry\_element} and \ {\bf control\_mesh\_delete\_geometry\_element\_group.}$ 

#### 6.167 control mesh delete geometry direct index switch

If *method* is set to **-yes** the elements inside the geometry will be deleted direct, as opposed to slowly over the timesteps.

# **6.168 control\_mesh\_delete\_geometry\_element** *index element\_name\_0 element\_name\_0* ...

Only elements with names *element\_name\_0* etc. will be deleted if the **control\_mesh\_delete\_geometry** (with the same index) is used. For example, *element\_name\_0* is **-quad4**, **-beam**, etc.

If this record is not specified all elements in the geometry will be deleted.

# **6.169 control\_mesh\_delete\_geometry\_element\_group** *index element\_group\_0 element\_group\_1* ...

Only elements from group *element\_group\_0* etc. will be deleted if the **control\_mesh\_delete\_geometry** (with the same index) is used.

#### 6.170 control\_mesh\_delete\_geometry\_factor index factor\_0 factor\_1

The elements deleted by **control\_mesh\_delete\_geometry** (with the same index), will be deleted by a factor *factor\_0* at the start of the timesteps up to a factor *factor\_1* at the end of the timesteps. If the **control\_mesh\_delete\_geometry** is not used in combination with timesteps, then directly *factor\_1* will be applied.

If  $factor_1$  exceeds 1. - 1.e - 10 an element will be completely deleted from the calculation, that is the **element** record will be removed and cannot be reactivated in any way later in the calculation.

If this record is not specified then *factor* 0 = 0 and *factor* 1 = 1.

#### 6.171 control mesh delete geometry method index method

Determines the condition on which an element will be considered part of the geometry to be deleted. If *method* is set to **-all** then all element nodes should be part of the geometry. If *method* is set to **-any** then any of the element nodes should be part of the geometry. If *method* is set to **-average** then the average element coordinate should be part of the geometry.

Default this record is **-all**.

. . .

See also control\_mesh\_delete\_geometry.

#### 6.172 control mesh delete geometry move node index switch

Determines if remaining nodes inside a deleted geometry, are moved onto the edge of the geometry (**-yes**) or not (**-no**). Moving nodes makes that the element mesh exactly fits the deleted geometry, but may also lead to heavily distorted elements. Default this record is **-no**.

See also control mesh delete geometry.

#### 6.173 control mesh delete geometry projection type index type

This record allows you to control what geometry will actually be deleted. Set *type* to -**project\_inside** or -**project\_exact**. For example if the geometry is a **geometry\_circle** then -**project\_inside** means that everything inside the circle will be deleted, whereas -**project\_exact** means that everything within a tolerance from the circle edge will be deleted. Default *type* is -**project\_exact**.

#### 6.174 control mesh delete geometry stop index switch

If *switch* is set to **-yes**, any deleting of elements in geometries will be stopped. That is, all remaining delete factors from **control\_mesh\_delete\_geometry\_factor** will be destroyed and all elements will become fully active again.

In combination with global\_element\_dof\_apply -yes, the elements which become active again will take their strains, stresses etc. of the moment just before being deleted! If you want to lower the stresses or strains or so, then consider using control reset dof.

In combination with **global\_element\_dof\_apply -no**, the elements which become active again will take their strains, stresses etc. from the nodes.

#### 6.175 control\_mesh\_delete\_geometry\_stop\_geometry index geometry entity name geometry entity index

Only do the **control\_mesh\_delete\_geometry\_stop** for elements part of the geometrical entity specified in this **control mesh delete geometry stop geometry**.

#### 6.176 control mesh delete small index eps

At the end of a timestep, an element will be deleted when its volume has become smaller than eps.

## 6.177 **control\_mesh\_duplicate\_element\_group** index element\_group\_old element\_group\_new

Use this command to duplicate elements from group *element\_group\_old* to new elements with group *element\_group\_new*. The new elements get the same nodes as the old original elements.

### 6.178 control\_mesh\_element\_group\_apply index group\_0 group\_1

If you specify this record, only the element groups specified will be evaluated in the timesteps with the same index. Default, if **control\_mesh\_element\_group\_apply** is not specified, all elements groups will be used.

#### 6.179 control mesh extrude index z0 z1 z2 ...

Option to extrude a 2D mesh to 3D. The 2D mesh has x,y,z coordinates, with z=0. The 3D mesh will have x,y,z coordinates. You need to specify in the initialisation part number of space dimensions to 3.

With z0, z1, z2 etc. you specify the coordinates of the layers to which the 2D coordinates will be extruded. With n0, n1, n2 you specify the number of elements that will be generated in each layer; n0 specifies the number of elements between z0 and z1, n1 specifies the number of elements between z1 and z2, etc.; for the last n-value you always should use a 1 (this is a dummy value, that is not used for any layer at all).

Extrusion must be done before doing mesh refinements, mesh splitting, etc.

#### 6.180 control mesh extrude direction index dir

Default extrusion is done in the global z-direction. Optionally you can set dir to  $-\mathbf{y}$  and then extrusion is done in global y-direction.

#### 6.181 control mesh extrude element index name

If you extrude -tria6 elements, you can set *name* either to -prism12 or -prism18. Then either the 12 node or 18 node prismatic elements will be generated. Default, if this control mesh extrude element is not set, then -prism18 is used for *name*.

See also control mesh extrude n.

#### 6.182 control\_mesh\_extrude\_contact\_spring\_element\_group index element\_group\_0 element\_group\_1 ...

See control mesh extrude contact spring element group new.

6.183 control\_mesh\_extrude\_contact\_spring\_element\_group\_new\_ index element\_group\_new\_0 element\_group\_new\_1 ...

If this record is specified, then a contact spring is generated between each start node and end node in the extrude direction. This option comes handy, when you want to use these contact springs to enforce that the nodes on the start plane get the same displacements as the nodes on the end plane, which models that the extruded mesh is in fact part of a very long domain with no variations in the longitudinal direction of the domain. The contact springs get group number *element\_group\_new\_0* when its node is attached to an element with old group *element\_group\_0*. The contact springs get group number *element\_group\_new\_1* when its node is attached to an element with old group *element\_group\_1*. Etc. The old groups are specified in the **control\_mesh\_extrude\_contact\_spring\_element\_** record. If the contact spring's node is attached to more than one old group, the first specified old group, and corresponding new group, will be used.

As a special option, if you specify in **control\_mesh\_extrude\_contact\_spring\_element\_group\_new** only one new element group number, then all contact springs will be placed on that group.

# **6.184 control\_mesh\_extrude\_element\_group\_new** index element\_group\_old\_0 element\_group\_old\_1 ... element\_group\_new\_00 element\_group\_new\_01 ... element\_group\_new\_10 element\_group\_new\_11 ...

With this option you set the element \_group number of the new extruded elements.

With *element\_group\_old\_0*, *element\_group\_old\_1* etc. you specify the old element\_group numbers of the 2D elements (which will be extruded). For these old groups, you specify for each layer in z-direction what the new element\_group numbers of the extruded 3D elements should be. For example, *element\_group\_new\_00*, *element\_group\_new\_01* etc. give for *element\_group\_old\_0* what the element\_group numbers of the new extruded elements will be (for each z layer).

You need to specify element group numbers for each and every z layer.

If a new element group is set to a negative number in this **control\_mesh\_extrude\_element\_group\_new** the elements will not be generated.

See also control mesh extrude.

6.185 control mesh extrude n index n0 n1 n2 ...

See control mesh extrude.

### 6.186 control\_mesh\_generate\_beam index element\_group geometry\_entity\_item geometry\_entity\_index

The same as control mesh generate truss, now for beams however.

#### 6.187 control\_mesh\_generate\_contact\_spring index element\_group geometry\_entity\_item geometry\_entity\_index

Generate -contact\_spring2 springs for nodes which have the same position in space. This can be used to connect these nodes with spring elements, so to model a contact area. Only nodes located on the specified geometry entity will be used.

The generated springs will get an **element\_group** record with value *element\_group*. So in that element group you can put the properties of the contact springs.

With the **control\_mesh\_generate\_contact\_spring\_element** record you can set between which elements the contact\_springs should be generated. For example use -**quad4** and -**truss beam** if you want to generate contact\_springs between those elements.

If control\_mesh\_generate\_contact\_spring\_element\_group (with the same index) is used, contact springs will only be generated between elements of the groups *element\_group\_0*, *element\_group\_1* etc.

#### 6.188 control\_mesh\_generate\_contact\_spring\_element index element\_0 element\_1

See control mesh generate contact spring.

6.189 control\_mesh\_generate\_contact\_spring\_element\_group index element\_group\_0 element\_group\_1 ...

See control mesh generate contact spring.

**6.190** control\_mesh\_generate\_interface index element\_group\_0 element\_group\_00 element\_group\_01 element\_group\_1 element\_group\_10 element\_group\_11 ...

With this record you can generate interface elements.

The interface elements will be given an **element\_group** record *element\_group\_0* if the interface is between *element\_group\_00* and *element\_group\_01*. The interface elements will be given an **element\_group** record *element\_group\_1* if the interface is between *element\_group\_10* and *element\_group\_11*. The interface elements will be given an **element\_group** record *element\_group\_21* if the interface is between *element\_group* record *element\_group\_220* and *element\_group\_220* and *element\_group\_21*. Etc, etc.

The groups element\_group\_00, element\_group\_10, element\_group\_20, etc. should be on one side. The groups element\_group\_01, element\_group\_11, element\_group\_21, etc.

should be on the opposite side.

Between two linear 2d elements -quad4 interfaces will be generated. Between two quadratic 2d elements -quad6 interfaces will be generated. Between two -hex8 elements a -hex8 interface will be generated. Between two -hex27 elements a -quad18 interface will be generated. Between two -tet4 elements a -prism6 interface will be generated. Between two -tet10 elements a -tria12 interface will be generated. Between two -prism6 elements a -prism6 interface will be generated a -prism6 elements a -hex8 elements a -hex8 interface will be generated on sides with 3 nodes. Between two -prism6 elements a -hex8 interface will be generated on sides with 4 nodes. For other situations no interface element will be generated.

Crossing interfaces are not allowed, eg in 2d you should not have locally two connecting lines with interfaces and in 3d you should not have locally two connecting surfaces with interfaces.

Interfaces can only be generated between exactly two elements. You cannot generate interface where three elements connect; for example ypu cannot generate an interface at the common side of two quad4 elements if there is also a truss along that common side.

If you want the interface to connect, you really should do for example:

```
control_mesh_generate_interface 10 20 30 31 20 40 41
```

which takes care that the interfaces generated by this command are connected together. If you would have used the following:

control\_mesh\_generate\_interface 10 20 30 31 control\_mesh\_generate\_interface 11 20 40 41

The interfaces generated by the two commands will not connect.

See also control mesh generate interface geometry and control mesh generate interface method

### **6.191 control\_mesh\_generate\_interface\_geometry** *index geometry\_item\_name geometry\_item\_index*

Restrict the generation of interfaces for the **control\_mesh\_generate\_interface** record with the same index to the geometry specified in this **control\_mesh\_generate\_interface\_geometry** record.

#### 6.192 control\_mesh\_generate\_interface\_method index method\_select method\_generate

If you set *method\_select* to **-element\_geometry** the **control\_mesh\_generate\_interface** will select with **element\_geometry** between which elements interfaces will be generated.

If you set *method\_generate* to **-element\_geometry** the **control\_mesh\_generate\_interface** will generate **element\_geometry** records for the interface elements, in stead of **element\_group** records.

So for example using -element\_geometry -element\_geometry tells that the control\_mesh\_generate\_interface in fact is index element\_geometry\_0 element\_geometry\_00 element\_geometry\_01 element\_geometry\_1 element\_geometry\_10 element\_geometry\_11 ....

Default, if **control\_mesh\_generate\_interface\_method** is not specified, it is set to **-element group**.

#### 6.193 control\_mesh\_generate\_spring1 index element\_group geometry\_entity\_item geometry\_entity\_index

Generate -**spring1** springs for nodes. Only nodes located on the specified geometry entity will be used.

The generated springs will get an **element\_group** record with value *element\_group*. So in that element group you can put the properties of the springs (see **group\_spring\_stiffness** etc.).

#### 6.194 control\_mesh\_generate\_spring2 index element\_group geometry\_entity\_item geometry\_entity\_index

Generate -spring2 springs for nodes which have the same position in space. This can be used to connect these nodes with spring elements. Only nodes located on the specified geometry entity will be used.

The generated springs will get an **element\_group** record with value *element\_group*. So in that element group you can put the properties of the springs (see **group\_spring\_stiffness** etc.).

Typically you can use this option to connect meshes which were generated with different **control mesh macro** records or so.

If you need interfaces, then afterwards use a **control\_mesh\_convert** to turn the generated surface elements into real interface elements.

#### 6.195 control\_mesh\_generate\_truss index element\_group geometry\_entity\_item geometry\_entity\_index

Generate trusses for nodes which are neighbor in space (that is, for nodes which are connected by an isoparametric finite element). Only nodes located on the specified geometry entity will be used.

The generated trusses will get an **element\_group** record with value *element\_group*. So in that element group you can put the properties of the trusses (see **group\_truss\_elasti\_young** etc.).

Typically you can use this option to put easy trusses somewhere in a mesh with isoparametric elements.

#### 6.196 control\_mesh\_generate\_truss\_beam index element\_group geometry\_entity\_item geometry\_entity\_index

The same as control mesh generate truss, now for truss\_beams however.

#### 6.197 control mesh generate truss beam loose index switch

This record works together with the**control\_mesh\_generate\_truss**,**control\_mesh\_generate\_beam**and**control mesh generate truss beam**records.

If *switch* is set to **-yes**, the truss or beam of truss \_ beam will not be connected to the existing nodes, but new nodes will be generated for the generated element.

Afterwards you can typically connect the truss or beam of truss\_beam to the existing mesh with constactsprings, so that the end result is that you can model frictional slip between isoparametric elements and structural elements.

See also control mesh generate contact spring.

#### 6.198 control\_mesh\_generate\_truss\_beam\_macro\_index macro\_0 macro\_1...

This record works together with the **control\_mesh\_generate\_truss**, **control\_mesh\_generate\_beam** and **control mesh generate truss beam** records.

With  $macro\_0$  etc. you can specify the indices of **control\_mesh\_macro\_\*** records. Then the trusses (or beams or truss\_beams) will only be generated for nodes coming from the mesh generated by the macro records with the specified indices.

This is handy in case you generate two neighboring meshes with macro's, and want to

generate the elements (trusses or beams or truss\_beams) in between these two meshes. Normally, both the meshes would get the extra truss (or ..) in case you use a geometry\_line or so to specify that the new elements should be generated between the two meshes (this is so, since the nodes of both meshes are located on the geometry\_line). With the present **control\_mesh\_generate\_truss\_beam\_macro** record however you can specify that the new elements should only be generated by looking at the nodes of some of the meshes, and so no double new elements will be generated in between the two meshes.

#### 6.199 control mesh generate truss beam separate index switch

This record works together with the **control\_mesh\_generate\_truss**, **control\_mesh\_generate\_beam** and **control mesh generate truss beam** records.

If *switch* is set to **-yes**, the truss or beam of truss\_beam will be generated for separate regions, not necessary connected by isoparametric finite elements.

A typical example is the generation of exactly one truss between two end points (thus no trusses along all of the isoparametric elements between the end points). For this, put the end points in a geometry set, and also use **-yes** for this **control\_mesh\_generate\_truss\_beam\_separate** record.

#### 6.200 control mesh gid batch index switch

If *switch* is set to **-yes** all **mesh\_gid\_\*** records are used to generate elements using GiD as preprocessor. In fact, the GiD program is called from inside Tochnog as batch. This makes it convenient to do everything that you need from inside the Tochnog input file, without even opening GiD as preprocessor. A simple example is as follows:

```
echo -yes
number_of_space_dimensions 2
end_initia
( define points )
mesh_gid_point_coord 1 0. 0.
mesh_gid_point_coord 2 1. 0.
mesh_gid_point_coord 3 0. 1.
mesh_gid_point_coord 4 1. 1.
( define lines, each line connects to two points )
mesh_gid_line_point 1 1 2
mesh_gid_line_point 2 2 4
mesh_gid_line_point 3 4 3
mesh_gid_line_point 4 3 1
```

( define closed surfaces, each surfaces connects to multiple lines )

```
mesh_gid_surface_line 1 1 2 3 4
( define that the elements in this surface get a group number )
mesh_gid_surface_element_group 1 -yes
( size of elements to be generated )
mesh_gid_size 0.1
( call gid in batch, this generates an extra file index.dat so 10.dat in this example )
control_mesh_gid 10 -yes
( read the mesh in 10.dat )
control_input 10 -yes
( add extra data )
( ... )
```

```
end_data
```

A more complex example which creates a circular hole in a rectangular zone is as follows:

```
echo -yes
number_of_space_dimensions 2
end_initia
( the circle will be used hollow )
mesh_gid_circle_coord 1 0.5 0.5
mesh_gid_circle_radius 1 0.3
mesh_gid_circle_hollow 1 -yes
mesh_gid_point_coord 1 0. 0.
mesh_gid_point_coord 2 1. 0.
mesh_gid_point_coord 3 0. 1.
mesh_gid_point_coord 4 1. 1.
mesh_gid_line_point 1 1 2
mesh_gid_line_point 2 2 4
mesh_gid_line_point 3 4 3
mesh_gid_line_point 4 3 1
( the surface edges are the line of the circle and the 4 specified lines )
mesh_gid_surface_line 1 1 2 3 4 5
mesh_gid_surface_element_group 1 1
mesh_gid_size 0.05
control_mesh_gid 10 -yes
control_input 10 -yes
target_item 0 -node 10 1
```

target\_value 0 0.8715 1.e-2
end\_data

For more examples see in your distribution test/other/mesh gid \*.dat.

<u>Attention</u>: Please realise that when generating gid objects (circle, cylinder, etc) also automatically points, lines, surfaces and volumes are generated. So you get more points, lines, surfaces and volumes then you specify yourself with mesh\_gid\_point\_coord, mesh\_gid\_line\_point, mesh\_gid\_surface\_line and mesh\_gid\_volume\_surface. Thus, you need to pay attention to which point, line, surface and volume numbers you specify in mesh\_gid\_line\_point, mesh\_gid\_surface\_line and mesh\_gid\_volume\_surface. To help you, this option control\_mesh\_gid\_batch will print for all mesh\_gid\_\* data the points, lines, surfaces and volumes numbers that it generates. So for each thing that you specify you get the corresponding point, line and surface numbers. It is these printed numbers you should use in the records which need these numbers: mesh\_gid\_line\_point, mesh\_gid\_volume\_surface.

<u>Attention</u>: When you specify group numbers for **mesh\_gid\_...\_group** records, you should apply the numbers 1, 2, 3, etc for the groups, and don't use gaps in the numbers.

<u>A</u>ttention: For this option to work you need to have GiD (of CIMNE) installed on your computer. Take care that the GiD path where the GiD executable is stored is set in your PATH environment symbol. Also take care that the **tochnog.gid** directory from your Tochnog distributions is copied to the GiD **problemtypes** directory.

#### 6.201 control mesh interface triangle index switch

See mesh interface triangle coordinates.

6.202 control mesh keep element index element\_0 element\_1 ...

With this option you can delete all elements except for the elements with numbers  $element\_0$ ,  $element\_1$ , etc. This enables you to clearly view some specific elements and nodes in a plot.

#### 6.203 control\_mesh\_keep\_element\_group index element\_group\_0 element\_group\_1...

With this option you can delete all elements except for the elements with group numbers element\_group\_0, element\_group\_1, etc. This enables you to clearly view some specific elements and nodes in a plot.

#### 6.204 control\_mesh\_keep\_geometry index geometry\_item\_name geometry\_item\_index

With this option you can delete all elements except for the elements present in the specified geometry. This enables you to clearly view some specific elements and nodes in a plot.

#### 6.205 control mesh keep node index node\_0 node\_1 ...

With this option you can delete all nodes except for the nodes with numbers  $node_0$ , node 1, etc. This enables you to clearly view some specific elements and nodes in a plot.

#### 6.206 control\_mesh\_macro\_index macro\_item element\_group n . . .

With this record and the **control\_mesh\_macro\_parameters** record you define a macro region. The macro region will automatically be divided into finite elements.

The type of macro region is defined by *macro\_item*. You can set *macro\_item* to a -sphere (3D), -cylinder (3D), -cylinder\_hollow (3D), -brick (3D), -rectangle (2D/3D), -circle (2D), -circle\_hollow (2D), -truss (1D/2D/3D), -truss\_beam (1D/2D/3D) and -bar (1D).

The elements to be generated will get **element** group element\_group.

With  $n \ldots$  you define how much nodes and elements will be generated. For a **-cylinder**, you need to specify the number of nodes in the length direction, the number of nodes in radial direction and the number of nodes in circ. direction (there is always only one element in radial direction). For a **-cylinder\_hollow**, you need to specify the number of nodes in the length direction. For a **-cylinder\_hollow**, you need to specify the number of nodes in circ. direction. For a **-brick**, you need to specify the number of nodes in x-direction, the number of nodes in y-direction and the number of nodes in z-direction. For a **-brick**, you need to specify the number of nodes in x-direction, the number of nodes is y-direction and the number of nodes in z-direction. For a **-circle** and **-sphere**, you need to specify 'fineness' of the mesh, which is a number 0, 1, 2, 3, ...; a higher number gives a higher fineness; typically use 3 or so. For a **-circle\_hollow**, you need to specify the number of nodes in tangential direction. For a **-rectangle**, you need to specify the number of nodes in first direction and the number of nodes. For a **-truss**, you need to specify the number of nodes. For a **-truss**, you need to specify the number of nodes. For a **-truss\_beam**, you need to specify the number of nodes.

In the following example a sphere is generated

number of space dimension 2

```
end_initia
...
control_mesh_macro 20 -sphere ...
control_mesh_macro_parameters 20 ...
```

#### 6.207 control mesh macro concentrate index ...

For the **-rectangle** macro you can specify with this **control\_mesh\_macro\_concentrate** record a mesh fineness concentration factor in the first direction and in the second direction. In each direction give a mesh fineness factor at the beginning and at the end (so two factors per direction). A smaller factor means smaller elements. The relative size of the factor determines where elements are concentrated, at the start or at the end.

#### 6.208 control mesh macro element index element\_type

With this option you can set the element type which will be generated with **control\_mesh\_macro** (with the same index). This option is only available in 2d and 3d.

For element you can use -tria3, -tria6, -quad4 and -quad9 in 2d. For element you can use -tet4, -tet10, -hex8 and -hex27 in 2d.

If this record is not specified then -bar2 (1d), -quad4 (2d) or -hex8 (3d) will be generated.

Attention: in case you choose a quadratic element the macro geometry may not be exactly followed. In this case, leave the default linear elements, and use a global mesh refinement to quadratic elements afterwards, including the geometry to follow.

#### **6.209** control mesh macro parameters index $x y \dots$

With this record you can specify the dimensions of the **control mesh macro** region.

For a **-sphere**, you need to specify the x, y, z coordinates of the middle of the sphere and the radius of the sphere. For a **-cylinder**, you need to specify the x, y, z coordinates at the start of the cylinder, the x, y, z coordinates at the end of the cylinder, the radius, the start angle and the end angle in degrees (which allows for an open section). For a **-cylinder\_hollow**, you need to specify the x, y, z coordinates at the start of the cylinder, the x, y, z coordinates at the start of the cylinder, the x, y, z coordinates at the start of the cylinder, the x, y, z coordinates at the end of the cylinder, the middle radius, the wall thickness, the start angle and the end angle in degrees (which allows for an open section). For a **-brick**, you need to specify the x, y, z coordinates at the length in x-direction, the length in y-direction, and the length in z-direction. For a **-circle**, you need to specify the x, y coordinates of the middle and also the radius. For a **-circle hollow**, you need to specify the same as for the **circle** 

and additionally the wall thickness, the start angle and the end angle in degrees (which allows for an open section). For a **-rectangle**, you need to specify the x, y coordinates of the middle, the width and the height respectively. For a **-bar**, you need to specify the x coordinate of the middle and the length respectively. For a **-truss**, you need to specify the coordinates of the start and of the end respectively.

### 6.210 control mesh map index switch

A typical piece of input file is

```
...
global_element_dof_apply -no
...
...
(input file with quadratic elements -hex20 or -hex27 or -tet10 or -prism15)
...
control_mesh_map ...-yes (map to linear elements -hex8 or -prism6 or -
tet4)
...
control_timestep ... (calculate with linear elements)
control_solver ...-matrix_pardiso (with pardiso solver)
...
control_mesh_map -yes (map back to quadratic elements)
...
control_timestep ... (calculate with quadratic elements)
...
control_solver ...-matrix_iterative_bicg (with bicg solver)
...
```

In this way, the last calculation with the quadratic elements gets as first guess for the bicg solver the solution field of the linear elements with the pardiso solver. This saves much computing time for bicg, especially in very large calculations. This strategy normally should only be used for large linear calculations. For this option always set **global\_element\_dof\_apply**-no.

#### 6.211 control mesh merge index switch

If *switch* is set to **-yes**, then nodes with the same coordinates are merged into one node.

## 6.212 control mesh merge\_eps\_coord index epsilon

Distance below which nodes will be merged. Default some small value.

#### 6.213 control mesh merge macro generate index macro\_0...

This record works together with the **control mesh merge** record.

With *macro\_0* etc. you can specify the indices of **control\_mesh\_macro\_\*** or **control\_mesh\_generate\_\*** records. Then the merging will only be done for nodes coming from the mesh generated by the macro or generate records with the specified indices.

## 6.214 control\_mesh\_merge\_geometry index geometry\_entity\_item geometry\_entity\_index

The mesh merging from **control\_mesh\_merge**, with the same index, will only be used for nodes in the geometry specified by *geometry entity item geometry entity index*.

## 6.215 control\_mesh\_merge\_geometry\_not index geometry\_entity\_item geometry\_entity\_index

The mesh merging from **control\_mesh\_merge**, with the same index, will not be used for nodes in the geometry specified by *geometry\_entity\_item geometry\_entity\_index*.

#### 6.216 control mesh mirror index axis

This option mirrors the mesh. Thus you get twice as much elements and nodes. The *axis* should either be set to  $-\mathbf{x}$ ,  $-\mathbf{y}$  or  $-\mathbf{z}$ .

## 6.217 control\_mesh\_move index move\_x\_constant move\_x\_linear\_x move\_x\_linear\_y move\_x\_linear\_z move\_y\_constant move\_y\_linear\_x move\_y\_linear\_y move\_y\_linear\_z move\_z\_constant move\_z\_linear\_x move\_z\_linear\_z move\_y\_linear\_z

This option movies the mesh. Thus you get the same amount of elements and nodes, but just moved in space.

In the x-direction a node is moved over  $move_x constant + move_x linear_x * x + move_x linear_y * y + move_x linear_z * z.$ 

Specify only data for the number of space dimensions.

#### 6.218 control mesh multiply index number\_of\_multiplications

The mesh is multiplied *number\_of\_multiplications* times. In each multiplication the mesh gets double the amount of elements, because for each element a new element is generated with the same nodes.

### 6.219 control mesh refine globally index refinement\_type

This record activates global mesh refinement or global mesh coarsening. This is not available for **-tria3** and **-tet4** elements. Either *refinement\_method* is **-h\_refinement** (more of the same elements) or *refinement\_method* is **-p\_refinement** (higher order elements).

As a special option for the -h\_refinement method, the format refine\_globally index -h\_refinement switch\_ $\xi$  switch\_ $\eta$  switch\_ $\zeta$  can be used. For example in 1D, only refine\_globally index -h\_refinement switch\_ $\xi$  should be specified. For example in the -hex8 element,  $\xi$  is the isoparametric coordinate running from the first node to the second node,  $\eta$  runs from the first node the third node and  $\zeta$  runs from the first node to the fifth node. A isoparametric direction will be refined if the corresponding switch is set to -yes. This option allows for refinement in specific directions. It should be used with care however, and only gives proper results if the  $\xi, \eta$  and  $\zeta$  directions of the elements match.

The **control\_mesh\_refine\_globally** will automatically merge nodes which have the same position in space.

Rules for old and new:

- A new generated element inherits its data items from the old element it is generated from.
- If a new generated node is placed on an old element edge it inherits those data items of the old nodes on that old edge that have a property in common; then arbitrarily the data item of one of the old nodes is taken.
- If a new generated node is placed inside an old element it inherits those data items of the old nodes of that old element that have a property in common; then arbitrarily the data item of one of the old nodes is taken.
- For all new nodes the **node\_dof** records are interpolated from the old element nodes **node\_dof** records by using the old element interpolation functions.

See also control mesh refine globally geometry.

## 6.220 control\_mesh\_refine\_globally\_geometry index geometry\_entity\_item geometry\_entity\_index

This record can be used together with the **control\_mesh\_refine\_globally** record with the same index. If all nodes of an element edge is part of the geometrical entity, the new generated nodes are also placed on that geometrical entity. This typically is used to follow curved edges of the domain.

The **control\_mesh\_refine\_locally** will automatically merge nodes which have the same position in space.

### 6.221 control mesh refine locally index percentage

An elements will be refined depending on the size of a solution variable. The solution variable can be chosen via **control mesh refine locally dof**.

The percentage of elements which will be refined is specified by *percentage*. Typically *percentage* is 10 or so.

This local mesh refinement is only available for **-bar2**, **-bar3**, **-tria3**, **-tria6**, **-tet4** and **-tet10** elements; there should be no other elements in the mesh.

See the rules for old and new at **control mesh refine globally**.

#### 6.222 control mesh refine locally dof index dof

With *dof* you can set which dofwill be used for deciding if an element should be refined. The size of the doffield will be used.

Possibilities for *dof* are: -materi\_damage, -materi\_displacement, -materi\_plasti\_kappa, -materi\_plasti\_kappa\_shear, -materi\_strain\_elasti, -materi\_strain\_plasti, -materi\_strain\_total, -materi\_stress, -materi\_velocity, -materi\_void\_fraction and

As a special option you can set *dof* to **-nothing**; then an element is refined always.

For finding localization zones (e.g. shear bands) choosing -materi\_strain\_plasti or - materi damage seems to be most robust.

See also control mesh refine locally geometry.

## 6.223 control\_mesh\_refine\_locally\_geometry index geometry\_entity\_item geometry\_entity\_index

This record can be used together with the **control\_mesh\_refine\_locally** record with the same index. If all nodes of an element edge is part of the geometrical entity, the new generated nodes are also placed on that geometrical entity. This typically is used to follow curved edges of the domain.

#### 6.224 control mesh refine locally minimal size index minimal\_size

Element with minimal size below the specified *minimal\_size* will not be refined. The minimal element size is defined as the largest node distance between nodes of the element. Default the minimal allowed size is 0.

## 6.225 control mesh refine locally not index geometry\_entity\_0 geometry\_entity\_index\_0

The refinement as specified in the **control\_mesh\_refine\_locally** record with the same index, will not be applied on the geometry specified by *geometry\_entity\_0 geometry\_entity\_index\_0*.

#### 6.226 control mesh refine locally not method index method

Set *method* to **-all** or **-any**. If *method* is set to **-all**, then the corresponding **control\_mesh\_refine\_locally\_not** is applied to elements for which all nodes are inside the specified geometry. If *method* is set to **-any**, then the corresponding **control\_mesh\_refine\_locally\_not** is applied to elements for which any of the nodes is inside the specified geometry. Default *method* is **-all**.

## **6.227 control\_mesh\_refine\_locally\_only** *index geometry\_entity\_0 geometry\_entity\_index\_0*

The refinement as specified in the **control\_mesh\_refine\_locally** record with the same index, will only be applied on the geometry specified by *geometry\_entity\_0 geometry\_entity\_index\_0*.

#### 6.228 control mesh refine locally only method index method

Set *method* to **-all** or **-any**. If *method* is set to **-all**, then the corresponding **control\_mesh\_refine\_locally\_only** is applied to elements for which all nodes are inside the specified geometry. If *method* is set to **-any**, then the corresponding **control\_mesh\_refine\_locally\_only** is applied to elements for which any of the nodes is inside the specified geometry. Default *method* is **-all**.

# **6.229 control\_mesh\_remove** index method element\_group\_0 element\_group\_1 element\_group\_2 ...

With *method* set to -**method1** you can remove elements of  $element\_group\_0$  if they are completely located inside a elements of groups *element group 1*, *element group 2* etc.

With *method* set to **-method3** you can remove elements if all nodes of these elements have an mpc (that is, **node\_mpc** exists for all nodes of an element). Thus for *method* set to **-method3** you do not need to specify *element\_group\_0*, *element\_group\_1*, *element\_group\_2* etc.

## 6.230 control\_mesh\_remove\_geometry index geometry\_item\_name geometry\_item\_index

With this record you can restrict to which geometry the **control\_mesh\_remove** will be applied.

## 6.231 control\_mesh\_remove\_keep\_geometry\_index geometry\_item\_name geometry\_item\_index

If elements are being removed once with the **control\_mesh\_remove** command, they keep on being removed in the future if they are part of the geometry as specified in this **control mesh remove keep geometry** command.

## 6.232 control mesh remove really index switch

If *switch* is set to **-yes** elements removed by the **control\_mesh\_remove** command are really removed. If *switch* is set to **-no** elements removed by the **control\_mesh\_remove** command are not really removed; they are made inactive instead, and will become active again at the moment that they are no longer being removed by the **control\_mesh\_remove** command.

## 6.233 control mesh\_remove\_really\_activate\_all index switch

See control mesh remove really activate factor.

#### 6.234 control mesh remove really activate factor index factor

When an element becomes active again, after being removed before, we need to decide what the stresses and pore pressures are in nodes of the activated element. For nodes of the element which were connected to already active elements nothing extra is done.

For nodes of the element which were not connected to already active elements, the average value of the stresses and pore pressures from already active nodes (connected to already active elements) are taken. However, with this *factor*, you can specify to which part the stresses and pore pressures of already nodes will actually be used. Set *factor* to 0 if you want to reactivate the nodal stresses and pore pressures with 0. Set *factor* to 1 if you want to reactivate the nodal stresses and pore pressures completely with the stresses and pore pressures from already active neighbour nodes. Or set a *factor* between 0 and 1 if you do something in between.

As an extra option you can set the *switch* in **control\_mesh\_remove\_really\_activate\_all** to **-yes**. Then the stresses and pore pressures in ALL nodes of the activated element will be reactivated using the *factor* (so not only the nodes which were not active before).

Default, if **control\_mesh\_remove\_really\_activate\_factor** is not specified, the *factor* is set to 1.

6.235 control mesh renumber index lowest\_element lowest\_node

The element numbers are made strictly sequential starting from *lowest\_element* and the node numbers are made strictly sequential starting from *lowest\_node*. Beware using **control\_renumber** in combination with, for example, node numbers in printing of **node\_dof** records; use **post\_point** records instead.

## 6.236 control\_mesh\_renumber\_element\_geometry\_offset index offset

While renumbering elements the element geometry number will be offset with offset.

## 6.237 control mesh renumber element group offset index offset

While renumbering elements the element group number will be offset with offset.

#### 6.238 control mesh rotate index n

After rotation n is the number of elements in rotational direction for a rotation over 360 degrees. After rotation the old y direction becomes the new z direction. The following data is transferred in the rotation process: element, element\_group, node and node\_dof. A 2D -tria3 element becomes a 3D -prism6 element and a 2D -quad4 element becomes

a 3D -hex8 element; other 2D elements can presently not be rotated. All data that is not valid in 3D, like for example a 2D line etc, will be deleted in the rotation process.

This **control\_mesh\_rotate** is convenient when the first part of calculation is axisymmetric, for example loading a pile vertically in a soil, and the second part of the calculation is 3D, for example loading the top of the pile in some horizontal direction. Then first an axi-symmetric calculation can be performed, and the results can be used to start a 3D calculation.

If a -quad4 elements has a side on the y-axis in the 2D mesh, the element is rotated to a -prism6 element; the -quad4 element should have the side with local node numbers 0 and 1 on the y-axis, which is the case if you generated the elements with a control\_mesh\_macro You should not use other elements with a side on the y-axis when rotating the mesh.

This control\_mesh\_rotate deletes all data, except element, element\_group, node, node\_dof, element\_interface\_strain and element\_interface\_stress will be rotated to 3D. Furthermore, control\_input will available afterwards, so that all 3d data can be set in an extra input file, which is read after the mesh rotation.

If you use any history variables in the model, these should be scalars (and thus not vectors or matrices); otherwise rotation will not go ok for the history variables.

## 6.239 control mesh rotate angle index angle

With *angle* you can specify an angle in degrees up to which the mesh rotation should be done for the **control\_mesh\_rotate** with the same index. Typically you could use 90 degrees or 180 degrees for *angle*. Default, if this **control\_mesh\_rotate\_angle** is not specified, *angle* will be set to 360.

#### 6.240 control mesh split index switch

If *switch* is set to **-yes** then each **-quad4** element is split into four **-tria3** elements and each **-hex8** element is split into twelve **-tet4** elements. Further, each **-quad9** element is split into four **-tria6** elements and each **-hex27** element is split into six **-tet10** elements. Further, each **-tria6** element is split into four **-tria3** elements.

See the rules for old and new at **control\_mesh\_refine\_globally**. See also **control\_mesh\_split\_element\_to** and **control\_mesh\_split\_only**.

Splitting a 3D mesh will only work correctly on certain regular grids; you need to check the splitted mesh.

#### 6.241 control mesh split element from index name

Split only elements with the specified *name*.

## 6.242 control mesh split element to index name

If you are splitting **-hex8** elements, then you can set **name** either to **-tet4** or **-prism6**. Default, if **control\_mesh\_split\_element\_to** is not specified, **-tet4** is used.

If you are splitting **-hex27** elements, then you can set **name** either to **-tet10** or **-prism18**. Default, if **control mesh split element to** is not specified, **-tet10** is used.

#### 6.243 control mesh split only index geometry entity geometry entity index

If this record is used, the corresponding **control\_mesh\_split** record will only be applied on elements which have at least one node on the geometry specified by *index geometry\_entity\_name geometry\_entity\_index*.

## 6.244 control mesh switch index dir0 dir1

With this option you can switch x, y and z values for all nodal coordinates. Each of  $dir\theta$  and dir1 can be set to either -x, -y or -z. Then the specified coordinates  $dir\theta$  and dir1 will be switched for all nodes.

#### 6.245 control mesh truss distribute mpc index switch

If *switch* is set to **-yes** the nodes of truss elements are fixed with multi point constraints (mpc's) to the isoparametric elements through which the trusses run. This typically can be used for modeling reinforcement bars in a concrete embedment, where the bars follow the displacements (and temperatures if present) of the concrete.

If **control\_mesh\_truss\_distribute\_mpc\_exact** switch is set to **-yes**, truss elements are redistributed (that is, more small truss elements will be made), in such way that each truss gets a node when it enters an isoparametric element or ends internally in an isoparametric element. This **control\_mesh\_truss\_distribute\_mpc\_exact** comes handy when you initially have large trusses relative to the isoparametric elements.

Truss below a minimum length as specified in **control\_mesh\_truss\_distribute\_mpc\_exact\_minimal\_leng** will not be generated; default the minimal length *tolerance* is set to some small value. With **control\_mesh\_truss\_distribute\_mpc\_exact\_minimal\_length\_connect** you can determine if the generated trusses jumping a space below the minimal length will be connected or will be not-connected (loose); set the *switch* to **-yes** if you want the truss to be connected in such case. Please realise that the connection is ensured only for the trusses generated from 1 old truss; connection is not ensured for trusses generated from different old truss elements.

This control\_mesh\_truss\_distribute\_mpc option is done for truss groups as specified in control\_mesh\_truss\_distribute\_mpc\_element\_group\_truss or in control\_mesh\_truss\_distribute\_mpc\_geometry\_truss. Only one of control\_mesh\_truss\_distribute\_mpc\_element\_group\_truss and control\_mesh\_truss\_distribute\_mpc\_geometry\_truss can be specified. If none of control\_mesh\_truss\_distribute\_mpc\_element\_group\_truss and control\_mesh\_truss\_distribute\_mpc\_element\_group\_truss and control\_mesh\_truss\_distribute\_mpc\_element\_group\_truss and control\_mesh\_truss\_distribute\_mpc\_geometry\_truss is specified the distribution will be done for all trusses.

Default Tochnog will look for all isoparametric elements how to distribute the trusses. To save computer time you can restrict the geometry or element group of the isoparametric elements where Tochnog will look with control\_mesh\_truss\_distribute\_mpc\_element\_group\_isoparamet end control mesh truss distribute mpc geometry isoparametric.

 $\label{eq:leasenvice} Please notice that if you are using geometries in$ **control\_mesh\_truss\_distribute\_mpc\_geometry\_truss**or

 $\begin{array}{c} {\bf control\_mesh\_truss\_distribute\_mpc\_geometry\_isoparametric\ these\ can\ in\ fact\ be\ a\ geometry\ set. \end{array}$ 

In case you specify both of the above **\***\_truss and **\***\_isoparametric, the number of specified values (groups or geometries) should be the same. Then the first value specified for the truss will be combined with the first value specified for the isoparametric elements, the second value specified for the truss will be combined with the second value specified for the isoparametric elements, etc. For example, if you specify two groups for **control\_mesh\_truss\_distribute\_mpc\_element\_group\_truss** and two groups for **control\_mesh\_truss\_distribute\_mpc\_element\_group\_isoparametric** the first specified truss group will be distributed over the first specified isoparametric group, and the second specified truss group will be distributed over the first specified isoparametric group.

If *switch* in **control\_mesh\_truss\_distribute\_mpc\_air** is set to **-yes**, trusses will also be generated in the center of the truss is not inside an isoparametric element. If *switch* in **control\_mesh\_truss\_distribute\_mpc\_air** is set to **-no**, trusses will not be generated in the center of the truss is not inside an isoparametric element. Default *switch* is **-yes**.

A typical input file looks like:

control\_mesh\_truss\_distribute\_mpc 10 -yes
control\_mesh\_truss\_distribute\_mpc\_exact 10 -yes
control\_mesh\_truss\_distribute\_mpc\_geometry 10 -element\_geometry
123

Only one **control\_mesh\_truss\_distribute\_mpc** record is allowed in the input file. As a special option you can also generate **truss\_beam** elements in stead of **truss** elements.

6.246 control mesh truss distribute mpc air index switch

See control mesh truss distribute mpc.

6.247 control mesh truss distribute mpc dof dof\_0 dof\_1 ...

The dof\_0 dof\_1 ... specify the dof's that should be set equal, e.g. -velx, -vely etc.

- 6.248 control\_mesh\_truss\_distribute\_mpc\_element\_group\_truss index element\_group\_0 element\_group\_1 ...
- $See \ {\bf control\_mesh\_truss\_distribute\_mpc}.$
- 6.249 control\_mesh\_truss\_distribute\_mpc\_element\_group\_isoparametric index element\_group\_0 element\_group\_1 ...

See control mesh truss distribute mpc.

6.250 control mesh truss distribute mpc exact index switch

See control mesh truss distribute mpc.

- $\begin{array}{ccc} \textbf{6.251} & \textbf{control\_mesh\_truss\_distribute\_mpc\_exact\_minimal\_length} \\ & \textit{index tolerance} \end{array}$
- See control mesh truss distribute mpc.

See control mesh truss distribute mpc.

**6.253 control\_mesh\_truss\_distribute\_mpc\_geometry\_truss** index geometry\_entity\_name\_0 geometry\_entity\_index\_0 geometry\_entity\_name\_1 geometry\_entity\_index\_1 ...

See control mesh truss distribute mpc.

6.254 control\_mesh\_truss\_distribute\_mpc\_geometry\_isoparametric index geometry\_entity\_name\_0 geometry\_entity\_index\_0 geometry\_entity\_name\_1 geometry\_entity\_index\_1 ...

See control mesh truss distribute mpc.

### 6.255 control mpc apply index switch

If *switch* is set to **-yes** then mpc conditions will be used for the control index. If *switch* is set to **-no** then mpc conditions will not be used for the control index. Default, if **control mpc apply** is not specified, *switch* is set to **-yes**.

## 6.256 control mpc element group index switch

If *switch* is set to **-yes** the **mpc\_element\_group** records will be evaluated at all timesteps for the current control index. If *switch* is set to **-no** the **mpc\_element\_group** records will only be evaluated when the mesh has been changed. This option can only be used if there are only mpc's coming from this option, and not from other mpc data.

Default, if control mpc element group is not specified, the *switch* is set to -no.

#### 6.257 control plasti apply index switch

If *switch* is set to **-no**, any plasticity data in the input file will be ignored. This is done for timestep records with the same index.

This option is convenient for testing your input file just linear, without the need to outcomment each and every part with plasticity data. See also **plasti apply**.

#### 6.258 control post index switch

If *switch* is set to **-yes** all post processing commands are evaluated for index *index*. This enforces evaluation of post processing command even if no timesteps are performed or so.

#### 6.259 control post apply index switch

Setting *switch* to **-no** prevents post processing commands to be evaluated for control commands with the same index. Postprocessing commands have **post** in the name (only the **post node rhside ratio** will be evaluated always, independent of **control post apply**).

#### 6.260 control post element force index switch

You can save CPU time in timesteps with the same index by setting *switch* to **-no**, which prevents **post\_element\_force** commands to be evaluated in timesteps with the same index.

#### 6.261 control print index data\_item\_name\_0 data\_item\_name\_1 ...

The is the normal printing command. A **control\_print** record causes the data items with name *data\_item\_name\_0*, etc. to be printed. Example

control print 1 -node -node dof

See also: **print** filter.

#### 6.262 control print beam force moment index switch

This option prints the beam forces and moments through a set of beams starting at place  $x_{start}, y_{start}, z_{start}$  and ending at  $x_{end}, y_{end}, z_{end}$  as specified in **control\_print\_beam\_force\_moment\_coord**. In 2D only x and y coordinates need to be specified. The forces and moments are printed in the file beam\_force\_moment.*index*. In fact, if the element contains a truss (either a truss element or a truss-beam element), the truss force will be used for the axial force. The first column in the file is the distance from the start point. The following columns contain in the local beam axes force\_x\_first\_node force\_y\_first\_node force\_z\_first\_node moment\_x\_first\_node moment\_y\_first\_node moment\_z\_first\_node force\_x\_second\_node force\_z\_second\_node moment\_z\_second\_node moment\_z\_second\_node moment\_z\_second\_node moment\_z\_sequential. See also control print beam\_force\_moment\_setup.

## 6.263 control\_print\_beam\_force\_moment\_coordinates index $x_{start}$ $y_{start} z_{start} x_{end} y_{end} z_{end}$

See control print beam force moment.

#### 6.264 control print beam force moment switch index switch

If you set *switch* to **-yes**, the definition of the beam forces and moments is changed (multiplied with a **-1**). So you can get exactly the definition that you want.

#### 6.265 control print database index switch

If *switch* is set to **-separate\_index**, the complete database is be printed. See the example below

#### control print database 6 -separate index

This database contains the complete status of the calculation. For example if *index* is 6, the data base is printed in the file input\_file\_name6.dbs. As a special option, you can print databases with sequential numbers by setting *switch* to **-separate sequential**.

If tochnog exists with an error, for example due to non-convergence, a complete database is printed in **input\_file\_name\_error.dbs**. Otherwise, a complete database will be printed at the end of the calculation.

## 6.266 control print database\_method index method

If *method* is set to **-all** then all database base records will be printed in the database. If *method* is set to **-size\_tot** then the size of all database base records will be printed in the database. If *method* is set to **-size\_tot\_large** then the size of database base records larger then 1 Mb will be printed in the database.

When using -size tot or -size tot large also the size of the system matrix is printed in the database.

Default, if control print database method is not specified, the *method* is set to -all.

## 6.267 control\_print\_data\_versus\_data index data\_item\_name\_0 index\_0 number\_0 data\_item\_name\_1 index\_1 number\_1 ...

This option prints columns of data for each time step. Print in the first column the *number\_0* value of *data\_item\_name\_0* with index *index\_0*. Similar in the second column for *data\_item\_name\_1 index\_1 number\_1*. Etc. (for all values). All results will be printed in the file *problemname*.dvd.

Typically, the data item names can be **-node\_dof** such that dof's can be printed against each other in time. If the data item names are **-node\_dof**, then  $number_0$  and  $number_1$ , etc. can be names of **dof\_label** (eg **-velx**).

Also typically, the data item names can be **-node\_dof\_calcul** such that post calculation results can be printed against each other in time. If the data item names are **-node\_dof\_calcul** or **post\_point\_dof\_calcul** or so, then *number\_0* and *number\_1*, etc. can be names of **post\_calcul** label (eg **-aept**).

Otherwise, for example, if  $number_0$  is 3 then the fourth value of  $data_item_name_0$  is printed.

Example:

control\_print\_data\_versus\_data 0 -node\_dof 2 -temp -node\_dof 2 -sigxx -node\_dof 2 -sigxx

Another example:

post\_point 0 0.0 1.0
post\_calcul -materi\_stress -average -materi\_stress -size\_dev
control\_print\_data\_versus\_data 20
-time\_current 0 0
-post\_point\_dof\_calcul 0 0 -post\_point\_dof\_calcul 0 1

In the last example, the **-post\_point\_dof\_calcul 0 0** stands for 'the **post\_point\_dof\_record** with index 0 and the 0'th number which is the first value so the average of the stresses'.

For data that is not present Tochnog will print a 0.

See also: **control print**.

6.268 control\_print\_data\_versus\_data\_factor index factor\_0 factor 1 ...

Optionally specify a multiplication factor for each of the data item of **control\_print\_data\_versus\_data**. This can, by example, be convenient if you want to change to sign or so. If you specify this record, a factor should be given for each data item.

#### 6.269 control print dof index switch

Results for the primary dof's will be printed, including also the coordinates at which the results hold. Also results for **node\_dof\_calcul** records will be printed. The printed files will contain lines like x, y, z and dof (where dof is the dof, e.g. **temp**). In 1D only x will be printed, etc.

As extra also the coordinates themselves are printed in files.

If *switch* is set to **-separate** index the filenames will be like *dof.index*.

If *switch* is set to **-separate\_sequential** then the filenames will be sequentially numbered, like *dof.0*, *dof.1*, etc.

## 6.270 control print dof id index switch

This record works in combination with the **control\_print\_dof** record. If *switch* is set to **-yes** also the node number ('identity') is written. So for example in 3D the file will contain lines with x, y, z, dof and node number.

Default, if control print dof id is not specified, *switch* is set to -yes.

#### 6.271 control print dof smooth dof index dof\_0 dof\_1 ...

This option allows you to smooth results for **control\_print\_dof**. With  $dof_0 dof_1 \dots$  you specify the dof's to be smoothed. As a special option you can specify **-all** so that all dof's will be smoothed.

The smoothing is done a number of times, with increasing smooth results. You can specify this number of times with the optional **control\_print\_dof\_smooth\_n**; if you don't specify this optional number of smoothings it is done 10 times.

#### 6.272 control print dof smooth n index number\_of\_smoothings

See control print dof smooth dof.

#### 6.273 control print dof line index switch

This control\_print\_dof\_line record together with the control\_print\_dof\_line\_coordinates and control\_print\_dof\_line\_n records print values of the node\_dof records and node\_dof\_calcul records along a line in space to files. The start point of the first line segment is given by  $x_0 y_0 z_0$ , and the end point of the first line segment is given by  $x_1 y_1 z_1$ , the start point of the second line segment is given by  $x_1 y_1 z_1$ , and the end point of the second line segment is given by  $x_2 y_2 z_1$ , etc.

In 1D only the x-coordinates of the start point and end point need to be specified, etc. The parameter n determines how many points will be printed along the line.

The printed files will contain lines like x, y, z and *dof* (where *dof* is the dof, e.g. **temp**). In 1D only x will be printed, etc.

If *switch* is set to **-separate** index the filenames will be like *dof.index*.

If *switch* is set to **-separate\_sequential** then the filenames will be sequentially numbered, like *dof.0*, *dof.1*, etc.

In control\_print\_dof\_line\_method you can set *node\_type* either to -node or -node\_start\_refined. Then the coordinates in the printed file will contain either the values of **node** or the values of **node\_start\_refined**. In case you use an updated lagrange formulation where the mesh nodes follow the material the values of **node** and **node\_start\_refined** will differ; in case you do a geometrically linear analysis the values will not differ. Default *node\_type* is set to -node start refined.

With **control\_print\_dof\_line\_element\_group** you can set a specific element groups for which the printing should be done.

With control\_print\_dof\_line\_eps\_iso you set the tolerance with which a point of the specified line is accepted to be part of an element. The default value is 1.e-3. You can increase the default value if if the mesh is not exactly adjusted to the line.

## 6.274 control print dof line coordinates index $x_0 y_0 z_0 x_1$ $y 1 z 1 x 2 y 2 z 2 \dots$

See control print dof line.

**6.275 control\_print\_dof\_line\_element\_group** *index element\_group\_0 element\_group\_1* ...

See control\_print\_dof\_line.

6.276 control print dof line eps iso index eps\_iso

See control print dof line.

#### 6.277 control print dof line method index node\_type

See control print dof line.

#### 6.278 control print dof line move index switch

If *switch* is set to **-yes** the **control\_print\_dof\_line\_coordinates** with the same index will be moved along with the velocity field. Thus, with this option you can follow with a **control print dof line** the dof's of material particles.

Please realise that this option should only be used if **materi\_velocity** is initialised in the initialisation part.

## 6.279 control print dof line n index n

See control print dof line.

#### 6.280 control print dof line time index switch

If *switch* is set to **-yes** the first line of each file will specify the **time\_current** at which the file is written (in gnuplot comment format).

#### 6.281 control print dof point index switch

This **control\_print\_dof\_point** record prints values of the **node\_dof** records and **node\_dof\_calcul** records in a point in space to files.

The point is given by x y z,

In 1D only the x-coordinates of the start point and end point need to be specified, etc.

The printed files will contain lines like x, y, z and dof (where dof is the dof, e.g. **temp**). In 1D only x will be printed, etc.

If *switch* is set to **-separate** index the filenames will be like *dof.index*.

If *switch* is set to **-separate\_sequential** then the filenames will be sequentially numbered, like *dof.0*, *dof.1*, etc.

#### 6.282 control print dof point coordinates index x y z

See control print dof line.

#### 6.283 control print dof point time index switch

If *switch* is set to **-yes** the first line of each file will specify the **time\_current** at which the file is written (in gnuplot comment format).

## 6.284 control print dof rhside index switch

If *switch* is set to **-yes** then results for right-hand-side for the primary dof's will be printed, including also the coordinates at which the results hold.

For example, for the file temp\_rhside.*index* will contain lines containing x, y, z and right-hand-side of -temp (that is, heat flux). In 1D only x will be printed, etc.

### 6.285 control print element index data\_item\_name

With this option you can print values from element data versus coordinates. Select either -element truss force or -element beam force moment for *data\_item\_name*.

The normal truss forces of the -element truss force records will be printed in the file element truss force n. *index*. This file will contain lines containing x, y, z and normal truss force. In 1D only x will be printed, etc.

The lateral beam shear forces of the **-element\_beam\_force\_moment** records will be printed in the file **element\_beam\_force\_moment\_q**.*index*. This file will contain lines containing x, y, z and lateral beam shear force. In 1D only x will be printed, etc. The shear force will always be calculated as an absolute value.

The beam moments of the **-element\_beam\_force\_moment** records will be printed in the file **element\_beam\_force\_moment**  $\underline{m.index}$ . This file will contain lines containing x, y, z and beam moment. In 1D only x will be printed, etc.

How the data is printed depends on how *method* is set in **control\_print\_element\_method**. If method is set to **-middle** then only the average value of the element data and the coordinate of the middle of the element is printed for each element. If method is set to **-node** then the two nodal values and nodal coordinates are printed for each element.

#### 6.286 control print element method index method

Set *method* to **-middle** or *-node*. If **control\_print\_element\_method** is not specified then **-middle** is used. See also **control print element**.

## **6.287 control\_print\_filter** *index print\_filter\_index\_0 print\_filter\_index\_1*

See print filter.

#### 6.288 control print frd index switch

Activate printing of results in Calculix output format **frd**. Only results for 2D and 3D isoparametric elements are printed presently. For structural elements (trusses, beams, ...) nothing is printed.

For example if the input file name is **excavation.dat** and *index* is 100 and *switch* is set to **-separate index** then results are printed in the file is **excavation 100.frd**.

For example if the input file name is **excavation.dat** and *switch* is set to -separate \_sequential then results are printed in the files is **excavation** 0.frd, **excavation1.frd**, etc.

For example if the input file name is **excavation.dat** and *switch* is set to **-yes\_sequential** then results are printed in the files is **excavation.frd**.

The frd files can be plotted by the **prepomax** pre- and postprocessor, see **http:**//lace.fs.unimb.si/wordpress/borovinsek/?page\_id=41. Since prepomax presently cannot plot 2d elements, Tochnog extrudes 2d calculations to a fictive thickness of 1 when printing the frd files.

You also can use the CGX postprocessor of calculix itself, see http://www.bconverged.com/download.php.

Since Freecad and Prepomax like specific names of results we write the following names:

- DISP for materi displacement or materi velocity integrated
- STRESS for materi stress
- TOSTRAIN for materi strain total
- NDTEMP for condif temperature
- **PEEQ** for **post\_calcul-groundflow\_pressure-total\_pressure** (if **print\_frd\_freecad** is **-yes**)

For other results the Tochnog names are used, up to the allowed 8 characters in frd files.

Freecad presently only reads frd files which contain displacements and stresses and total strains. Thus, if these are not initialised in your tochnog input file and if **print\_frd\_freecad** is **-yes** then tochnog will not write the frd file for freecad.

See also control print frd prepomax.

## 6.289 control print frd freecad index switch

If *switch* is set to **-yes** the frd files are written specifically suited for the **freecad** program. This is done for the **control\_print\_frd** command with the same index. If *switch* is set to **-no** the frd files are not written specifically suited for the **freecad** program. See also **print\_frd\_freecad**.

### 6.290 control print frd prepomax index switch

If *switch* is set to **-yes** the frd files are written specifically suited for the **prepomax** program. This is done for the **control\_print\_frd** command with the same index. If *switch* is set to **-no** the frd files are not written specifically suited for the **prepomax** program. See also **print\_frd prepomax**.

## 6.291 control print frequency timeinterval index timeinterval

This control\_print\_frequency\_timeinterval record causes control\_print\_\* other than control\_print, control\_print\_history and control\_print\_data\_versus\_data to be done each time after a time interval has passed, and always also at the end of the time increment. This control\_print\_frequency\_timeinterval record should only be used in combination with control\_timestep (with the same index). All control\_print\_\* printing will be influenced except control\_print, control\_print\_history and control\_print\_data\_versus\_data printing.

Example:

control\_timestep 10 0.04 0.41 control\_print\_gid 10 -yes control\_print\_frequency\_timeinterval 10 0.15

In this example gid data is written at times 0.16, 0.32, 0.41

#### 6.292 control print frequency timestep index timestep

This control\_print\_frequency\_timestep record causes control\_print\_\* other than control\_print, control\_print\_history and control\_print\_data\_versus\_data to be done each time after a number of time timesteps has passed, and always also at the end of the time increment. This control\_print\_frequency\_interval record should only be used in combination with control\_timestep (with the same index). All control\_print\_\* printing will be influenced except control\_print, control\_print\_history and control\_print\_data\_versus\_data printing.

Example:

control\_timestep 10 0.04 0.41 control\_print\_gid 10 -yes control\_print\_frequency\_timestep 10 5

In this example gid data is written at times 0.20, 0.40, 0.41

#### 6.293 control print gid index switch

Print the mesh and the dof's in a file which can be plotted with the GID pre-post processor if *switch* is set to **-yes**. For example, if the input file is called turbine.dat then the mesh is written in the turbine flavia.msh file. The results are written in the turbine flavia.res.

The mesh and results for dof's will always be written at the end of the calculation.

Since GID gets confused when the number of elements changes between several **control\_print\_gid** records, Tochnog will only print GID results for the last mesh used.

Coordinates for nodes will be written in the original configuration. If **materi\_velocity** is initialized, also a vector **mesh\_deform** will be written for GID which contains the deformation between the original mesh configuration and the deformed mesh configuration. Use the **deform mesh** menu in GID, to draw the deformed configuration by applying the vector **mesh\_deform** with a factor 1. If **materi\_displacement** is initialised the **mesh\_deform** contains the nodal displacements. If **materi\_velocity\_integrated** is initialised the **mesh\_deform** contains the integrated nodal velocities. Else **mesh\_deform** contains the current value of **node** minus the start value of the nodal coordinates.

For 2D interface elements which have strains and stresses, the normal stress **interface\_sign**, the tangential shear stress **interface\_sigt**, the normal strain **interface\_epsn** and the tangential shear strain **interface\_epsn**, are written to the GID results file.

The following data is written also to the gid file and can serve as a help to check the validity of your input file. This data is only available after one or more time steps are taken.

- condif bounda dof, boundary conditions on temperature.
- condif heat edge normal, distributed prescribed heat flow on an edge.
- condif convection edge normal, distributed convection heat flow on an edge.
- condif radiation edge normal, distributed convection heat flow on an edge.
- **groundflow\_bounda\_dof**, boundary conditions on groundflow hydraulic pressure head.
- materi bounda force, discrete forces on nodes.
- materi force edge, distributed forces on nodes.
- materi force edge normal, distributed normal forces on nodes.
- materi force edge projected, distributed projected forces on nodes.
- materi force edge water, distributed water forces on nodes.
- materi force volume, distributed volume forces on nodes.
- materi force gravity, distributed gravity forces on nodes.
- materi bounda dof, boundary conditions on materi velocity on nodes.
- materi support edge normal, distributed support forces on nodes.
- **materi\_rhside\_free**, unbalance forces for **materi\_velocity** ( for free displacements) on nodex.
- materi\_rhside\_fixed, reaction forces for materi\_velocity ( for fixed displacements) on nodex.
- element materi force edge, norm of distributed forces on elements.
- element <u>materi</u> force edge normal, norm of distributed normal forces on an edge on elements.
- element <u>materi</u> force edge water, norm of distributed water forces on an edge on elements.
- **plasti\_reduction\_factor**, reduction factor for plasticity parameters from **group\_materi\_plasti\_eleme** etc.

If you have specified **print\_node\_geometry\_present** then the gid files will contain **geometry** ... values which are 1 on nodes present in a geometry.

The materi\_bounda\_dof you can view in gid with View results, Display vectors, materi bounda dof, All. The other data you can view in GID for example with View results, Display vectors, force edge normal, | force edge normal | . Above with 'distributed' we mean that results are per unit area.

For isoparametric elements the element group number will be printed.

For all **post\_point** a point will be plotted in the gid files, with element group number 1000 + the index of the post point.

As a special option, you can set *switch* to **-separate\_index**. Then the mesh and results will be printed in separate files for GID, numbered with *index*. The option comes handy when the mesh changes during the calculation; GID cannot plot that if the mesh and results are in the same file.

As a further special option, you can set *switch* to **-separate \_sequential**. Then the mesh and results will be printed in separate files for GID, number sequentially.

#### 6.294 control print gid batch index switch

If *switch* is set to **-yes** Tochnog calls GiD as batch program to generate some typical png files of results. This works i.c.w. a **control print** gid with the same index.

The following files are generated: mesh\_index.png, element\_group\_index.png, element\_group\_index.png, materi\_velocity\_integrated\_veliy\_index.png, materi\_stress\_sigyy\_index.png and ground-flow\_pressure\_total\_pressure\_to\_pres\_index.png.

For this GiD needs to be installed on your computer. You also need to take care that GiD can be found, so you need to adjust your PATH environment symbol.

#### 6.295 control print gid beam vectors index switch

If *switch* is set to **- yes**, force and moment vectors will be plotted for **-beam** and **-truss\_beam** elements. The force and moment vectors will be plotted perpendicular to the length direction and a user specified plane-normal vector, see **control\_print\_gid\_beam\_vectors\_normal**. The length of the plotted vectors measures the size of the forces and moment.

The vectors will be plotted as element result, so not as nodally averaged result.

Attention: this **control\_print\_gid\_beam\_vectors** is a special plotting option, to get each beam force and moment result as vector plot, with the possibility to influence the direction of the vectors with **control\_print\_gid\_beam\_vectors\_normal**. Default Tochnog plots the beam result already as scalar values for each beam element.

## 6.296 control\_print\_gid\_beam\_vectors\_normal index normal\_x normal\_y normal\_z

This record gives you the possibility to influence the plane in which the moment vectors generated by the **control\_print\_gid\_beam\_vectors** will be plotted. In fact this **control\_print\_gid\_beam\_vectors\_normal** specifies the normal to the plotting plane. If this **control\_print\_gid\_beam\_vectors\_normal** is not specified then **0 0 1** is taken

as normal.

#### 6.297 control print gid contact spring2 index number\_of\_nodes

Set number\_of\_nodes to 2 if you want to draw contact\_spring2 with two nodes, and to 1 if you want to draw contact\_spring2 with one node. Default, if control\_print\_gid\_contact\_spring2 is not specified, then 1 node is used.

#### 6.298 control print gid coord index switch

If *switch* is set to **-yes** the coordinates of nodes is plotted in gid. If *switch* is set to **-no** the coordinates of nodes is not plotted in gid. Default *switch* is set to **-yes**.

# **6.299 control\_print\_gid\_dof** *index initialisation\_name\_0 initialisation\_name\_1*

When you specify this record only the solution fields *initialisation\_name\_0*, *initialisation\_name\_1* etc will be printed to the gid files. So the files become smaller in size. This is especially convenient for very large calculations. The names *initialisation\_name\_0*, *initialisation\_name\_1* are names from the initialisation part like -condif\_temperature, -materi\_velocity, -materi\_stress etc. In case you do not want any field to be printed in the gid file use control print gid dof index -none.

See also control print gid other.

## 6.300 control print gid dof calcul index calcul\_0 calcul\_1...

When you specify this record only the post fields  $calcul_0$ ,  $calcul_1$  etc will be printed to the gid files. So the files become smaller in size. This is especially convenient for very large calculations. See **post\_calcul\_label** for the allowed names of  $calcul_0$ ,  $calcul_1$  etc. In case you do not want any post field to be printed in the gid file use **control print gid dof calcul** *index*-**none**.

See also control print gid other.

# **6.301 control\_print\_gid\_element\_group** *index element\_group\_0 element\_group\_1* ...

Select specific element groups for the gid files. If this record is not specified all element groups will be used.

#### 6.302 control print gid element mpc index switch

If *switch* is set to **-yes** also elements which have mpc'd nodes will be printed in the gid files. If *switch* is set to **-no** aelements which have mpc'd nodes will not be printed in the gid files. Default, if **control print gid element mpc** is not specified, *switch* is set to **-yes**.

## 6.303 control print gid empty index switch

If *switch* is set to **-yes**, empty elements will be show in GID plots. If *switch* is set to **-no**, empty elements will not be shown. Default *switch* is set to **-no**.

See also **element empty**.

#### 6.304 control print gid group index switch

If *switch* is set to **-yes** the element groups are plotted in gid as result field; in gid you can do a **contour\_fill** to visualize this result field. If *switch* is set to **-no** the element groups are not plotted in gid as result. Default *switch* is set to **-yes**.

## 6.305 control print gid mesh activate gravity index switch

See also mesh activate gravity time.

## 6.306 control print gid method index method

If *method* is set to **-node**, results will be written for global nodes to the gid files. Gid will interpolate between the nodes, to fill contour plots, etc. Hence, you get continuous plots fields.

If *method* is set to **-element**, results will be written element-by-element to the gid files, so that any discontinuity in fields can be seen. The results will be written using the integration point values.

If *method* is set to **-element\_average**, results will be written element-by-element to the gid files, so that any discontinuity in fields can be seen. The results will be written using the average of the integration point values.

If *method* is set to **-node\_elemen**, results will be written with continuous fields to the gid files, but at group jumps discontinuous fields are allowed.

For -element and -node elemen gid cannot plot some results like 'contour fill' for all elements if there are several type of elements (quad4, tria3, ...) in the mesh. You can only

select on specific element type for the plot.

If this **control\_print\_gid\_method** record is not specified then *method* is set to **-node**. If **global\_element\_dof** is set to **-no** the element-by-element stresses are not available, so then **-element average** and **-node elemen** should not be used.

#### 6.307 control print gid node method index method

If *method* is set to **-node**, these coordinates are written. If *method* is set to **-node**\_**start\_refined**, these coordinates are written. If *method* is set to **-node**\_**deformed**\_**mesh**, these coordinates are written. Default *method* is **-node**\_**start\_refined**.

#### 6.308 control print gid other index switch

If *switch* is set to **-yes** also other things like boundary conditions, mesh deformation etc. are printed in the gid files. If *switch* is set to **-no** these other things are not printed in the gid files. Default *switch* is set to **-yes**.

#### 6.309 control print gid save difference index switch

If *switch* is set to **-yes** then data differences relative to a saved status will be plotted. See **control data save**.

#### 6.310 control print gid safety slip critical index switch

If *switch* is set to **-yes**, then for a safety analysis with **control\_safety\_slip** only the critical slip surface will be plotted. Default, if *switch* is not set, all evaluated slip surfaces will be plotted. The critical surface is either determined over all safety surfaces, or otherwise in case sets are specified a critical surface is determined for each set.

Furthermore, always the normal stresses and shear stresses on the slip surfaces will be plotted.

#### 6.311 control print gid smooth dof index dof\_0 dof\_1 ...

This option allows you to smooth results in gid files. With  $dof_0 dof_1 \dots$  you specify the dof's to be smoothed. As a special option you can specify **-all** so that all dof's will be smoothed.

The smoothing is done a number of times, with increasing smooth results. You can specify this number of times with the optional **control print gid smooth n**; if you don't

specify this optional number of smoothings it is done 10 times.

## 6.312 control print gid smooth n index number\_of\_smoothings

See control print gid smooth dof.

#### 6.313 control print gid spring2 index number\_of\_nodes

Set *number\_of\_nodes* to 2 if you want to draw **spring2** with two nodes, and to 1 if you want to draw **spring2** with one node. Default, if **control\_print\_gid\_spring2** is not specified, then **print\_gid\_spring2** is used.

#### 6.314 control print gid truss vector index switch

Same as control\_print\_gid\_beam\_vector, however now for the normal force in - truss and - truss beam elements.

Attention: this **control\_print\_gid\_truss\_vector** is a special plotting option, to get the truss force result as vector plot, with the possibility to influence the direction of the vectors with **control\_print\_gid\_truss\_vector\_normal**. Default Tochnog plots the truss force result already as scalar values for each truss element.

## 6.315 **control\_print\_gid\_truss\_vector\_normal** index normal\_x normal\_y normal\_z

Same as control\_print\_gid\_beam\_vectors\_normal, however now for the normal force in -truss and -truss beam elements.

## 6.316 control print gmsh index switch

We discuss as an example the printed file naming convention if the input file name is **excavation.dat** 

If *switch* is set to **-yes** the results are printed into the **excavation.msh** file. In case the mesh (elements and nodes) have not been printed before in this file, the file will be emptied, and the mesh will be printed. This will also be done if the mesh is changed.

If *index* is 100 and *switch* is set to **-separate\_index** then the mesh and results are printed in the file is **excavation 100.msh**.

If *switch* is set to -**separate\_sequential** then the mesh and results are printed in the files **excavation\_0.msh**, **excavation\_1.msh**, etc. So each time that a **control\_print\_gmsh** with -**separate\_sequential** is evaluated a new file is generated with number increased by one.

A dummy point element is put in each node in the gmsh file. Gmsh needs that for plotting vector fields in the nodes. The dummy element group 1234 is used for these dummy point elements. You can suppress these dummy point elements by setting **control print gmsh dummy** to **-no**.

All element data starts with **element** in the plots. All node data starts with **node** in the plots.

Scalar data with more then one value is given the extension \_0, \_1 etc. for each of the values. For example the record **node** (which contains coordinates in each space direction) is plotted as scalar **node\_0**, **node\_1** and **node\_2** which contain the x-coordinate, y-coordinate and z-coordinate respectively. For example the record **group\_groundflow\_permeability** (which contains permeability in each space direction) is plotted as scalar **group\_groundflow\_0**, **group\_groundflow\_1** and **group\_groundflow\_2** which contain the x-permeability, y-permeability and z-permeability respectively.

For nodes the presence in geometries is plotted as **node\_geometry\_\***. For elements the presence in geometries is plotted as **element\_geometry\_\***.

You can plot this file with the program  $\mathbf{gmsh}$ ; see http://www.geuz.org/gmsh. You can also plot this file with the program  $\mathbf{mecway}$ ; see https://mecway.com/.

See also input\_gmsh.

#### 6.317 control print gmsh dummy index switch

#### See control print gmsh.

Default, if this record is not set and **print\_gmsh\_dummy** is not specified, *switch* is set **-yes**.

#### 6.318 control print gmsh element data index switch

If you set *switch* to **-yes** data for elements (like element strains, stresses, etc.) is written averaged over the element; this corresponds to **ElementData** in the gmsh format.

If you set *switch* to **-no** this data is written for all element nodes; this corresponds to **ElementNodeData** in the gmsh format.

Default, if this record is not set, *switch* is set -yes.

#### 6.319 control print gmsh node method index method

If *method* is set to **-node**, these coordinates are written. If *method* is set to **-node**\_**start**\_**refined**, these coordinates are written. If *method* is set to **-node**\_**deformed**\_**mesh**, these coordinates are written.

# **6.320 control\_print\_history** *index data\_item\_name\_0 data\_item\_index\_0 number\_0* ...

Print the time history for each of the sets data\_item\_name\_0 data\_item\_index\_0 number\_0 ....

For example, if **-node** dof is used, *number*  $\theta$  is one of the names of dof label (eg -velx).

For example, if **-node\_dof\_calcul** is used, *number\_0* is one of the names of **post\_calcul\_label** (eg **-aept**).

Otherwise,  $number_0$  should be an integer specifying the number of the value in the record (for instance number 2 means the third value).

The following history is printed in the file node\_dof\_112\_temp.his

#### control print history 0 -node dof 112 -temp

### 6.321 control print history factor index factor\_0 factor\_1 ...

Optionally specify a multiplication factor for each of the data item of **control\_print\_history**. This can, by example, be convenient if you want to change to sign or so. If you specify this record, a factor should be given for each data item.

#### 6.322 control print history relative time index tr

If you specify this record, the time printed in the history files will not be the actual time, but instead the actual time minus the relative time *tr*. This is especially convenent when something happens suddenly after a long time, in which case the time-axis in the history plot would be not clear. Using this relative time the time-axis in the plots will become clear (since the long initial time is not part of the time axis).

#### 6.323 control print history smooth index smooth\_0 smooth\_1 ...

You can smooth history results with this option. Each of  $smooth_0smooth_1$  etc. represent an integer number for each data value printed with **control\_print\_history**. By example if  $smooth_0$  is 10, for the first data value the average of the last 10 values will be printed (as opposed to the current value). You should specify an integer for each of the data values being printed. As a special option you can specify one integer only which then will be used for all data values being printed. The smoothed results will be printed in a separate history file starting with **smooth**.

#### 6.324 control print interface stress index switch

#### 2D analysis

This option prints in 2D the interface stresses through a set of interfaces starting at place  $x_{start}, y_{start}$  and ending at  $x_{end}, y_{end}$  as specified in **control\_print\_interface\_stress\_coordinates**. The *switch* needs to be set to **-separate\_index** or **-separate\_sequential**. The stresses are printed in the file interface\_stress.*index*. The first column in the file is the distance from the start point. The following columns contain interface\_sign and interface\_sigt. A line is written for each node of each interface element. Crossing interfaces are not allowed. From the start point up to the end point the interfaces needs to be connected without gaps.

#### 3D analysis

This option prints in 3D the average interface stresses in the middle of interface elements. The *switch* needs to be set to -separate \_index or -separate \_sequential. The interface element middles and average stresses are printed in the file interface \_stress.*index*. The first three columns in the file are the coordinates of the middle of the interface element. The following columns contain interface\_sign and interface\_sigt1 and interface\_sigt2. A line is written for each interface element. If you specify control \_print\_interface\_stress\_3d\_geometry then only interfaces elements located on the geometry will be printed. If you don't specify control\_print\_interface\_stress\_3d\_geometry then all interfaces elements will be printed. You can specify the order of printing of the interfaces in the file with *method* in control\_print\_interface\_stress\_3d\_order. If you set *order* to -x the interfaces will be ordered according to x-coordinate. If you set *order* to -y the interfaces will be ordered according to y-coordinate. If you set *order* to -z the interfaces will be ordered according to zcoordinate. If you don't use control\_print\_interface\_stress\_3d\_order the interfaces will be ordered will be ordered according to z-coordinate. If you set *order* to -z the interfaces will be ordered according to zcoordinate. If you don't use control\_print\_interface\_stress\_3d\_order the interfaces will be ordered according to z-coordinate. If you set *order* to -z the interfaces will be ordered according to zcoordinate. If you don't use control\_print\_interface\_stress\_3d\_order the interfaces will be ordered according to element number.

## 6.325 control\_print\_interface\_stress\_2d\_coordinates index $x_{start}$ $y_{start} x_{end} y_{end}$

For 2D only. See control print interface stress.

## 6.326 control\_print\_interface\_stress\_3d\_geometry index geometry\_item\_name geometry\_item\_index

For 3D only. See control print interface stress.

## 6.327 control print interface stress 3d order index order

For 3D only. See control print interface stress.

## 6.328 control print materi stress force index method

This option prints forces and moments as calculated by **post\_calcul\_materi\_stress\_force**. It prints in special purpose ascii files, convenient for further external postprocessing. For example, the name of the file will be **materi\_stress\_force.100** if the *index* is 100. The files themselves will contain comments explaining the detailed structure of the files.

The *method* can be set either to **-all** if all results should printed in the file (so including the averaged results) or to **-primary** if only the primarily calculated results should be printed in the file (so not including the averaged results).

#### 6.329 control print mesh dof index switch

See print mesh dof.

. . .

## 6.330 control\_print\_node index data\_item\_name number\_0 number\_1

With this record you can print nodal data like **node\_dof**, **node\_dof\_calcul** etc. to files. As an example in 2D you can use **control\_print\_node index -node\_dof -velx -velx** to get the files **velx**.*index* and **vely**.*index*; these files contain in columns for all nodes x y velx and x y vely.

For *data\_item\_name* you can apply any nodal data record for which the name starts with **node**. For *number\_0 number\_1* you can specify which parts of the data record should be printed; you can either specify numbers 0, 1, etc. or for **node\_dof** you can specify the names of **dof\_label** like **-vely**, **-vely** etc., or for **node\_dof\_calcul** you can specify the names of **post\_calcul label** like **-to\_pres**, **-dy\_pres** etc.

#### **6.331** control print node angular index switch\_x switch\_y switch\_z

With this record you can specify that an angle will be included in the files (in stead of coordinates). With *switch\_x switch\_y switch\_z* set to **-yes -yes -no** the angle will measure the number of degrees from the positive global x-coordinate directed to the positive global y-direction. With *switch\_x switch\_y switch\_z* set to **-no -yes -yes** the angle will measure the number of degrees from the positive global y-coordinate directed to the positive global z-direction. With *switch\_x switch\_y switch\_z* set to **-yes -no -yes** the angle will measure the number of degrees from the positive global y-coordinate directed to the positive global z-direction. With *switch\_x switch\_y switch\_z* set to **-yes -no -yes** the angle will measure the number of degrees from the positive global x-coordinate directed to the positive global z-direction. In 1D you cannot use this **control\_print\_node\_angular** record. In 2D you should not specify *switch\_z* and you can only use **-yes -yes**.

The middle point of the axes in which the angle is determined should be specified with **control\_print\_node\_angular\_middle**. For example in 2D the angle follows from  $tan(angle) = \frac{y-y\_middle}{x-x\_middle}$ . In 1D you cannot use this **control\_print\_node\_angular\_middle** record. In 2D you should not specify  $z\_middle$  and you should only specify  $x\_middle$   $y\_middle$ .

See also control print node.

6.332 **control\_print\_node\_angular\_middle** *index*  $x_middle$   $y_middle$   $z_middle$ 

See control print node angular.

## 6.333 control\_print\_node\_geometry index geometry\_item\_name geometry\_item\_index

With **control\_print\_node\_geometry** you can restrict the printing to be done only on nodes located on the specified geometry. See also **control print node**.

### 6.334 control print node sort index sort\_method

With control\_print\_node\_sort you can set if the printed results should be sorted. In case you use -angular for control\_print\_node\_method, you can set the *sort\_method* to -angle. Otherwise you can set the sort method to -x, -y or -z (-y is only allowed for 2D or 3D, and -z is only allowed for 3D). The results will be sorted starting from small values (of the -angle, -x, -y or -z) up to high values.

#### 6.335 control print node zero index switch

With **control\_print\_node\_zero** you can can suppress or activate printing of results with value zero. If you set *switch* to **-yes** then zero valued results will also be printed. If you set *switch* to **-no** then zero valued results will not be printed. Default *switch* is **-yes**. See also **control print node**.

#### 6.336 control print number iterations index switch

If *switch* is set to |bf - yes, Tochnog will print the iteration number while doing equilibrium iterations in a time step. This comes convenient in very large calculations, where you want to monitor the evolution of the calculation.

# **6.337 control\_print\_partialname** index data\_item\_name\_0 data\_item\_name\_1

This printing command is similar to the normal **control\_print** command. With this record, however, all records starting with *data\_item\_name\_0*, *data\_item\_name\_1*, etc. will be printed. Thus, for example, **control\_print\_partialname 10 -element** will print all records stating with **element** (as opposed to **control\_print 10 -element** will only print the **element** record).

## 6.338 control print tecplot index switch

If *switch* is set to **-yes** a tecplot plot file is printed, and each time results are added to the same file. You can also set *switch* to **-separate\_index**; then a new file using the index number will be printed. And also you can set *switch* to **-seperate\_sequential**; then sequential tecplot files will be printed.

These files contain:

- the primary doffields from **node dof**
- post calculated results from **node** dof calcul

Tecplot uses **zones** to collect data. Zones with nodal results are given names **nodal\_...**. Zones with element averaged results are given names **element\_averaged\_...**. Tecplot uses a strandid integer to select which data is visualised. Tochnog generates in the tecplot file this strandid as follows:

• for nodal results the strandid is the group number and extra 1 is placed at the end

• for element averaged results the strandid is the group number and extra 2 is placed at the end

For example for group 100, the strandid is 1001 for nodal results, and the strandid is 1002 for element averaged results.

Tecplot files are less complete as GID files and GMSH files. Tecplot files can be plotted with the tecplot program, a trademark of Amtec Eng., Inc.

#### 6.339 control print vtk index switch

Activate printing of results in the Visual Toolkit unstructured grid format, which can be plotted by the **paraview** plotting program. See **www.paraview.org**.

For example, if the input file name is **excavation.dat** and *index* is 100 and *switch* is set to **-separate index** then results are printed in the file is **excavation100.vtk**.

For example, if the input file name is **excavation.dat** and *switch* is set to **-separate sequential** then results are printed in the files is **excavation0.vtk**, **excavation1.vtk**, etc.

In **paraview** elements are called 'cells' and nodes are called 'points'.

How to get a nice contour plot for the yy-stress:

- File open ..... choose file and hit apply button
- Coloring ..... choose node \_\_materi \_ stress and set 4 in stead of magnitude
- Edit hit the Choose preset button and select something nice.
- Edit set number of table values to e.g. 80
- Color Legend change legend text etc.
- File Save Screenshot save picture

How to get a vector plots for velocities:

- File open ..... choose file and hit apply button
- **Glyph** add glyphs for vectors
- Glyph type choose arrow
- Scale mode choose vectors
- Set scale factor choose factor to get nice vector lengths
- Coloring choose node materi velocity and choose magnitude

How to find the number of elements depicted in the plot:

- Split the screen at the top right of the layout window, and select spreadsheet view on the second screen
- View and then Selection display inspector
- In the inspector select ID for Cell labels and Point labels
- Activate the small **select cells on** button in the layout
- With the left mouse button click and drag to select the cells

How to see only elements of a certain groups:

- In Filters select Common and then select Threshold
- In Scalars select element group
- In Minimum set the mininum group number that you want to see
- In Maximum set the maximum group number that you want to see
- In Coloring select the data that you want to see

#### 6.340 control print vtk coord index switch

If *switch* is set to **-yes** the coordinates of nodes is plotted in vtk. If *switch* is set to **-no** the coordinates of nodes is not plotted in vtk. Default *switch* is set to **-yes**.

# **6.341 control\_print\_vtk\_dof** *index initialisation\_name\_0 initialisation\_name\_1*

When you specify this record only the solution fields *initialisation\_name\_0*, *initialisation\_name\_1* etc will be printed to the vtk files. So the files become smaller in size. This is especially convenient for very large calculations. The names *initialisation\_name\_0*, *initialisation\_name\_1* are names from the initialisation part like -condif\_temperature, -materi\_velocity, -materi\_stress etc. In case you do not want any field to be printed in the vtk file use control print vtk dof index -none.

See also control print vtk other.

#### 6.342 control print vtk dof calcul index calcul\_0 calcul\_1 ...

When you specify this record only the post fields  $calcul_0$ ,  $calcul_1$  etc will be printed to the vtk files. So the files become smaller in size. This is especially convenient for very large calculations. See **post\_calcul\_label** for the allowed names of  $calcul_0$ ,  $calcul_1$  etc. In case you do not want any post field to be printed in the vtk file use **control print vtk dof calcul** *index*-none.

See also control print vtk other.

#### 6.343 control print vtk empty index switch

If *switch* is set to **-yes**, empty elements are included in the vtk file. If *switch* is set to **-no**, empty elements are not included in the vtk file. Default, if **control\_print\_vtk\_empty** is not specified, *switch* is set to **-yes**.

#### 6.344 control print gid group index switch

If *switch* is set to **-yes** the element groups are plotted in vtk as result field. If *switch* is set to **-no** the element groups are not plotted in vtk as result. Default *switch* is set to **-yes**.

#### 6.345 control print vtk node method index node\_type

If *method* is set to **-node**, these coordinates are written. If *method* is set to **-node\_start\_refined**, these coordinates are written. If *method* is set to **-node\_deformed\_mesh**, these coordinates are written.

#### 6.346 control print vtk other index switch

If *switch* is set to **-yes** also other things like boundary conditions, mesh deformation etc. are printed in the vtk files. If *switch* is set to **-no** these other things are not printed in the vtk files. Default *switch* is set to **-yes**.

#### 6.347 control repeat index number of repeats control index

If *number\_of\_repeats* is larger than 0 the calculation repeats from the *control\_index*. The value of *number\_of\_repeats* is decreased by 1.

A first application is to do many time steps, but print only once in a while:

control\_timestep 10 1. 100. ... control\_print 20 -node\_dof ... control\_repeat 30 80 10

In the latter example, first 100 timesteps are taken, then results for **node\_dof** are printed; this is repeated 80 times.

Also, this **control\_repeat** can typically be used to perform a number of refinements combined with time stepping to a new, refined, solution. This is done a fixed number of times.

In case the repeat jumps back to a **control\_timestep** record for which the index equals *control\_index*, then that the previous timestep will be used (instead of the timestep specified by the **control\_timestep** record).

See also control repeat until item.

#### 6.348 control\_repeat\_save index data\_item\_name\_0 data\_item\_index\_0 data\_item\_number\_0 data\_item\_name\_1 data\_item\_index\_1 data\_item\_number\_1 ....

This record specifies data that should be saved while repeats are performed with **con-trol\_repeat**. The saved results are stored in the records **repeat\_save\_result** (subsequent repeats write in subsequent indices of repeat\_save\_result).

#### 6.349 control repeat save calculate index switch

Perform a statistical analysis on data of **repeat\_save\_result**. The statistical results are placed in **repeat calculate result**. The average value and variance will be calculated.

#### 6.350 control\_reset\_dof index dof\_0 dof\_1 ...

The dof's as specified in this record are set to a some new value. For example,  $dof_0$  is -sigxx, etc. As a special option you can use -all to reset all dof's.

With control <u>reset</u> value <u>constant</u> you can specify the new value to which the dof's should be set. Additionally you can specify values depending on space coordinates with control <u>reset</u> value <u>linear</u> etc. The records control <u>reset</u> value <u>constant</u>, control <u>reset</u> value <u>linear</u> etc. can be arbitrarily combined so that complex dependency of the value of space coordinates is possible. If none of these records is specified then a new value 0 is used.

As a typical example, you can set displacements and strains to zero in a geotechnical calculation, with an **-updated** material description, after the gravity load has been applied. In

this way the strains for further deformations can de distinguished more clearly.

The dof's will be reset on all nodes of elements which are completely inside the geometry specified in **control\_reset\_geometry**, or of elements which have all their nodes specified in **control\_reset\_node**.

As a special option for groundflow calculations, you can set an dof to **-total\_pressure** to reset the physical groundflow pore pressure (total pressure).

Attention: this **control\_reset\_dof** should not be used to reset displacements (or velocity integrated) if also **support\_edge\_normal** is present. This is because those **support\_edge\_normal** supports calculate forces directly from total displacements, and so you would in fact set the support forces also to zero. Also in the presence of interface elements the displacements (or velocity integrated) should not be reset. Normal isoparametric elements use an incremental formulation for stresses however (new stress = old stress + incremental stress from stiffness), so that resetting displacements to zero does not influence the stresses.

Attention: with this **control\_reset\_dof** option you cannot reset the strains, stresses, forces, etc. in structural elements (springs, interfaces, trusses, ...) This option only works for isoparametric elements.

#### 6.351 control reset element dof index switch

If *switch* is set to **-yes** only dofs in **element\_dof** and **element\_intpnt\_dof** will be set; dofs in **node\_dof** will not be set.

### **6.352 control\_reset\_element\_group** *index element\_group\_number\_0 element\_group\_number\_1*...

Specifies the specific element groups on which the **control\_reset\_dof** record with the same index should be applied. If this record is not specified, the **control\_reset\_dof** record will be done for all element groups (in the specified geometry).

6.353 control reset geometry index geometry item name geometry item index

Specifies the geometry on which the **control\_reset\_dof** record with the same index should be applied.

#### **6.354** control reset interface index geometry item name geometry item index

Reset all interface data like strains, stresses, etc. to 0 for interface elements located in the geometry with name geometry item name and index geometry item index.

#### 6.355 **control\_reset\_interface\_strain** index geometry\_item\_name geometry\_item\_index

Reset all interface strains to 0 for interface elements located in the geometry with name *geometry\_item\_name* and index *geometry\_item\_index*. The interface stresses at this moment of resetting will be remembered by Tochnog. In the next time steps the new interface strains start with 0, and change when the interfaces deform further. And in the next time steps the new interface stresses are calculated from the interface stresses at this moment of resetting plus stress due to additional deformation (from the specified stiffness properties).

6.356 control reset node index node\_0 node\_1 ...

Specifies nodes on which the **control\_reset\_dof** record with the same index should be applied.

#### 6.357 control reset value constant index value

Specifies the value to which dof's of the **control\_reset\_dof** record are reset. A constant *value* will be used.

#### 6.358 control reset value dof index dof

This record allows you to make the dofs of **control\_reset\_dof** dependent on a specified *dof* of this **control\_reset\_value\_dof** record. The dependency should be given in table form with **control\_reset\_value\_dof\_diagram**.

By example you can make a **-hyhis0** in the hypo laws dependent on the vertical stress **-sigyy**:

- control reset dof 10 -hyhis0
- control reset value dof 10 -sigyy
- control reset value dof diagram 10 -1.e2 0.4 -2.e2 0.38 -1.e3 0.3

This specifies that for vertical stress -1.e2 the void ratio is 0.4, for vertical stress -2.e2 the void ratio is 0.38, etc.

#### 6.359 control reset value dof diagram index ....

See control reset value dof.

#### 6.360 control reset value exponent index $a_x b_x c_x d_x e_x a_y b_y c_y d_y e_y a_z b_z c_z d_z e_z$

Specifies the exponential space distribution to which dof's of the **control\_reset\_dof** record are reset. The dependency  $a_x e^{\frac{b_x+c_xx}{d_x+e_xx}} + a_y e^{\frac{b_y+c_yy}{d_y+e_yy}} + a_z e^{\frac{b_z+c_zz}{d_z+e_zz}}$  will be used. In 1D only  $a_x b_x c_x d_x e_x$  should be specified. In 2D only  $a_x b_x c_x d_x e_x a_y b_y c_y d_y e_y$  should be specified.

#### 6.361 control reset value linear index $a_x a_y a_z$

Specifies the linear space distribution to which the dof's of the **control\_reset\_dof** record are reset. The dependency  $a_x x + a_y y + a_z z$  will be used. In 1D only  $a_x$  should be specified. In 2D only  $a_x a_y$  should be specified.

#### 6.362 control reset value logarithmic first index $a_x b_x c_x d_x e_x a_y b_y c_y d_y e_y a_z b_z c_z d_z e_z$

Specifies the logarithmic space distribution to which dof's of the **control\_reset\_dof** record are reset. The dependency  $a_x \ln(\frac{b_x + c_x x}{d_x + e_x x}) + a_y \ln(\frac{b_y + c_y y}{d_y + e_y y}) + a_z \ln(\frac{b_z + c_z z}{d_z + e_z z})$  will be used. In 1D only  $a_x b_x c_x d_x e_x$  should be specified. In 2D only  $a_x b_x c_x d_x e_x a_y b_y c_y d_y e_y$  should be specified.

#### **6.363** control reset value logarithmic second index $a_x b_x c_x d_x e_x f_x g_x a_y b_y c_y d_y e_y f_y g_y a_z b_z$

Specifies the logarithmic space distribution to which dof's of the **control** reset dof record are reset. The dependency  $(a_x + b_x)(e^{c_x \ln(d_x(x+e_x)/f_x)}) + g_x + (a_y + b_y)(\overline{e^{c_y \ln(d_y(\overline{y}+e_y)/f_y)}}) + g_y + (a_z + b_z)(e^{c_z \ln(d_z(z+e_z)/f_z)}) + g_z$  will be used. In 1D only  $a_x b_x c_x d_x e_x f_x g_x$  should be specified. In 2D only  $a_x b_x c_x d_x e_x f_x g_x a_y b_y c_y d_y e_y f_y g_y$  should be specified.

**6.364** control reset value multi linear  $index z_0 value_0 z_1 value_1 \dots$ 

Specifies the multi-linear space distribution in vertical direction to which the dof's of the **control\_reset\_dof** record are reset. A multilinear table of value versus z coordinate should be given; at  $z_0$  the value is  $value_0$  etc. The  $z_0$ ,  $z_1$  etc. should have increasing values from low to high; the values should cover all coordinates in the FE mesh for with the reset is done. In 1D not a z coordinate but x coordinate is used instead. In 2D not a z coordinate but y coordinate is used instead.

#### **6.365** control reset value power index $a_x b_x a_y b_y a_z b_z$

Specifies the power space distribution to which the dof's of the **control\_reset\_dof** record are reset. The dependency  $a_x x^{b_x} + a_y y^{b_y} + a_z z^{b_z}$  will be used. In 1D only  $a_x b_x$  should be specified. In 2D only  $a_x b_x a_y b_y$  should be specified.

#### 6.366 control reset value method index method

If *switch* is set to **-multiply** the values as specified by **control\_reset\_value** etc. are used as relative factor by which the dof's are changed. So for example if 0.1 is given in **control reset value constant**, then the dof's will be multiplied with 0.1.

If *switch* is set to **-add** the values as specified by **control\_reset\_value** etc. are used as additional values by which the dof's are changed. So for example if 0.1 is given in **control reset value constant**, then the dof's will be added with 0.1.

#### 6.367 control reset value square root $index a_x b_x c_x a_y b_y c_y a_z b_z c_z$

Specifies the power space distribution to which the to which dof's of the **control** reset dof record are reset. The dependency  $a_x\sqrt{b_x + c_xx} + a_y\sqrt{b_y + c_yy} + a_z\sqrt{b_z + c_zz}$  will be used. In 1D only  $a_xb_x$  should be specified. In 2D only  $a_xb_xa_yb_y$  should be specified.

#### 6.368 control restart index switch

If *switch* is set to **-yes** then the calculation continues with the undeformed mesh. The dof's (in the **node\_dof** records) are reset to the initial values. And **time\_current** is set to the initial time.

This allows you to calculate some path dependent behavior completely from the start with a refined mesh.

#### 6.369 control safety slip index switch

If *switch* is set to **-yes** a slip safety factor calculation will be performed with the method as described in [4]. The calculated safety factor  $F_s$  is:

$$F_s = \frac{\int \tau_{mc} dA}{\int \tau dA}$$

where  $\tau_{mc}$  is the maximum possible shear stress according to the mohr-coulomb condition using the actual normal stress,  $\tau$  is the actual shear stress and dA is the surface area in the integral. The advantage of this safety factor definition is that it can be evaluated at any stress state, for example the gravity stress state, without any further timesteps with friction angle and cohesion reduction. The definition simply compares the actual current shear stress relative to the maximum possible shear stress following from mohr-coulomb and the current normal stresses.

The user needs to specify over which surface the integration of the safety factor needs to be performed. See **safety slip circle grid \***, etc.

A critical slip surface will be calculated for each set of **safety\_slip\_circle\_grid\_\***, etc. (thus for each separate index of these a critical surface will be calculated). You can specify also **safety\_slip\_set** however, which defines the indices of **safety\_slip\_circle\_grid\_\***, etc. belonging to a specific set. The overall minimal safety factor will be determined for all safety geometries belonging to the set.

#### This control\_safety\_slip is available for group\_materi\_plasti\_mohr\_coul, group\_materi\_plasti\_mohr\_coul\_direct, group\_materi\_plasti\_druck\_prag and group\_materi\_plasti\_hypo\_wolffersdorff.

As a special option you can set the *switch* not to **-yes** but to a number 1, 2, 3, ... instead. Then this number 1, 2, 3, ... is used by tochnog as the number of automatic safety calculations of the critical slip surface. For example if you use slip circles (specified by middle points and radii) after the first safety calculations a specific middle point and radius will have the lowest safety factor. Then in the next safety calculation tochnog will reduce the area of middle points and the set of radii to a smaller zone around that critical middle point and radius. With this smaller zone a new safety analysis will lead to a new critical middle point and radius somewhere in the reduced zone. Then again a smaller zone will be used, leading to again a new critical middle point and radius, etc. etc. This repetition of reducing the zone of middle points and radii with will done such many times as set in the number, so 1, 2, 3, ... Typically the number 2 could be used.

Slip surfaces will be drawn in GID plots (see **control\_print\_gid** for GID plotting). For each slip surfaces the safety factor can be plot. Moreover, also a local safety factor can be plot, which is the local ratio of shear stress and maximum possible shear stress.

Slip surfaces crossing a boundary with prescribed displacements (or velocities) non valid since the slip velocities are in general not compatible with the prescribed velocities on such boundary.

#### 6.370 control slide damping apply index switch

If *switch* is set to **-yes** then any **slide\_damping** records will be applied. If *switch* is set to **no** then any **slide\_damping** records will be not applied. Default if **control\_slide\_damping\_apply** is not specified then *switch* is **-yes**.

#### 6.371 control slide plasti apply index switch

If *switch* is set to -yes then any slide \_plasti \_\* records will be applied. If *switch* is set to no then any slide \_plasti \_\* records will be not applied. Default if control \_slide \_plasti \_apply is not specified, plasti \_apply or control \_plasti \_apply will be used.

#### 6.372 control slide stiffness apply index switch

If *switch* is set to **-yes** then any **slide\_stiffness** records will be applied. If *switch* is set to **no** then any **slide\_stiffness** records will be not applied. Default if **control\_slide\_stiffness\_apply** is not specified then *switch* is **-yes**.

#### 6.373 control solver index solver\_type

If *solver\_type* is set to **-diagonal** then only the main diagonal of the system matrix will be used for the solution of all dof's. This gives the program an explicit like structure. In fact, if **control\_timestep\_iterations** is set to 1, then a classical explicit finite element program is obtained.

If *solver\_type* is set to **-matrix\_iterative\_bicg** then the complete system matrix will be used for solution of the principal dof's (see the initialization section for an explanation on principal dof's). A diagonal Preconditioned Biconjugate Gradient method is applied.

If *solver\_type* is set to **-matrix\_pardiso** then the pardiso solver will be used for solution of the principal dof's.

If *solver\_type* is set to **-none** then only the matrices and right-hand sides are setup, but the equations are not really solved.

#### 6.374 control solver bicg error index error

With *error* you set the termination error ratio between the initial and final error in the bicg iterations. Default *error* is set to **1.e-13**.

See also **solver\_bicg\_error**. This **control\_solver\_bicg\_error** record overrules **solver\_bicg\_error** if both are specified.

#### 6.375 control solver bicg restart index nrestart

With *nrestart* you set the number of restarts in the bicg iterations. Default *nrestart* is set to 0.

See also solver bicg restart. This control solver bicg restart record overrules solver bicg restart if both are specified.

#### 6.376 control solver bicg stop index switch

If *switch* is set to **-yes**, the calculation is stopped if the bicg solver does not converge. If *switch* is set to **-no**, the calculation is not stopped if the bicg solver does not converge. Default *switch* is set to **-yes**.

See also **solver\_bicg\_stop**. This **control\_solver\_bicg\_stop** record overrules **solver\_bicg\_stop** if both are specified.

#### 6.377 control solver matrix save index switch

If *switch* is set to **-yes**, the solver saves and applies the decomposed matrix, but not in case Tochnog thinks for some reason that the matrix needs to be decomposed at each timestep. This can save CPU time, since further decompositions of the matrix are not required anymore (only backsubstitution to find the solution vector).

If *switch* is set to **-no**, the solver does not save the decomposed matrix.

If *switch* is set to **-always**, the solver saves and applies the decomposed matrix, even in case Tochnog thinks for some reason that the matrix needs to be decomposed at each timestep.

This option is only available in combination with the pardiso solver.

Side remark: Tochnog mostly uses a linear matrix in iterations (no plasticity effect in the matrix). Only in special cases like hypoplasticity, user supplied routines, etc. the current stiffness matrix is used.

#### 6.378 control solver pardiso out of core index switch

If switch is set to **-yes** the pardiso solver is called with a 'out of core' option. See for further the pardiso solver in the the intel mkl library. Default switch is **-no**.

#### 6.379 control solver pardiso ordering index ordering

Set the number *ordering* to one of the following:

- 0 The minimum degree algorithm.
- 2 The nested dissection algorithm from the METIS package.

• 3 The parallel (OpenMP) version of the nested dissection algorithm.

Default ordering is 3. For more information see pardiso info at intel.

#### 6.380 control support edge normal damping apply index switch

If *switch* is set to **-yes** then all **support\_edge\_normal\_damping** records will be applied. If *switch* is set to **-no** then all **support\_edge\_normal\_damping** records will not be applied. Default, if **control\_support\_edge\_normal\_damping\_apply** is not specified, then *switch* is set to **-yes**.

#### 6.381 control support edge normal stiffness freeze index switch

If *switch* is set to **-yes**, tochnog freezes the stiffness forces generated by **support\_edge\_normal**. The stiffness forces remain at their present value and will not change anymore. A typical application is earthquake or vibration analysis where you first impose gravity including stiffness at supports, then freeze the forces at the supports, and then in the earthquake or vibration analysis use only damping at the supports to model absorbing boundaries which absorb further force changes at the boundaries.

```
( support properties )
support _edge _normal 10 ...
support _edge _normal _damping 10 ...
...
( calculate gravity stresses )
control _timestep 10 ...
control _support _edge _normal _damping _apply 10 -no
...
( freeze stiffness forces at boundary )
control _support _edge _normal _stiffness _freeze 20 -yes
...
( calculate earthquake or vibrations )
control _timestep 30 ...
control _support _edge _normal _damping _apply 30 -yes
```

#### 6.382 control system call index integer\_value

Specifying this record tochnog calls a system command. You need to program that command yourself. On linux provide a **tochnog\_system\_call.sh** file which is executable. On MS Windows provide a **tochnog\_system\_call.bat** file.

In the command you can place commands that you want to be executed. For example, if you put in the linux file the command **date** >> **system\_call.out** you get the output of the **date** command appended to **system\_call.out**. Another example is sending you an automatic email indicating that the calculation reached a certain point or is almost finished.

The command is called with *integer\_value* as first argument. You can use this integer value in your command (eg by using \$1 in the linux shell script command).

This command is executed at the end of each timestep and also at the end of each control index.

### **6.383 control\_timestep** *index step\_size time\_increment step\_size time\_increment ...*

These records define sets of time steps of size  $step\_size$  which are to be taken till the time is increased by  $time\_increment$ . In the example below time steps of 0.1 are taken from time 0.0 up to time 1.0. Then time steps of 0.2 are taken up to time 2.0

#### control timestep 0 0.1 1. 0.2 1.

#### 6.384 control timestep adjust minimum iterations index switch

If *switch* is set to **-yes** Tochnog will increase the minimum number of iterations in a timestep if it thinks that is helpful for the specific input file that you are running; this is done in combination with **control\_timestep\_iterations\_automatic**. If *switch* is set to **-no** Tochnog will not do so, and keep 2 as the minimum number of iterations. Default, if **control\_timestep\_adjust\_minimum\_iterations** is not specified, *switch* is set to **-yes**.

#### 6.385 control timestep iterations index number\_of\_iterations

This sets a fixed number of equilibrium iterations in each time step (for time steps of the **control\_timestep** record with the same index). For many iterations, the time stepping is Euler implicit. For few iterations the time stepping becomes explicit. Default *number\_of\_iterations* is 2.

In dynamic analysis, with the default number of 2 iterations you gain numerical stability, at the expense of numerical damping however. To prevent this numerical damping use 1 iteration instead.

As an alternative, you can use control timestep iterations automatic.

#### 6.386 control\_timestep\_iterations\_automatic index ratio\_criterium minimal\_timestep maximum\_timestep

After specification of this record, iterations will be performed until *ratio* in **post\_node\_rhside\_ratio** is less than *ratio* criterium. Typically, set *ratio* criterium to 0.001.

The time step size is increased if the number of iterations is substantially lower then the wished (preferred) number of iterations. The time step size is decreased if the number of iterations is substantially larger than the wished (preferred) number of iterations.

The time step specified in **control\_timestep** is used as initial step. The time step is not allowed to become higher then *maximum\_timestep*. The time step is not allowed to become lower then *maximum\_timestep*.

The initial step as specified in **control\_timestep**, should be sufficient small so that this automatic algorithm can fulfill the *ratio* criterium in that initial step.

After the iterations in a step are finished, Tochnog performs one extra iterations to update strains, stresses, etc with the last velocity fields. In this extra iteration also the **post\_node\_rhside\_ratio** will be recalculated, and thus may become different from the previous value that was used to determine if the iterations should be stopped.

See also control timestep iterations automatic stop, and control timestep iterations automatic

#### 6.387 control\_timestep\_iterations\_automatic\_minimum\_maximum\_wished index minimum iterations maximum iterations wished iterations

This sets the minimum number of allowed iterations, the maximum number of allowed iterations, and the wished (preferred) number of iterations for the automatic time stepping mechanism as specified by **control\_timestep\_iterations\_automatic** with the same index. The default for this record is 2.8.4. The maximum number of allowed iteration should be 2 or larger.

#### 6.388 control timestep iterations automatic stop index switch

If you set *switch* in **control\_timestep\_iterations\_automatic\_stop** to **-yes** then the calculation does stop if the minimal timestep size is reached. If you set *switch* in **control\_timestep\_iterations\_automatic\_stop** to **-no** then the calculation does not stop if the minimal timestep size is reached, and the present timestepping will be finished. If you set *switch* in **control\_timestep\_iterations\_automatic\_stop** to **-continue** then the calculation does not stop if the minimal timestep size is reached, and the present timestepping will be finished. If you set *switch* in **control\_timestep\_iterations\_automatic\_stop** to **-continue** then the calculation does not stop if the minimal timestep size is reached, and the present timestepping will not be finished. Default, if **control\_timestep\_iterations\_automatic\_stop** is not specified, then *switch* is set to **-yes**.

#### 6.389 control timestep multiplier index multiplier maximum timestep

If this record is specified, each new time step size is multiplier \* old time step size. The step size as specified in **control timestep** will only be used as the initial time step.

The timestep is not allowed to exceed the maximum specified timestep maximum\_timestep.

This option is handy to study physical processes which develop more slowly when time proceeds. A typical example is consolidation analysis in geotechnics.

# **6.390 control\_timestep\_until\_data** index data\_item\_name\_0 data\_item\_index\_0 data\_item\_number\_0 data\_item\_name\_1 data\_item\_index\_1 data\_item\_number\_1 ...

With this record you can specify conditions for which the timesteps with the same index should be stopped. For each specified data item name, index and number you can specify a mnimum value in **control\_timestep\_until\_minimum** and a maximum value in **control\_timestep\_until\_minimum** and a maximum value in **control\_timestep\_until\_minimum**. A typical example:

control\_timestep 10 ... control\_timestep\_until\_data 10 -post\_point\_dof 3 -velx control\_timestep\_until\_minimum 10 -120. control\_timestep\_until\_maximum 10 +120.

### **6.391 control\_timestep\_until\_maximum** *index maximum\_0 maximum\_1*

See control timestep until data.

### 6.392 control\_timestep\_until\_minimum index minimum\_0 minimum\_1

See control timestep until data.

#### 6.393 control truss rope apply index switch

If *switch* is set to **-no**, any truss rope data in the input file will be ignored. This is done for timestep records with the same index.

This option is convenient for testing your input file just linear, without the need to outcomment each and every part with truss rope data. See also **truss rope apply**.

#### 6.394 control zip index switch

If *switch* is set to **-yes** all **\*flavia\***, **\*msh**, **vtk**, **\*.plt** and **\*dbs** files are zipped with the **gzip** program. The **gzip** program should be installed on your computer.

This comes convenient in large calculation with lots of output, where you want to use results later and save disk space during the calculation.

#### 6.395 convection apply switch

If *switch* is set to **-yes**, the convection of a material with respect to the mesh is allowed. If *switch* is set to **-no**, the convection of a material with respect to the mesh is not allowed. This is done for all timesteps.

Default *switch* is set to **-no**. See also **control convection apply**.

#### 6.396 convection stabilization switch

Because of finite discretisation sometimes unrealistic results may be obtained (wiggles, etc.). If *switch* is set to **-yes** results are stabilized with a minimal amount of artificial diffusion. If *switch* is set to **-maximal** results are stabilized with a maximal amount of artificial diffusion. If *switch* is set to **-no** results are not stabilized.

Default, if **convection** stabilization is not specified, *switch* is set to -yes.

6.397 data\_activate index data\_item\_name\_0 data\_item\_name\_1 ... switch

With this record you can set data items to become activated if *switch* is set to **-yes** or de-activated if *switch* is set to **-no**. The *data item name* specifies a data record name.

#### 6.398 data\_activate\_time index time

Time point at which the record **data** \_activate with the same index is evaluated. If this record is not specified, the **data** activate is evaluated at the start of the calculation.

#### **6.399** data delete index data\_item\_name index\_range

Similar to control data delete, but now not as control record however.

#### 6.400 data delete time index time

Time point at which the record **data\_delete** with the same index is evaluated. If this record is not specified, the **data\_delete** is evaluated at the start of the calculation.

#### 6.401 data ignore data\_item\_name

With this option Tochnog will ignore all data items with name *data\_item\_name*. This **data\_ignore** can also be set as environment symbol. As a typical example you can set the environment symbol **data\_ignore** to **-print\_apply**. Then when you run the tochnog example tests all **-print\_apply -no** in the tests will be ignored, so you you get all printed output for the tests.

#### 6.402 dependency apply switch

If *switch* is set to **-yes**, dependencies like specified in **dependency\_diagram** and **dependency\_item** are included included in the calculation. If *switch* is set to **-no**, these dependencies are not included. This is done for all timestep records.

Default *switch* is set to **-yes**. See also **control dependency apply**.

### **6.403** dependency\_diagram index dof\_value\_0 ... data\_item\_value\_0

See dependency\_item.

#### 6.404 dependency method index method

See dependency\_item.

6.405 dependency geometry index geometry item name geometry item index

See dependency item.

#### 6.406 dependency item index data\_item element\_group dof n

This record allows you to make an element data item  $group_*$  dependent on one of the dof's, see dof\_label for dofnames, or on one of the post calculation results, see **post\_calcul\_label** for post calculation names. This is done for *n* values of the dof(*n* should be at least 2). The dependency should be specified in the **dependency\_diagram** record (same index) with a multi linear diagram. In the diagram first a set of dof's values should be specified. Second the set of data item values for those dof values should be specified. Please realise that the record which you want to make dependent needs to be specified itself also.

Some examples are given below.

Temperature dependent Young's modulus of element\_group 1 (E = 1.e10 at temperature 1, etc.):

#### dependency\_item 1 -group\_materi\_elasti\_young 1 -temp 4 dependency\_diagram 1 1. 2. 3. 4. 1.e10 1.e9 1.e8 3.e5

Temperature dependent Young's moduli in two maxwell chains of element\_group 1 (for the first chain the moduli  $1.e10, 1.e9, \ldots$  for the second chain the moduli  $1.e12, 1.e11, \ldots$  all relaxation times are  $1. 10^{-2}$ .):

```
dependency_item 1 -group_materi_maxwell_chain 1 -temp 4
dependency_diagram 1
1. 2. 3. 4.
1.e10 1.e9 1.e8 3.e5
1.e-2 1.e-2 1.e-2 1.e-2
1.e12 1.e11 1.e10 3.e7
1.e-2 1.e-2 1.e-2 1.e-2
```

As a special option, dof can be set to -time\_current. This allows for time-dependent properties (aging). The example below shows time dependent Young's modulus of element\_group 1 (E = 1.e10 at time 0, etc.):

dependency\_item 1 -group\_materi\_elasti\_young 1 -time\_current 4 dependency\_diagram 1 0. 1. 2. 3. 1.e10 1.e9 1.e8 3.e5

As a special option, *element\_group* can be set to **-all**, so that the dependency diagram will be used for all groups.

As another special option, dof can be set to  $-\mathbf{x}$ ,  $-\mathbf{y}$  or  $-\mathbf{z}$ . This allows for dependency on one of the space coordinates. The example below shows a von-mises stress dependent on the z-coordinate for element group 1:

dependency\_item 1 -group\_materi\_plasti\_vonmises 1 -z 4 dependency\_diagram 1 -300. -200. -100. 0. 1.e5 1.e4 1.e3 1.e2

In 1D only  $-\mathbf{x}$  can be used, in 2D only  $-\mathbf{x}$  and  $-\mathbf{y}$  can be used, and in 3D all of  $-\mathbf{x}$ ,  $-\mathbf{y}$  and  $-\mathbf{z}$  can be used.

The dependencies are available only for real precision data (and thus not for integer data). The **dependency diagram** values should be specified from low to high values for the dof.

The **dependency\_method** can be set to either **-use** or **-multiply**; with **-use** you specify that the values of **dependency\_diagram** will overwrite specified values for the data item; with **-multiply** you specify that the values of **dependency\_diagram** will multiply specified values for the data item; default, if **dependency\_method** is not specified, **-use** will be used.

With the **dependency** type record you can require that the cosinus, sinus or tangent of a data value is used in the dependency (in stead of the data value directly itself). The type can be set to either -cosinus, -sinus or -tangent. This is typically convenient for geotechnical safety factor calculations where you want that for a mohr coulomb law the cohesion and tangent of the friction angle are decreased at the same ratio in time. If you don't specify **dependency** type the value itself will be changed. To be clear we give the following four examples. If dependency method is set to -use and dependency type is not specified, then the value specified in the dependency diagram will be used for the data. If **dependency** method is set to -use and **dependency** type is set to -tangent, then the arc-tangent of the value specified in the dependency diagram will be used for the data. If dependency method is set to -multiply and dependency type is not specified, then the value specified in the dependency diagram will be multiplied with the original value for the data, and the result will be used as new value for for the data. If dependency method is set to -multiply and dependency type is set to -tangent, then the value specified in the dependency diagram will be multiplied with the tangent of the original value for the data, the arc-tangent of the result will be taken, and the final result will be used as new value for for the data.

With the **dependency\_number** record you can require that you only want to make one specific number of the data (0 for the first value, 1 for the second value, etc.) dependent. As a special option you can specify **-all** for the **dependency\_number** record, so that tochnog understands that you want to make all values in a record dependent. If you don't specify **dependency\_number** then all values of the record are made dependent.

The **dependency\_geometry** can be set to select a geometry for which the dependency is valid; outside the geometry the dependency will not be used; default, if **dependency\_geometry** is not specified, no geometry selection will be used.

The following gives as example lowering the tangent of the mohr coulomb friction angle with

a factor in time, for the elements of all groups within a radius distance from a point:

geometry\_point 10 ...
dependency\_item 1 -group\_materi\_plasti\_mohr\_coul-all-time\_current
dependency\_number 1 0 (only for the friction angle)
dependency\_method 1 -multiply (use specified diagram as multiplication
factor)
dependency\_type 1 -tangent (for the tangent, so not for the value itself)
dependency\_diagram 1 10. 11. 1. 0. (lower the tangent of friction angle
between time 10 to time 11 from original value to 0)
dependency\_geometry 1 -geometry\_point 10 (do that only within a certain radius of a point)

You can use **print\_group\_data** to get the result for the calculated values using the dependency diagram. In fact, most **group\_\*** records can be used in the dependency diagram, but not all. Thus checking if things go like you want with the **print\_group\_data** is stringly adviced.

#### 6.407 dependency number index number

See dependency item.

6.408 dependency type index type

See dependency item.

```
6.409 dof element dof dof_per_element_0 dof_per_element_1 ...
```

This record is for printing only. It is not meant as user input record. After the calculation the  $dof\_per\_element\_0$ ,  $dof\_per\_element\_1$  etc. contain a **-yes** or **-no**. In case a dof is default calculated per element, so the field is non-continuous, a **-yes** is set. In case a dof is default calculated as continuous field a **-no** is set. This default calculation can be overruled by **global element dof apply**.

**6.410** dof label *dof\_0 dof\_1*...

This record will be filled with labels of the dof's in the correct order. This information is required for understanding records like **node dof** etc. The sequential order for the primary

dof's will match the order in which they are specified in the initialization part.

The total list of possible doflabels is:

-accx acceleration in x-direction, -accy, -accz,

-cchis0, -cchis1 cam clay history variables,

-dam damage,

-dens density,

-dipriscohisv, -dipriscohis1, ..., di prisco plasticity history variables,

-disx displacement in x-direction, -disy, -disz,

-rdisx relative displacement in x-direction, -rdisy, -rdisz,

-ener material strain energy,

-epexx xx-strain elastic, -epexy, -epexz, -epeyy, -epeyz, -epezz,

-epixx xx-strain intergranular, -epixy, -epixz, -epiyz, -epiyz, -epizz,

 $-episa\_cxx xx-strain is a intergranular, -episa\_cxy, -episa\_cxz, -episa\_cyy, -episa\_cyz, -episa\_czz,$ 

-episa eacc is a intergranular accumulated strain,

-eppxx xx-strain plastic, -eppxy, -eppxz, -eppyz, -eppyz, -eppzz,

-eppcaxx xx-strain plastic cap model, -eppcaxy, -eppcayz, -eppcayz, -eppcazz,

-eppcoxx xx-strain plastic compression model, -eppcoxy, -eppcoyz, -eppcoyz, -eppcozz,

-eppdixx xx-strain plastic diprisco model, -eppdixy, -eppdiyz, -eppdiyz, -eppdizz,

-eppdrxx xx-strain plastic druckprag model, -eppdrxy, -eppdryz, -eppdryz, -eppdrzz,

-eppgencamxx xx-strain plastic generalised non associate cam clay for bonded soils model, -eppgencamxy, -eppgencamyz, -eppgencamyz, -eppgencamyz, -eppgencamzz,

-epphaxx xx-strain plastic hardsoil model, -epphaxy, -epphayz, -epphayz, -epphazz,

-eppmoxx xx-strain plastic mohr-coulomb model, -eppmoxy, -eppmoyz, -eppmoyz, -eppmozz,

-epptexx xx-strain plastic tension model, -epptexy, -epptexz, -eppteyz, -epptezz,

-eppvoxx xx-strain plastic von-mises model, -eppvoxy, -eppvoyz, -eppvoyz, -eppvozz,

-eptxx xx-strain total, -eptxy, -eptxz, -eptyz, -eptyz, -eptzz,

-f plasticity yield rule,

-fn nonlocal plasticity yield rule,

-fscal time derivative of scalar,

-gvelx ground water velocity in x-direction, -gvely, -gvelz.

-hisv0, -hisv1, ..., material history variables,

-kap plastic hardening parameter kappa,

-kapsh shear plastic hardening parameter kappa,

-phimob mobilized friction angle plasticity in degrees,

-pres hydraulic pressure head,

-pres gradx gradient hydraulic pressure head in x direction, -pres grady, -pres gradz

-rhoxx xx plastic kinematic hardening, -rhoxy, -rhoxz, -rhoyz, -rhozz,

-rotx rotation around x-direction, -roty, -rotz,

-scal scalar,

-sigxx xx-stress, -sigxy, -sigxz, -sigyy, -sigyz, -sigzz,

-sigmkxx xx-stress in the k-th maxwell chain, -sigmkxy, -sigmkxz, -sigmkyy, -sigmkyz, -sigmkzz,

-strtokap total strain hardening parameter,

-strtocokap compression part of total strain hardening parameter,

-strtoshkap shear part of total strain hardening parameter,

-strtotekap tension part of total strain hardening parameter,

-temp temperature,

-velx velocity in x-direction, -vely, -velz,

-velix integrated velocity in x-direction, -veliy, -veliz,

-void material void fraction.

-work material second order work.

Furthermore, **-xvelx** denotes the spatial *x*-derivative of **-velx** in *x*-direction, etc.. Finally, **-tvelx** denotes the first time derivative of **-velx**, etc.. The time derivative and the space derivatives are only calculated if **derivatives** is included in the initialization part.

For example, the following might be seen after a print of the database

echo - yes number\_of\_space\_dimensions 2 derivatives condif\_temperature end\_initia ... dof\_label - temp - xtemp - ytemp - ttemp ...

Or, for example, the following might be seen after a print of the database

echo - yes number\_of\_space\_dimensions 2 condif\_temperature end\_initia ... dof\_label - temp ...

6.411 dof limit lower\_dof\_0 upper\_dof\_0 lower\_dof\_1 upper\_dof\_1 ...

With this record you can specify the lower and upper allowed values for all primary dof's. With  $lower\_dof\_0$  you specify the lower allowed value for the first dof. With  $upper\_dof\_0$ 

you specify the upper allowed value for the first dof. Etc.

#### 6.412 dtime dt

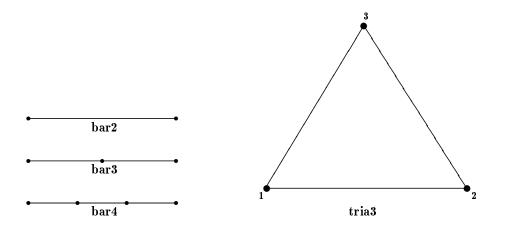
This record contains after the calculation the last timestep used in the calculation. This record is meant for printing only.

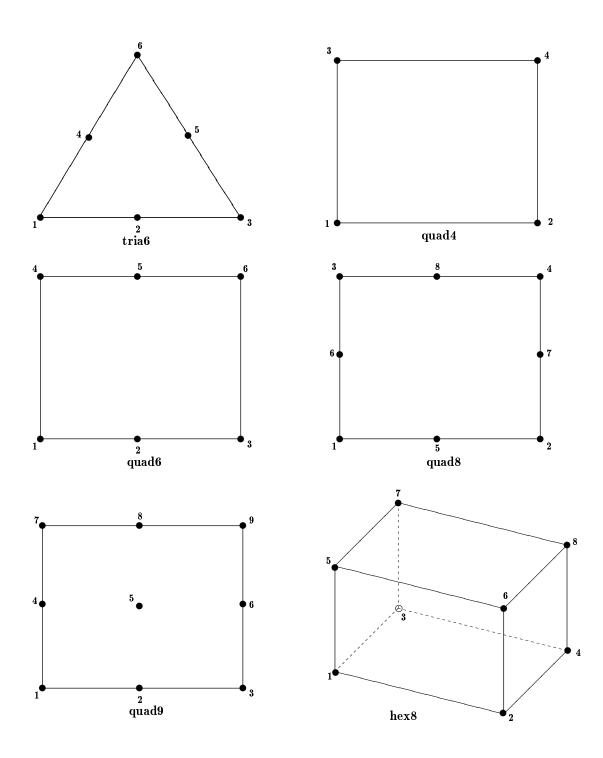
#### 6.413 element index element\_name node\_0 node\_1 node\_2 ...

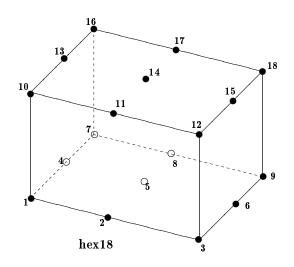
Nodal connective of element *index*. In 1D, *element\_name* is **-bar2** (2 noded bar), **-bar3**, **-bar4**. In 2D, *element\_name* is **-tria3** (3 noded triangle), **-tria6** (6 noded triangle), **quad4** (4 noded quadrilateral), **-quad6** (6 noded quadrilateral, 2 sides of 3 nodes), **-quad8**, **-quad9**, **-quad16**. In 3D, *element\_name* is **-tet4** (4 noded tetrahedral), **-prism6** (6 noded prismatic), **-prism12** (12 noded prismatic), **-prism15** (15 noded prismatic), **-prism18** (18 noded prismatic), **-tet10** (10 noded tetrahedral), **-hex8** (8 noded hexahedral), **-hex18** (18 noded hexahedral, 2 sides of 9 nodes), **-hex20** (20 noded hexahedral, not formally available yet, still being tested, use with care), **-hex27**.

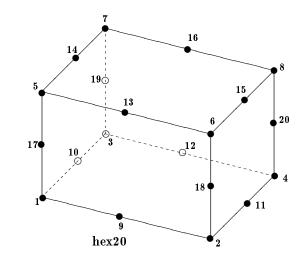
Further possibilities for *element\_name* are: -spring2 (2 noded spring), -contact\_spring1 (1 noded contact element), -contact\_spring2 (2 noded contact element), the two nodes may have the same position in space. -truss (truss element), -beam (beam element), -truss beam (combined truss-beam element).

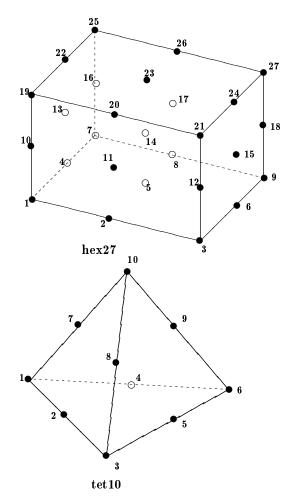
Some of the elements are drawn below.

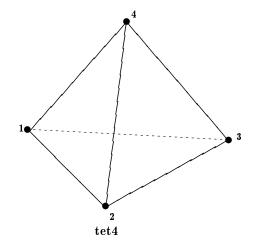


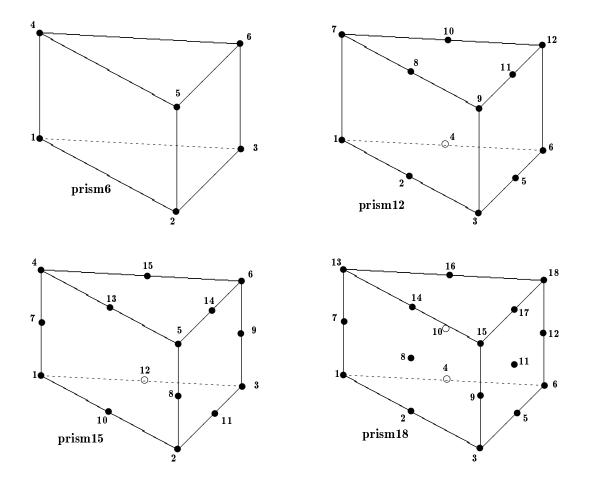












See also: group type and group integration points.

# 6.414 element beam direction index $dir_x, x \ dir_x, y \ dir_x, z \ dir_y, x \ dir_y, z \ dir_z, x \ dir_z, y \ dir_z, z$

After the calculation, this record will be filled with the direction of a beam in space. The first three values give the direction of the local beam x direction, that is the beam torsion axis. The second three values give the direction of the local beam y direction, that is the beam y bending axis. The third three values give the direction of the local beam z direction, that is the beam z bending axis.

The *index* specifies the beam element number.

#### **6.415** element beam direction z index $dir_{z,x} dir_{z,y} dir_{z,z}$

The *index* specifies the beam element number.

Sate as group beam direction z, but now per element however.

#### 6.416 element\_beam\_force\_moment index force\_x\_first\_node force\_y\_first\_node force\_z\_first\_node moment\_x\_first\_node moment\_y\_first\_node moment\_z\_first\_node force\_x\_second\_node force\_y\_second\_node force\_z\_second\_node moment\_x\_second\_node moment\_y\_second\_node moment\_z\_second\_node ...

After the calculation, this record will be filled with the forces and moments of a beam in the local beam axes x, y, z.

The *index* specifies the beam element number.

Attention: the values at the first node have a minus in their definition as compared with the values in the second node. For example in a beam number 20 with constant z moment of 10 you will find:

element beam force moment 20 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 10.

#### 6.417 element boundary index switch

The *switch* will be set to **-yes** if the element with index *index* is located on the boundary of the mesh.

This record will only become available if **mesh\_boundary** is set to **-yes**. This record is meant for printing only, it should not be set by the user.

## 6.418 element contact spring direction index $dirN_x dirN_y dirN_z dirT_1_x dirT_1_y dirT_1_z dirT_2_x dirT_2_y dirT_2_z$

In the input file, you can specify with this record the directions of a contact spring. If not specified, after the calculation this record will be filled with the used directions. The *index* specifies the spring element number.

#### 6.419 element contact spring strain index strain\_N strain\_T1 strain\_T2

After the calculation, this record will be filled with the normal and tangential elongation in a contact\_spring element. The *index* specifies the spring element number. The tangential strain *strain\_T2* only is present in 3D.

#### 6.420 element contact spring force index force\_N force\_T1 force\_T2

After the calculation, this record will be filled with the normal and tangential forces in a contact\_spring element. The *index* specifies the spring element number. The tangential force *force\_T2* only is present in 3D.

#### **6.421** element dof *index dof*\_0 *dof*\_1 ...

Unknowns as saved per element in the element nodes. First dof's in the first node. Then dof's in the second node. Etc.

This is done optionally by tochnog, only when needed for the calculation. The *index* specifies the element number.

#### 6.422 element dof initial index dof\_0 dof\_1 ...

When an element comes the first time to live, it assumes that it had in the past the dof's specified in this **element\_dof\_initial** record. You can either specify one value for each dofor you can specify values for the dof's for all nodes (specify first all dof's for the first node, then specify the dof's for the second node, etc.). The *index* specifies the element number.

This record will influence inertia terms (like mass acceleration, temperature capacity, etc). As an example you can set so the initial temperature of a part that is connected to the mesh at some time.

6.423 element dof initial specific number index number

With this record you can an initial value for one specific dof. The *number* specifies the dof number, for example -velx or -sigxx, etc. The initial value for the dof needs to be specified with element dof initial specific value. The *index* specifies the element number.

### 6.424 element\_dof\_initial\_specific\_value index value\_0 value\_grad\_x value\_grad\_y value\_grad\_z

This specifies for the **element\_dof\_initial\_specific\_number** record the initial value. Here  $value_0$  is the value at coordinate x = y = z = 0,  $value_grad_x$  is the x-gradient,  $value_grad_y$  is the y-gradient and  $value_grad_z$  is the z-gradient. In 1D you only need to specify for the gradients the  $value_grad_x$  and in 2D you only need to specify for the gradients the  $value_grad_y$ . As special option you can specify no gradients at all, and then a constant value in space of size  $value_0$  will be used.

#### 6.425 element empty index switch

If Tochnog believes an element is empty, then it will set automatically *switch* to **-empty** for **element empty**.

#### 6.426 element geometry index geometry\_set

This data item specifies for element *index* a geometrical set number *geometry\_set*. Elements with the same geometrical set number together form a geometry, which can be referenced by functionality selecting elements by a geometry. The syntax for referring is **-element** geometry geometry\_set.

A typical application would be changing material data (groups) in time for different sets of elements. In the example below element 1 belongs to geometrical set 10. The elements of geometrical set 10 get in time respectively groups 100, 101, 102 and 103.

```
element 1 -bar2 1 2
element 2 -bar2 2 3
element _geometry 1 10
element _geometry 2 20
...
area_element _group_sequence_time 11 0. 1. 2. 3.
area_element _group_sequence _geometry 11 -element _geometry 10
area_element _group_sequence_element _group 11 100 101 102 103
...
area_element _group_sequence _time 12 0. 1. 2. 3.
area_element _group_sequence _geometry 12 -element _geometry 20
area_element _group_sequence _element _group 12 200 201 202 203
...
```

The element geometry cannot be used in a geometry set.

6.427 element\_geometry\_present index geometry\_item\_name\_0 geometry\_item\_index\_0 geometry\_item\_name\_1 geometry\_item\_index\_1 ....

This record lists for element *index* the geometries in which it is present. So it is a print record only, for checking if the geometries include exactly the elements that you want. You can switch on or off filling of these records by setting **print\_element\_geometry\_present** to **-yes** or **-no**.

#### 6.428 element group index element\_group

This data item is specified which element data items should be taken for the element *index*. Example: elements 0 and 1 get density 1024 while element 2 gets density 1236

element 0 0 1 2 element 1 1 2 3 element 2 2 3 4 ... element group 0 1 element group 1 1 element group 2 2 ... density 1 1024. density 2 1236.

If no **element** group records are specified, all element data should use *index* is 0.

See also area element group and element geometry.

## 6.429 element\_group\_apply index element\_group\_0 element\_group\_1

This is yet another option to change the group of elements. It works in combination with **control\_element\_group\_apply**. We explain it by means of an example:

element 43 ... element 44 ... element\_group\_apply 43 1 7 4 element\_group\_apply 44 21 22 8 ... **control\_element\_group\_apply 10 0** (element 43 gets group 1, element 44 gets group 21)

. . .

. . .

. . .

control\_element\_group\_apply 20 2 (element 43 gets group 4, element 44
gets group 8)

**control\_element\_group\_apply 30 1** (element 43 gets group 7, element 44 gets group 22)

### **6.430** element\_interface\_intpnt\_direction index normal\_x\_0 normal\_y\_0 normal\_z\_0 first\_tangential\_x\_0 first\_tangential\_y\_0 first\_tangential\_z\_0 second\_tangential\_x\_0 second\_tangential\_y\_0 second\_tangential\_z\_0

After the calculation this record will be filled with the direction vectors in interface element. Here  $normal\_x\_0$  is the x-component of the normal direction in the first integration point, etc.

#### 6.431 element interface intput gap status index status

After the calculation, this record will be filled with the gap status in an interface element. The status is either **-opened** of **-closed**. The *index* specifies the interface element number.

#### 6.432 element\_interface\_intpnt\_materi\_tension\_status index status

After the calculation, this record will be filled with the materi tension status in an interface element. The status is either **-opened** of **-closed**. The *index* specifies the interface element number.

# **6.433** element interface intput strain index strain, normal, 0 strain, shear, first, 0 strain, shear, second, 0 strain, normal, 1 strain, shear, first, 1 strain, shear, second, 1 ...

After the calculation, this record will be filled with the normal strain, the first shear strain and second shear strain in the integration points of an an interface element. The values with subscript 0 are for the first integration point; The values with subscript 1 are for the second integration point; etc. For a 2D interface element the second shear strain will not be set. In fact, the normal strain is the normal displacement difference, and the shear strains are half of the shear displacement differences.

This **element\_interface\_intpnt\_strain** record will only be filled if **materi\_strain\_total** is initialised. The *index* specifies the interface element number.

**6.434 element\_interface\_intpnt\_strain\_average** index strain, normal, 0 strain, shear, first, 0 strain, shear, second, 0

Average of element interface intput strain.

**6.435** element\_interface\_intpnt\_stress index stress, normal, 0 stress, shear, first, 0 stress, shear, second, 0 stress, normal, 1 stress, shear, first, 1 stress, shear, second, 1 ...

After the calculation, this record will be filled with the normal stress, the first shear stress and the second shear stress in the integration points of an an interface element. The values with subscript 0 are for the first integration point; The values with subscript 1 are for the second integration point; etc. For a 2D interface element the second shear stress will not be set.

The *index* specifies the interface element number.

See **control\_reset\_interface** on how to reset strains and stresses somewhere in a calculation.

**6.436** element\_interface\_intpnt\_stress\_average index stress, normal, 0 stress, shear, first, 0 stress, shear, second, 0

Average of element interface intput stress.

6.437 element intput dof index dof\_0 dof\_1 ...

Unknowns as saved per element in the element integration points. The *index* specifies the element number.

#### 6.438 element intput h index ...

This record is meant for printing only. It contains for each node of the element the value of the interpolation polynomial in the integration points.

#### 6.439 element intput iso coord index ...

This record is meant for printing only. It contains for each node of the element the value of the isoparametric coordinates in the integration points.

### 6.440 element\_intpnt\_materi\_plasti\_hardsoil\_gammap\_initial index gammap\_initial\_integration\_point\_0 gammap\_initial\_integration\_point\_1

See theory section on hardsoil.

#### 6.441 element intput materi undrained pressure index undrained total pressure

Total pressure from undrained analysis. See group materi undrained capacity.

#### 6.442 element intput method index method

This record is meant for printing only. It shows the space integration method that is actually used for element *index*. See also **group integration method**.

#### 6.443 element intput npoint index npoint

This record is meant for printing only. It shows the number of space integration method points that are actually used for element *index*. See also **group** integration points.

**6.444 element middle** *index middle*\_*x middle*\_*y middle*\_*z* 

After the calculation, this record will be filled with the middle coordinates of an element. The *index* specifies the element number.

#### 6.445 element normal index normal x normal y normal z

This record will be generated in case it is needed during a calculation. It will contain the normal of an element in the inward normal direction of the mesh boundary. In 1d only  $normal_x$  will be filled. In 2d only  $normal_x$   $normal_y$  will be filled. In 3d all of  $normal_x$  normal y normal z will be filled.

The side of an element is on the mesh boundary if no other element is connected to that side.

#### 6.446 element print group data values index ....

Values as required by **print\_group\_data**. The first value as required by **print\_group\_data** is placed in the first value of **element\_print\_group\_data\_values**. The second value as required by **print\_group\_data** is placed in the second value of **element\_print\_group\_data\_values**. Etc. Please realise that some group data requires more than one value, so that more than one value is filled in the **element print group data values** record.

#### 6.447 element spring force index force

After the calculation, this record will be filled with the force in a spring element. The *index* specifies the spring element number.

#### 6.448 element spring strain index strain

After the calculation, this record will be filled with the strain in a spring element. In fact the strain in a spring element is the elongation of the spring. The *index* specifies the spring element number.

In case you perform a geotechnical analysis and want to set all strains in the model to 0 after gravity has been imposed, then do a **control\_data\_delete** on all **element\_spring\_strain** records. In such way the **element\_spring\_strain** records will contain in the remaining part of the calculation strains relative to the gravity status.

#### 6.449 element truss direction index dir\_x dir\_y dir\_z

After the calculation, this record will be filled with the direction of a truss in space. The *index* specifies the truss element number.

#### 6.450 element truss force index force

After the calculation, this record will be filled with the normal force in a truss element. The *index* specifies the truss element number.

#### 6.451 element truss strain index strain

After the calculation, this record will be filled with the strain in a truss element (length increase divided by length). The *index* specifies the truss element number.

#### 6.452 element truss strain temperature index strain

After the calculation, this record will be filled with the normal thermal strain in a truss element (thermal length increase divided by length). The *index* specifies the truss element number.

#### 6.453 element volume index volume

This record contains the volume of the isoparametric element number *index* after the calculation. In fact for 1D elements it contains the element length, for 2D elements it contains the element area, and for 3D elements it contains the element volume.

#### 6.454 force edge index force\_0 force\_1 ...

Distributed edge forces. These distributed forces are translated into equivalent nodal force terms on the edges of elements. You should specify a force term for each direction. Also the record force \_edge \_geometry should be specified, and optionally the records force \_edge \_factor, force edge multi linear factor x and force edge time can be specified.

<u>Attention</u>: if this **force\_edge** option is used INSIDE a FE mesh, then the elements on each side of the geometry will get the force. So you may need to specify only half of the physical force value.

Attention: this option is only available for linear and quadratic isoparametric elements.

#### 6.455 force\_edge\_diagram index vertical\_displacement\_0 factor\_0 vertical\_displacement\_1 factor\_1 ...

This is a special purpose record, to make the specified forces dependent on vertical displacements. It actually contains multiplication factors for the force values as specified in the **force\_edge** record with the same index. You need to specify sets of vertical displacements and corresponding multiplication values. Please realise the following:

- In many geotechnical calculations vertical displacements are negative (so eg vertical\_displacement\_0 is -3.e-2, vertical\_displacement\_1 is -2.e-2, etc.)
- The vertical displacements should be specified from low to high in the diagram.
- You need to specify all possible displacement values in the diagram.

6.456 force edge element index element 0 element 1 ...

Selects the element for which the **force** edge record with the same *index* should be applied.

### 6.457 force\_edge\_element\_group index element\_group\_0 element\_group\_1

Selects the element group for which the **force\_edge** record with the same *index* should be applied.

#### 6.458 force edge element node index element node\_0 node\_1 ...

Selects the element and local node numbers for which the **force\_edge** record with the same *index* should be applied.

#### 6.459 force edge element side index element 0 element 1 ... side

Selects the elements and local side number for which the **force\_edge** record with the same *index* should be applied.

#### **6.460** force edge factor index $a_0 a_1 \ldots a_n$

This data item defines a polynomial in space. This polynomial gives a factor which is used as a multiplication factor for **force\_edge** records (with the same index). In this way, you can obtain coordinate dependent forces.

In 1D the polynomial is  $a_0 + a_1x$  (specify 2 values). In 2D the polynomial is  $a_0 + a_1x + a_2y$  (specify 3 values). In 3D the polynomial is  $a_0 + a_1x + a_2y + a_3z$  (specify 4 values).

#### 6.461 force edge geometry index geometry\_entity\_name geometry\_entity\_index

Selects the area for which the **force\_edge** record with the same *index* should be applied. For example, **-geometry\_line 1** can be used in 2D, indicating that the nodes on line 1 get the distributed force. The total edge of an element must be inside the geometry for the force to become active. For 2D elements the border lines are edges. For 3D elements the border surfaces are edges.

## **6.462 force\_edge\_multi\_linear\_factor\_x** index x\_0 factor\_0 x\_1 factor\_tor\_1 ...

This factor is given as a multilinear diagram in x direction. This factor is used as a multiplication factor for **force \_edge** records (with the same index). In this way, you can obtain x-coordinate dependent forces. Outside the specified x-range the factor will be taken as 0.

#### 6.463 force edge node index node\_0 node\_1 ...

Selects the nodes for which the **force \_edge** record with the same *index* should be applied. The *node*  $\theta$  etc. specify global node numbers.

**6.464** force edge node factor index  $factor_0 factor_1 \dots$ 

Nodal multiplication factors with which the force of **force\_edge** will be applied to the nodes of **force\_edge\_node**. You need to specify a factor for each node. Here  $factor_0$  is the multiplication factor for the first node, etc.

#### 6.465 force\_edge\_sine index start\_time end\_time freq\_0 amp\_0 freq\_1 amp\_1...

The **force\_edge** record with the same *index* is imposed with the sum of the sine functions; the first sine function has frequency  $freq_0$  and amplitude  $amp_0$ , the second sine function has frequency  $freq_1$  and amplitude  $amp_1$ , etc.. The sine functions start at time 0. More general behavior in time can be imposed by using **force\_edge\_time** records. For a specific *index* only one of **force\_edge\_sine** and **force\_edge\_time** can be specified.

The sine loads will be only imposed after start time, and only up to end time.

More general time behavior can be specified with **force** edge time.

#### 6.466 force edge time index time load time load ....

This record specifies a diagram which contains the factors with which the **force\_edge** record with the same index is applied. Linear interpolation is used to extend the *time load* values to the intervals between these pairs. Outside the specified time range a factor 0 is used.

If this record is not specified, and the **force\_edge\_sine** record is not specified, the force is applied at all times with a factor 1.

If no external forces like **force\_edge\_time** are specified, the internal element forces become zero at free edges to satisfy equilibrium. This causes, for example, temperature gradients to become zero at free edges in heat problems.

#### 6.467 force edge normal index force

Distributed normal force in the direction of the outward normal at the edge of a element. This distributed term is translated into equivalent nodal force terms on the edges of elements.

Also the record **force\_edge\_normal\_geometry** should be specified, and optionally the record **force\_edge\_normal\_time** can be specified.

Attention: this option is only available for linear and quadratic isoparametric elements.

<u>Attention</u>: if this **force\_edge\_normal** option is used INSIDE a FE mesh, then the elements on each side will get the force. So the total force will normally become zero since the normals of the elements at the side of the geometry are opposite.

6.468 force edge normal element index element\_0 element\_1 ...

Restricts the element to which the **force\_edge\_normal** record with the same *index* should be applied.

#### 6.469 force edge normal element node index element node\_0 node\_1

Selects the element and local node numbers for which the **force\_edge\_normal** record with the same *index* should be applied.

6.470 force\_edge\_normal\_element\_group index element\_group\_0 element\_group\_1 ...

Restricts the element group to which the **force\_edge\_normal** record with the same *index* should be applied.

#### 6.471 force\_edge\_normal\_element\_side index element\_0 element\_1 ...side

Selects the elements and local side number for which the **force\_edge\_normal** record with the same *index* should be applied.

#### 6.472 force edge normal factor index $a_0 a_1 \ldots a_{n-1}$

This data item defines a polynomial in space. This polynomial gives a factor which is used as a multiplication factor for **force\_edge\_normal** records (with the same index). In this way, you can obtain coordinate dependent forces.

In 1D the polynomial is  $a_0 + a_1x$  (specify 2 values). In 2D the polynomial is  $a_0 + a_1x + a_2y$  (specify 3 values). In 3D the polynomial is  $a_0 + a_1x + a_2y + a_3z$  (specify 4 values).

#### 6.473 force\_edge\_normal\_geometry index geometry\_entity\_name geometry\_entity\_index

Selects the area for which the **force\_edge\_normal** record with the same *index* should be applied. For example, **-geometry\_line 1** can be used in 2D, indicating that the nodes on line 1 get the distributed force. The total edge of an element must be inside the geometry for the force to become active. For 2D elements the border lines are edges. For 3D elements the border surfaces are edges.

#### 6.474 force edge normal node index node\_0 node\_1 node\_2 ...

Selects the nodes for which the **force\_edge\_normal** record with the same *index* should be applied. The *node*  $\theta$  etc. specify global node numbers.

#### **6.475** force edge normal node factor index $factor_0 \ factor_1 \ \dots$

Nodal multiplication factors with which the force of **force\_edge\_normal** will be applied to the nodes of **force\_edge\_normal\_node**. You need to specify a factor for each node. Here *factor*<sub>0</sub> is the multiplication factor for the first node on the side, etc.

# **6.476 force\_edge\_normal\_sine** *index start\_time end\_time freq\_0 amp\_0 freq\_1 amp\_1 ...*

Same as **force** edge sine, now for normal edge loads however.

#### 6.477 force edge normal time index time load time load ....

This record specifies a diagram which contains the factors with which the **force\_edge\_normal** record with the same index is applied. Linear interpolation is used to extend the *time load* values to the intervals between these pairs. Outside the specified time range a factor 0 is used.

If this record is not specified, the force is applied at all times with a factor 1.

#### 6.478 force\_edge\_projected index force ph(0,0,0) ph\_grad\_x ph\_grad\_y ph\_grad\_z pv(0,0,0) pv\_grad\_x pv\_grad\_y pv\_grad\_z factor\_normal factor\_tangential vertical\_dir\_downward\_x vertical\_dir\_downward\_y vertical\_dir\_downward\_z tunnel\_dir\_x tunnel\_dir\_y tunnel\_z

Distributed projected force on the edge of a element. This distributed term is translated into equivalent nodal force terms on the edges of elements.

This record typically can be used to model soil normal and tangential loading on tunnels. With ph(0,0,0) you specify the horizontal ground stress at x=0,y=0,z=0. With  $ph\_grad\_x$ ,  $ph\_grad\_y$  and  $ph\_grad\_z$  you specify the gradients of the horizontal stress (such that a linear horizontal stress field can be modeled). With pv(0,0,0) you specify the vertical ground stress at x=0,y=0,z=0. With  $pv\_grad\_x$ ,  $pv\_grad\_y$  and  $pv\_grad\_z$  you specify the gradients of the horizontal stress (such that a linear horizontal stress field can be modeled). With pv(0,0,0) you specify the vertical ground stress at x=0,y=0,z=0. With  $pv\_grad\_x$ ,  $pv\_grad\_y$  and  $pv\_grad\_z$  you specify the gradients of the vertical stress (such that a linear vertical stress field can be modeled).

The vertical and horizontal stresses are projected on the edge of the element so that the radial stress sig\_radial and the tangential stress sig\_tangential of the edge of the element are obtained. You can decide to apply the radial stress sig\_radial only with a factor factor\_normal (between 0 and 1). Likewise, you can decide to apply the tangential shear stress sig\_tangential only with a factor factor\_tangential (between 0 and 1).

As extra information for Tochnog to determine the correct radial stress and tangential shear stress on the edge of an element you need to specify the downward vertical direction with vertical\_dir\_downward\_x, vertical\_dir\_downward\_y and vertical\_dir\_downward\_z.

Only in 3D, you also need to specify the length direction of the tunnel axis with tunnel\_dir\_x, tunnel\_dir\_y and tunnel\_z.

In 2D you should not specify the 3D information  $ph_grad_z$ ,  $pv_grad_z$ ,  $vertical_dir_downward_z$ , tunnel\_dir\_x, tunnel\_dir\_y and tunnel\_z.

Also the record **force\_edge\_projected\_geometry** should be used to specify where the force should be applied, and optionally the record **force\_edge\_projected\_time** can be specified.

Attention: notice that horizontal soil stress in length direction of the tunnel is not included.

Attention: this option is only available for linear and quadratic isoparametric elements.

<u>Attention</u>: if this **force\_edge\_projected** option is used INSIDE a FE mesh, then the elements on each side will get the force. So the total force will protectedly become zero since the projected of the elements at the side of the geometry are opposite.

#### 6.479 force edge projected element index element\_0 element\_1...

Restricts the element to which the **force\_edge\_projected** record with the same *index* should be applied.

### 6.480 force\_edge\_projected\_element\_node index element node\_0 node\_1

Selects the element and local nodes for which the **force\_edge\_projected** record with the same *index* should be applied.

### **6.481 force\_edge\_projected\_element\_group** *index element\_group\_0 element\_group\_1*...

Restricts the element group to which the **force\_edge\_projected** record with the same *index* should be applied.

#### 6.482 force\_edge\_projected\_element\_side index element\_0 element\_1 ...side

Selects the elements and local side number for which the **force\_edge\_projected** record with the same *index* should be applied.

#### **6.483** force edge projected factor index $a_0 a_1 \ldots a_n$

This data item defines a polynomial in space. This polynomial gives a factor which is used as a multiplication factor for **force\_edge\_projected** records (with the same index). In this way, you can obtain coordinate dependent forces.

In 1D the polynomial is  $a_0 + a_1x$  (specify 2 values). In 2D the polynomial is  $a_0 + a_1x + a_2y$  (specify 3 values). In 3D the polynomial is  $a_0 + a_1x + a_2y + a_3z$  (specify 4 values).

#### 6.484 force\_edge\_projected\_geometry index geometry\_entity\_name geometry\_entity\_index

Selects the area for which the **force\_edge\_projected** record with the same *index* should be applied. For example, **-geometry\_line 1** can be used in 2D, indicating that the nodes on line 1 get the distributed force. The total edge of an element must be inside the geometry for the force to become active. For 2D elements the border lines are edges. For 3D elements the border surfaces are edges.

#### 6.485 force edge projected node index node\_0 node\_1 node\_2 ...

Selects the nodes for which the **force\_edge\_projected** record with the same *index* should be applied. The *node*  $\theta$  etc. specify global node numbers.

**6.486** force edge projected node factor index  $factor_0$  factor<sub>1</sub>...

Nodal multiplication factors with which the force of **force\_edge\_projected** will be applied to the nodes of **force\_edge\_projected\_node**. You need to specify a factor for each node. Here *factor*<sub>0</sub> is the multiplication factor for the first node, etc.

#### 6.487 force\_edge\_projected\_sine index start\_time end\_time freq\_0 amp\_0 freq\_1 amp\_1 ...

Similar to force edge sine, now for projected edge loads however.

#### 6.488 force edge projected time index time load time load ....

This record specifies a diagram which contains the factors with which the **force \_edge \_projected** record with the same index is applied. Linear interpolation is used to extend the *time load* values to the intervals between these pairs. Outside the specified time range a factor 0 is used.

If this record is not specified, the force is applied at all times with a factor 1.

#### 6.489 force edge water index switch

If switch is set to **-yes**, distributed water pressure force is added to the model. This distributed term is translated into equivalent nodal force terms on the edges of elements. The distributed force is automatically calculated as density\_water g  $\Delta z$  where g is the gravitational acceleration, and  $\Delta z$  is the distance to the phreatic level. The water pressure force acts normal to the element edge, in inward direction. You need to specify also force edge water geometry.

The water density is given by **groundflow** density. The gravity acceleration is given by the vertical component of **force** gravity. The water height is relative to the water height is given by **groundflow** phreatic level.

Attention: if this force edge water option should be used with care INSIDE a FE mesh.

The total edge of an element must be inside the geometry for the force to become active. For 2D elements the border lines are edges. For 3D elements the border surfaces are edges.

#### 6.490 force edge water element index element\_0...

Selects the element for which the **force\_edge\_water** record with the same *index* should be applied.

6.491 force edge water element group index element\_group\_0...

Selects the element groups for which the **force\_edge\_water** record with the same *index* should be applied.

### 6.492 force\_edge\_water\_element\_node index element node\_0 node\_1

Selects the element and local nodes for which the **force\_edge\_water** record with the same *index* should be applied.

### 6.493 force\_edge\_water\_element\_side index element\_0 element\_1 ... side

Selects the elements and local side number for which the **force\_edge\_water** record with the same *index* should be applied.

#### **6.494** force\_edge\_water\_factor index $a_0 a_1 \dots a_n$

This data item defines a polynomial in space. This polynomial gives a factor which is used as a multiplication factor for **force\_edge\_water** records (with the same index). In this way, you can obtain coordinate dependent forces.

In 1D the polynomial is  $a_0 + a_1x$  (specify 2 values). In 2D the polynomial is  $a_0 + a_1x + a_2y$  (specify 3 values). In 3D the polynomial is  $a_0 + a_1x + a_2y + a_3z$  (specify 4 values).

#### 6.495 force\_edge\_water\_geometry\_index geometry\_item\_name geometry\_item\_index

Selects the area for which the **force\_edge\_water** record with the same *index* should be applied. For example, **-geometry\_line 1** can be used in 2D, indicating that the nodes on line 1 get the distributed water pressure force.

#### 6.496 force edge water node index node\_0 node\_1 ...

Selects the nodes for which the **force\_edge\_water** record with the same *index* should be applied. The *node*  $\theta$  etc. specify global node numbers.

#### 6.497 force edge water time index time load time load ...

This record specifies a diagram which contains the factors with which the **force \_edge \_water** record with the same index is applied. Linear interpolation is used to extend the *time load* values to the intervals between these pairs. Outside the specified time range a factor 0 is used.

If this record is not specified, the force is applied at all times with a factor 1.

#### 6.498 force gravity $g_x g_y g_z$

Gravitational acceleration.

In 1D, only the gravity in x-direction needs to be specified. In 2D, the gravity in x-direction and y-direction needs to be specified. In 3D, the gravity in x-direction, y-direction and z-direction needs to be specified.

See also force gravity time.

#### 6.499 force gravity geometry geometry\_item\_name geometry\_item\_index

With this record you can specify a geometrical entity on which the gravity force should be used. Only elements inside the geometry get the gravity force.

If this record is not specified all elements can get the gravity force.

See also force gravity time.

#### 6.500 force gravity time time load time load ....

This record specifies a multi-linear diagram which contains the factors with which the **force\_gravity** record is applied. This allows you to impose the gravity on a structure slowly, which might be needed for path dependent problems. Outside the specified time range a factor 0 is used.

If this record is not specified, the gravity is applied at all times with a factor 1.

#### 6.501 force point index coord\_0 coord\_1 coord\_2 force\_0 force\_1 force\_2

Point force in space; it needs not to be exactly in a node. The force will be distributed over the element nodes in which the point force is located. This will be done consistent with the element interpolation functions (linear or quadratic).

The coord\_ $\theta$  specifies the x-coordinate in space. The coord\_1 specifies the y-coordinate in space (only in two or 3 dimensions). The coord\_2 specifies the z-coordinate in space (only in 3 dimensions). The force\_ $\theta$  specifies the x-force in space. The force\_1 specifies the y-force in space (only in two or 3 dimensions). The force\_2 specifies the z-force in space (only in 3 dimensions).

You can use the usual **change\_dataitem\_time** to change the position (or value) of the point force in time.

#### 6.502 force volume index force\_0 force\_1 ...

Distributed volume forces for each direction. Here  $force_{-}\theta$  is the distributed force in the x-direction, etc. Consider the example with distributed volume force in x-direction for a 2D material:

force volume 0 1.0.

The force volume record can be used in dependency diagram records (just like element group data)/

See also force volume factor, force volume geometry, and force volume time.

6.503 force volume element index element 0 element 1 ...

Specifies the elements for which the **force\_volume** record with the same *index* should be applied.

### 6.504 force\_volume\_element\_group\_0 element\_group\_1 ... $in-dex \ element\_group$

Specifies the element group for which the **force\_volume** record with the same *index* should be applied.

#### **6.505** force volume factor index $a_0 a_1 \ldots a_n$

This polynomial gives a factor which is used as a multiplication factor for **force\_volume** records (with the same index). In this way, you can obtain coordinate dependent forces.

In 1D the polynomial is  $a_0 + a_1x$  (specify 2 values). In 2D the polynomial is  $a_0 + a_1x + a_2y$  (specify 3 values). In 3D the polynomial is  $a_0 + a_1x + a_2y + a_3z$  (specify 4 values).

6.506 force volume geometry index geometry item name geometry item index

Specifies the area for which the **force\_volume** record with the same *index* should be applied. For example, **-geometry\_quadrilateral 1** can be used in 2D, indicating that the elements on quadrilateral 1 get the distributed force.

If both the **force\_volume\_element** and **force\_volume\_geometry** are not specified, then a geometry which encloses the whole model will be applied.

### 6.507 force\_volume\_sine index start\_time freq\_0 amp\_0 freq\_1 amp\_1

Same as force volume sine, now for volume loads however.

#### 6.508 force volume time index time load time load ....

This record specifies a multi-linear diagram which contains the factors with which the **force volume** record with the same index is applied.

If this record is not specified, the force is applied at all times with a factor 1.

#### 6.509 geometry boundary index switch

With this record you can restrict a geometry to the boundary of the mesh, or to the inside of the mesh. If *switch* is set to **-yes** only nodes which are at the boundary of the mesh are actually used for the geometry with the same index. If *switch* is set to **-no** only nodes which are not at the boundary of the mesh are actually used for the geometry with the same index.

Attention: for this option to work correctly, the mesh should not contain badly shaped elements. See the section at the end of this manual for more information on bad element shapes.

#### 6.510 geometry bounda sine x index a b

This sets, for the geometrical entity with the same index an extra factor which is used for the **bounda** dof and the **bounda** force records. The factor gives a sinus variation in x-direction. The size of the factor is sin(a + b \* x).

#### 6.511 geometry bounda sine y index a b

This sets, for the geometrical entity with the same index an extra factor which is used for the **bounda** dof and the **bounda** force records. The factor gives a sinus variation in y-direction. The size of the factor is sin(a + b \* y).

#### 6.512 geometry bounda sine z index a b

This sets, for the geometrical entity with the same index an extra factor which is used for the **bounda\_dof** and the **bounda\_force** records. The factor gives a sinus variation in z-direction. The size of the factor is sin(a + b \* z).

#### **6.513** geometry brick index $x_c \ y_c \ z_c \ l_x \ l_y \ l_z$ tolerance

This data item defines a brick in space. Other data items can check if nodes are located on this geometry. The coordinate of the center is  $x_c c y_c c z_c$ . The length in respectively x, y and z direction are  $l_x l_y l_z$ . All **node** within a distance *tolerance* are considered to be part of the brick.

#### **6.514 geometry circle** index $x_c \ y_c \ \dots \ radius$ tolerance

This data item defines a circle in space. Other data items can check if nodes are located on this geometry. The coordinate of the center is  $x_c \ y_c \ c$ . In 2D you need to specify  $x_c \ y_c \ c$  radius tolerance. In 2D all **node** within a distance tolerance of the radius are considered to be part of the circle. In 3D you need to specify  $x_c \ y_c \ c \ z_c \ normal_x \ normal_y \ normal_z \ radius \ tolerance$ , where normal\_x normal\_y normal\_z specifies the direction normal to the surface. In 3D all **node** within a distance tolerance of the circle surface are considered to be part of the circle.

#### 6.515 geometry\_circle\_part index x\_c y\_c angle\_start angle\_end radius tolerance

This data item defines a circle in 2D space. Other data items can check if nodes are located on this geometry. The coordinate of the center is  $x_c c y_c$ . All **node** within a distance

tolerance of the radius are considered to be part of the circle. The circle part starts at angle angle\_start, measured in radians from the positive x-axis. The circle part ends at angle angle end, measured in radians from the positive x-axis.

### 6.516 geometry\_circle\_segment index x\_c y\_c radius side\_x side\_y tolerance

This data item defines a circle segment in space. Other data items can check if nodes are located on this geometry. The coordinate of the center is  $x_c c y_c c$ . If  $side_x$  is set to a positive value, say +1., then only x-values larger then  $x_c$  are considered to be part of the geometry. If  $side_x$  is set to a negative value, say -1., then only x-values smaller then  $x_c$  are considered to be part of the geometry. If  $side_x$  is set to a negative value, say -1., then only x-values smaller then  $x_c$  are considered to be part of the geometry. If  $side_x$  is set to 0, then all x-values are considered to be part of the geometry. Likewise remarks hold for y-values. All **node** within a distance *tolerance* of the radius are considered to be part of the circle segment.

### 6.517 geometry\_cylinder index $x_0 y_0 z_0 x_1 y_1 z_1$ radius tolerance

This data item defines a cylinder segment in space. Other data items can check if nodes are located on this geometry. The coordinate of the center point at the bottom is  $x_0 y_0 z_0$ . The coordinate of the center point at the top is  $x_1 y_1 z_1$ . The cylinder can only be used in 3D. All **node** within a distance *tolerance* of the radius are considered to be part of the cylinder.

**6.518** geometry cylinder part index x 0 y 0 z 0 x 1 y 1 z 1 radius angle\_start 0 angle\_end\_0 angle\_start 1 angle\_end\_1 ... tolerance

This data item defines parts of a cylinder in space. Other data items can check if nodes are located on this geometry.

The index  $x_0 y_0 z_0 x_1 y_1 z_1$  radius are the same as in **geometry** cylinder.

The  $angle\_start\_0$   $angle\_end\_0$  defines the first valid part of the cylinder, where  $angle\_start\_0$  is the start angle of the part and  $angle\_end\_0$  is the end angle. The angles are measured in the x-y plane, starting from the positive x-axis towards the positive y-axis. Likewise, the  $angle\_start\_1$   $angle\_end\_1$  defines a second valid part of the cylinder. You should define at least one valid part, and optionally you can specify several valid parts.

Start angles and end angles should be non-negative. End angles should be larger than start angles.

Angles will be measured relative to the vector as specified in **geometry\_cylinder\_part\_start\_vector**, if that vector is specified. This **geometry\_cylinder\_part\_start\_vector** should be specified perpendicular to the cylinder axes. This **geometry\_cylinder\_part\_start\_vector** 

should be exactly in the middle of the angle range that you want to select. With **geome-**try\_cylinder\_part\_start\_vector only one angle range is allowed, and the start angle should be 0. All nodes with an angle smaller or equal to the end angle are accepted as valid (thus, you get a total angle range of twice the end angle size as valid range).

If geometry\_cylinder\_part\_start\_vector is not specified, the geometry\_cylinder\_part should be either along the x-direction, y-direction or z-direction; then the angle is measured relative to the axes (for example for a cylinder along the z-direction the angle starts at the x-axes).

All **node** within a distance *tolerance* of the radius and inside a valid part are considered to be part of the cylinder part.

#### 6.519 geometry cylinder part start vector $index v_x v_y v_z$

See geometry cylinder part.

#### 6.520 geometry\_cylinder\_segment index x\_0 y\_0 z\_0 x\_1 y\_1 z\_1 radius side x side y side z tolerance

This data item defines a cylindrical segment in space. Other data items can check if nodes are located on this geometry. The coordinate of the center point at the bottom is  $x_0 y_0 z_0$ . The coordinate of the center point at the top is  $x_1 y_1 z_1$ . If *side\_x* is set to a positive value, say +1., then only x-values larger then  $x_c$  are considered to be part of the geometry. If *side\_y* is set to a negative value, say -1., then only x-values smaller then  $x_c$  are considered to be part of the geometry. If *side\_y* is set to a negative value, say -1., then only x-values smaller then  $x_c$  are considered to be part of the geometry. If *side\_x* is set to 0, then all x-values are considered to be part of the geometry. Likewise remarks hold for y- and z-values. The cylinder segment can only be used in 3D. All **node** within a distance *tolerance* of the radius are considered to be part of the cylinder.

# **6.521** geometry\_exclude index geometry\_item\_name\_0 geometry\_item\_index\_0 geometry\_item\_name\_1 geometry\_item\_index\_1 ...

With this record you can exclude geometries from the geometry with the same index. The next 2D example excludes a circular area with radius 0.3 inside a quadrilateral:

geometry\_quadrilateral 10 0. 0. 1. 0. 0. 1. 1. 1. geometry\_exclude 10 -geometry\_point 20 geometry\_point 20 0.5 0.5 0.3 You are not allowed to let a geometry \* use a geometry exclude which contains itself.

### **6.522 geometry\_element\_geometry** *index element\_geometry\_0 element\_geometry\_1*

Similar to geometry\_element\_group, but now using element\_geometry i.s.o. element group however.

#### 6.523 geometry element geometry method index method

Similar to geometry element group method.

### **6.524** geometry\_element\_group index element\_group\_0 element\_group\_1

With this record you can restrict the geometry as specified in the geometry record with the same *index*. For example for the geometry as specified by

geometry\_quadrilateral 10 ... geometry\_element\_group 10 ...

nodes which are located on the **geometry\_quadrilateral 10**, but at the same time are also a node of elements of one of the specified element groups *element\_group\_0 element\_group\_1* etc., belong to the geometry. Nodes which are not a node of elements of one of the groups do not belong to the geometry, even if such nodes are located on the **geometry\_quadrilateral 10**.

See also geometry element group method.

#### 6.525 geometry element group method index method

With this record you can set the *method* that the **geometry\_element\_group** record uses. If *method* is set to **-all** then a node should be attached to all the specified element groups, to be part of the geometry. If *method* is set to **-any** then a node should be attached to any of the specified element groups, to be part of the geometry. If *method* is set to **-only** then a node should be attached to only the specified element groups, to be part of the geometry. Default, if *method* is not specified then **-any** is assumed.

#### **6.526 geometry ellipse** index $x_c \ y_c \ a \ b$ tolerance

The coordinate of the center is  $x_c y_c$ . The equation for the ellipse is:

$$\left(\frac{x-x\_c}{a}\right)^2 + \left(\frac{y-y\_c}{b}\right)^2 = 1$$

Other data items can check if nodes are located on this geometry. The ellipse can only be used in 2D. All **node** within a distance *tolerance* of the ellipse are considered to be part of the ellipse.

#### **6.527** geometry factor index factor $0 \dots$

This sets for some geometries extra factors which are used for the **bounda\_dof**, **bounda\_force** and **force\_edge\_\*** records. For a **geometry\_line** either 2 or 3 factors should be specified; 2 factors define a linear variation where the factors hold at the start and end of the line respectively; 3 factors define a parabolic variation where the factors hold at the start, at the middle and at the end of the line respectively. For a **geometry\_triangle 3** factors should be specified (a linear variation with factors for the first, second and third corner point respectively). For a **geometry\_quadrilateral** 4 factors should be specified (a linear variation with factors for the first, second, third and fourth corner point respectively). For a **geometry\_point** 1 factor should be specified; a multiplication with a half sine wave will be used, with the specified factor in the middle (exactly at the point) creasing to factor 0 at a distance *tolerance* from the point,

In the example below, node 2 will get temperature 20 \* 1.6 and node 3 will get temperature 20 \* 2.2.

```
number _of _space _dimensions 2
condif _temperature
...
end _initia
node 2 0.2 0
node 3 0.4 0.
...
geometry _line 1 0. 0. 1. 0. 0.01
geometry _factor 1 1. 4.
bounda _dof 0 -geometry _line 1 -temp
bounda _time 0 0. 20. 1.e6 20.
...
end _data
```

6.528 geometry\_hexahedral index  $x_0 y_0 z_0 x_1 y_1 z_1 x_2 y_2 z_2 x_3 y_3 z_3 x_4 y_4 z_4 x_5 y_5 z_5 x_6 y_6 z_6 x_7 y_7 z_7$ 

This data item defines a hexahedral in space. Other data items can check if nodes are located on this geometry (everything inside the hexahedral belongs to the geometry). The coordinates of the corner points are  $x_0 y_0 z_0$  etc.. The points of the hexahedral should be specified in the correct order; the order is clarified in the example below.

Example

number\_of\_space\_dimensions 3 .... geometry\_hexahedral 0 0. 0. 0. 1. 0. 0. 0. 1. 0. 1. 1. 0. 0. 0. 1. 1. 0. 1. 0. 1. 1. 1. 1. 1. ...

Notice the order in which the points are to be specified.

#### **6.529 geometry line** index $x_0 y_0 z_0 x_1 y_1 z_1$ radius

This data item defines a line in space. Other data items can check if nodes are located on this geometry. Coordinates of the end points are denoted by  $x_0$ , etc.. In 1D, only the *x*-coordinates should be specified, etc.. All **node** within a distance *radius* are considered to be part of the line.

In the example, a line in 2D space is defined and is used by a **convection\_geometry** record (nodes located on the line will convect heat)

geometry\_line 2 1. 0. 1. 1. 0.01
...
group\_condif\_convection\_edge\_normal\_geometry 0 -geometry\_line
2
...

#### 6.530 geometry line eps iso index iso\_tolerance

With this parameter you can ask Tochnog to accept points just outside the line in direction of the line. Typically try 1.e-3 for *iso tolerance*.

#### 6.531 geometry list index number\_0 number\_1 ....

This is a list of numbers which can be used in geometry selection options.

For example

. . .

geometry\_list 10 1 45 43 26 27
...
bounda\_dof 200 -geometry\_list 10 ... (set the boundary condition on the
nodes of the list)
...

#### 6.532 geometry method index method

For selecting elements with a geometry enity you can set the *method* either to **-all**, **-any** or **-average**. With **-all** all nodes of an element should be inside the geometry entity for the element to be selected (completely inside). With **-any** any node of an element should be inside the geometry entity for the element to be selected (at least partially inside). With **-average** the middle coordinate of an element should be inside the geometry entity for the element to be selected. Default if this record is not specified the *method* is set to **-all**.

#### 6.533 geometry moving index geometry\_entity

This option comes handy when you want to model moving excavations in complex FE meshes. For complex geological regions it is not possible to make the FE mesh a priori in such way that the excavation zones can be easily defined in terms of element groups or otherwise. In such case this option allows geometrical entities to move through the complex FE mesh. The geometrical entity will determine automatically which elements become part of the entity at which times, and then will automatically excavate those elements. Elements which are at a moment in time only partly inside the geometrical entity, will be only partly excavated to the same amount as which they are inside the geometrical entity. Elements which are at a moment in time completely inside the geometrical entity, will be fully excavated.

We first explain the usage of this option by means of an example:

geometry\_moving 10 -geometry\_point (geometical point that will move in space and excavate the mesh) geometry moving parameter 10 -1. -1. 2.e-1 (start x-coordinate of

```
point, start y-coordinate of point, radius of point)
geometry moving operat 10 -translate (translate the point in space)
geometry moving operat parameter 10 1. 1. (velocity of point in x-
direction, velocity of point in y-direction)
geometry moving operat time 10 0. 3. (start time of point moving,
end time of point moving)
geometry moving n 10 200 200 (number of time discretision, number of
space distretisation)
. . .
mesh delete geometry moving 10 10 (tell Tochnog to use the geome-
try moving records with index 10 to excavate the mesh)
control geometry moving 25 -initialise (prepare the point for moving
through the mesh)
. . .
control timestep 30 ...
. . .
```

 $The geometrical entity can be one of -geometry\_point, -geometry\_triangle, -geometry\_quadrilateral, -geometry\_hexahedral and -geometry\_tethrahedral.$ 

#### 6.534 geometry moving parameter index parameters of entity

For each vertex of the geometry you need to specify the initial x-coordinate, y-coordinate (only for 2D and 3D) and z-coordinate. A point has one vertex, a triangle three, a quadrilateral four, a hexahedral eight and a tethrahedral six. The sequence of the vertices is the same as the sequence of nodes for the similar finite elements (see element). For a -geometry point you need to specify additionally the radius.

See also geometry moving.

#### 6.535 geometry moving operat index operator

This operator specifies how the geometry changes between time\_start and time\_end. If operator is set to **-translate** the geometry moves in space with a constant velocity vector.

See also **geometry\_moving**.

#### 6.536 geometry\_moving\_operat\_parameter index parameters of operator

For a **-translate** you need to specify the x-velocity, y-velocity (only for 2D and 3D) and z-cvelocity.

See also geometry moving.

#### 6.537 geometry moving operat time index start\_time end\_time

The *start\_time* specifies when the geometry comes into existance and starts to move. The *end\_time* specifies when the geometry stops moving but remains in existance.

See also geometry moving.

#### 6.538 geometry moving n index ntime nspace

The **geometry\_moving** command needs to find out for each element at each time point which part of the element is part of the geometrical entity. In order to found this out, it is checked for many points inside each element if the point is inside the geometrical entity. In fact, for a 1D element *nspace* points will be used, for a 2D element *nspace* \* *nspace* points will be used and for *nspace* \* *nspace* \* *nspace* \* *nspace* \* *nspace*.

This checking which part of each element is inside the geometrical entity will be done at *ntime* time moments between *time* start and *time* end.

Default, if **geometry** moving is not specified, we use *ntime* is 100 and *nspace* is 5.

See also geometry moving.

#### 6.539 geometry mpc index switch

With this record you can restrict the geometry as specified in the geometry record with the same *index*. Only nodes which have mpc's are considered to be part of the geometry.

#### 6.540 geometry\_node\_type node\_type

If *node\_type* is set to **-node\_start\_refined** the values of **-node\_start\_refined** are used for evaluating the geometry. If *node\_type* is set to **-node** the values of **-node** are used for evaluating the geometry. If *node\_type* is set to **-plus\_displacement** the values of **-node** plus nodal displacements are used for evaluating the geometry. Default *node\_type* is set to **-node\_start\_refined**.

#### 6.541 geometry point index x y z radius

This data item defines a point in space. Other data items can check if nodes are located on this geometry. The coordinate of the point is  $x \ y \ z$ . In 1D, only x should be specified, etc.. All **node** within a distance *radius* are considered to be part of the point.

#### **6.542** geometry\_polynomial index $a_0 a_1 \ldots a_n x_0 x_1 y_0 y_1$ tolerance

This data item defines a polynomial in space in 2D or 3D. Other data items can check if nodes are located on this geometry.

In 2D the polynomial is  $a_0 + a_1x + a_2y$  (specify 3 values). In 3D the polynomial is  $a_0 + a_1x + a_2y + a_3z$  (specify 4 values).

In 2D  $x_0 - x_1$  defines the domain of x. In 3D  $x_0 - x_1$  defines the domain of x and  $y_0 - y_1$  defines the domain of y. All **node** with a distance (that is the y-distance in 2D or the z-distance in 3D) not more than *tolerance* are considered to be part of the polynomial.

#### 6.543 geometry projection type index type

This record allows you to control what geometry will actually be used. Set *type* to -**project\_inside** or -**project\_exact**. For example if the geometry is a **geometry\_circle** then -**project\_inside** means that everything inside the circle will be used, whereas -**project\_exact** means that everything within a tolerance from the circle edge will be used. Default *type* is -**project\_exact**.

#### 6.544 geometry\_quadrilateral index $x_0 y_0 z_0 x_1 y_1 z_1 x_2$ $y_2 z_2$ $x_3 y_3 z_3$ tolerance

This data item defines a quadrilateral in space. Other data items can check if nodes are located on this geometry. The coordinates of the corner points are  $x_0 y_0 z_0 etc.$  In 2D, only  $x_0, y_0 etc.$  should be specified etc.. The points of the quadrilateral should be specified in the correct order; the order is clarified in the example below.

In 2D all **node** inside the quadrilateral (the tolerance is neglected). In 3D all **node** within a distance *tolerance* are considered to be part of the quadrilateral (this is a brick with thickness *tolerance*). All **node** within a distance *tolerance* are considered to be part of the quadrilateral (in 2D this gives a quadrilateral with corners nodes specified by the corners points, in 3D this gives a brick corners nodes specified by the corners points and with thickness *tolerance*). Internally in TOCHNOG, the quadrilateral is divided into two **geometry triangles**, which is only approximately true if the quadrilateral is twisted. Example number\_of\_space\_dimensions 2 ... geometry\_quadrilateral 0 0. 0. 1. 0. 0. 1. 1. 1. 1.e-3

Notice the order in which the points are to be specified.

#### 6.545 geometry quadrilateral eps iso index iso tolerance

With this parameter you can ask Tochnog to accept points just outside the quadrilateral in direction of the quadrilateral plane. Typically try 1.e - 3 for *iso\_tolerance*.

# 6.546 geometry\_set index geometry\_entity\_0 geometry\_entity\_index\_0 geometry\_entity\_1 geometry\_entity\_index\_1 ...

This set combines a number of geometrical entities (e.g. **geometry\_circle**, **geometry\_line**, etc.) into a new entity. You cannot use another geometry set for the geometrical entities (that is, geometry sets cannot be nested).

Other data items can check if nodes are located on this geometry.

#### 6.547 geometry sphere index x\_c y\_c z\_c radius tolerance

This data item defines a sphere in space. Other data items can check if nodes are located on this geometry. The coordinate of the center is  $x_c \ y_c \ z_c$ . All **node** within a distance *tolerance* of *radius* are considered to be part of the sphere.

### **6.548** geometry\_sphere\_segment index x\_c y\_c z\_c radius side\_x side\_y side\_z tolerance

This data item defines a spherical segment in space. Other data items can check if nodes are located on this geometry. The coordinate of the center is  $x_c \ y_c \ z_c \ z_c$ . If  $side_x$  is set to a positive value, say +1, then only x-values larger then  $x_c$  are considered to be part of the geometry. If  $side_x$  is set to a negative value, say -1, then only x-values smaller then  $x_c$  are considered to be part of the geometry. If  $side_x$  is set to 0, then all x-values are considered to be part of the geometry. Likewise remarks hold for y- and z-values.

All **node** within a distance *tolerance* of *radius* are considered to be part of the spherical segment.

#### **6.549** geometry\_tetrahedral index x\_0 y\_0 z\_0 x\_1 y\_1 z\_1 x\_2 y\_2 z\_2 x\_3 y\_3 z\_3

This data item defines a tetrahedral in space. Other data items can check if nodes are located on this geometry. The coordinates of the corner points are  $x \ 0 \ y \ 0 \ z \ 0$  etc..

#### 6.550 geometry\_triangle index $x_0 y_0 z_0 x_1 y_1 z_1 x_2 y_2 z_2$ tolerance

This data item defines a triangle in space. Other data items can check if nodes are located on this geometry. The coordinates of the corner points are  $x_0 y_0 z_0$  etc.. In 2D the z coordinates should not be specified. All **node** within a distance *tolerance* are considered to be part of the triangle (this gives a wedge with thickness 2*tolerance*).

#### 6.551 geometry triangle eps iso index iso\_tolerance

With this parameter you can ask Tochnog to accept points just outside the triangle in direction of the triangle plane. Typically try 1.e - 3 for *iso tolerance*.

#### 6.552 global node dof empty switch

If *switch* is set to **-yes**, the dof's in a node are set to zero when the node is only connected to empty elements. If *switch* is set to **-no**, the dof's in a node are not set to zero when the node is only connected to empty elements. Default *switch* is set to **-yes**.

#### 6.553 global element dof apply switch

If you set **switch** to **-yes**, then dof's like strains, stresses, etc. will be saved in the element integration points in the records **element\_intpnt\_dof**. So, not the dof's from the global nodes will be used, but each element uses its own values for these dof's in the integration points. This will only be done for dof's like strains, stresses, history variables, etc. Other principal dof's like velocities, temperature, etc. are used from the global nodes.

If you set **switch** to **-no**, then elements will actually use the averaged nodal results, and will not remember its own values.

Default, if global\_element\_dof\_apply is not specified, global\_element\_dof\_apply is set to -yes. See also global\_element\_dof\_from\_node\_dof.

Realise that even when the dofs like strains, stresses etc are remembered and stored in integration points, they still be be added to the global node dofs to facilitate, for example, plot programs using the nodal results for plotting. As a special option you can suppressing this adding to the global nodel dofs by using **-only** for the *switch*; then again the dofs like strains, stresses etc. are remembered in integration points, but now also these dofs are not added to the global node dofs.

#### 6.554 global element dof from node dof switch

If global\_element\_dof\_apply is set to -yes, and the element\_intpnt\_dof record does not exist, but node\_dof records exist in the input file, you can either require that the element\_intpnt\_dof records will be initialised from the node\_dof records, or will not be initialised from the node\_dof records. If you set *switch* to -yes the element\_intpnt\_dof records will be initialised from the node\_dof records. If you set *switch* to -no the element\_intpnt\_dof records will not be initialised from the node\_dof records. Default, if global\_element\_dof from\_node\_dof is not specified, *switch* is set to -no.

#### 6.555 global\_post\_point\_node\_type node\_type

With this record you can determine how records like **post\_point**, **control\_print\_dof\_point** and **control\_print\_dof\_line** are evaluated. If *node\_type* is set to **-node** the current nodal coordinates for elements are used to determine for which material point inside elements the dof's should be determined; if you do an updated lagrange calculation in which the coordinates of nodes change, so the **node** records change, you get dof results for the material at the current moment presented on the point or line. If *node\_type* is set to **-node** the initial start nodal coordinates for elements are used to determine for which material point inside elements for elements are used to determine for which material at the initial start nodal coordinates for elements are used to determine for which material point inside elements the dof's should be determined; thus you get dof results for the material at the initial start moment presented on the point or line.

Default, if this record is not set, *node\_type* is set to **-node\_start\_refined**.

#### 6.556 groundflow apply switch

If *switch* is set to **-no**, then the groundflow equation is skipped, and all groundflow data is ignored. This is done for all timesteps.

#### 6.557 groundflow consolidation apply switch

If *switch* is set to **-no**, then the material divergence part in the groundflow equation is skipped. This is done for all timesteps.

Default *switch* is **-no**.

#### 6.558 groundflow density $\rho$

Density of ground water.

#### 6.559 groundflow flux edge normal index flux

Distributed prescribed water flux normal to the edge of an element. This distributed flux is translated into equivalent nodal flux on the edges of elements. Also the record **groundflow\_flux\_edge\_normal\_geometry** should be specified, and optionally the record **groundflow flux edge normal time** can be specified.

Attention: this option is only available for linear and quadratic isoparametric elements.

<u>Attention</u>: if this option is used INSIDE a FE mesh, then the elements on each side will get the distributed flux. So the total water flux will normally become zero since the normals of the elements at the side of the geometry are opposite.

#### 6.560 groundflow\_flux\_edge\_normal\_element index element\_0 element\_1 ...

Restricts the elements to which the **groundflow\_flux\_edge\_normal** record with the same *index* should be applied.

### **6.561** groundflow\_flux\_edge\_normal\_element\_group index element\_group\_0 element\_group\_1 ...

Restricts the element groups to which the **groundflow\_flux\_edge\_normal** record with the same *index* should be applied.

#### **6.562** groundflow\_flux\_edge\_normal\_element\_node index element node\_0 node\_1 ...

Selects the element and local node numbers for which the **groundflow\_flux\_edge\_normal** record with the same *index* should be applied.

# 6.563 groundflow\_flux\_edge\_normal\_element\_node\_factor index $factor_0 \ factor_1 \ \dots$

Nodal multiplication factors with which the **groundflow\_flux\_edge\_normal** will be applied to the element of **groundflow\_flux\_edge\_normal\_element\_node**. You need

to specify a factor for each node on the side. Here  $factor_0$  is the multiplication factor for the first node on the side, etc.

#### **6.564** groundflow\_flux\_edge\_normal\_element\_side index element\_0 element\_1 ... side

Selects the elements and local side number for which the **groundflow\_flux\_edge\_normal** record with the same *index* should be applied.

#### 6.565 groundflow flux edge normal factor index $a_0 a_1 \dots a_n$

This data item defines a polynomial in space. This polynomial gives a factor which is used as a multiplication factor for **groundflow\_flux\_edge\_normal** records (with the same index). In this way, you can obtain coordinate dependent water fluxes.

In 1D the polynomial is  $a_0 + a_1x$  (specify 2 values). In 2D the polynomial is  $a_0 + a_1x + a_2y$  (specify 3 values). In 3D the polynomial is  $a_0 + a_1x + a_2y + a_3z$  (specify 4 values).

### **6.566 groundflow\_flux\_edge\_normal\_geometry** *index geometry\_entity\_name geometry\_entity\_index*

Selects the area for which the **groundflow\_flux\_edge\_normal** record with the same *index* should be applied. For example, **-geometry\_line 1** can be used in 2D, indicating that the nodes on line 1 get the distributed flux. The total edge of an element must be inside the geometry for the force to become active. For 2D elements the border lines are edges. For 3D elements the border surfaces are edges.

### 6.567 groundflow\_flux\_edge\_normal\_node index node\_0 node\_1 node\_2

Selects the nodes for which the **groundflow\_flux\_edge\_normal** record with the same *index* should be applied. The *node\_0* etc. specify global node numbers.

# **6.568** groundflow\_flux\_edge\_normal\_sine index start\_time end\_time freq\_0 amp\_0 freq\_1 amp\_1 ...

Similar to force edge sine, now for water flux however.

### 6.569 groundflow\_flux\_edge\_normal\_time index time load time load

This record specifies a diagram which contains the factors with which the **groundflow\_flux\_edge\_normal** record with the same index is applied. Linear interpolation is used to extend the *time load* values to the intervals between these pairs. Outside the specified time range a factor 0 is used.

If this record is not specified, the flux is applied at all times with a factor 1.

#### 6.570 groundflow nonsaturated apply index switch

If *switch* is set to **-no**, then nonsaturated groundflow data (eg van Genuchten) will not be applied; only saturated data will be used. This is done for all timesteps.

Default *switch* is **-yes**.

#### 6.571 groundflow phreatic bounda switch

If *method* is set to **-yes**, the phreatic level is used to automatically prescribe the hydraulic pressure head of nodes which are located on or above the phreatic level.

Default, if groundflow phreatic bounda is not specified, *method* is set to -yes,

#### 6.572 groundflow phreatic level ...

Groundwater level.

In a 1D calculation this record should be given x value of the groundwater level. The groundwater is below that x-value.

In a 2D calculation this record should be given sets of x - y which specify the y level of the groundwater at several x locations; In 2D you need to give the x - y sets as follows:

• specify x - y sets for increasing x

In 3D the phreatic line is specified as follows. Denote the lowest x with  $x_0$ , the next higher x with  $x_1$  etc. Denote the lowest y with  $y_0$ , the next higher y with  $y_1$  etc. Denote the phreatic level z value for x i y j with z ij. Then give the following:

- x\_0 y\_0 z\_00 x\_1 y\_0 z\_10 etc.
- $x_0 y_1 z_0 x_1 y_1 z_{11}$  etc.

• etc.

In 3d, the number of points in x and y direction respectively should be set with nx and ny of the **groundflow phreatic level n** record.

In nodes above the phreatic level the total pressure will be set to zero during the calculation.

As a special option in 2D and 3D, you can specify one value only, which sets a constant phreatic level of that value everywhere. In this special case, you do not need to specify **groundflow phreatic level n**.

#### 6.573 groundflow phreatic level n nx ny

See groundflow phreatic level.

#### 6.574 groundflow phreatic level static switch

If *switch* is set to **-yes**, total pressures (pore pressures) in nodes for which the **ground-flow phreatic level** holds, will be set equal to the static pressure.

This is convenient if the phreatic line is located above the mesh part to which it belongs; then nodes of the mesh will not get a boundary condition, so that the correct hydraulic pressure head can not be determined. And thus, it is necessary to determine the total pressure (pore pressure) from the difference of nodal coordinates and phreatic line height.

It is also convenient if you do not want to initialise and solve the hydraulic pressure heads with the groundflow storage equation. This saves computer memory and CPU time.

In the **group\_type** for elements which should get the static groundflow pressure you need to add **-groundflow**.

#### 6.575 groundflow phreatic level multiple index ...

The same as **groundflow\_phreatic\_level**, but now however several groundwater levels can be specified. For each **groundflow\_phreatic\_level\_multiple** you should specify a separate value for *index*.

This option typically can be used if you have in vertical direction non-permeable layers separating the total domain in independent parts with each its own groundwater level.

You can specify with one of groundflow\_phreatic\_level\_multiple\_element or groundflow\_phreatic\_level\_multiple\_element\_geometry or groundflow\_phreatic\_level\_multiple\_element or groundflow\_phreatic\_level\_multiple\_node the parts of the domain that belong to the groundwater level of **groundflow\_phreatic\_level\_multiple** with the same index. Only one of these record can be used, you cannot combine them.

With groundflow phreatic level multiple n you specify nx ny in 3D again.

In the **group\_type** for elements which should get the static groundflow pressure you need to add **-groundflow**.

#### 6.576 groundflow\_phreatic\_level\_multiple\_element index element\_0 element 1...

Element numbers for groundflow phreatic level multiple with the same index.

#### 6.577 groundflow\_phreatic\_level\_multiple\_element\_group index element group 0 element group 1...

Element group numbers for **groundflow** phreatic level multiple with the same index.

#### 6.578 groundflow\_phreatic\_level\_multiple\_element\_geometry index element\_geometry 0 element\_geometry 1 ...

 $Element \ geometry \ numbers \ for \ {\bf groundflow\_phreatic\_level\_multiple} \ with \ the \ same index.$ 

#### 6.579 groundflow phreatic level multiple n nx ny

See groundflow phreatic level multiple.

# 6.580 groundflow\_phreatic\_level\_multiple\_node index node\_0 node\_1

Node numbers for groundflow phreatic level multiple with the same index.

#### 6.581 groundflow phreatic level multiple static index switch

If *switch* is set to **-yes**, total pressures (pore pressures) in nodes for which the **ground-flow phreatic level multiple** holds, will be set equal to the static pressure.

This is convenient if the phreatic line is located above the mesh part to which it belongs; then nodes of the mesh will not get a boundary condition, so that the correct hydraulic pressure head can not be determined. And thus, it is necessary to determine the total pressure (pore pressure) from the difference of nodal coordinates and phreatic line height.

It is also convenient if you do not want to initialise and solve the hydraulic pressure heads with the groundflow storage equation. This saves computer memory and CPU time.

#### 6.582 groundflow phreatic only switch

If *switch* is set to **-yes** groundflow data is removed for groups which are not part of **ground-flow\_phreatic\_level\_multiple\_element\_group** records. Thus only groundflow data is retained for groups for which a multiple phreatic level is defined.

#### 6.583 groundflow phreatic project switch

If *switch* is set to **-yes**, the hydraulic pressure head which is imposed on nodes above the phreatic level uses the project coordinate on the phreatic level (smallest distance); thus not simply the distance in vertical direction. For some calculations that gives better groundwater velocities.

Default, it groundflow phreatic project is not specified, switch is set to -no.

#### 6.584 groundflow pressure factor factor

With this record you can specify a factor which is used to add the pore pressure to the effective stress to get the total stress: total stress = effective stress + factor \* pore pressure.

If you calculate with the van genuchten law, automatically the factor will be set to Se of that law.

#### 6.585 groundflow seepage eps eps

The eps specifies the tolerance if the groundflow seepage condition should be applied or not. If the inner product of the groundflow water flow direction with the normal outside the material is smaller then eps, the seepage status will be set to closed, and the total pressure condition will not be applied (so that the boundary is really closed for water flow). If not specified, eps is set to 0.1.

#### 6.586 groundflow\_seepage\_geometry index geometry\_item\_name geometry\_item\_index

This record specifies an edge of the groundflow domain for which the groundwater is only allowed to flow outwards of the domain; flow into the domain is not allowed on that edge. The geometrical entity should be specified such that the normal of the geometry points outwards the material (so outwards the groundflow domain). This option comes handy when the point of groundwater flow exit is not known in advance of the calculation; it will be a result of the calculation instead.

Example:

groundflow\_seepage\_geometry 10 -geometry\_line 100 ... bounda\_dof 20 -geometry\_line 100 -total\_pressure bounda\_time 20 0.0

In this example the total pressure (pore pressure) is set to 0 on the geometry line number 100, to account for free air at that edge. Since at that edge water cannot enter the domain the seepage option is applied to that edge. The result of these combined options is that on nodes with outward flow a total pressure 0 boundary condition is imposed, whereas on other nodes no boundary conditions is imposed (so that the flow is 0 at those nodes). The transition point between these outflow nodes and nodes with zero flow will be found automatically as a result of the calculation.

#### 6.587 groundflow seepage node index node\_0 node\_1 ...

This record does the same as the **groundflow\_seepage\_geometry** record, but now however you specify node numbers at which the seepage condition holds. The  $node_0$  is the first node number, the  $node_1$  is the second node number, etc.

#### 6.588 groundflow total pressure limit limit

With this record you can specify the maximum allowed total pressure value. Any higher value resulting from the groundflow equations will be cutoff to this value. Default the *limit* is set to 0.

#### 6.589 group axisymmetric index switch

If *switch* is set to **-yes**, the calculation becomes axi-symmetrical for the group *index*. Each specified x coordinate becomes a radius and y becomes the length (=vertical) direction. The z-direction is the axi-symmetric direction. Specify only non-negative x coordinates, i.e. define the computational domain in the right half-plane.

This option is onloy available for groups with isoparametric 1D elements (bar2, ...), or isoparametric 2D elements (tria3, quad4, ...), or for 2D interface elements (quad4 interface, ...).

#### 6.590 group\_beam\_force\_moment\_plasti index force\_x\_plasti\_first\_node force\_y\_plasti\_first\_node force\_z\_plasti\_first\_node moment\_x\_plasti\_first\_node moment\_y\_plasti\_first\_node moment\_z\_plasti\_first\_node force\_x\_plasti\_second\_node force\_y\_plasti\_second\_node force\_z\_plasti\_second\_node moment\_x\_plasti\_second\_m moment\_y\_plasti\_second\_node moment\_z\_plasti\_second\_node

With this record you can set ideally plastic limits on forces and moments in beam elements. You can specify different values for each of the local x, y and z-directions, so that a different plastic behavior can be specified for beams having different properties in the different directions. You can also different values for each of the two nodes.

#### 6.591 group beam inertia index Iyy Izz J

Bending and torsion properties for beam elements. Here Iyy is the area moment of inertia for bending along the local beam y axis, and Izz is the area moment of inertia for bending along the local beam z axis, and J is the polar moment of inertia for torsion along the local beam x axis.

See also **beam** rotation in the initialisation part.

The *index* specifies the element group, see **element** group.

6.592 group beam memory index memory\_type

Memory model for beam; either **-updated\_linear**, **-updated** or **-total\_linear**. The **-updated** model is a geometrically nonlinear model which takes large beam rotations into account. The *index* specifies the element group, see **element** group.

#### 6.593 group beam direction z index dir\_z, x dir\_z, y dir\_z, z

This record specifies the local beam z direction in global space. If **group\_beam\_direction\_z** is not specified in 2D then  $0 \ 0 \ 1$  will be used. If **group\_beam\_direction\_z** is not specified in 3D then a arbitrary direction perpendicular to the beam length axes will be used.

The local beam axes will be placed in the **element\_beam\_direction** record after the calculation.

The *index* specifies the element group, see **element** group.

See also group beam direction z reference point for automatic beam z-axis towards a reference point.

### **6.594** group\_beam\_direction\_z\_reference\_point index point\_x point\_y point\_z

This data record defines a reference point that allows you to influence the local beam zdirection. The local beam z-direction will be setup as follows:

- The length direction of the beam is determined, that is the local beam x-axis.
- A vector is taken from the beam middle point to the reference point.
- The part of this vector perpendicular to the length direction defines the local beam z-axis.

The above procedure ensures that the beam z-axis is perpendicular to the length direction, and that the z-axis points as much as possible to the reference point. As a typical example, you can use this option to take care that the local beam z-axis points to the middle of a tunnel, which is convenient if a tunnel lining with the local z-axis towards the tunnel middle; to do so specify the middle point of the tunnel axis as reference point  $point_x$  point\_y point\_z.

#### **6.595 group beam young** index E

Young's modulus for a beam (for bending moment calculation). The *index* specifies the element group, see **element** group.

#### **6.596** group beam shear index G

Shear modulus for a beam (for torsion moment calculation). The *index* specifies the element\_group, see **element** group.

#### 6.597 group condif absorption index a

Absorption coefficient. The *index* specifies the element\_group, see **element** group.

#### 6.598 group condif capacity index C

Heat capacity. The *index* specifies the element group, see **element** group.

#### 6.599 group condif conductivity index $k_x k_y k_z$

Heat conductivity in x, y and z direction respectively. As a special option you can also specify one value only, which then will be used in each direction. The *index* specifies the element\_group, see **element\_group**.

#### 6.600 group condif density index density

Density for convection-diffusion equation. The *index* specifies the element\_group, see **element\_group**.

#### **6.601** group condif flow index $beta_1 beta_2 beta_3$

Known flow field. In 1D only  $beta_1$  should be specified, etc. The *index* specifies the element\_group, see **element** group.

#### 6.602 group\_contact\_spring direction index $dirN_x dirN_y dirN_z$

Normal direction of a contact\_spring. The *index* specifies the element\_group, see **element** group.

As an alternative, you can specify **element\_contact\_spring\_direction** which allows for specification of the direction for each element separately.

As yet another alternative you can set *switch* in **group**\_**contact**\_**spring**\_**direction**\_**automatic** to **-yes**. Then the contact spring will automatically determine the directions.

#### 6.603 group contact spring direction automatic index switch

See group contact spring direction.

#### 6.604 group contact spring plasti cohesion index c

The normal contact force  $F_N$  is not allowed to become larger than cohesion c in tension (positive values of  $F_N$ ). If it would become larger, then the contact is broken, a gap is assumed and the contact force  $F_N$  is put to 0. To have really a positive  $F_N$  for extension of the contact spring, the order of the two nodes as specified in the **element** record for the contact spring should be correct.

Notice that when you use **control\_mesh\_generate\_contact\_spring** to obtain the contact spring elements, you are not sure what the first and what the second node of an element will be, and thus you should not use this **group\_contact\_spring\_cohesion** record. Otherwise, it is not important what you use as first and second node, so that **control mesh generate contact spring** can be used safely.

If this group\_contact\_spring\_plasti\_cohesion is not specified, infinite cohesion is assumed.

The *index* specifies the element\_group, see **element** group.

#### 6.605 group contact spring plasti friction index f

With this record you can specify a fixed friction coefficient for contact springs. If this record is not specified, a very large value for f will be applied.

The *index* specifies the element group, see **element** group.

See also group contact spring stiffness and group contact spring friction automatic.

#### 6.606 group contact spring plasti friction automatic index switch

If *switch* is set to **-yes**, the friction coefficient for contact springs will be determined from the plasticity law angle of neighboring elements. For a neighboring **group\_materi\_plasti\_mohr\_coul** the friction coefficient f will be set to  $f = (2./3.)\tan\phi$  with  $\phi$  the friction angle in the mohr-coulomb law of the neighboring elements. For a neighboring **group\_materi\_plasti\_diprisco** the friction coefficient f will be set to a value depending on the parameter  $\gamma$  of that law.

If no neighbor elements with appropriate material law are found, then f will be set to 0.2.

The *index* specifies the element group, see element group. See also group contact spring direction aut

#### 6.607 group\_contact\_spring\_direction\_automatic\_planes index switch\_x switch\_y switch\_z

With this option you can help the **group\_contact\_spring\_friction\_automatic** by telling in which planes the automatically determined spring direction is allowed to be. If a switch is set to **-yes**, then the direction may have a component in that plane. If a switch is set to **-no**, then the direction may not have a component in that plane. Default all switches are **-yes**.

The *index* specifies the element\_group, see **element** group.

#### 6.608 group contact spring memory index memory\_type

Memory model for contact\_spring; either -updated\_linear, -total\_linear. The *index* specifies the element\_group, see element group.

#### 6.609 group contact spring stiffness index $k_N k_T$

Stiffnesses for contact springs. The force  $F_N$  in normal direction of the contact spring is determined from  $F_N = k_N u_N$  where  $u_N$  is the normal displacement difference of the two nodes (that is, the displacement of the second node in normal direction minus the displacement of the first node in normal direction). The first tangential force  $F_{T1}$  of the contact spring is determined from  $F_{T1} = k_T u_{T1}$  where  $u_{T1}$  is the tangential displacement difference of the two nodes in the first tangential direction; the same is done for the second tangential force. The total tangential force  $\sqrt{F_{T1}^2 + F_{T2}^2}$  cannot exceed  $f F_N$  with f friction coefficient; then frictional slip occurs and the total tangential force is set to  $f F_N$  To model continuing stick between two bodies just put the friction coefficient f very high.

In 1D the parameters  $k_T$  and f will not be used (but should be specified as dummies nevertheless).

The *index* specifies the element\_group, see **element** group.

See also group contact spring friction and group contact spring friction automatic.

6.610 group dof initial index dof\_0 dof\_1 ...

Same as element dof initial, now specified for a group of elements however.

#### 6.611 group dof initial specific number index dof

Same as **element\_dof\_initial\_specific\_number**, now specified for a group of elements however.

**6.612** group\_dof\_initial\_specific\_value index value\_0 value\_grad\_x value\_grad\_y value\_grad\_z

Same as **element\_dof\_initial\_specific\_value**, now specified for a group of elements however.

#### 6.613 group\_groundflow capacity index C

Capacity in ground water flow equation. The *index* specifies the element\_group, see **element** group.

#### 6.614 group groundflow consolidation apply index switch

If *switch* is set to **-no** consolidation will not be applied for the elements of the group.

#### 6.615 group groundflow expansion index $\alpha$

Thermal expansion coefficient for ground water, for a combined groundwater with temperature analysis. The *index* specifies the element\_group, see **element** group.

#### 6.616 group groundflow nonsaturated eps permeability index eps

The nonsaturated law lowers the permability relative to the specified linear permeability as specified in **group\_groundflow\_permeability**. With this **group\_groundflow\_nonsaturated\_eps\_perme** you can specify the lowest allowed factor for reducing the permeability.

#### 6.617 group\_groundflow\_nonsaturated\_vangenuchten index $S_{residu}$ $S_{sat} g_a g_l g_n$

Parameters for non-saturated van Genuchten ground water flow, see the theory section. The *index* specifies the element\_group, see **element** group.

Since the van-Genuchten law is highly nonlinear, convergence of the calculation can be difficult. Always check if the calculation converges by printing **post\_node\_rhside\_ratio**. You can try including inertia to improve convergence.

#### 6.618 group groundflow permeability index $pe_x pe_y pe_z$

Permeability coefficient in ground water flow, in each space direction. In 1D you only should specify  $pe_x$ , etc. If you specify only value, then that will be used in each direction. The *index* specifies the element group, see **element** group.

#### 6.619 group\_groundflow\_permeability\_vertical\_stress index a b $\sigma_0$ minimum maximum

This is a special purpose equation for the permeability coefficient in ground water flow:

$$k^p = \frac{a}{\left(\sigma_v / \sigma_0\right)^b}$$

where a and b and  $\sigma_0$  are user specified parameters, and  $\sigma_v$  is the vertical effective stress. The permeability calculated this way is not allowed to become smaller than *minimum* and larger than *maximum*.

This option should only be used in 2D or 3D. This option should be used in combination with **group\_groundflow\_permeability**; the permeability as specified in **group\_groundflow\_permeability** will then be used in regions where the vertical stress is extremely small, to prevent division by zero; in all other regions the permability as specified by this **group\_groundflow\_permeability\_vertical\_stress** law will be used.

The *index* specifies the element\_group, see **element** group.

# 6.620 group\_groundflow\_total\_pressure\_tension index plastic\_tension\_minimum water\_height

Using this option you can control that the water pressure in an element is at least the value as determined from the specified *water\_height*. More precise, if the static water pore pressure as determined from the water density, the gravity and the *water\_height* exceeds the pore water pressure from the groundflow equation (in absolute terms), this static water pressure actually is used. This is only done if the largest eigenvalue of **ma**-teri\_strain\_plastic\_tension exceeds *plastic\_tension\_minimum*. To calculate the eigenvalues of materi\_strain\_plastic\_tension you need to include post\_calcul-materi\_strain\_plasti\_tension -prival in the input file.

This option comes handy to take care that in cracks in concrete actually the largest water pressure from an environment is used. It so ensures that a critical safety analysis for concrete cracking is obtained.

## 6.621 group integration method index method

Here *method* sets the integration method for bars, quad en hex elements. You can either set *method* to **-gauss**, **-lobatto** or **-reduced**.

The gauss and lobatto integration method are explained in text books on finite element methods. The reduced method is a weighted combination of lobatto and 1-point cell centered integration. The weighting is set by the **group\_integration\_method\_reduced\_factor** record.

If this record is not set, the default method as described in **group\_integration\_points** is chosen.

It is advised to keep the default method, so not specify this **group\_integration\_method** record, unless you know what you are doing.

#### 6.622 group integration method reduced factor index factor

When factor=1 the reduced method is in fact lobatto integration. When factor=infinite the reduced method is in fact 1-point cell centered integration. When the factor is in between 1 and infinity the intergation is done with a weighted combination of the lobatto and 1-point cell centered integration.

Default the *factor* is set to 10.

## 6.623 group integration points index type

Here *type* sets the number of integration points in an element. It should be set to **-normal**, **-minimal** or to **-maximal**.

For **-tria3** elements the integration point will be located in the middle with **-minimal** integration, or a four-point integration scheme will be used with **-maximal** integration.

For -tria6 elements a seven-point scheme will be used for -maximal and a four-point scheme will be used with -minimal integration.

For **-tet4** elements the integration point will be located in the middle with **-minimal** integration, or a five-point integration scheme will be used with **-maximal** integration.

For **-tet10** elements a five-point scheme will be used for **-minimal** and a ten-point scheme will be used with **-maximal** integration.

For other elements, if **-minimal** is used then the number of integration points in a direction is set equal to the number of nodes in the direction minus 1, and gauss integration is used. If for the other elements **-maximal** is used then the number of integration points in a direction is set equal to the number of nodes in the direction; gauss integration is used, but in case inertia is applied then lobatto integration will be used.

Default -minimal is used for -bar2, -tria3, and -tet4 elements; it is default -maximal otherwise.

If type is set to **-normal**, the default integration will be used.

The above is valid for normal isoparametric elements. For interface elements default lobatto integration is used (integration points in nodes).

It is advised to keep the default method, so not specify this **group\_integration\_points** record.

The *index* specifies the element\_group, see **element** group.

## 6.624 group interface index switch

With this record, you set that the element with element group *index* will act as an interface element by setting *switch* to **-yes**. This is available for **-quad4**, **-quad6**, **-hex8**, **-hex18**, **-prism6** and **-prism12**.

See group interface \* which data can be set for interfaces.

In interfaces strains are displacement differences between the opposite interface sides.

## 6.625 group interface condif conductivity index k

The 'index' specifies the group number. The conductivity k specifies the heat flow in interface thickness direction per unit temperature difference. Thus the conductivity is not the material conductivity but the conductivity of the layer simulated by the interface incorporating the thermal thickness of the interface, The 'conductivity' has units [power]/[temperature\*length] in 2D, and [power]/[temperature\*length\*length] in 3D.

## 6.626 group interface gap index gap

By specifying this record you can account for initial empty space between the sides of an interface element. Only when the sides displacements are such that the normal strain becomes lower then the specified gap value the interface will be closed and start to generate stresses.

So typically you set *gap* to a negative number if there is a physical gap.

If you want to allow always tension stresses in an interface set gap to, by example, 1.e20.

#### 6.627 group interface groundflow capacity index C

This record specifies the capacity for interface elements.

## 6.628 group interface groundflow permeability index pe

This record specifies the permeability per unit length in 2D or unit area in 3D for interface elements.

## 6.629 group interface materi elasti stiffness index kn kt, first kt, second

This record allows you to specify a normal stiffness and tangential shear stiffnesses for discrete interface elements with **-materi** in **group type**. Normal stresses in the interface element follow from normal strains multiplied with kn (stress,normal = kn \* strain,normal). Shear stresses in the interface element in the first tangential direction follow from shear strains in the first tangential direction multiplied with kt, first (stress,shear,first = kt,first \* shear,gamma,first = 2 \* kt,first \* strain,shear,first). Shear stresses in the interface element in the second tangential direction follow from shear strains in the second tangential direction follow from shear strains in the second tangential direction multiplied with kt,second (stress,shear,second = kt,second \* shear,gamma,second = 2 \* kt,second \* strain,shear,second). The kt,second should be specified for 3D interfaces only.

Too high values for interface stiffness will cause convergence problems in calculations. Thus, if you are running a calculation with interface elements and you are experiencing convergence problems please try lower values for the interface stiffnesses. Typically the normal interface stiffness can be chosen as 10 times the Young's modulus of the neigbouring isoparametric element divided by the length of that element in normal direction. Typically the tangential interface stiffness can be chosen as half of the normal interface stiffness.

Attention: an interface will only build up stresses if the interface normal strain is not larger then the gap specified in **group\_interface\_gap**. If the interface normal strain is larger then the gap , then the interface status is considered to be opened, and the interface does not have stresses.

A 3d example:

. . .

number\_of\_space\_dimensions 3

 $\begin{array}{l} group\_interface\_materi\_elasti\_stiffness \ 0 \ 0.10000e+11 \ 0.50000e+10 \\ 0.50000e+10 \end{array}$ 

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#### 6.630 group interface materi expansion normal index expansion\_coefficient\_normal

The 'index' specifies the group number. The *expansion\_coefficient\_normal* specifies the thermal strain expansion in interface thickness direction per unit temperature in the interface. The temperature is the average of the temperature of the both sides at the location of the integration point. This option is only available if group\_interface\_materi\_memory is set to **-total\_linear** or **-updated\_linear**. Furthermore, **materi\_strain\_elasti** should be initialised.

#### 6.631 group interface materi memory index memory\_type

Either *memory\_type* should be set to **-updated** linear or **-total** linear.

## 6.632 group\_interface\_materi\_plasti\_mohr\_coul\_direct index phi c\_phiflow

Mohr-coulomb plasticity model for interfaces. The angles are in radians. The cohesion c has stress unit (so just the same as for **group\_materi\_plasti\_mohr\_coul** in normal isoparameteric elements). The maximum friction force in the interface is  $c + F_n * tan(phi)$  where c is the cohesion, phi is the friction angle in radians and  $F_n$  is the normal force (which is a negative value under compression).

## 6.633 group interface materi plasti tension direct index tension\_limit

With this record you can specify the tension limit in interface elements. You may want to set **group interface gap** to a large value if you want a tensile stress for opening interfaces.

## 6.634 group interface materi residual stiffness index factor

With *factor* you can set the part of the original stiffness to be used as stiffness in opened interfaces. Default, if this record is not specified, the *factor* is set to some small value.

## 6.635 group\_interface\_groundflow\_total\_pressure\_tension index strain\_normal\_minimum water\_height

Using this option you can control that the water pressure in an interface element is at least the value as determined from the specified *water\_height*. More precise, if the static water pore pressure as determined from the water density, the gravity and the *water\_height* exceeds the pore water pressure from the groundflow equation (in absolute terms), this

static water pressure actually is used. This in only done if the interface normal strain (displacement different between interface sides) exceeds *strain normal minimum*.

This option comes handy to take care that in cracks in concrete actually the largest water pressure from an environment is used. It so ensures that a critical safety analysis for concrete cracking is obtained.

## **6.636** group\_interface\_tangential\_reference\_point index point\_x point\_y point\_z

This data record defines a reference point that allows you to influence the tangential directions in a 3D interface element. The tangential directions will be setup as follows:

- The normal direction to the interface plane is determined.
- A vector is taken from the integration point in the interface element to the reference point.
- The part of this vector perpendicular to the normal direction defines the first tangential direction.
- The outer product of the normal direction and the first tangential direction gives the second tangential direction.

The above procedure ensures that the tangential directions are perpendicular to the normal direction, and that the first tangential directions points as much as possible to the reference point. As a typical example, you can use this option to take care that the first tangential direction points to the middle of a tunnel, so the first tangential interface direction equals in fact the tunnel radial direction; to do so specify the middle point on the tunnel axis as reference point *x point x point z*.

If this **group\_interface\_tangential\_reference\_point** is not specified, it is only certain that the tangential directions are in plane of the interface (perpendicular to the normal direction), but are not defined otherwise.

See also element interface intput direction.

#### 6.637 group materi damage mazars index $epsilon_0a_t b_t a_c b_c \beta$

Parameters for the Mazars damage law. The *index* specifies the element\_group, see **element** group.

## 6.638 group materi damping index d

Material damping coefficient d. See also the dynamics section near the end of this manual.

The *index* specifies the element\_group, see **element** group.

## 6.639 group materi damping method index method

See group materi damping.

## 6.640 group materi density index density

Density for material flow equation. The *index* specifies the element group, see **element** group.

## 6.641 group materi density groundflow index density\_wet density\_dry

Density for material flow equation when a calculation is performed in combination with groundflow. If the element is filled with groundwater the *density\_wet* will be used and otherwise the *density\_dry* will be used. To determine if an element is filled with water, tochnog does not the following: If **post\_calcul -groundflow\_pressure -total\_pressure** is put in the input file then total pressures (pore pressures) are calculated. Then if the pore pressure in an element is negative the wet density is taken. Otherwise the dry density. If **post\_calcul -groundflow\_pressure** is NOT put in the input file the total pressure -total\_pressure is NOT put in the input file the total pressures are not calculated. Then tochnog looks if a phreatic level is given; if so, then if an element is below the phreatic level the wet density is used, otherwise if an element is above the phreatic level the dry density is used.

Here *density\_wet* is the amount of kg of soil + water in a unit volume. And *density\_dry* is the amount of kg of soil in a unit volume.

The *index* specifies the element\_group, see **element** group.

In case total pressures are calculated

In case total pressures are calculated from the **post\_calcul groundflow\_pressure** - **total\_pressure** command, the *density\_wet* will be used if the total pressure is smaller then 0, whereas *density\_dry* will be used if the total pressure is larger or equal to 0.

In case total pressures are not calculated but a phreatic level is specified

In case an element is above a specified phreatic level the  $density\_dry$  will be used. In case an element is below a specified phreatic level, the  $density\_wet$  will be used.

In other cases

In other cases *density* dry will be used.

## 6.642 group materi elasti borja tamagnini index $G_0 \alpha k p_r$

Elastic data for the modified Borja Tamagnini model, see [1]. The *index* specifies the element group, see **element** group.

## 6.643 group materi elasti c index 81 values

With this record you can directly specify the 81 values of the linear material stiffness  $C_{ijkl}$  which will be used to calculate stresses from strains with  $\sigma_{ij} = C_{ijkl}\epsilon_{kl}$ . Here  $\sigma_{ij}$  is the stress matrix and  $\epsilon_{kl}$  is the strain matrix.

The sequence of strains and stresses is xx, xy, xz, yx, yy, yz, zx, zy and zz.

The 81 values should be specified row by row, where each row contains 9 values. See also **group materi elasti c direction**.

## 6.644 group\_materi\_elasti\_c\_direction index dir\_0 dir\_1 dir\_2

This record specifies local axes for which the **group\_materi\_elasti\_c** is specified. In total 9 values need to be specified, first the 3 values for dir\_0, then the 3 values for dir\_1 and then the 3 values for dir\_2. Default, if this record is not specified, the global axes will be used.

## 6.645 group materi elasti camclay g index G

Elastic data G for the modified CamClay model. The *index* specifies the element\_group, see **element** group.

## 6.646 group materi elasti camclay poisson index $\nu$

Elastic data  $\nu$  for the modified CamClay model. This option is alternative to the **group\_materi\_elasti\_camclay** option (so, only one of both can be defined). With this option the poisson ratio  $\nu$  is assumed constant, and is used as follows:

$$G = \frac{3}{2}K(1-2\nu)/(1+\nu)$$

The *index* specifies the element\_group, see **element** group.

#### 6.647 group materi elasti camclay pressure min index pressure\_min

This specifies a minimal allowed value for the pressure in the calculation of the bulk modulus for the camclay model. In the calculation pressures below  $pressure\_min$  will be set to pressure min. This prevents numerical problems for very low bulk modulus K values.

The *index* specifies the element group, see **element** group.

## 6.648 group materi elasti compressibility index co

Compressibility for materials. A positive value should be used. The *index* specifies the element group, see **element** group.

## 6.649 group materi elasti hardsoil index $E_{50}^{ref}$ sigma<sub>50</sub><sup>ref</sup> $\nu_{50}$ m $E_{ur}^{ref}$ sigma<sub>ur</sub><sup>ref</sup> $\nu_{ur}$

Elasticity data for Hardening Soil model. The *index* specifies the element\_group, see **element** group.

## 6.650 group materi elasti k0 index K0

Elastic data K0. When this data is specified, and also **control\_materi\_elasti\_k0** is set to **-yes**, then the K0 parameter will be used in the elastic stress law with **group\_materi\_elasti\_young** or **group\_materi\_elasti\_young\_power** and **group\_materi\_elasti\_poisson**, or with **group\_materi\_elasti\_hardsoil**. In fact it will be used to determine the poisson coefficient consistent with the K0; this poisson coefficient is used in the elastic stress law.

This group \_materi\_elasti\_k0 in combination with control \_materi\_elasti\_k0 is a convenient method to get 'K0 stresses' when imposing gravity in a geotechnics calculation. After gravity is imposed simply do not set the control \_materi\_elasti\_k0 anymore, so that the normal group materi elasti poisson will be used in the remaining steps.

For K0>0.95 Tochnog will take 0.95. K0 exceeding 1 (or 0.95) may lead to ill-conditioned calculations.

### 6.651 group materi elasti lade index $B R \lambda$

Elastic data B - 0, R,  $\lambda$  for the Lade model. The *index* specifies the element\_group, see **element\_group**.

#### 6.652 group materi elasti poisson index poisson

Poisson ratio for solid. The *index* specifies the element group, see **element** group.

#### 6.653 group materi elasti poisson power $index \nu_0 \nu_1 \nu_2 p_1 \alpha$

Power law poisson ratio for solid material. See the theory part. The *index* specifies the element\_group, see **element\_group**.

## 6.654 group materi elasti shear factor index factor

Specifying this record causes the shear stiffness following from a specified young and poisson to be multiplied with *factor*. This provides a convenient way to test in a calculation what the effect of low shear stresses is. The *index* specifies the element group, see **element** group.

#### 

This record allows you to model a different soil stiffness when first loading or unloading/reloading instead. The **materi\_stress\_pressure\_history** should be initialised, which records the maximum soil pressure that occurred in history. If the current pressure is smaller then the largest pressure in history, the material is unloading or reloading, and the stiffness will be multiplied with *factor*. If the current pressure is the larger then the largest pressure from history, then this current pressure becomes the maximum history pressure, and the stiffness will not be multiplied with *factor*.

This group <u>materi</u> <u>elasti</u> <u>stress</u> <u>pressure</u> <u>history</u> <u>factor</u> can be combined with the young as specified by group <u>materi</u> <u>elasti</u> <u>young</u> or the young calculated from group <u>materi</u> <u>elasti</u> <u>young</u> <u>power</u>.

Typically the factor has a value of about 3.

## 6.656 group materi elasti transverse isotropy index $E_1$ $E_2$ $\nu_1$ $\nu_2$ $G_2$ dir x dir y dir z

Specifies the unique direction  $(dir_x dir_y dir_z)$  and elastic moduli in the transverse isotropic model. Here  $E_2$  is the young modulus in the unique direction,  $E_1$  is the young modulus in both directions perpendicular to the unique direction, etc. The *index* specifies the element group, see **element** group.

#### 6.657 group materi elasti volumetric poisson index $\nu$

 $See \ {\bf group\_materi\_elasti\_volumetric\_young\_values}$ 

## **6.658** group materi elasti volumetric young order index n

See group materi elasti volumetric young values

## 6.659 group materi elasti volumetric young values index epsilon\_0 sigma\_0 epsil

This is a special record to model the volumetric stress part of a nonlinear material, given the experimental results of a volumetric compression test (compression in one direction, fixed size in other two directions).

The table  $epsilon_0 sigma_0 epsilon_1 sigma_1 \dots$  specifies the strain-stress results for the volumetric compression test. Together with the poisson ratio as specified in **group\_materi\_elasti\_volumetric** an isotropic law in a nonlinear Young's modulus and a constant poisson ratio is fitted to this experiment. The Young modulus in fact is taken as the polynomial expansion  $E_0 + E_1 epsilon + E_2 epsilon^2 + \ldots + E_{n-1} epsilon^{n-1}$  where *n* denotes the order of the polynomial expansion (as given in **group materi elasti volumetric young order**).

The poisson ratio should be taken very high, say 0.4999999 or so, to ensure that the resulting law only models volumetric stresses. Then afterwards a normal young-poisson isotropic law (group\_materi\_elasti\_young and group\_materi\_elasti\_poisson) can be added to get an extra deviatoric part.

#### 6.660 group materi elasti young index E

Young's modulus for solid material. The *index* specifies the element \_ group, see **element** \_ group.

## 6.661 group materi elasti young polynomial index $E_0 E_1 \dots$

Polynomial parameters for strain dependent Young's modulus for solid material. See the theory part. The *index* specifies the element \_group, see **element** \_group.

## **6.662** group materi elasti young power $index E_0E_1E_2E_3p_1\alpha$

Power law Young's modulus for solid material. See the theory part. The *index* specifies the element\_group, see **element\_group**.

If you want to get the calculated young as output, initialise with **materi\_history\_variable** 1; the history variable will be filled with the calculated young, and can be plotted for example in GID.

## 6.663 group materi elasti young user index switch

If *switch* is set to **-yes** the user supplied routine **user\_young** will be called. There the youngs modulus should be calculated from the solution fields and the stress history. Typically degradation of material stiffness for cyclic loading can be programmed with this user specified routine.

You can plot in gid the values for the young as follows:

print\_group\_data -group\_materi\_elasti\_young

## 6.664 group materi expansion linear index $\alpha$

Linear expansion coefficient. The *index* specifies the element group, see **element** group.

## 6.665 group materi expansion volume index $\beta$

Volume expansion coefficient. The *index* specifies the element group, see **element** group.

## 6.666 group materi factor index factor

This factor comes convenient if your material stress law is specified in other units then you actually want in your calculation. Then you can specify *factor* to take care that your material stresses become consistent with the remaining part of the input file. For example, if you want your input file to work with kPa but your material stress law works with MPa then simply set *factor* to 1000.

### 6.667 group materi failure crunching index threshold delete\_time

If the compression strain in an element exceeds *threshold*, the element is considered to be fail. The element will be slowly deleted. It is totally deleted if the *delete time* has passed.

The *index* specifies the element group, see **element** group.

#### 6.668 group materi failure damage index threshold delete\_time

If the damage in an element exceeds *threshold*, the element is considered to be fail. The element will be slowly deleted. It is totally deleted if the *delete time* has passed.

The *index* specifies the element group, see **element** group.

## 6.669 group materi failure plasti kappa index threshold delete\_time

If the plastic parameter kappa in an element exceeds *threshold*, the element is considered to be fail. The element will be slowly deleted. It is totally deleted if the *delete\_time* has passed.

The *index* specifies the element\_group, see **element** group.

## 6.670 group materi failure rupture index threshold delete\_time

If the tensile strain in an element exceeds *threshold*, the element is considered to be fail. The element will be slowly deleted. It is totally deleted if the *delete time* has passed.

The *index* specifies the element\_group, see **element** group.

## 6.671 group materi failure void fraction index threshold delete\_time

If the void fraction in an element exceeds *threshold*, the element is considered to be fail. The element will be slowly deleted. It is totally deleted if the *delete time* has passed.

The *index* specifies the element\_group, see **element\_group**.

## 6.672 group materi history variable user index switch

Set *switch* to **-yes** if you want to activate the user supplied routine for material history variables. The *index* specifies the element group, see **element** group.

## 6.673 group materi history variable user parameters index...

Specify parameters for the user supplied routine for material history variables. The *index* specifies the element \_group, see **element** \_group.

#### 6.674 group materi hyper besseling index $K_1K_2\alpha$

Parameters for Besseling Hyper elastic rubber model. The *index* specifies the element\_group, see **element\_group**.

## 6.675 group materi hyper blatz ko index $G\beta$

Parameters for Blatz-Ko model. The *index* specifies the element group, see **element** group.

## 6.676 group materi hyper mooney rivlin index $K_1K_2$

Parameters for Mooney-rivlin hyper elastic rubber model. The *index* specifies the element\_group, see **element** group.

## 6.677 group materi hyper neohookean index $K_1$

Parameter for Neo-Hookean hyper elastic rubber model. The *index* specifies the element\_group, see **element\_group**.

## 6.678 group materi hyper reduced polynomial index $K_1 K_2 \ldots$

Parameters for reduced polynomial hyper elastic rubber model. The *index* specifies the element group, see **element** group.

## 6.679 group materi hyper volumetric linear index K

Parameter for the linear volumetric hyperelasticity model. The *index* specifies the element\_group, see **element** group.

## 6.680 group materi hyper volumetric murnaghan index $K\beta$

Parameter for the murnaghan volumetric hyperelasticity model. The *index* specifies the element\_group, see **element\_group**.

#### 6.681 group materi hyper volumetric ogden index $K\beta$

Parameter for the ogden volumetric hyperelasticity model. The *index* specifies the element\_group, see **element group**.

**6.682** group materi hyper volumetric polynomial  $index K_0 K_1 \dots$ 

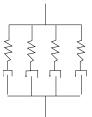
Parameters for the polynomial volumetric hyperelasticity model. The *index* specifies the element\_group, see **element\_group**.

## 6.683 group materi hyper volumetric simo taylor index K

Parameter for the simo-taylor volumetric hyperelasticity model. The *index* specifies the element\_group, see **element\_group**.

**6.684** group materi maxwell chain  $index E_0 t_0 \dots E_{n-1} t_{n-1}$ 

In total *n* parallel maxwell chains are defined with stiffness E = 0, relaxation time t = 0, etc...



The number *n* should equal **materi maxwell stress** in the input initialization part. The *index* specifies the element group, see **element group**.

## 6.685 group materi membrane index switch

If switch is set to -yes the zz stress becomes zero in 2D and the yy and zz stress become zero in 1D (in combination with axi-symmetry in 1D, only the yy stress becomes zero since zz is the axi-symmetric direction). So this option models plane stress conditions.

If group <u>materi</u> <u>membrane</u> is not used the plane strain conditions are used. Always the z-thickness is 1. in 3D, and the y, and z-thickness are 1. in 2D; see however also **volume factor**.

The group \_\_materi \_\_membrane option cannot be used in combination with group \_\_materi \_\_elasti \_\_compress group \_\_materi \_\_hyperelasticity and group \_\_materi \_\_viscosity.

The *index* specifies the element\_group, see **element** group.

## 6.686 group materi memory index memory type

Either *memory\_type* should be set to **-updated\_jaumann**, **-updated\_linear**, **-total** or **-total linear**. See the theoretical part for some explanation.

For an linear total Lagrange solid the input file may look like, and is <u>recommended</u> for most solid calculations:

```
materi_velocity
materi_displacement
materi_strain_total
materi_stress
end_initia
...
node 1 ...
node 2 ...
group_materi_memory 0 -total_linear
group_materi_elasti_young 0 ...
...
end_data
```

For a large deformation total Lagrange solid with a straightforward decomposition of the deformation tensor into a rotation tensor and a stretch tensor the input file may look like

```
materi_velocity
materi_displacement
materi_strain_total
materi_stress
end_initia
...
group_materi_memory 0 -total
group_materi_elasti_young 0 ...
...
end_data
```

For an updated Lagrange solid the input file may look like

• • •

```
materi_velocity
materi_velocity_integrated
materi_stress
end_initia
...
mesh -follow_material ...
node 1 ...
group_materi_memory 0 -updated
group_materi_elasti_young 0 ...
...
end data
```

Notice that for an updated Lagrange formulation you should always set that the mesh follows the material.

For a fluid the input file may look like

```
. . .
materi_velocity
materi_stress
end initia
. . .
(use Eulerian mesh)
mesh-fixed in space ...
timestep predict velocity-yes
. . .
node 1 ...
node 2 ...
. . .
group materi memory 0 -updated linear
group materi viscosity 0 ...
group materi elasti compressibility 0...
. . .
end data
```

The *index* specifies the element\_group, see **element** group.

## 6.687 group materi plasti bounda index index\_0 index\_1 ...

With this option, you can model reduction of friction of soil material and alike granular materials on walls. Set *index\_0*, *index\_1* etc. to the index of the **bounda\_dof** records

for which you want to use this reduction. We define an element to be on a wall when at least one of the velocities (displacements) of the elements is prescribed (via **bounda\_dof**). As a special option, you can use **-all** which indicates that the **bounda\_dof** records for all indeces will be used.

The reduction of friction is done for group\_materi\_plasti\_mohr\_coul, group\_materi\_plasti\_mohr\_coul\_direct, group\_materi\_plasti\_druck\_prag, group\_materi\_plasti\_hardsoil, if specified, by reducing the friction angle *phi* and dilatancy angle *phiflow* and cohesion *c* of the granular material with a factor (2./3.).

This is done for **group\_materi\_plasti\_camclay**, if specified, by reducing M with a factor (2./3.).

This is done for **group\_materi\_plasti\_hypo\_\***, if specified, by reducing deviatoric stress increments with a factor (2./3.).

The *index* specifies the element group of the granular material, see **element** group.

See also group materi plasti bounda factor and group materi plasti mohr coul direct wall.

## 6.688 group materi plasti bounda factor index factor

With this record you can specify a factor other then the default 2./3. used by the **group\_materi\_plasti\_bounda** record. You need to specify a factor for each of *index\_0*, *index\_1* etc. If you specify one factor only it will be used for all boundaries.

The *index* specifies the element group of the granular material, see **element** group.

## 6.689 group materi plasti coord limit index coord\_limit

To prevent plasticity problems near free surfaces, you can require that Tochnog neglects plasticity laws if the vertical coordinate exceeds *coord\_limit*. This option is not available for hypoplasticity laws, since for these laws nonlinear elasticity and plasticity are defined by one law, so the plasticity part cannot be suppressed by itself.

## 6.690 group materi plasti camclay index $M \kappa \lambda$

Plastic data M,  $\kappa$  and  $\lambda$  for the modified CamClay model. The *index* specifies the element\_group, see **element\_group**.

## 6.691 group materi plasti cap1 index $\phi \ c \ M \ \lambda^* \ \kappa^* \ K^{ref} \ p^{ref}m$

Plastic data for the cap1 plasticity model.

The *index* specifies the element\_group, see **element** group.

## 6.692 group materi plasti cap2 index $c \phi \alpha R epsilon_v^p p_b \dots$

Plastic data for the cap2 plasticity model. The  $epsilon_v^p p_b \dots$  represents a table with  $epsilon_v^p$  versus  $p_b$  values; at least two sets of values need to be specified.

The *index* specifies the element group, see **element** group.

## 6.693 group materi plasti compression index sigy

Yield data for compression plasticity. The *index* specifies the element\_group, see **element** group. Condition: materi strain plasti should be initialized.

## 6.694 group materi plasti compression direct index sigy

Compression limit. Principal stresses lower than *sigy* are not allowed and will be cut of by Tochnog. This model uses directly a cut-off of stresses, and does not use plastic strains. The *index* specifies the element group, see **element** group.

You can apply softening with a dependency diagram on materi strain total compression kappa.

#### 6.695 group materi plasti compression direct visco index tm

Characteristic relaxation time for visco plasticity with **group**\_**materi**\_**plasti**\_**compression**\_**direct**. Choose *tm* small if you want to have fast relaxation of the plasticity to the plastic state.

Thus the group <u>materi</u> <u>plasti</u> <u>compression</u> <u>direct</u> can only use this group <u>materi</u> <u>plasti</u> <u>compression</u> for visco-plasticity, and no other visco-plasticity model.

See also group materi plasti mohr coul direct visco.

6.696 group\_materi\_plasti\_diprisco index  $\gamma \ \hat{\beta}_f \ b_p \ c_p \ t_p \ \hat{\theta}_c \ \hat{\theta}_e \ \xi_c \ \xi_e \ \beta_f^0$ 

Yield data for di Prisco plasticity. The *index* specifies the element \_group, see **element \_group**. Condition: **materi \_strain \_plasti** and **materi \_plasti \_diprisco \_history 11** should be initialized.

## 6.697 group\_materi\_plasti\_diprisco\_density index $\gamma_l \ \hat{\beta}_{lf} \ b_{lp} \ c_{lp} \ t_{lp}$ $\hat{\theta}_{lc} \ \hat{\theta}_{le} \ \xi_{lc} \ \xi_{le} \ \beta_{lf^0} \ \gamma_d \ \hat{\beta}_{df} \ b_{dp} \ c_{dp} \ t_{dp} \ \hat{\theta}_{dc} \ \hat{\theta}_{de} \ \xi_{dc} \ \xi_{de} \ \beta_{df^0} \ e_l \ e_d$

Yield data for di Prisco plasticity with varying density. All data with an l in the subscript holds for loose soil, whereas all data with an d in the subscript holds for dense soil. The actually used data will be interpolated between the loose and dense data using the current density.

The *index* specifies the element \_group, see **element \_group**. Condition: **materi \_strain \_plasti** and **materi plasti diprisco history 12** should be initialized.

## 6.698 group materi plasti druck prag index phi c phiflow

Both yield data and flow data (indicated by the word flow) for Drucker-Prager plasticity. Choose *phi* and *phiflow* in between 0 and  $\frac{\pi}{2}$ . The *index* specifies the element\_group, see **element\_group**. Condition: **materi\_strain\_plasti** should be initialized.

## 6.699 group\_materi\_plasti\_element\_group index group\_0 group\_1

With this record you can model frictional slip of soil material and alike granular materials on other materials like concrete, steel, etc.

## $This is done for \ {\bf group\_materi\_plasti\_mohr\_coul}, \ {\bf group\_materi\_plasti\_druck\_prag}, \\ {\bf group\_materi\_plasti\_hardsoil}, \ if \ {\rm specified}, \\ \end{array}$

by reducing the friction angle phi and dilatancy angle phiflow and cohesion c of the granular material with a factor (2./3.).

This is done for **group\_materi\_plasti\_camclay**, if specified, by reducing M with a factor (2./3.).

This is done for **group\_materi\_plasti\_tension**, if specified, by reducing sigy with a factor (2./3.).

This is done for **group\_materi\_plasti\_hypo\_\***, if specified, by reducing the deviatoric stress increments with a factor (2./3.).

With  $group\_0$ ,  $group\_1$  etc. you can specify the groups of the concrete material, steel material etc. The reduction of the friction angle and dilatancy angle will only be applied to the granular elements (of *element\_group*) which are a direct neighbor of an element which has one of the groups  $group\_0$ ,  $group\_1$  etc.

Please realise that this method only works well if the finite elements are not too large.

The *index* specifies the element group of the granular material, see **element** group.

See also group materi plasti element group factor.

# **6.700** group\_materi\_plasti\_element\_group\_factor index factor\_0 factor\_1 ...

With this record you can specify a factor other then the default 2./3. used by the **group\_materi\_plasti\_element** record. You need to specify a factor for each group.

As a special option you can specify one value only, which will then be used for all factors.

The *index* specifies the element group of the granular material, see **element** group.

#### 

Yield data for the Generalised Non Associate Cam Clay for Bonded Soils plasticity model. The *index* specifies the element\_group, see **element** group.

## 6.702 group materi plasti gurson index sigy q1 q2 q3

Yield data (also used as flow data) for Gurson plasticity. The *index* specifies the element\_group, see **element\_group**.

## 6.703 group materi plasti hardsoil index $\phi \ c \ \psi \ R_f$

Plasticity data for Hardening Soil model. The *index* specifies the element\_group, see **element\_group**.

This model requires sufficient small timesteps; in case of trouble try smaller timesteps.

## 6.704 group materi plasti heat generation factor

This *factor* specifies how much of the plastic energy loss is transformed into heat (this only makes sense if **condif\_temperature** is initialized). The *factor* should be between 0 and 1. The *index* specifies the element group, see **element\_group**.

#### 6.705 group materi plasti hypo cohesion index c

Cohesion parameter in hypoplastic law. First, the cohesion in this record is subtracted from the normal stresses in the hypo law evaluation, so that the hypo law is evaluated at a lower pressure level. Second, if the pressure level then still exceeds the -3.\*c, the hypo law only will use its linear contribution. These actions increase numerical stability of the calculation in zones of small stresses (free surfaces).

The *index* specifies the element group, see **element** group.

## 6.706 group materi plasti hypo masin index $\varphi_c \lambda^* \kappa^* N r$

Masin hypoplasticity parameters; see the theory section. The angle  $\varphi_c$  should be specified in degrees. The  $\lambda^*$  should be bigger than the  $\kappa^*$ .

## 6.707 group\_materi\_plasti\_hypo\_masin\_clay index $\varphi_c \ \lambda^* \ \kappa^* \ N \ \nu_{pp}$

Masin clay hypoplasticity parameters. The angle  $\varphi_c$  should be specified in degrees. The  $\lambda^*$  should be bigger than the  $\kappa^*$ . Typical values are  $\varphi_c = 25$ ,  $\lambda^* = 0.1$ ,  $\kappa^* = 0.01$ , N = 1 and  $\nu_{pp} = 0.2$ .

# 6.708 group\_materi\_plasti\_hypo\_masin\_clay\_advanced\_parameters index $\alpha_G \alpha_f a_y o_c$

Optional advanced parameters  $\alpha_G \alpha_f a_y o_c$  for masin lay law. If not specified the defaults are:  $\alpha_G = 1$ ,  $\alpha_f$  from governing equation in [13],  $a_y = 0.3$  and  $o_c = 2$ .

# 6.709 group materi plasti hypo masin clay avanced direction $index \ dir_i$

Optional advanced integer parameter for Masin clay law. If not specified the default for the vertical direction  $dir_i$  is: 0 in 1D calculation, 1 in 2D calculation and 2 in 3D calculation.

## 6.710 group materi plasti hypo masin clay ocr index OCR

OCR in masin hypoplastic law; the initial void ratio will be calculated from this. A typical value is OCR = 1.5. You need to set **control\_materi\_plasti\_hypo\_masin\_clay\_ocr\_apply** to **-yes**.

## 6.711 group\_materi\_plasti\_hypo\_masin\_clay\_structure index k $A s_f$

Masin clay hypoplasticity structure parameters; see the theory section. The k should be at least 0. The A should be greater or equal to 0, and lower than 1. The  $s_f$  should be greater or equal to 1. Typical values are k = 0.4, A = 0.1 and  $s_f = 1.5$ .

## 6.712 group materi plasti hypo masin clay visco index $D_r I_v$

Masin clay hypoplasticity visco hypoplasticity parameters; see the theory section. Typical values are  $D_r = 1.e - 6$  and  $I_v = 0.1$ .

Remark: this law should not be combined with intergranular strains.

## 6.713 group materi plasti hypo masin ocr index OCR

OCR in masin hypoplastic law; the initial void ratio will be calculated from this. You need to set **control\_materi\_plasti\_hypo\_masin\_ocr\_apply** to **-yes**.

## 6.714 group materi plasti hypo masin structure index $k A s_f$

Masin hypoplasticity structure parameters; see the theory section. The k should be at least 0. The A should be greater or equal to 0, and lower than 1. The  $s_f$  should be greater or equal to 1.

## 6.715 group\_materi\_plasti\_hypo\_minimum\_void\_ratio index minimum\_void\_ratio

This option sets a minimum for the void ratio as calculated by hypoplasticity laws. If the void ratio would become lower than this *minimum\_void\_ratio* value it will be reset to this *minimum\_void\_ratio* value.

This option becomes handy when in difficult nonlinear calculations the void ratio may become negative in local zones, preventing the remainder of the calculation to continue. You should check the final results of the calculation if they are what you want.

The *index* specifies the element\_group, see **element** group.

## 6.716 group\_materi\_plasti\_hypo\_strain\_intergranular index $R m_R$ $m_T \beta_r \chi \theta$

Intergranular strain parameters in hypoplastic law. The *index* specifies the element \_group, see **element** \_group.

## 6.717 group materi plasti hypo strain intergranular masin clay index $R A_q n_q m_{rat} \beta_r \chi \theta$

Intergranular strain parameters for masin clay hypoplastic law. Typical values are R = 5.e-5,  $A_g = 270$ ,  $n_g = 1$ ,  $m_r at = 0.5$ ,  $\beta_r = 0.08$ ,  $\chi = 7$  and  $\theta = 7$ . The *index* specifies the element \_group, see **element \_group**.

## 6.718 group materi plasti hypo strain isa index $\chi_{max}$ $C_a$

ISA-Intergranular strain parameters in hypoplastic law; see the theory section. The *index* specifies the element\_group, see **element** group.

6.719 group materi plasti hypo wolffersdorff  $index \varphi h_s n e_{c0} e_{d0}$  $e_{i0} alpha beta$ 

Von-Wolffersdorff parameters in hypoplastic law; see the theory section. Here  $\varphi$  is in degrees. The *index* specifies the element group, see **element** group.

## 6.720 group\_materi\_plasti\_hypo\_wolffersdorff\_niemunis index $\nu$

Von-Wolffersdorff-Niemunis extended parameters in hypoplastic law; see the theory section. The *index* specifies the element\_group, see **element\_group**.

## 6.721 group\_materi\_plasti\_hypo\_niemunis\_visco index $\varphi$ nu $D_r$ $I_v \ e_{e0} \ p_{e0} \ lambda \ \beta_R \ kappa$

Parameters  $\varphi \nu D_r I_v e_{e0} p_{e0} \lambda \beta_R \kappa$  for the visco part of hypoplasticity; see the theory section.

The history variables are the same as for group materi plasti hypo wolffersdorff. You also need to specify control materi plasti hypo niemunis visco ocr apply.

The *index* specifies the element\_group, see **element** group.

#### 6.722 group materi plasti hypo niemunis visco ocr index OCR

OCR in visco hypoplastic law. The initial void ratio will be calculated from this; see the theory section.

In case you would like to have an OCR dependent on space coordinate you can use **dependency diagram** and **dependency item**.

The *index* specifies the element\_group, see **element** group.

#### 6.723 group materi plasti hypo void ratio linear index switch

Normally the void ratio change in hypoplasticity is calculated by using the trace of the deformation tensor. That in fact uses a linear approximation of the volume change. This linear approximation is convenient to compare results with analytical theories which are based on such linear approximation, and also to use parameters in hypoplastic laws which are calibrated with such approximation.

Optionally, using -no in this group \_materi \_plasti \_hypo \_void \_ratio \_linear record, the exact volume change of the material is calculated using the determinant of the deformation tensor, and used to calculate the exact new void ratio. However, hypoplasticity laws are usually fitted on the linear approximation of the volume change. So you should not use -no unless you are perfectly certain what you are doing.

#### 6.724 group materi plasti kinematic hardening index a

This record specifies the size of the rate of the kinematic hardening matrix  $\rho_{ij}$ . The *index* specifies the element\_group, see **element\_group**.

## 6.725 group materi plasti mohr coul index phi c phiflow

Both yield data and flow data (indicated by the word flow) for Mohr-Coulomb plasticity. Choose *phi* and *phiflow* in between 0 and  $\frac{\pi}{2}$ . The *index* specifies the element\_group, see **element\_group**.

It is advised to use **group\_materi\_plasti\_tension** or preferably with **group\_materi\_plasti\_tension\_direc** for tension cutoff of large tension stresses.

## 6.726 group materi plasti mohr coul direct index phi c phiflow

Both yield data and flow data (indicated by the word flow) for Mohr-Coulomb plasticity. Choose *phi* and *phiflow* in between 0 and  $\frac{\pi}{2}$ . The *index* specifies the element group, see

#### element group.

Principal stress differences higher than allowed by the mohr-coulomb criterium are not allowed and will be cut of by Tochnog. This model uses an alternative programming of the mohr-coulomb law, which tends to be very stable.

You must specify also group materi plasti tension direct.

You can apply softening with a **dependency** diagram on materi strain total shear kappa.

## 6.727 group materi plasti mohr coul direct normal index normal\_x normal\_y normal\_z

This option i.c.w. **group\_interface\_materi\_plasti\_mohr\_coul\_direct** ensures that the friction stress does not exceed the mohr coulomb shear stress limit on specifically the plane with normal vector normal x normal y normal z. In 1d only specify normal x, etc.

If this option is used i.c.w. group\_materi\_plasti\_element\_group or group\_materi\_plasti\_bounda it will only be actually used in elements with satisfy the condition listed in group\_materi\_plasti\_element\_gro or group\_materi\_plasti\_bounda.

To limit tension stress on specifically the plane with a normal vector use **group materi plasti tension direct normal**.

#### 

This option is similar to group materi plasti mohr coul direct normal, however now the normal direction is automatically taken from the element normal element normal. You need to set *switch* to **-yes**.

## 6.729 group materi plasti mohr\_coul\_direct\_visco index tm

Characteristic relaxation time for visco plasticity with **group**\_**materi**\_**plasti**\_**mohr**\_**coul**\_**direct**. Choose *tm* small if you want to have fast relaxation of the plasticity to the plastic state.

Thus the **group\_materi\_plasti\_mohr\_coul\_direct** can only use this **group\_materi\_plasti\_mohr\_coul** for visco-plasticity, and no other visco-plasticity model.

For group\_materi\_plasti\_...\_direct\_visco laws visco plasticity works as follows. Suppose the stress at time t is  $\sigma_t$ . Calculate the elastic response (non-plastic) at time t + dt as  $\sigma_{e,t+dt}$ . Calculate the plastic non-viscous response at time t + dt as  $\sigma_{p,t+dt}$ . Calculate the plastic viscous response at time t + dt as

 $\sigma_{vp,t+dt} = \sigma_{e,t+dt} + factor * (\sigma_{p,t+dt} - \sigma_{e,t+dt})$ 

$$factor = 1 - exp(-dt/tm)$$

Notice that for dt is very small factor = 0 so  $\sigma_{vp,t+dt} = \sigma_{e,t+dt}$ , thus purely elastic response (which is the stress at the previous time plus the elastic stress increment). Notice that for dt is very large factor = 1 so  $\sigma_{vp,t+dt} = \sigma_{p,t+dt}$ , thus purely plastic response (which is the stress completely mapped to the plastic surface). And for dt in between the response is in between the elastic response and purely plastic response.

## 6.730 group\_materi\_plasti\_mohr\_coul\_direct\_wall index phi c phiflow

As a special option for group \_materi \_plasti \_mohr \_coul \_direct you can use a record group \_materi \_plasti \_mohr \_coul \_direct \_wall index phi c phiflow. These values will be used when an element is attached to a wall. The input file needs to contain group \_materi \_plasti \_element \_group, group \_materi \_plasti \_bounda or group \_materi \_plasti \_mp to specify when an element is at the wall. Thus a typical input file looks like:

group\_materi\_plasti\_mohr\_coul\_direct 0 ... (values for elements not
attached to wall)
group\_materi\_plasti\_mohr\_coul\_direct\_wall 0 ... (values for elements attached to wall)
group\_materi\_plasti\_mpc 0 -yes (elements with mpc's are attached to
wall)
...
end data

See also group \_\_materi \_\_plasti \_\_element \_\_group, group \_\_materi \_\_plasti \_\_bounda and group \_\_materi \_ plasti \_\_mpc.

## 6.731 group\_materi\_plasti\_mohr\_coul\_hardening\_softening index phi\_0 c\_0 phiflow\_0 phi\_1 c\_1 phiflow\_1 kappashear\_crit

Both yield data and flow data (indicated by the word flow) for Mohr-Coulomb hardeningsoftening plasticity. See the theoretical part. Choose each of the angles  $phi_0 phiflow_0$  $phi_1 phiflow_1$  in between 0 and  $\frac{\pi}{2}$ . It is advised to use **group\_materi\_plasti\_tension** or preferably with **group\_materi\_plasti\_tension\_direct** for tension cutoff of large tension stresses. The *index* specifies the element group, see **element\_group**.

where

## 6.732 group\_materi\_plasti\_mohr\_coul\_direct\_wall index phi c phiflow

As a special option for group <u>materi</u> <u>plasti</u> <u>mohr</u> <u>coul</u> <u>direct</u> you can use a record group <u>materi</u> <u>plasti</u> <u>mohr</u> <u>coul</u> <u>direct</u> <u>wall</u> <u>index</u> <u>phi</u> <u>c</u> <u>phiflow</u>. These values will be used when an element is attached to a wall. The input file needs to contain group <u>materi</u> <u>plasti</u> <u>bounda</u> or group <u>materi</u> <u>plasti</u> <u>mpc</u> to specify when an element is at the wall. Thus a typical input file looks like:

...
group\_materi\_plasti\_mohr\_coul\_direct 0 ... (values for elements not
attached to wall)
group\_materi\_plasti\_mohr\_coul\_direct\_wall 0 ... (values for elements attached to wall)
group\_materi\_plasti\_mpc 0 -yes (elements with mpc's are attached to
wall)
...
end\_data

See also group materi plasti bounda and group materi plasti mpc.

## 6.733 group materi plasti mpc index switch

Same as group\_materi\_plasti\_bounda, but now for mpc\_... records however. If you set *switch* to -yes, the reduction factor will be applied if there is any mpc at the node of an element.

See also group materi plasti mpc factor.

### 6.734 group materi plasti mpc factor index factor

Same as group\_materi\_plasti\_bounda\_factor, but now for group\_materi\_plasti\_mpc however.

## 6.735 group materi plasti pressure limit index pressure\_limit

To prevent plasticity problems near free surfaces, you can require that Tochnog neglects plasticity laws if the pressure exceeds *pressure\_limit*. This option is not available for hypoplasticity laws, since for these laws nonlinear elasticity and plasticity are defined by one law, so the plasticity part cannot be suppressed by itself.

See also group materi plasti pressure limit method.

## 6.736 group materi plasti residual stiffness index factor

With *factor* you can set the part of the original shear stiffness stiffness to be used as stiffness in plastic elementss. Default, if this record is not specified, the *factor* is set to 1.

## 6.737 group materi plasti tension index sigy

Yield data for tension plasticity. The *index* specifies the element group, see **element group**. Condition: **materi strain plasti** should be initialized.

It is encouraged to use **group\_materi\_plasti\_tension\_direct** instead, which tends to give more stable calculations.

## 6.738 group materi plasti tension direct index sigy

Tension limit. Principal stresses higher than *sigy* are not allowed and will be cut of by Tochnog. This model uses directly a cut-off of stresses, and does not use plastic strains. The *index* specifies the element group, see **element** group.

If this option is not specified *sigy* is set to 0 if also the **group\_materi\_plasti\_mohr\_coul\_direct** is available for the group.

You can apply softening with a dependency diagram on materi strain total tension kappa.

## **6.739** group materi plasti tension direct normal index normal\_x normal\_y normal\_z

This option i.c.w. **group\_materi\_plasti\_tension\_direct** limits the tension stress specifically on specifically the plane specified by the normal *index normal\_x normal\_y normal\_z*. In 1D only *normal\_x* should be specified etc.

If this option is used i.c.w. group\_materi\_plasti\_element\_group or group\_materi\_plasti\_bounda it will only be actually used in elements with satisfy the condition listed in group\_materi\_plasti\_element\_gro or group\_materi\_plasti\_bounda.

#### 

This option is similar to group\_materi\_plasti\_tension\_direct\_normal, however now the normal direction is automatically taken from the element normal element\_normal. You need to set *switch* to **-yes**.

#### 6.741 group materi plasti tension direct visco index tm

Characteristic relaxation time for visco plasticity with **group**\_materi\_plasti\_tension\_direct. Choose *tm* small if you want to have fast relaxation of the plasticity to the plastic state.

Thus the **group \_\_materi \_\_plasti \_\_tension \_\_direct** can only use this **group \_\_materi \_\_plasti \_\_tension \_\_direct** for visco-plasticity, and no other visco-plasticity model.

See also group materi plasti mohr coul direct visco.

## 6.742 group materi plasti tension direct wall index sigy

As a special option for group \_\_materi \_\_plasti \_\_tension \_\_direct you can use a record group \_\_materi \_\_plasti \_\_tension \_\_direct \_\_wall index sigy. This value will be used when an element is attached to a wall. The input file needs to contain group \_\_materi \_\_plasti \_\_element \_\_group, group \_\_materi \_\_plasti \_\_bounda or group \_\_materi \_\_plasti \_\_mpc to specify when an element is at the wall.

## 6.743 group materi plasti user index switch

If *switch* is set to **-yes** the user supplied routine for plasticity is called.

See also the file **user.cpp** in the distribution.

The *index* specifies the element\_group, see **element** group.

## 6.744 group materi plasti visco exponential index $\gamma \alpha$

This record specifies visco-plasticity data for the exponential model. It should be used in combination with a plasticity model.

### 6.745 group materi plasti visco exponential limit index limit

This record defines the limit for the exponential viscoplasticity argument alphaf. If the argument alphaf becomes larger than limit then actually limit will be used instead as argument for the exponent.

Default, if group\_materi\_plasti\_visco\_exponential\_limit is not specified, then 3 will be used as limit.

This record specifies visco-plasticity data for the exponential model. It should be used in combination with a plasticity model.

**6.746** group\_materi\_plasti\_visco\_exponential\_name index name\_0 name\_1 ...

Same as group\_materi\_plasti\_visco\_power\_names, now for the exponential law however.

## 6.747 group\_materi\_plasti\_visco\_exponential\_values index $\gamma_0 \alpha_0$ $\gamma_1 \alpha_1 \dots$

See group materi plasti visco exponential name.

## 6.748 group materi plasti visco power $index \eta p$

This record specifies visco-plasticity data for the power model. It should be used in combination with a plasticity model.

The *index* specifies the element\_group, see **element** group.

## 6.749 group\_materi\_plasti\_visco\_power\_name index name\_0 name\_1

This group\_materi\_plasti\_visco\_power\_name together with group\_materi\_plasti\_visco\_power\_value allows you to specify different viscoelastic parameters for each of the plasticity models.

Set each of the names  $name_0$ ,  $name_1$ , etc. to the plasticity models that you use (eg -group\_materi\_plasti\_mohr\_coul etc.) Set the visco parameters for  $name_0$  in  $\eta_0$  and  $p_0$ , set the visco parameters for  $name_1$  in  $\eta_1$  and  $p_1$ , etc.

In case a plasticity model is used, but is not present in the names  $name_0$ ,  $name_1$ , etc. then that model will be evaluated elasto-plastic (and thus not elasto-viscoplastic).

The *index* specifies the element\_group, see **element** group.

## 6.750 group\_materi\_plasti\_visco\_power\_value index $\eta_0 p_0 \eta_1 p_1$

See group materi plasti visco power name.

#### 6.751 group materi plasti vonmises index $sigma_{y0}$

Yield data for Von-Mises plasticity.

The *index* specifies the element\_group, see **element\_group**. Condition: **materi\_strain\_plasti** should be initialized.

## 6.752 group materi plasti vonmises nadai index $C \kappa_0 n$

Data for Von-Mises Nadai hardening. The  $sigma_{y0}$  of the **group\_materi\_plasti\_vonmises** record is taken as  $sigma_{y0}$  in the nadai law.

The *index* specifies the element\_group, see **element\_group**. Condition: **materi\_plasti\_kappa** should be initialized.

## 6.753 group materi stokes index switch

If *switch* is set to **-yes**, then stokes flow is used. If *switch* is set to **-no**, then stokes flow is not used. Default it is set to **-yes**. The *index* specifies the element \_group, see **element** \_group.

## 6.754 group\_materi\_stress null index dir ...

With this option you can set the stress in the material in direction *dir* to zero. For *dir* you can use each of the local directions -**x**, -**y**, -**z** or the local shear directions -**xy**, -**xz**, -**yz**. You can specify multiple directions, for example **group\_materi\_stress\_null index -y** -**z**.

With group <u>materi</u> <u>stress</u> <u>null</u> <u>direction</u> you can specify local frame directions (nine values) -x, -y, -z</u>. Take care that these directions are perpendicular and of unit length.

If group\_materi\_stress\_null\_direction\_automatic is specified, the local frame -x, -y, -z is made out of the element normal direction element\_normal and two directions perpendicular to it.

If group \_\_materi \_\_stress \_\_null \_\_direction and group \_\_materi \_\_stress \_\_null \_\_direction \_\_automatic are not specified, the global frame is used as local frame.

## 6.755 group\_materi\_stress\_null\_direction index dir0\_x dir0\_y dir0\_z dir1\_x dir1\_y dir1\_z dir2\_x dir2\_y dir2\_z

See group materi stress null.

#### 6.756 group materi stress null direction automatic switch

#### See group materi stress null.

#### 6.757 group materi umat index switch

If switch is set to **-yes** then the user supplied umat routine is called for the element group *index*.

See also the section about user supplied routines at the end of this manual.

**6.758** group\_materi\_umat\_parameters index parameter\_0 parameter 1 ...

User supplied parameters for group materi umat.

## 6.759 group materi umat pardiso decompose index switch

If *switch* is set to **-yes** and an umat routine is present, Tochnog will ask the pardiso solver to decompose the system matrix each and every iteration of each and every timestep. If *switch* is set to **-no** and an umat routine is present, Tochnog will ask the pardiso solver to decompose the system matrix only once (please realise, however, that because of other input file options the decomposition possibly can be done more than once). Default, if *switch* is not defined, it is set to **-yes**.

## 6.760 group materi undrained capacity index C

Capacity for undrained analysis. See the theory section for details on undrained analyses.

## 6.761 group materi viscosity index $\nu$

Dynamic viscosity for nearly incompressible Newtonian flow. The *index* specifies the element group, see **element group**.

## 6.762 group materi viscosity heatgeneration index switch

If *switch* is set to **-yes**, then viscous dissipation will be used as a heat generation source. See also the theoretical part at the start of this manual. The *index* specifies the element\_group, see **element\_group**.

#### 6.763 group plasti apply index switch

If *switch* is set to **-no** any plasticity data in the group *index* will be neglected. Default, if **group plasti apply** is not specified, *switch* is set to **-yes**.

## 6.764 group porosity index n

Porosity in material. For example needed for group groundflow nonsaturated vangenuchten. The *index* specifies the element group, see element group.

## 6.765 group spring direction index $dir_x dir_y dir_z$

Direction of a spring. If for a -spring2 this record is not specified, the direction is taken to be from the first node of the spring to the second node. The *index* specifies the element\_group, see **element\_group**.

## 6.766 group spring memory index memory\_type

Memory model for spring; either -**updated\_linear**, -**total\_linear** or -**updated**. The -**updated** model is a geometrically nonlinear model which takes large spring rotations into account fro two-noded springs. The *index* specifies the element group, see **element group**.

## 6.767 group spring plasti index $F_y$

Maximum force in a spring. The *index* specifies the element group, see **element** group.

## 6.768 group spring stiffness index k

Stiffness of a spring. It is multiplied with the elongation of the spring to calculate the spring force. The *index* specifies the element\_group, see **element** group.

## **6.769** group spring stiffness nonlinear index $epsilon_0 k_0 epsilon_1 k_1$

Diagram with spring stiffness dependent on total spring strain (= total spring elongation). Here  $epsilon_0 k_0$  is the first point in the diagram, with  $epsilon_0$  the total spring strain and  $k_0$  the spring stiffness. Likewise for the next points in the diagram. Take care that you specify diagram values with a strain range that includes all spring strain that actually occur in the calculation.

The *index* specifies the element\_group, see **element** group.

## 6.770 group time index birth death

With this option you can set the time of birth of the elements (in group *index*) and the time of death of the elements.

Out of the range *birth* - *death* the elements of the group will not be used in the calculation (the starting *birth* limit itself is not included in the range, whereas the ending *death* limit itself is included).

#### 6.771 group time fill index birth\_empty birth\_filled death

With this option you can set the time of birth of the elements (in group *index*) and the time of death of the elements.

Out of the range *birth\_empty* - *death* the elements of the group will not be used in the calculation (the starting *birth\_empty* limit itself is not included in the range, whereas the ending *death* limit itself is included).

Between *birth\_empty* and *birth\_filled* the elements will be 'slowly filled with material'. This means that the density of the element and the total pressure (pore pressure), in case groundflow is present, will be scaled with a factor 0 at time *birth\_empty* up to a factor 1 at time *birth\_filled*. To prevent numerical problems at low gravity, any plasticity data will be ignored when an element is being filled; after the element is completely filled plasticity will become active (plasticity data will be applied).

## 6.772 group truss area index A

Cross-sectional area for a truss. The *index* specifies the element\_group, see **element** group.

## 6.773 group truss density index $\rho$

Density for a truss. The *index* specifies the element\_group, see **element** group.

# **6.774** group\_truss\_elasti\_elongation\_force\_diagram index l\_0 F\_0 l\_1 F\_1 ...

With this record you can specify a force versus elongation diagram for a truss. Here each  $l_i$  is the ratio of the truss elongation divided by the initial truss length. And each  $F_i$  is the corresponding force. This group truss elasti elongation force diagram cannot be combined with group truss elasti young.

## 6.775 group truss elasti young index E

Young's modulus for a truss. The truss force F is  $F = EA\Delta u$ , where  $\Delta u$  is the elongation of the truss. The *index* specifies the element group, see **element** group.

See also group truss area.

## 6.776 group truss expansion index alpha

Thermal expansion coefficient for trusses. A temperature increment dT leads to a thermal incremental length of the size alpha \* dT \* initial length;

## 6.777 group truss initial force index initial\_force

Initial truss force in truss elements.

#### 6.778 group truss memory index memory\_type

Memory model for truss; either **-updated\_linear**, **-updated** or **total\_linear**. The **-updated** model is a geometrically nonlinear model which takes large truss rotations into account. The *index* specifies the element group, see **element** group.

## 6.779 group truss rope index switch

The truss will act as a rope if *switch* is set to **-yes**. This means that negative forces will not be allowed (the force remains zero in compression). The *index* specifies the element\_group, see **element\_group**.

#### 6.780 group truss plasti index $sigma_c sigma_t$

Compressive and tension yield stress for truss. The actual stress cannot become lower than the  $sigma_c$  in compression, and the actual stress cannot become higher than the  $sigma_t$  in tension. The *index* specifies the element group, see **element** group.

6.781 group type index type\_name\_0 type\_name\_1 ...

With this record a differential equation is specified for the element group *index*. Allowed type names are **-condif**, **-groundflow**, **-materi**, **-wave**, **-spring**, **-contact\_spring**, **-truss**, **-beam**, **-truss\_beam**. Also **-empty** is allowed; it indicates that the element is empty.

For the **-truss \_beam** type you need to set parameters with **group \_truss \_\*** and **group \_beam \_\*** records. For the **-truss** type you need to set parameters with **group \_truss \_\*** records. For the **-beam** type you need to set parameters with **group \_beam \_\*** records. For the **-condif** type you need to set parameters with **group \_condif \_\*** records. For the **-materi** type you need to set parameters with **group \_materi \_\*** records. Etc etc.

See also **element** group.

## 6.782 group volume factor index factor

In 1D or 2D you can specify the cross-section and thickness respectively, for elements of the element group *index* (see **element group**).

See also volume factor.

## 6.783 group wave speed of sound index c

Speed of sound in wave equation. The *index* specifies the element\_group, see **element\_group**.

#### 6.784 icontrol icontrol

With this record you can set the control index which already have been performed. Thus if you set it to 10, all **control\_\*** records up to and including those with index 10 will be skipped, and the control indices starting from 11 will be performed.

#### 6.785 incremental driver ...

All incremental driver options are given in the example input file below:

```
echo -no
number_of_space_dimensions 2
materi_velocity
materi_velocity_integrated
materi_strain_total
materi_strain_plasti
materi_strain_intergranular
materi_strain_isa_c
materi_strain_isa_eacc
materi_plasti_hypo_history 4
materi_stress
groundflow_pressure
end_initia
( RUNNING:
  Run this file with the normal tochnog executable.
  Results are written in incremental_driver_result.txt.
  EQUILIBRIUM:
  At the start equilibrium is assumed (an external force is assumed which
  makes equilibrium with the internal stress and internal pore water
  pressure if specified).
  UNITS:
  The word force below means a force unit (eg kN).
  The word stress below means a stress unit (eg kPa).
  The incremental driver does not use specific units; all units can be
  used which are consistent. As an exception the hypoplastic masin law
  uses specific units (see the tochnog users manual).
  EXPERIMENTS:
  -INCREMENTAL_DRIVER_OEDOMETRIC
   'group_axisymmetric index -yes' is allowed; then x=r and y=theta.
   'incremental_driver_volume_constant index -yes' is not allowed
   x: zero displacement in x-direction
   y: zero displacement and strain in y-direction
   z: compression in z-direction
   user specified: displacement or force on top plane in z-direction,
                   the specified values should be negative for compression
  -INCREMENTAL_DRIVER_TRIAX
   'group_axisymmetric index -yes' is allowed; then x=r and y=theta.
   'incremental_driver_volume_constant index -yes' is allowed;
     then the volume is kept constant (undrained water pressure is not needed)
   x: fixed stress in x-direction
   y: if axisymmetric then y-strain comes from calculation,
      if not axisymmetric then constant stress in y direction
```

```
z: compression in z-direction
 user specified: displacement or force on top plane in z-direction,
                 the specified values should be negative for compression
-INCREMENTAL_DRIVER_DIRECT_SHEAR
 'group_axisymmetric index -yes' is not allowed.
 'incremental_driver_volume_constant index -yes' is allowed;
   then the volume is kept constant (undrained water pressure is not needed)
 x: zero strain in x-direction
 y: zero displacement and strain in y-direction
 z: shear of top z-plane in x-direction, fixed stress in z-direction
 user specified: shear displacement or shear force on top plane in x-direction
-INCREMENTAL_DRIVER_HYDROSTATIC
 'group_axisymmetric index -yes' is allowed.
 'incremental_driver_volume_constant index -yes' is not allowed
 equal compression in x- and y- and z-direction
 user specified: volumetric displacement
   ( (x- plus y- plus z-displacement ) / 3. ) or volumetric stress
 notice: not a force but instead a stress is specified.
COMBINED EXPERIMENTS:
Only combined experiments will be analysed, and only output will be printed for
these combined experiments; the experiments by themselves will not be analysed.
The start time of each experiment in a combined experiment should be equal to
the end time of a previous experiment in that combined experiment.
Only for the first experiment the variables incremental_driver_void_ratio,
incremental_driver_stress , incremental_driver_pressure and
incremental_driver_intergranular_strain can be specified;
for all subsequent experiments these values will be taken from the end values
of the previous experiment.
If no combined experiments are specified, automatically a combined experiment will
be generated for each experiment, containing that experiment only.
Notice that prescribed displacements and forces in an experiment are additional
to the last values of the previous experiment.
In an experiment all forces that are not prescribed and don't belong to a
prescribed displacement remain equal to those of the previous experiment.
UNDRAINED:
```

You can obtain undrained behavior in two different ways:

- 1. Use groundwater in the test specimen. The groundwater capacity physically causes the test specimen to behave undrained.
- Apply 'incremental\_driver\_volume\_constant index -yes'. This causes the deformations of the test specimen to be such that the volume remains constant.
   You should choose one of both methods. You should not combine both methods.

#### **GROUNDFLOW:**

In case of 'group\_type index -groundflow' groundwater is present in the test specimen.

```
Then all groundflow initialisation and data needs to be specified (search for
  groundflow in this file to see what is needed).
 MATERIAL LAWS:
 group_materi_plasti_mohr_coul_direct*
  group_materi_plasti_hypo*
  group_materi_umat*
 See the tochnog users manual.
 MATERIAL MEMORY:
  If you use -updated_linear then small deformation theory is used, and the area
  change of the surface where forces are applied is not taken into account.
  If you use -updated_area then small deformation theory is used, but the area
  change of the surface where forces are applied is taken into account.
  If you use -updated then large deformation theory is used, and the area change
 of the surface where forces are applied is taken into account.
 AXISYMMETRIC:
 Either all experiments are axisymmetric or all experiments are not axisymmetric.
 Do not mix axisymmetric and non-axisymmetric experiments in a combined experiment.
)
post_calcul -groundflow_pressure -total_pressure
group_type
                                       0 -materi
group_axisymmetric
                                       0 -no
group_materi_memory
                                       0 -updated_area
group_materi_plasti_hypo_cohesion
                                      0 1.
group_materi_plasti_hypo_wolffersdorff 0 30. 5800.e3 0.28 0.84 0.53 1.00 0.13 1.05
group_materi_plasti_hypo_strain_intergranular 0 1.e-4 5.0 2.0 0.50 6.0 6.0
group_materi_plasti_hypo_strain_isa
                                      0 20.0 0.017
group_type
                                     1 -materi -groundflow
                                     1 -no
group_axisymmetric
group_materi_memory
                                     1 -updated_area
```

group\_materi\_elasti\_young

1 6000.

group\_materi\_elasti\_poisson 1 0.4 group\_materi\_plasti\_mohr\_coul\_direct 1 0.4 1.e1 0.1 (gamma\_0 c\_0 gamma\_1 c\_1 ..., optional) group\_materi\_plasti\_mohr\_coul\_direct\_incremental\_driver\_c 1 0. 10. 0.01 9. 100. 0. (gamma\_0 phi\_0 gamma\_1 phi\_1 ..., optional) group\_materi\_plasti\_mohr\_coul\_direct\_incremental\_driver\_phi 1 0. 0.4 0.01 0.3 100. 0.1 (gamma\_0 psi\_0 gamma\_1 psi\_1 ..., optional) group\_materi\_plasti\_mohr\_coul\_direct\_incremental\_driver\_psi 1 0. 0.1 0.01 0.05 100. 0.02 group\_groundflow\_capacity 1 1.e-4 (etc) (number of timesteps to be used for each experiment, default 1000. In these steps the total time table as specified by incremental\_driver\_experiment\_time is performed) incremental\_ntime 1000 (set to experiment name) incremental\_driver 0 -incremental\_driver\_direct\_shear (set to element group used) incremental\_driver\_element\_group 0 0 (set to -yes for force controlled experiment, set to -no for displacement controlled) incremental\_driver\_experiment\_force\_controlled 0 -no (time table for additional displacement or additional force at top y-plane in x-direction relative to start time experiment) incremental\_driver\_experiment\_time 0 0.0 0.0 1.0 0.02 (initial stress: sigxx sigxy sigxz sigyy sigyz sigzz) incremental\_driver\_stress 0 -100 0 0 -100 0 -100 (initial pore pressure for undrained analysis for groundflow) incremental\_driver\_pressure 0 -10. (initial void\_ratio for hypo laws) incremental\_driver\_void\_ratio 0 0.75

```
(initial intergranular strain for hypo: epixx epixy epixz epiyy epiyz epizz)
incremental_driver_intergranular_strain 0 -0.0000577 0. 0. -0.0000577 0. -0.0000577
(initial isa intergranular back-strain for hypo: episa_cxx episa_cxy episa_cxz episa_cyy episa_
incremental_driver_isa_strain 0 -0.0000404145 0. 0. -0.0000404145 0. -0.0000404145
(initial length in x-direction)
incremental_driver_length_x
                                0 0.1
(initial length in y-direction, not needed in axisymmetric)
incremental_driver_length_y
                                0 0.1
(initial length in z-direction)
                               0 0.2
incremental_driver_length_z
(set to experiment name)
incremental_driver
                                1 -incremental_driver_direct_shear
(set to element group used)
incremental_driver_element_group 1 0
(set to -yes for force controlled experiment,
set to -no for displacement controlled)
incremental_driver_experiment_force_controlled 1 -no
(time table for additional displacement or additional force at top y-plane
in x-direction relative to start time experiment)
incremental_driver_experiment_time
                                   1 1.0 0.0 2.0 0.02
(initial length in x-direction)
incremental_driver_length_x
                               1 0.1
(initial length in y-direction, not needed in axisymmetric)
incremental_driver_length_y
                               1 0.1
(initial length in z-direction)
                             1 0.2
incremental_driver_length_z
(etc)
(a combined experiment which only contains experiment 0)
incremental_driver_combined 0 0
(a combined experiment which contains both experiment 0 and 1)
incremental_driver_combined 1 0 1
(etc)
```

end\_data

#### 6.786 inertia apply switch 0 switch 1 ...

If  $switch_0$  is set to -yes, the corresponding inertia term is included (material mass, heat capacity, ..). The same for the other switches. A switch should be specified for each of the principal dof's. See the 'input file - data part - introduction - types of dof's' section for an explanation about principal dof's. The sequence of the principal dof's is in the order as initialised in the **initia** ... **end initia** part.

As a special option you can specify only one switch, and then the specified value will automatically be used for all principal dof's.

This **inertia apply** is applied for all timestep records.

Default, if **inertia\_apply** is not specified, then each of *switch\_0*, *switch\_1* etc. is set to **-no**.

See also control inertia apply.

#### 6.787 input abaqus switch

Set *switch* to **-yes** for reading the abaqus input file **abaqus.inp**. Tochnog will use it to generate a tochnog input file **tochnog\_abaqus.dat**. This can typically be done by making an input file like:

echo -yes
number \_of \_space \_dimensions 3
materi \_velocity
materi \_stress
end \_initia
input \_abaqus -yes
input \_abaqus \_mesh -no
input \_abaqus \_continue -yes
...
include tochnog \_abaqus.dat
...
( other data , you can use the abaqus sets of tochnog \_abaqus.dat )
...
end \_data

You need to initialise the fields like materi velocity, materi stress, etc that you will

actually use later in the calculation. Only a limited set of data is transferred from the abaqus input file to the tochnog input file; you need to check if the Tochnog input file is like you want. Abaqus element sets and node sets are evaluated and can be used in the tochnog calculation.

ABAQUS is a registered trademark or trademark of Dassault Systemes. Abaqus input files can be generated by many programs, among which **mecway** is relatively affordable, see https://mecway.com/.

## 6.788 input abaques continue switch

If *switch* is set to **-yes** then after **tochnog\_abaqus.dat** is generated the remainder of the input file read and the calculation continues. If *switch* is set to **-no** then after **tochnog\_abaqus.dat** is generated the remainder of the input file will not be read and the calculation aborts. The **input\_abaqus\_continue** record should always be present as last record of the **input\_abaqus\_\*** records.

## 6.789 input abaqus group switch

If *switch* is set to **-yes** then also **group**<sup>\*</sup> is written to **tochnogabaqus.dat**. If *switch* is set to **-no** then no **group**<sup>\*</sup> is written to **tochnogabaqus.dat**. So you can set *switch* to **-no** in case you want to provide the **group**<sup>\*</sup> yourself, and don't want it to be taken from the **abaqus.inp**.

Default, if **input abaqus group** is not specified, the *switch* is set to **-yes**.

#### 6.790 input abaqus mesh switch

If *switch* is set to **-no** then only mesh data is written to **tochnog\_abaqus.dat**; so timestep information and post processing prints are not written. This record should be placed before the **input abaqus continue** record.

Default the switch is set to **-no**.

## 6.791 input abaques set set\_0 set\_1 ...

With this option you can specify for which set numbers the elements should be written. See the generated **tochnog abaqus.dat** for the set numbers.

#### 6.792 input abaques name name\_0 name\_1 ...

With this option you can specify which abaqus element types should be converted into tochnog elements. For example specify **-tria3** if you want to include **tria3** elements in the Tochnog input file. In case you do not specify **input\_abaqus\_name** all elements will be converted into tochnog elements. However, not all abaqus elements are available as tochnog element; if a non-available element is encountered it will be skipped.

#### 6.793 input feflow mesh switch

If switch is set to -yes the mesh is read from a FEFLOW file. FEFLOW is a dedicated groundwater analysis program, see https://www.mikepoweredbydhi.com/products/feflow. The Tochnog input file itself should not contain a mesh (elements and nodes). Presently, in a 3D analysis the thickness of each layer should be uniform over the nodes (no varying layer thickness in x- and y-direction). Default elements of the FEFLOW file get the element group 0 in the Tochnog calculation; however, for each set of elements specified inside the feflow file with  $K_x$  inside MAT I FLOW an increasing element group number is generated (1 for the elements specified in the first  $K_x$ , 2 for the elements specified in the second  $K_x$ , etc.).

## 6.794 input feflow fem switch

If the *switch* in **input\_feflow\_fem** is set to **-yes** the mesh is read from the file **feflow.fem**. If the *switch* in **input\_feflow\_fem** is set to **-no** the mesh is read from the file **feflow.dac**. Both files should be in ASCII text format. Default, if **input\_feflow\_fem** is not specified, the *switch* is set to **-no**.

See also input feflow mesh.

## 6.795 input feflow mesh hydraulic head switch

If the *switch* in **input\_feflow\_hydraulic\_head** is set to **-yes** then also the hydraulic head is read from the **feflow.dac** file. The **feflow.dac** should contain hydraulic pressure head results for all time points in the Tochnog calculation. The hydraulic pressure head as calculated by FEFLOW is then used as prescribed value in the Tochnog calculation, so that Tochnog only calculates stress (influenced however by the presence of the hydraulic pressure head). This option allows you to calculate hydraulic safety factors using FEFLOW hydraulic pressure heads and Tochnog stresses (see **safety\_piping** and **safety\_lifting** in **post\_calcul**).

See also input feflow mesh.

#### 6.796 input gmsh switch

Set *switch* to **-yes** for reading the **gmsh** mesh file **tochnog\_in.msh**. Only linear and quadratic elements are read.

The  ${\bf gmsh}$  program is a free external pre- and postprocessor. See http://www.geuz.org/gmsh

Only the data **element**, **element group** and **node** is read.

#### 6.797 interface gap apply switch

If *switch* is set to **-yes** then any **group**\_interface\_gap will be applied. If *switch* is set to **-no** then any **group** interface gap will be ignored.

Default, if **interface** gap apply is not specified, *switch* is set to -yes.

This interface \_gap\_apply record will be overruled by the control\_interface \_gap\_apply record if specified.

#### 6.798 linear calculation apply switch

If you set the *switch* to **-yes**, Tochnog will skip nonlinearities from the input file. This option is convenient for testing and problem search. Simple set **linear\_calculation -yes** so that the calculation should run without any trouble, and use a **control\_print** for **-post\_node\_rhside\_ratio**. The printed **-post\_node\_rhside\_ratio** should be very small, typically 1.e-10 or lower, since the calculation is linear now. If that is not the case, there may be a problem with the boundary conditions or some other problem.

A typical sequence for testing very large calculations may be following: first run with **solver** -**none** and check the mesh at all times; second run with **linear\_calculation\_apply** -**yes** to check if good linear solutions fields are obtained (check the linear results carefully); finally run your actual calculation without any special options.

The following specific actions are taken:

- Any control plasti apply is deleted, and plasti apply is set to -no.
- mesh is set to -fixed in space.
- For all group <u>\*</u> memory the memory type is set to -total linear if materi displacement is initialised, and it is set to -updated linear otherwise.
- Any **dependency\_item**, **dependency\_diagram** containing **group\_\*** data dependening on one of the dof's of **dof label** is deleted.

- Any group <u>materi</u> <u>elasti</u> <u>hardsoil</u> is deleted and substituted by a group <u>materi</u> <u>elasti</u> <u>young</u> with *E50ref* as Young's modulus.
- Any group <u>materi</u> <u>elasti</u> <u>polynomial</u> is deleted and substituted by a group <u>materi</u> <u>elasti</u> <u>young</u> with *E0* as Young's modulus.
- Any group materi elasti young power is substituted by a linear group materi elasti young.
- Any group <u>materi plasti hypo</u> wolffersdorff is deleted and substituted by a group <u>materi elasti young</u> with *hs* as Young's modulus, and a group <u>materi elasti poisson</u> with value 0.2.
- Any group \_spring \_stiffness \_nonlinear is deleted and substituted by a group \_spring \_stiffness with the stiffness value at strain 0.
- Any group groundflow nonsaturated is deleted.
- Any group interface gap is deleted.
- Any group interface materi hardening is deleted.
- Any group interface materi elasti stiffness tangential diagram is deleted.
- Any group materi damage is deleted.
- Any group materi failure is deleted.
- Any group truss rope is deleted.
- Any **contact** \* is deleted.
- Any groundflow seepage \* is deleted.

## 6.799 materi damage apply switch

If *switch* is set to -no, any damage data in the input file will be ignored. This is done for all timesteps.

This option is convenient for testing your input file just linear, without the need to outcomment each and every part with damage data. See also **control materi damage apply**.

## 6.800 materi\_dynamic factor

This option is specifically meant for dynamic calculations. When the solution is known at time t and is being solved in the new timestep for time d + dt the stresses at time t are used with 1 - factor and the stress at time t + dt are used with factor. Thus using a factor less than 1 makes the scheme less implicit, and thus less numerical damping will occur. Default, if this option is not specified, factor is set to 1 (so fully implicit).

For quasi static calculations this option should not be used since it results in a loss of stability for strong nonlinearities. Also in case of dynamic calculations with heavy nonlinearity it is better to use the default timestepping method with sufficient small steps. You can try if by using this option the calculation converges; if the calculation converges it is better to use this option; if the calculation diverges by using this option it should be removed.

Using this **materi\_dynamic** option the value of **post\_node\_rhside\_ratio** is not usable anymore. So this option is typically used in combination with fixed timestepping. In fact, you should perform the dynamic calculation with typical timestep, and afterwards perform the calculation again with a smaller timestep to see if results change significantly.

In solid mechanics dynamics we always use for velocities and accelerations  $velocity^{t+dt} = (displacement^{t+dt} - displacement^t)/dt$  and  $acceleration^{t+dt} = (velocity^{t+dt} - velocity^t)/dt$ . In fluid calculations where velocities are calculated and displacements of particles are not relevant, we only have  $acceleration^{t+dt} = (velocity^{t+dt} - velocity^t)/dt$ .

## 6.801 materi elasti young power apply switch

If *switch* is set to **-no**, any nonlinearity in young dependent on a power law will be ignored; simply the constant young as encountered in the **group\_materi\_elasti\_young\_power** records will be applied at all times.

#### 6.802 materi failure apply switch

If *switch* is set to **-no**, any failure data in the input file will be ignored. This is done for all timesteps.

This option is convenient for testing your input file just linear, without the need to outcomment each and every part with failure data. See also **control materi failure apply**.

## 6.803 materi plasti hypo substepping index switch

If *switch* is set to **-yes** substepping will be applied in hypoplasticity routines. If *switch* is set to **-no** substepping will not be applied in hypoplasticity routines.

If the record **control\_materi\_plasti\_hypo\_substepping** is specified that record will be used. If none record is not specified *switch* is set to **-yes**.

#### 6.804 materi plasti maximum iterations maximum\_iterations

Plasticity needs iterations on integration point level. Default at maximum 20000 iterations are allowed. With this **materi plasti maximum iterations** options you can increase

the number of allowed iterations, if it looks like results are inaccurate and more iterations are needed. This might be the case in calculations with very heavy plasticity.

## 6.805 materi plasti tension apply switch

If switch is set to -no, any tension-plasticity data in the input file will be ignored. This is done for all timesteps.

See also control materi plasti tension apply.

## 6.806 materi plasti visco apply switch

If switch is set to -no, any visco-plasticity data in the input file will be ignored. This is done for all timesteps.

See also control materi plasti visco apply.

**6.807** mesh specifier\_x specifier\_y specifier\_z

If  $specifier_x$  is set to  $-fixed_in_space$ , the nodal points of the mesh remain fixed in space in x-direction. If a  $specifier_x$  is set to  $-follow_material$ , the nodal points of the mesh will follow material displacements in x-direction. The same holds for the other directions. In 1D, you only need to give  $specifier_x$ , etc.

Default each specifier is set to **-fixed** in **space**.

This record **mesh** only is used if **materi\_velocity** is initialised. If **materi\_displacement** is initialized each specifier is automatically set to **-fixed** in **space**.

6.808 mesh activate gravity element index element\_range

See mesh activate gravity time.

## **6.809 mesh\_activate\_gravity\_element\_group** *index element\_group\_0 element\_group\_1*...

See mesh activate gravity time.

## 6.810 mesh\_activate\_gravity\_geometry index geometry\_item\_name geometry item index

See mesh activate gravity time.

#### 6.811 mesh activate gravity method index method

Set to -method1 or -method2. Default Tochnog will use -method2.

See mesh activate gravity time.

#### 6.812 mesh activate gravity stiffness factor index factor

See mesh activate gravity time.

## 6.813 mesh\_activate\_gravity\_time index time\_start time\_end

With this record you can slowly activate gravity for elements between *time\_start* and *time\_end*.

You can specify an element range with **mesh\_activate\_gravity\_element**. The elements you need to specify as elements range. Possible formats for the elements are a number (eg. 5), a number range (eg. **-ra** 5 4 8 **-ra**), or all elements (**-all**).

Or otherwise, you can specify element group numbers with mesh activate gravity element group.

Or otherwise, you can specify a geometry with **mesh\_activate\_gravity\_geometry** so that elements completely in the geometry will be used.

Tochnog will activate the elements from the bottom to the top. For each specific element the start time of activation is interpolated from the global *time\_start* and *time\_end* and the lowest coordinate of the element. Likewise, for the element end time of activation the highest coordinate is used. Typically take care that the timestep is so small that each timestep only about 10 percent of an element gets filled.

This option comes handy to slowly build dams or so, starting at the bottom and building upwards to the top.

If mesh\_activate\_gravity\_method is set to -method1, before the element start time of activation, the element is not active in the calculation. After the element end time of activation, the element is fully active in the calculation. Between these times the element is active, but the gravity is 0 at the element start time of activation, the gravity gets its full value at the element end time of activation, and the gravity is interpolated in between. With this -method1 the displacements for activated nodes are 0 at the moment of activation,

and grow later in time. Thus the displacements in the activation area are relative to the moment of material activation, and not relative to the moment of start of the calculation.

If mesh\_activate\_gravity\_method is set to -method2, before the element start time of activation, the element is active in the calculation, but has no gravity force yet. After the element end time of activation, the element is fully active in the calculation with full gravity force. Between these times the element is active, but the gravity is 0 at the element start time of activation, the gravity gets its full value at the element end time of activation, and the gravity is interpolated in between. With this -method2 the displacements for activated nodes are not 0 at the moment of activation, but already have values resulting from activation of material below. For elements which are not activated yet, Tochnog will reduce the stiffness so that it will not really influence displacements inside the elements which are already being activated; the stiffness reduction factor can be specified by mesh\_activate\_gravity\_stiffness\_factor, and is 1.e-6 by default. For elements which are not activated yet, Tochnog will not print the elements to the gid postprocessing files; however you can demand that these elements will also be printed by specifying -yes in print\_gid\_mesh\_activate\_gravity or control\_print\_gid\_mesh\_activate\_gravity (default -no).

You can set with **mesh\_activate\_gravity\_time\_initial** when elements become active in a calculation; before the specified *time\_of\_birth* an element will not take part of the calculation.

You can only use this option not in combination with *materi\_displacement*. See also **control mesh activate\_gravity\_apply**.

#### 6.814 mesh activate gravity time initial index time\_of\_birth

See mesh activate gravity time.

#### 6.815 mesh activate gravity time strain settlement index switch

If *switch* is set to **-yes** then strain settlement should be used for the **mesh\_activate\_gravity\_time** record with the same index.

### 6.816 mesh boundary switch

If *switch* is set to **-yes**, Tochnog determines the boundary of the mesh and sets **element\_boundary** and **node\_boundary** records. If *switch* is set to **-no**, Tochnog does not determine the boundary of the mesh. Default, if **mesh\_boundary** is not specified, the *switch* is **-no**.

#### 6.817 mesh correct switch

If *switch* is set to **-yes**, Tochnog checks that the connectivity list for quadrilateral and hexahedral interfaces is correct. If the connectivity list would not be correct (that is, according to the required sequence in Tochnog for such elements), the connectivity list will be corrected.

Default *switch* is set to **-no**.

#### 6.818 mesh delete geometry moving index geometry\_moving\_index

With *geometry\_moving\_index* you tell Tochnog to excavate the mesh with the geometrical entity as specified by the **geometry\_moving\*** records with index *geometry\_moving\_index*.

## 6.819 mesh element group apply index group\_0 group\_1...

If you specify this record, only the element groups specified will be evaluated in all timesteps. Default, if **mesh\_element\_group\_apply** is not specified, all elements groups will be used.

If control\_mesh\_element\_group\_apply is specified it overrules this mesh\_element\_group\_apply record.

6.820 mesh gid assign conditions line index line\_0 line\_1 ...

Assign nodes on gid lines to tochnog **geometry** list *index*. You can use geometry lists to impose geometrical conditions, see **geometry** list. The *index* needs to be 11, 12, ...., 20 (start with index 11, then index 12, up to maximal 20).

A simple example is:

mesh\_gid\_assign\_conditions\_line 11 3 4 (assign gid line 3 and 4 to geometry\_list 11)

bounda\_dof 10 -geometry\_list 11 -velx -vely (set velocities to 0 on geometry\_list 11)

See also control mesh gid batch.

#### 6.821 mesh gid assign conditions point index point\_0 point\_1...

Assign nodes on gid points to tochnog **geometry**\_list *index*. You can use geometry lists to impose geometrical conditions, see **geometry**\_list. The *index* needs to be 1, 2, ..., 10 (start with index 1, then index 2, up to maximal 10).

See also control mesh gid batch.

# 6.822 mesh\_gid\_assign\_conditions\_surface index surface\_0 surface\_1

Assign nodes on gid surfaces to tochnog **geometry\_list** *index*. You can use geometry lists to impose geometrical conditions, see **geometry\_list**. The *index* needs to be 21, 22, ...., 30 (start with index 21, then index 22, up to maximal 30).

See also control mesh gid batch.

## **6.823** mesh gid arc coord index $x_0 y_0 x_1 y_1 x_2 y_2$

Arc coordinates of first, second and third point. See also control mesh gid batch.

#### **6.824** mesh gid circle coord index x y

Circle middle coordinates. See also **control mesh gid batch**.

#### 6.825 mesh gid circle element group index group

Group number. See also control mesh gid batch.

#### 6.826 mesh gid circle hollow index switch

SWitch for hollow circle.

### 6.827 mesh gid circle radius index radius

Circle radius. See also **control mesh gid batch**.

## 6.828 mesh gid cylinder coord index x y z

Circle base middle coordinates. See also control mesh gid batch.

#### 6.829 mesh gid cylinder element group index group

Group number. See also control mesh gid batch.

#### 6.830 mesh gid cylinder height index height

Cylinder height. See also control mesh gid batch.

#### 6.831 mesh gid cylinder hollow index switch

Set *switch* to **-yes** if you want the cylinder to act as a hollow zone, typically inside another zone (for example a volume zone). So this option comes handy when you want to study, for example, cylindrical excavation zones in a soil.

6.832 mesh gid cylinder normal index normal x normal y normal z

Cylinder normal. See also control mesh gid batch.

#### 6.833 mesh gid cylinder radius index radius

Cylinder radius. See also control mesh gid batch.

#### 6.834 mesh gid line structured concentrate index weight start weight end

Concertate elements for structured mesh near start or end of line. The  $weight\_start$  and  $weight\_end$  can be specified between -1 and +1. These are the mesh concentration factors for the line start and end respectively. With 0 there is no concentration. With -1 the concentration is low. With +1 the concentration is high.

Dedault, if **mesh\_gid\_line\_structured\_concentrate** is not specified, both weight factors are 0. See also **control mesh gid batch**.

#### 6.835 mesh gid line element group index group

Group number. See also control mesh gid batch.

#### 6.836 mesh gid line point index point\_0 point\_1

Line points. See also control mesh gid batch.

## 6.837 mesh gid line size index size

Size of elements at line. See also control mesh gid batch.

#### 6.838 mesh gid line structured nel index nel

Number of elements for structured line. See also **control mesh gid batch**.

## 6.839 mesh gid line structured size index size

Size of elements at structured line. See also control mesh gid batch.

### 6.840 mesh gid point coord index x y z

Point coordinates. In 2D you only need to specify x and y. See also control mesh gid batch.

## **6.841** mesh gid rectangle coord index $x_0 x_1 y_0 y_1$

Rectangle coordinates of first and second point. See also control mesh gid batch.

## 6.842 mesh gid rectangle element group index group

Group number. See also control mesh gid batch.

## 6.843 mesh gid rectangle hollow index switch

Set *switch* to **-yes** if you want the rectangle to act as a hollow zone, typically inside another zone (for example a surface zone). So this option comes handy when you want to study, for example, rectangular excavation zones in a soil.

#### 6.844 mesh gid size element\_size

Size of elements. See also control mesh gid batch.

### 6.845 mesh gid sphere coord index x y z

Sphere middle coordinates. See also control mesh gid batch.

#### 6.846 mesh gid sphere element group index group

Group number. See also control mesh gid batch.

## 6.847 mesh gid sphere hollow index switch

Set *switch* to **-yes** if you want the sphere to act as a hollow zone, typically inside another zone (for example a volume zone). So this option comes handy when you want to study, for example, a hollow space in a soil.

#### 6.848 mesh gid sphere radius index radius

Sphere radius. See also control mesh gid batch.

## 6.849 mesh gid surface element index element\_type

Element type for surface. You can use one of **-tria3**, **-tria6**, **-quad4**, **-quad8**, **-quad9**, **-tet4**, **-tet10**, **-hex8**, **-hex20** and **-hex27**. If this record is not specified linear elements are generated as default, triangles in 2D and tets in 3d. See also **control mesh gid batch**.

## 6.850 mesh gid surface element group index group

Group number. See also control mesh gid batch.

#### 6.851 mesh gid surface line index line\_0 line\_1 ...

Surface lines. This option will generate 1 new surface. See also control mesh gid batch.

#### 6.852 mesh gid surface structured nel index nel\_0 nel\_1 ...

Number of elements on lines of structured surface. Specify a number for each of the lines. Specify a positive number if you want to set the number of elements for the line. Specify a 0 for each of the opposing lines. See also **control mesh gid batch**.

### 6.853 mesh gid surface structured size index size

Size of elements on structured surface. See also control mesh gid batch.

## 6.854 mesh gid volume element group index group

Group number. See also control mesh gid batch.

6.855 mesh gid volume surface index surface\_0 surface\_1 ...

Volume surfaces. This option will generate 1 new volume. See also control mesh gid batch.

**6.856** mesh\_interface\_triangle\_coordinates index coord\_x\_0 coord\_y\_0 coord\_z\_0 coord\_x\_1 coord\_y\_1 coord\_z\_1 coord\_z\_2 coord\_y\_2 coord\_z\_2 ...

With this option you can generate interface elements in a 3d mesh with tet4 elements. You specify the triangulated plane of the interface as sets of triangles in 3d space. For each triangle you specify for the three corner points the coordinates. For example  $coord\_x\_0$   $coord\_y\_0$   $coord\_z\_0$  are the coordinates of the first corner point,  $coord\_x\_1$   $coord\_y\_1$   $coord\_z\_1$  are the coordinates of the second corner point and  $coord\_x\_2$   $coord\_y\_2$  coord\_z\_2 are the coordinates of the third corner point. The combination of all triangles specifies the plane which will be intersected with the 3d tet4 mesh to generate the interface elements.

With **mesh\_interface\_triangle\_element\_group** you specify the group which will be attributed to the interface elements. With **control\_mesh\_interface\_triangle** you specify the control index for which the generation should be done.

A typical input file looks like:

group\_type 1 -materi group\_interface 1 -yes group\_interface materi\_memory 1 -total\_linear group\_interface\_materi\_elasti\_stiffness 1 1.e11 0.5e11 0.5e11 ... mesh\_interface\_triangle\_coordinates 0. 0. 0.6 100. 0. 0.6 0. 100. 0.6 mesh\_interface\_triangle\_element\_group 1 ... control\_mesh\_interface\_triangle 10 -yes ...

#### 6.857 mesh interface triangle element group index element\_group

See mesh interface triangle coordinates.

#### 6.858 message switch

If *switch* is set to **-yes** the introduction message (executable name, date, conditions) will be printed. If *switch* is set to **-no** it will not be printed. Default *switch* is **-yes**.

You can also set this **message** as environment symbol to **-no** if you want to suppress messages for all calculations.

#### 6.859 mpc apply switch

If *switch* is set to **-yes** then mpc conditions will be used. If *switch* is set to **-no** then mpc conditions will not be used. Default, if **mpc apply** is not specified, *switch* is set to **-yes**.

See also control mpc apply.

## 6.860 mpc element group index element\_group\_0 element\_group\_1

Each node of element of group *element\_group\_0* that is also located in an element of group *element\_group\_1* will be tied to that group by means of multi point constraints. The multi point constraints will be consistent with the shape functions at the specific isoparametric coordinates of the location of that node in the element of group *element\_group\_1*. For *element\_group\_1* you can only use isoparametric elements. As a special option you can use **-all** for *element\_group\_1*, so that all other element groups are selected.

 $See also \mathbf{mpc} \ \mathbf{element} \ \mathbf{group} \ \mathbf{always}, \mathbf{mpc} \ \mathbf{element} \ \mathbf{group} \ \mathbf{closest} \ \mathrm{and} \ \mathbf{control} \ \mathbf{mpc} \ \mathbf{element} \ \mathbf{group} \ \mathbf{closest} \ \mathbf{and} \ \mathbf{control} \ \mathbf{mpc} \ \mathbf{element} \ \mathbf{group} \ \mathbf{closest} \ \mathbf{and} \ \mathbf{control} \ \mathbf{mpc} \ \mathbf{closest} \ \mathbf{mpc} \ \mathbf{closest} \ \mathbf{mpc} \ \mathbf{closest} \ \mathbf{mpc} \ \mathbf{closest} \ \mathbf{mpc} \ \mathbf{mpc} \ \mathbf{mpc} \ \mathbf{closest} \ \mathbf{mpc} \ \mathbf{m$ 

#### 6.861 mpc element group always index switch

If *switch* is set to **-yes** the mpc's will be generated always. If *switch* is set to **-no** the mpc's will only be generated if the considered node is not a member of the node list of the element of group *element\_group\_1* (this ensures that mpc's will only be generated if the node is completely loose from the other element). You can use the *switch* is **-no** option if you are not sure if *element\_group\_0* is connected to, or not connected to, *element\_group\_1*; with **-no** you will not get mpc's if the groups are connected; see **mpc\_\*** in the dbs file to check if mpc's are generated. So if you are not sure if *surfaces* are connected in gid, a typical strategy would be:

- run tochnog with mpc\_element\_group ... and mpc\_element\_group\_always -no
- if you get mpc\_\* records in the dbs, run again with mpc\_element\_group ... and mpc\_element\_group\_always yes
- if you do not get mpc \* records remove mpc element group and mpc element group always

Default, if mpc element group always is not specified, *switch* is set to -yes.

#### 6.862 mpc element group closest index switch

If *switch* is set to **-yes** the mpc's will also be generated if no element of *element\_group\_1* is found; then the closest element of *element\_group\_1* will be used.

### 6.863 mpc element group coord geometry index switch

If *switch* is set to **-yes** the mpc's will only be generated if the coordinate of the node of *element\_group\_0* is above the lowest node of *element\_group\_1*.

A typical example is when you mpc the nodes of soil with  $element\_group\_0$  to pile elements with  $element\_group\_1$ . Then only soil nodes above the toe of the pile should be mpc'd. A piece of input file will look like:

```
...
start_define
pile_group 1
end_define
start_define
soil_interface_group 2
end_define
...
start_define
```

```
pile_edge geometry_line 10
end_define
pile_edge .....
...
mpc_element_group 10 soil_interface_group pile_group (mpc soil nodes to pile)
mpc_element_group_geometry 10 -pile_edge (for soil nodes located on the pile edge)
mpc_element_group_closest 10 -yes (the soil node should be mpc'd to the closest pile element)
mpc_element_group_coord_geometry 10 -yes (do this only for soil nodes above the toe of the pil
mpc_element_group_dof 10 -velx -vely (in 2D mpc the velocities, and thus displacements)
...
```

### 6.864 mpc element group dof index dof\_0 dof\_1 ...

The dof\_0 dof\_1... in mpc\_element\_group\_dof specify the dof's that should be set equal, e.g. -velx, -vely etc. Default, if mpc\_element\_group\_dof is not specified, all principal dofs will be set equal.

### 6.865 mpc element group eps iso index eps

With *eps* you can specify the tolerance on the isoparametric coordinates for the element of *element\_group\_1* below which a node of *element\_group\_0* is considered to be located in *element\_group\_1*. Default, if **mpc\_element\_group\_eps\_iso** is not specified, *eps* is set to **1.e-4**.

## **6.866 mpc\_element\_group\_geometry** index geometry\_entity\_item geometry entity\_index

Select a geometry for nodes of  $element\_group\_0$ .

#### 6.867 mpc element group keep index switch

If *switch* is set to **-yes** the generated mpc's will kept in the remainder of the calculation. As a typical application you can use this option to save computing time if you know that the mpc's need not to be changed during time steps. Or as another example you can use this option if you want that the mpc remains active even if the element for which the mpc is generated gets outside the specified geometry of **mpc element group geometry**.

## **6.868** mpc\_geometry index geometry\_entity\_item\_0 geometry\_entity\_index\_0 geometry\_entity\_item 1 geometry\_entity\_index\_1

#### See also mpc geometry method.

If method in mpc\_geometry\_method is set to -method0 the following mpc's will be generated. This record automatically generates mpc\_node\_number and mpc\_node\_factor records such that dof's in the second geometry geometry\_entity\_item\_1 geometry\_entity\_index\_1 become equal to the dof's in the first geometry geometry\_entity\_item\_0 geometry\_entity\_index\_0. The switch\_x switch\_y switch\_z in mpc\_geometry\_switch specify the coordinates that should be checked to judge if a node in the second geometry is considered to have the same position as a node in the first geometry, and thus should get the same dof's. Only the coordinate for which the corresponding switch is set to -yes will be checked. For example in 3D if -yes -no -no are used then a node in the second geometry gets the same dof's of a node in the first geometry in it has (almost) equal x-coordinate; the y and z-coordinate are irrelevant. In 2D only switch\_x switch\_y need to be specified. With mpc\_geometry\_tolerance you can set the tolerance beneath which nodes of the first geometry and second geometry are assumed to have the same coordinate. If mpc\_geometry\_tolerance is not specified then a tolerance of 1.e-4 is used.

If *method* in **mpc\_geometry\_method** is set to **-method1** the following mpc's will be generated. You should only specify the first geometry. The dof's of the nodes in this first geometry become equal. The first node of this first geometry becomes the master, all other nodes in this first geometry become slave. If you want to know which node is the first node in this first geometry, use a **control\_print ... -node** with a **print\_filter** for the first geometry.

If *method* in **mpc\_geometry\_method** is set to **-method2** the following mpc's will be generated. You should only specify the first geometry. Unknowns of the nodes with equal coordinate in this first geometry become equal.

#### 6.869 mpc geometry method index method

See mpc\_geometry. If this mpc\_geometry\_method is not specified then *method* will be set to -method0.

6.870 mpc geometry switch index switch\_x switch\_y switch\_z

See mpc geometry.

#### 6.871 mpc geometry tolerance index tolerance

See mpc geometry.

The dof\_0 dof\_1 ... in mpc\_geometry\_dof specify the dof's that should be set equal, e.g. -velx, -vely etc.

## 6.873 mpc linear quadratic switch

If *switch* is set to **-yes** this option is activated.

If you have a mesh with both linear elements and quadratic elements, the mesh is not compatible at the places where the linear elements and quadratic elements meet at a common interface. There some of the quadratic element nodes are not attached to the linear elements, and so non-compatible solution fields occur.

This **mpc\_linear\_quadratic** option allows you to automatically prevent the non-compatible solution fields. Tochnog imposes a multi point constraint on all non-compatible solution fields between the linear and quadratic elements, so that the extra nodes of the quadratic elements are forced to follow the solution field of the linear elements, and so compatibility is ensured again.

This option typically can be used to model structural parts like beams, sheet piles, tunnel shells etc with quadratic elements, and the surrounding soil with linear element. Use one quadratic element in the structural part thickness direction, and extra one quadratic soil element attached to the structural element. For the remaining soil elements use linear elements. In this way, the stiff structural elements can deform flexible enough, and you save computer time by modeling most of the soils with linear elements.

# **6.874** mpc\_node\_factor\_index\_factor\_10\_factor\_11 ... factor\_20\_factor\_21 ...

See mpc node number.

## **6.875** mpc\_node\_number index node\_0 dof\_0 node\_1 dof\_10 dof\_11 ...node\_2 dof\_20 dof\_21 ...

This Multi Point Constraint record  $\mathbf{mpc}$ \_node\_number allows you to set constraints between dof's at different nodes. The  $dof_0$  specifies the dofat node number  $node_0$  which will be constrained. It will be constrained to dof's  $dof_10$ ,  $dof_11$ , ... of  $node_1$  and  $dof_20$ ,  $dof_21$ , ... of  $node_2$ , etc. Only principal dof's can be specified. Principal dof's are material velocities, groundflow pressure, temperature in the convection diffusion equation, etc.; see the start of the data section for a definition of principal dof's. With mpc\_node\_factor you can set multiplication factors for the constraints. If you don't specify mpc\_node\_factor a 1 is used for all factors.

Example:

. . .

mpc\_node\_number 10 1 -velx 2 -velx 3 -vely mpc\_node\_factor 10 7. 9.

In this example the velx\_1 = 7. \* velx\_2 + 9. \* vely\_3 where velx\_1 is the x-velocity at node 1 etc. Node number  $node_0$  is this slave node which depends on nodes  $node_1$  etc. which are the master nodes.

Boundary conditions with **bounda\_dof** and **bounda\_time** cannot be specified for slave nodes.

See also mpc geometry for easy generation of multi point constraints.

#### 6.876 node index coord\_0 coord\_1 coord\_2

Coordinates of node index. In 1D, only coord 0 should be specified, etc..

You are not allowed to put free nodes (not attached to any element) in your model. These free nodes will be removed automatically.

#### 6.877 node boundary index switch

The *switch* will be set to **-yes** if the node with index *index* is located on the boundary of the mesh.

This record will only become available if **mesh\_boundary** is set to **-yes**. This record is meant for printing only, it should not be set by the user.

6.878 node bounded index indicator\_dof\_0 indicator\_dof\_1 ...

This record is for printing only, it is not an input record. This record indicates if dof's in the node are bounded via a **bounda\_dof** record; then the corresponding indicator is set to 1, else it remains 0.

## **6.879** node\_bounded\_index index bounda\_dof\_index\_0 bounda\_dof\_index\_1

This record is for printing only, it is not an input record. This record list the index of the **bounda\_dof** record by which the dof's are bounded. This index is only filled if the dof's really bounded, so if the corresponding value in the **node bounded** record is set to 1.

#### 6.880 node convection apply index switch

. . .

If *switch* is set to **-no** convection contributions in node *index* are de-activated in case they are activated for the whole mesh by **convection apply** or **control convection apply**.

#### **6.881** node damping index damping\_x damping\_y damping\_z

This record adds a discrete damper to node *index* in x, y and z direction respectively. In 1D only *damping\_x* needs to be specified, etc. The damper will lead to a nodal force of the size *damping\_x* \*  $v_x$  where  $v_x$  is the velocity in x direction. The same holds for the y and z direction.

#### 6.882 node deformed mesh index coord 0 coord 1 coord 2

After the calculation, this record will contain deformed coordinates of node *index*.

If **materi\_displacement** is initialised these are the initial start coordinates **node\_start\_refined** plus the displacements.

If materi\_velocity\_integrated is initialised and the node follows the material with - follow material these are the current coordinates of the node node.

If **materi** velocity integrated is initialised and the node is fixed in space with -fixed in space these are the initial start coordinates **node** start refined plus the integrated velocities.

Else these are equal to the current node **node**.

In 1D, only *coord*  $\theta$  is filled, etc..

## **6.883** node dof *index dof*\_0 *dof*\_1 ....

 $dof_0 dof_1 \dots$  are the degrees of freedom (dof's) at the node with number *index*. The total number and type of the dof's depends on the initialization part. Each node has the same dof's.

Unknowns like pressure, temperature, etc. are primary dof's. The other dof's, space derivatives and the time derivative, are not primary dof's. In the example below, **-temp** is 1., **-xtemp** is 0.2 and **-ttemp** is 0.1 in node 6

number \_of \_space \_ dimensions 1
derivatives
condif \_temperature
end \_initia
...
node \_dof 6 1.0 0.2 0.1
...

Default all values in the **node** dof records are set zero at the start of the calculation.

These **node** dof records contain principal dof's for all elements (displacements, temperatures, etc). Other dof's like strains, stresses etc. are only filled for the normal isoparametric elements; thus, for example, strain and stress results for interfaces elements are not placed in the **node** dof records.

See also: dof label and post point.

## 6.884 node dof calcul index ...

See post calcul.

## 6.885 node dof start refined index dof\_0 dof\_1 ...

This record will be filled with  $dof_0 dof_1 \dots$ , which are the degrees of freedom (dof's) as specified at the start of the calculation. at the node with number *index*.

If the mesh has been refined, these start values hold for the refined mesh.

See also node dof and node start refined.

## 6.886 node dynamic pressure index value

With this record you can specify for node *index* the dynamic pressure. Thus, the dynamic pressure as normally calculated will be overruled with this *value*.

#### **6.887** node force index force\_x force\_y force\_z

With this record you can input a discrete nodal force at node *index*. In 1D you only should specify the force in x-direction. In 2D you only should specify the force in x- and y-direction.

## 6.888 node\_geometry\_present index geometry\_item\_name\_0 geometry\_item\_index\_0 geometry\_item\_name\_1 geometry\_item\_index\_1 ....

This record lists for node *index* the geometries in which it is present. So it is a print record only, for checking if the geometries include exactly the nodes that you want. You can switch on or off filling of these records by setting **print\_node\_geometry\_present** to **-yes** or **-no**.

#### **6.889** node inertia index inertia\_dof\_0 inertia\_dof\_1 ...

This record will be filled with calculated inertia terms degrees of freedom (dof's) as specified at the start of the calculation. at the node with number *index*. For material velocity that is the mass inertia term in the node.

## 6.890 node mass index mass\_x mass\_y mass\_z

This record adds a discrete mass to node *index* in x, y and z direction. In 1D only the x-mass needs to be specified, etc. The mass will lead to a nodal force of the size  $mass\_x * \dot{v}$  where  $\dot{v}\_x$  is the acceleration, and to a gravity force if **force**\_**gravity** is specified. The same holds for the y and z direction.

#### **6.891** node mesh *index* ...

Same as **mesh**, but now specified per node however. The *index* specifies the node number. If this **node mesh** record is specified for a node, it overrules the **mesh** record.

## 6.892 node rhside index rhside\_0 rhside\_1 ...

This record will contain after the calculations the unbalance forces.

For the temperature equation, this will give the heat flow normal to the outer surface (the heat flux to the environment) at prescribed temperatures. For velocity dof's, this will give the force vector at prescribed displacements. For the pressure in the ground flow equation,

this will give the ground flow to the environment at prescribed pressures. For non-prescribed dofs the record will contain the numerical unbalance, and should be close to zero.

The *index* is the node number.

#### 6.893 node slide index slide\_number

With **node slide** you can specify of a specific node *index* if it belongs to a sliding geometry with index *slide\_number*. For the sliding geometry **slide\_geometry** is not needed anymore because the **node slide** already specifies which nodes belong to the sliding geometry.

#### 6.894 node static pressure index value

With this record you can specify for node *index* the static pressure. Thus, the static pressure as normally calculated will be overruled with this *value*.

#### 6.895 node start refined index coord\_0 coord\_1 coord\_2

After the calculation, this record will contain coordinates of node *index* as specified at the start of the calculation. If the mesh has been refined this record with contain the start coordinates for the refined mesh. In 1D, only *coord*\_ $\theta$  is filled, etc..

## **6.896** node stiffness index stiffness\_x stiffness\_y stiffness\_z

This record adds a discrete stiffness to node *index* in x, y and z direction respectively. In 1D only *stiffness\_x* needs to be specified, etc. The stiffness will lead to a nodal force of the size *stiffness\_x* \*  $u_x$  where  $u_x$  is the displacement in x direction. The same holds for the y and z direction. Condition: also **materi\_velocity\_integrated** or **materi displacement** should be initialized.

#### 

This record will contain after a calculation the status of a node for the **support\_edge\_normal\_plasti\_tension** or **support\_edge\_normal\_plasti\_tension\_double** option. If the node is opened due to tension plasticity the status is set to **-opened**. If the node is closed the status is set to **-closed**.

#### 6.898 node total pressure index value

With this record you can specify for node *index* the total pressure. Thus, the total pressure as normally calculated will be overruled with this *value*.

#### 6.899 nonlocal nonlocal\_radius

By specifying this record in combination with a viscoplastic model, like group \_materi \_plasti \_visco\_power, a nonlocal yield rule fn will be used in the viscoplastic law. The nonlocal yield rule needs to be initialized as dof by the materi \_plasti \_f \_nonlocal record in the initialization part. The nonlocal yield rule fn in a point is determined by an averaging of the local yield rule fn in a point is determined by an averaging of the local yield rule fn in neighboring points and using gauss weighting functions for this (i.e. the larger the distance the less the neighboring point contributes to the nonlocal yield rule). The averaging is done over a region with radius nonlocal radius.

In this way, you can prevent unlimited localization and so mesh dependency, in calculations with softening plasticity.

See also **nonlocal name**.

#### 6.900 nonlocal name name

With *name* you specify the name of the plasticity model that should be treated nonlocal, eg **-group\_materi\_plasti\_mohr\_coul**. You can only specify one name, so only one plasticity model can be used as nonlocal model.

#### 6.901 plasti apply switch

If *switch* is set to **-no**, any plasticity data in the input file will be ignored. This is done for all timesteps.

This option is convenient for testing your input file just linear, without the need to outcomment each and every part with plasticity data. See also **control plasti apply**.

#### 6.902 post apply index switch

Setting *switch* to **-no** prevents post processing commands to be evaluated. Postprocessing commands have **post** in the name (only the **post\_node\_rhside\_ratio** will be evaluated always, independent of **post\_apply**).

Default, if **post** apply is not specified, the switch is set to -yes.

#### **6.903** post calcul dof\_0 operat\_0 dof\_1 operate\_1 ...

This records activates calculation post results. The  $dof_0 dof_1$  etc. refers to a dof, and possibilities for dof are listed below. The  $operat_0 operat_1$  etc. refers to an operat, and possibilities for operat are listed below.

Here dof can be one of the matrices

-materi\_stress,
-materi\_strain\_elasti,
-materi\_strain\_plasti,
-materi\_strain\_plasti\_compression,
-materi\_strain\_plasti\_diprisco,
-materi\_strain\_plasti\_druckprag,
-materi\_strain\_plasti\_hardsoil,
-materi\_strain\_plasti\_tension,
-materi\_strain\_plasti\_tension,
-materi\_strain\_plasti\_vonmises,
-materi\_strain\_total or dof can be one of the vectors -materi\_velocity, -materi\_displacement,

or dof can be one of the scalars -condif\_temperature, -groundflow\_pressure.

The results of these calculations are stored for each **node\_dof** record in a **node\_dof\_calcul** record, and are stored for each **post\_point\_dof** record in a **post\_point\_dof\_calcul** record, and are stored for each **post\_line\_dof** record in a **post\_line\_dof\_calcul** record, and are stored for each **post\_quadrilateral\_dof** record in a **post\_quadrilateral\_dof\_calcul** record, and are stored for each **post\_quadrilateral\_dof** record in a **post\_quadrilateral\_dof\_calcul** record.

We denote a matrix dof with  $A_{ij}$  and denote a vector dof with  $A_i$ , and denote a scalar dof with a. If operat is -absol then the absolute value of a scalar a is calculated.

If operat is -average then  $\frac{1}{3}(A_{11} + A_{22} + A_{33})$  is calculated for a matrix or  $\frac{1}{3}(A_1 + A_2 + A_3)$  is calculated for a vector.

If operat is -negative then the average of the negative principal values for a matrix is calculated. If materi\_strain\_plasti is taken for the matrix  $A_{ij}$ , then this operator typically can be used as a measure for the amount of compression failure (crunching).

If operat is **-positive** then the average of the positive principal values for a matrix is calculated. If **materi\_strain\_plasti** is taken for the matrix  $A_{ij}$ , then this operator typically can be used as a measure for the amount of tensile failure (cracking).

If operat is **-prival** then three principal values of a matrix  $A_{ij}$  are calculated. Each principal value contains the size of the principal vector. The principal values are ordered (the first value is the smallest one, and the last value is the largest one).

If operat is **-privec** then three principal vectors of a matrix  $A_{ij}$  are calculated. Each principal vector contains the x, y and z component of the principal vector. The same ordering as used for **-prival** is used here also.

If operat is -size tot then  $\sqrt{A_{ij}A_{ij}}$  is calculated for a matrix or  $\sqrt{A_iA_i}$  is calculated for a vector. This measures the size of a matrix or the size of a vector.

If operat is -size dev then  $\sqrt{B_{ij}B_{ij}}$  is calculated where  $B_{ij}$  is the deviatoric part of a matrix  $A_{ij}$ :  $B_{ij} = A_{ij} - \delta_{ij} \frac{A_{11} + A_{22} + A_{33}}{3}$  where  $\delta_{ij}$  is 1 if i = j and is 0 otherwise. This measures the size of the deviatoric part of the matrix.

Specially for -quad4, -quad9, -hex8 and -hex27 elements you can set *operat* to -force in case *dof* is -materi \_stress. Then forces and moments are calculated in these isoparametric elements. See also post calcul materi stress force element group.

Specially for geotechnics you can set *operat* to **-total\_pressure** in case *dof* is **-materi\_stress**. Then the total stress is calculated from the effective stress and the groundflow total pressure. This option is not valid in combination with undrained pressures as obtained by **group materi undrained capacity**.

Specially for geotechnics you can set *operat* to **-static\_pressure** in case *dof* is **-groundflow\_pressure**. Then the static pressure is calculated.

Specially for geotechnics you can set *operat* to **-dynamic\_pressure** in case *dof* is **-groundflow\_pressure**. Then the dynamic pressure is calculated.

Specially for geotechnics you can set *operat* to -**k0** in case *dof* is -**materi\_stress**. Then the ratio of horizontal and vertical stresses is calculated. If 2D this is the ratio  $0.5 \frac{\sigma_{xx} + \sigma_{zz}}{\sigma_{yy}}$ .

If 3D this is the ratio  $0.5 \frac{\sigma_{xx} + \sigma_{yy}}{\sigma_{zz}}$ .

Specially for geotechnics you can set *operat* to **-young\_apparent** in case *dof* is **-materi\_stress**. Then the apparant Young modulus is calculated from the incremental strains and incremental stresses. If determination is not possible (for almost zero incremental strains) the apparent young is put to 0. See also **post\_calcul\_apparent\_total**.

Specially for geotechnics you can set *operat* to **-poisson\_apparent** in case *dof* is **-materi\_stress**. Then the apparent Poisson ratio is calculated from the incremental strains and incremental stresses. If determination is not possible (for almost zero incremental strains) the apparent poisson is put to 0. See also **post\_calcul apparent\_total**.

Specially for geotechnics you can set *operat* to **-total\_pressure** in case *dof* is **-groundflow\_pressure**. Then the total pressure is calculated.

Specially for geotechnics you can set *operat* to **-safety\_lifting** in case *dof* is **-materi\_stress**. Then the hydraulic safety factor  $\frac{\sigma_{\text{vertical}} + p\_\text{total}}{p\_\text{total}}$  is calculated. In 1D  $\sigma_{\text{vertical}} = \sigma_{\text{xx}}$ , in 2D  $\sigma_{\text{vertical}} = \sigma_{\text{yy}}$  and in 3D  $\sigma_{\text{vertical}} = \sigma_{\text{zz}}$ ; see also **post\_calcul\_safety\_method**.

Specially for geotechnics you can set *operat* to **-safety\_piping** in case *dof* is **-materi\_stress**. Then the hydraulic safety factor  $\frac{\sigma_{\text{vertical}}+p\_dynamic}{p\_dynamic}$  is calculated; see also **post\_calcul\_safety\_method**.

The next piece of input file

materi stress

. . .

```
materi_strain_plasti
end_initia
...
post_calcul-materi_stress-size_dev-materi_strain_plasti-size_tot
...
control_timestep 1 ...
control print 1 -node dof calcul
```

will print records like

node dof calcul index 0.2 1.1e-4

Here the 0.2 is the equivalent Von Mises stress and 1.1e-4 measures the plastic strain matrix.

See also post calcul absolute and post calcul label.

## 6.904 post calcul absolute switch

If *switch* is set to **-yes** all results of **post\_calcul** are set to be positive values. This may be done if you prefer positive values in your presentation of results.

#### 6.905 post calcul apparent total switch

If *switch* is set to **-yes**, the total strains and total stresses (as opposed to incremental) will be used to calculate the apparent young modulus and poisson ratio. If *switch* is set to **-no**, the incremental strains and incremental stresses will be used. Default, if **post calcul apparent total** is not specified, the *switch* is set to **-no**.

## 6.906 post calcul label doflabel\_0 label\_1 ...

This record will be filled with the names of the data that is calculated by means of the **post\_calcul** option. The first name comes from the first *dofoperat* in **post\_calcul**, the second name comes from the second *dofoperat* in **post\_calcul**, etc. You can find this record in the dbs file after a calculation.

6.907 post calcul limit lower\_0 upper\_0 lower\_1 upper\_1 ...

With this record you can specify the lower and upper allowed values for all calculated results. With  $lower\_dof\_0$  you specify the lower allowed value for the first result. With

upper dof 0 you specify the upper allowed value for the first result. Etc.

#### 6.908 post calcul materi stress force average switch

See first post calcul materi stress force element group.

This **post\_calcul\_materi\_stress\_force\_average** option is only available for **quad9** and **hex27** elements. It can be used if forces and moments are primarily calculated in two opposing end faces of the **quad9** and **hex27** element. If *switch* set to **-yes**, the forces and moments of nodes in the plane between the two end faces will be set to the averaged values from the forces and moments on the two opposing end faces. If *switch* set to **-no** this is not done. Default *switch* is **-yes**.

## 6.909 post\_calcul\_materi\_stress\_force\_direction\_exclude $dir_x$ $dir_y dir_z$

#### See first post calcul materi stress force element group.

In 3D, Tochnog needs to know for which element sides it should determine forces and moments. For this purpose you need to specify this direction  $dir_x dir_y dir_z$ . All element sides with normals in this direction will be neglected; no forces and moments will be determined for such sides.

Typically, in a tunnel calculation you take the tunnel length direction as dir\_x dir\_y dir\_z.

#### 

With eps you can influence which normals are considered to be in the specified exclude direction. A small eps specifies that only very precise normals in the specified direction will be excluded. A large eps specifies that also not precise normals in the specified direction will be excluded. In fact eps is the difference from inproduct between the specified exclude direction with the normal direction and 1. Default eps is 1.e - 8.

## 6.911 post\_calcul\_materi\_stress\_force\_direction\_include $dir_x dir_y dir_z$

#### See first post calcul materi stress force element group.

In 3D, Tochnog needs to know for which element sides it should determine forces and moments. For this purpose you need to specify this direction  $dir_x dir_y dir_z$ . All

element sides with normals perpendicular to this direction will be neglected; no forces and moments will be determined for such sides.

Typically, in a sheet pile calculation you take the sheet pile height direction as  $dir_x dir_y dir_z$ .

## $\begin{array}{cccc} \textbf{6.912} & \textbf{post\_calcul\_materi\_stress\_force\_direction\_include\_epsilon} \\ & eps \end{array}$

With eps you can influence which normals are considered to be perpendicular to the specified include direction. A small eps specifies that only normals precisely perpendicular to the specified direction will be excluded. A large eps specifies that also normals not precisely perpendicular to the specified direction will be excluded. In fact eps is the difference from inproduct between the specified include direction with the normal direction and 0. Default eps is 1.e - 8.

## 6.913 post\_calcul\_materi\_stress\_force\_element\_group\_element\_group\_0 element\_group\_1...

With the **post\_calcul -materi\_stress -force** option the normal force, shear force and moment(s) are calculated for the isoparametric elements **-quad4**, **-quad9**, **-hex8** and **-hex27**. This option is meant for structures like sheet piles, tunnel shells, etc. where there is only 1 element over the thickness of the structure. Thus the element has a thickness equal to the complete thickness of the structure, and the length of the element is a part of the total length of the structure (e.g. tunnel length).

In the following definitions of forces and moments, n denotes the normal to an element side, t denotes the thickness direction in the side, and l denotes the length direction. The 2D and 3D normal force **nor** results is defined by the normal stresses  $sigma_{nn}$  integrated over the thickness. The 2D and 3D shear force **she** results is defined by the shear stresses  $sigma_{nt}$  integrated over the thickness. The 2D moment **mom** and 3D moment **mom1** are defined by the moment contributions of normal stresses  $sigma_{nn}$  with a distance in thickness direction  $d_t$  relative to the middle of the element, integrated over thickness direction (radial bending moment in tunnel shell, thickness bending moment in sheet pile, etc.). The 3D moment **mom2** is defined by the moment contributions of normal stresses  $sigma_{nn}$  with a distance in length direction  $d_l$  relative to the middle of the element, integrated over thickness direction (bending moment in tunnels, sheet piles, etc.).

The forces and moments will be calculated per unit length l of the isoparametric element, where l is the size of the element in length direction. In a 3D calculation, the length of an element is determined from the nodal coordinates differences in length direction. In a axi-symmetric 2D calculation, the length of the elements is set to 2 \* PI \* radius by Tochnog (notice that with this definition values cannot be calculated at the symmetry axis with zero radius). In a plane 2D calculation, the length of the elements is set to 1 by Tochnog.

The normal force and moment(s) are given the proper sign (plus or minus). For example, a positive normal force means that the structure is under tension. For the shear force, however, always a positive value is calculated by Tochnog, so only the size of the shear force is available (and not the direction of the shear force).

For all of the forces and moment vectors, we want to display the vector in thickness direction of the structure, to get a clear view in postprocessors (e.g. GID). Thus, the components in global x- and y-direction are determined such that the vector direction is in thickness direction of the structure. Because of this, the components by themselves are not the real physical components of the force or moment; they are only convenient values for getting clear plots in postprocessors. However, the size of the vector formed by these components (square root of components squared), indeed is the real physical size of the force or moment, so the size can indeed be used for design purposes. For your convenience, the size of each vector is also calculated automatically be Tochnog. For example, for the normal forces Tochnog calculates **-norx\_sig, -nory\_sig** and **-nors\_sig** which are the global plot vector x-component, y-component and the physical real size respectively.

The enable a correct force or moment direction in either the positive of negative thickness direction, Tochnog wants you to specify **post calcul materi stress force reference point**.

In 3D, you need to specify either **post\_calcul\_materi\_stress\_force\_direction\_exclude** or **post\_calcul\_materi\_stress\_force\_direction\_include**. With these records you can determine for which element sides forces and moments should be determined. The direction and element should be such that for each element for which you want to determine forces and moments exactly 4 sides should be consistent with the specified direction. Otherwise the present option for determination of forces and moments is not available for the element. Only one of **post\_calcul\_materi\_stress\_force\_direction\_exclude** and **post\_calcul\_materi\_stress\_force\_direction\_exclude** and

The *element\_group\_0 element\_group\_1*... of this **post\_calcul\_materi\_stress\_force\_element\_group** specify the groups of isoparametric elements for which the forces and moments should be determined by Tochnog.

Summary of conditions for the post calcul -materi stress -force option to work well:

- Only 1 element in thickness direction.
- Elements in 3D should be regular shaped in length direction. That is, the element sides perpendicular to the length direction should be completely parallel.
- At least 1 timestep should be done (since element forces needed for this option are setup in a timestep)

# **6.914** post\_calcul\_materi\_stress\_force\_reference\_point $x_0 y_0 z_0 x_1 y_1 z_1 \dots$

#### See first post calcul materi stress force element group.

For example tunnels typically are of circular or piecewise circular geometry. To get a correct direction of the calculated forces and moments, Tochnog needs to know the approximate middle point of the tunnel, so that it can put all negative forces and moments and positive forces and moments consistently outwards or inwards in thickness direction of the structure. Thus, you need to specify with this **post\_calcul\_materi\_stress\_force\_reference\_point** record the approximate middle point of the tunnel that you are evaluating for each of the element groups. In case you have a sheet pile, you should specify a reference point on a large perpendicular distance away from the sheet pile.

You need to specify a reference point for each element group specified in **post calcul materi stress force ele** 

In 3D you need to specify the x, y and z value for each reference point. In 2D you only need to specify the x and y value for each reference point.

See also post calcul materi stress force plot switch.

#### 6.915 post calcul materi stress force outer switch

If *switch* is set to **-yes**, the forces and moments are only calculated for the nodes at the outer sides of the elements; these are the nodes which have the furthest distance relative to the reference point. This will give a bit more nice vector plots.

Default, if **post\_calcul\_materi\_stress\_force\_outer** is not specified, *switch* is set to **-no**. This will give a bit more nice contour fill plots.

# 6.916 post\_calcul\_materi\_stress\_force\_plot\_switch\_switch\_0 switch\_1

If you don't like the direction in which tochnog draws the vectors (outward or inward), you can switch the direction by setting the corresponding switch to **-yes**. In 2D you need to specify a switch for the normal force, shear force and moment. In 3D you need to specify a switch for the normal force, shear force and two moments.

# **6.917 post\_calcul\_materi\_stress\_force\_thickness\_switch** *switch\_element\_group\_0 switch\_element\_group\_1*...

See first post calcul materi stress force element group.

In 3D Tochnog normally assumes that the shortest element direction in the side where forces

and moments are calculated is the structure thickness direction. If that is not the case, e.g. if you have very short elements in a tunnel length direction, then you need to explain Tochnog that it should switch to the longest element direction as structural thickness direction, by setting a to **-yes**.

This ensures that the shear force is always really calculated over the structural thickness, and the first moment is really the moment over the structural thickness.

If you specify **post\_calcul\_materi\_stress\_force\_thickness\_switch** you need to give a switch for each element group of **post\_calcul\_materi\_stress\_force\_element\_group**.

#### 6.918 post calcul multiply factor\_0 factor\_1 ...

With this record you can specify a multiplication factor for each calculated item. This comes handy when you prefer another definition. If you specify **post\_calcul\_multiply**, you need to give a factor for each item.

# 6.919 post calcul safety default eps value

Specifically for safety lifting and piping calculations division by pressure values equal to zero can occur. With this **post\_calcul\_safety\_default** record you can prevent such division. If the absolute of the pressure value for the division is smaller than the specified *eps*, the safety factor will be set to the user specified *value*.

In case this record is not specified we set *eps* to something very small, and *value* to 0.

# 6.920 post calcul safety maximum value

Specifically for safety lifting and piping calculations you can limit the calculated safety factor to this specified maximum *value*. This comes convenient if you get excessive large values in a calculation due to numerical accuracy in the calculation.

In case this record is not specified we set *value* to something very large.

#### 6.921 post calcul safety method method

You can determine with this option how the hydraulic piping and lifting safety should be determined. If you set *method* to **-vertical** the safety factors will be determined using the vertical stress (zz-stress in 3D, yy-stress in 2D, xx-stress in 1D); this is the default, as described in the **post\_calcul** command. Thus you get one value for the safety piping and one value for the safety lifting.

If you set *method* to **-prival** the safety factors will be determined using the three principal stresses (principal stress 0, principal stress 1, principal stress 2). Thus you get three values for the safety piping and three values for the safety lifting.

If you set *method* to **-global** the safety factors will be determined using the three global normal stresses (xx-stress, yy-stress, zz-stress). Thus you get three values for the safety piping and three values for the safety lifting.

Setting *method* to -vertical is the classical definition used in most text books. However since the critical direction in complex calculations will not always be in the vertical direction, it is also of interest to study the hydraulic safety factors with the principal stress values (**prival**). And for some situation, like soil near a retaining wall, it may be of interest to study the hydraulic safety factors with global normal stresses (-global).

In the dbs file after a calculation you can see in the **post\_calcul\_label** record the naming of the calculated hydraulic safety factors.

# 6.922 post calcul static pressure static\_pressure

With this option you can directly specify the static pressure to be used in postprocessing by **post calcul -groundflow pressure -static pressure** for the complete domain.

See also **post\_calcul\_static\_pressure\_height** if you want to specify a static pressure height instead.

# 6.923 post\_calcul\_static\_pressure\_height coord\_min,0 coord\_max,0 height\_ref,0 coord\_min,1 coord\_max,1 height\_ref,1 ...

Using this option the static pressure as required by **post\_calcul -groundflow\_pressure** -static\_pressure is determined relative to the reference height, and not anymore to a groundwater level. Thus, the  $\Delta z$  in the equation for  $p_{\text{static}} = \rho g \Delta z$  is taken relative to the specified reference height in this **post\_calcul static\_pressure\_height** record.

You can specify multiple regions. The first region is between vertical coordinate  $coord\_min, 0$  and  $coord\_max, 0$ . The  $coord\_min, 0$  and  $coord\_max, 0$  themselves are included as part the region. If a node is inside this region the  $height\_ref, 0$  is used as phreatic level height in the equation for the static pressure. The second region is between vertical coordinate  $coord\_min, 1$  and  $coord\_max, 1$ . The  $coord\_min, 1$  and  $coord\_max, 1$  themselves are included as part the region. If a node is inside this region the  $height\_ref, 1$  is used as phreatic level height in the equation for the static pressure.

If a node is not inside any of the regions, and if the groundflow phreatic level itself is not specified, the static pressure cannot be determined and remains zero.

 $See also \ \textbf{post\_calcul\_static\_pressure\_height\_element\_group}.$ 

# **6.924 post\_calcul\_static\_pressure\_height\_element\_group** *element\_group\_0 element\_group\_1* ...

Restrict the regions of **post\_calcul\_static\_pressure\_height** to specific element groups. The region between *coord\_min*, 0 and *coord\_max*, 0 is valid for element group *element\_group\_0*. The region between *coord\_min*, 1 and *coord\_max*, 1 is valid for element group *element\_group\_1*. Etc.

You need to specify an element group for each and every region. As a special option you can specify **-all** for an element group number; then the corresponding region is valid for all element groups.

### 6.925 post count dataitem\_name\_0 dataitem\_name\_1 ...

With this **post\_count** record you can specify data items for which the number of active indices should be counted. The results will be placed in the record **post count result**.

For example count the number of active elements, nodes and geometry points by:

post\_count -element -node -geometry\_point ...

# **6.926** post\_data index dataitem\_name\_0 dataitem\_index\_0 dataitem\_number\_0 dataitem\_name\_1 dataitem\_index\_1 dataitem\_number\_1 ...

The specified data items are taken, and each is multiplied with its corresponding factor in **post\_data\_factor** and added to **post\_data\_result**. This allows you to conveniently follow the sum of data item, each multiplied with some factor.

6.927 post data factor index factor 0 factor 1 ...

See post data.

6.928 post data result index result

See post data.

# 6.929 post\_element\_force index dir\_normal\_x dir\_normal\_y dir\_normal\_z dir\_shear0\_x dir\_shear0\_y dir\_shear0\_z dir\_shear1\_x dir\_shear1\_y dir\_shear1\_z middle\_x middle\_y middle\_z

With this record you can calculate the normal force, shear force and moments in cross sections. Only cross sections at the side of elements are allowed; so that typically is the common side between two elements, or the side at the edge of a domain; a cross section through the interior of elements is not allowed. Below we will describe how you can select elements. For the combination of selected elements nodal forces will be used to determine cross section forces and moments.

The *middle\_x middle\_y middle\_z* should contain the exact middle coordinates of the cross section. You can use this **post\_element\_force** option however also to get the soil forces on the shaft or toe of a pile; then use a *middle\_x middle\_y middle\_z* point below the bottom of the pile.

The nodal force components in the  $dir\_normal\_x \ dir\_normal\_y \ dir\_normal\_z$  direction are summed to give a normal force  $normal\_force$ . The nodal force components in the  $dir\_shear0\_x \ dir\_shear0\_y \ dir\_shear0\_z$  direction are summed to give the first shear force  $shear0\_force$ . The nodal force components in the  $dir\_shear1\_x \ dir\_shear1\_y \ dir\_shear1\_z$ direction are summed to give the second shear force  $shear1\_force$ . The nodal force components in the  $dir\_normal\_x \ dir\_normal\_y \ dir\_normal\_z$  direction are multiplied with the distance in  $dir\_shear0\_x \ dir\_shear0\_y \ dir\_shear0\_z$  direction as measured from the  $middle\_x \ middle\_y \ middle\_z$  vector, and this is summed to give the first bending moment moment0. The nodal force components in the  $dir\_normal\_x \ dir\_normal\_x \ dir\_normal\_z \ direction$  as measured from the direction are multiplied with the distance in  $dir\_shear1\_x \ dir\_shear1\_y \ dir\_normal\_z \ direction$  as measured from the dir\\_normal\\_x \ dir\\_shear1\\_x \ dir\\_shear1\\_z \ direction as measured from the dir\\_shear0\\_x \ dir\\_shear1\\_x \ dir\\_shear1\\_z \ direction as measured from the dir\\_shear0\\_x \ dir\\_shear1\\_x \ dir\\_shear1\\_z \ direction as measured from the dir\\_shear0\\_x \ dir\\_shear0\\_y \ dir\\_shear1\\_z \ direction as measured from the dir\\_shear0\\_x \ dir\\_shear0\\_y \ dir\\_shear0\\_z \ dir\\_shear0

In 3D you need to specify the complete **post\_element\_force** record and you get the normal force, two shear forces and two bending moments in the **post\_element\_force\_result** record. The directions  $dir\_shear0\_x dir\_shear0\_y dir\_shear0\_z$  and  $dir\_shear1\_x dir\_shear1\_y$  $dir\_shear1\_z$  should be perpendicular.

In 2D you need to specify only a partial record **post\_element\_force** as *index dir\_normal\_x dir\_normal\_y dir\_shear0\_x dir\_shear0\_y middle\_x middle\_y* and you get the normal force, one shear force and one bending moment in the **post\_element\_force\_result** record.

In 1D you need to specify only a partial record **post\_element\_force** as *index dir\_normal\_x*  $middle_x$  and you get the normal force in the **post\_element\_force\_result** record.

You can restrict with **post\_element\_force\_geometry** with the same index that the **post\_element\_force** is only evaluated for nodes on a specific geometry. This **post\_element\_force** option always checks if the initial node location is located on this geometry (so not the location of moved nodes after displacement of the nodes).

You can restrict with **post\_element\_force\_group** with the same index that the **post\_element\_force** is only evaluated for certain element groups.

You can restrict with **post\_element\_force\_number** with the same index that the **post\_element\_force** is only evaluated for certain element numbers.

You can restrict with **post\_element\_force\_normal** with the same index that the **post\_element\_force** is only evaluated for elements in positive normal direction *dir\_normal\_x dir\_normal\_y dir\_normal\_z*. If you don't specify **post\_element\_force\_normal** elements on both sides will be used if present.

You can require by setting the *switch* in **post\_element\_force\_force** with the same index that also the external forces (like gravity and edge loads etc.) are added to the result.

You can require by setting the *switch* in **post\_element\_force\_inertia** with the same index that also the inertia forces is added to the result.

If you are not hapy with the sign or units with which the forces are calculated, you can use a multiply factor in **post\_element\_force\_multiply\_factor** with the same index to get what you want.

Please realise that in calculation with groundwater the calculated forces contain the force due to effective stresses and also due to groundwater total pressure (pore pressure).

We now give some examples for a 2D vertical pile driven into the soil in a dynamic inertia ... calculation, and including gravity force\_gravity ... and an external force force\_element\_edge ... at the top of the pile. Below  $x_pile$  is the x-coordinate at the middle of the pile,  $y_pile_middle$  is the y-coordinate at the middle of the pile,  $y_pile_bottom$ is the y-coordinate at the bottom of the pile and *pile group* is the group number of the pile.

The force in a cross section (force resulting from normal stress in cross section):

post\_element\_force 10 0. 1. 1. 0. x\_pile y\_pile
post\_element\_force\_geometry 10 -pile\_cross\_section
post\_element\_force\_group 10 pile\_group
....

. . .

Here *pile\_cross\_section* is a geometry line through the cross section of the pile,

The force along the shaft (force resulting from shear stress along shaft):

post element force 10 0. 1. 1. 0. x pile y pile bottom

```
post_element_force_geometry 10 -pile_shaft
post_element_force_group 10 pile_group
post_element_force_force 10 -yes
post_element_force_inertia 10 -yes
....
```

Here *pile* shaft is a geometry line containing only nodes of the pile shaft,

The force at the pile toe (force resulting from normal stress at pile tip):

```
post_element_force 10 0. 1. 1. 0. x_pile y_pile_bottom
post_element_force_geometry 10 -pile_toe
post_element_force_group 10 pile_group
post_element_force_force 10 -yes
post_element_force_inertia 10 -yes
...
```

Here *pile\_toe* is a geometry line containing only nodes of the pile toe,

The complete force on the pile:

```
post_element_force 10 0. 1. 1. 0. x_pile y_pile_bottom
post_element_force_geometry 10 -pile_complete
post_element_force_group 10 pile_group
post_element_force_inertia 10 -yes
post_element_force_inertia 10 -yes
...
```

Here *pile\_complete* is a geometry line containing all nodes of the pile, Also see the example calculation **force14.dat** and **force17.dat**.

6.930 post element force force index switch

See post element force.

- 6.931 post\_element\_force\_geometry index geometry\_item\_name geometry\_item\_index
- See post element force.
- 6.932 post element force group index element\_group\_0 element\_group\_1

See post element force.

. . .

6.933 post element force inertia index switch

See post element force.

6.934 post element force multiply factor index multiply\_factor

See post element force.

6.935 post element force normal index switch

Set *switch* to -yes if you want to select elements in positive normal direction. See **post** element force.

6.936 post element force number index number\_0 number\_1 ...

See post element force.

6.937 post\_element\_force\_result index normal\_force shear0\_force shear1\_force moment0\_moment1

See post element force.

#### 6.938 post global switch

With this **post\_global** you can ask for global information to be determined if you set *switch* to **-yes**. The following information will then be determined:

- -post\_bounda\_force\_summed (total force following from -bounda\_force records, number of principal dofvalues)
- -post element mass summed (total global mass)
- -post element summed (total number of elements)
- -post element volume summed (total elements volume without empty elements)
- -post group summed (total number of elements in group 0, group 1, etc.)
- -post materi inertia summed (sum of material nodal inertia, so of node inertia)
- -post slide force summed (sum of slide forces in global axes, so of node slide force)
- -post node summed (total number of nodes)
- -post node dof average (average values for dof's)
- -post node dof maximum (maximum values for dof's)
- -post node dof minimum (minimum values for dof's)
- -post\_force\_edge\_summed (total force following from -force\_edge integrated over edges in x,y,z directions, number\_of\_space\_dimensions values)
- -post\_force\_edge\_normal\_summed (total force following from force\_edge\_normal integrated over edges in x,y,z directions, number of space dimensions values)
- -post\_force\_edge\_projected\_summed (total force following from -force\_edge\_projected integrated over edges in x,y,z directions, *number of space dimensions* values)
- -post\_support\_edge\_normal (total force following from -support\_edge\_normal integrated over edges in x,y,z directions, *number\_of\_space\_dimensions* values)
- -post\_solver\_diagonal\_minimum\_value (minimum diagonal term total matrix, only for pardiso solver)
- -post\_solver\_diagonal\_minimum\_node (node number at which the minimum value is found)
- -post\_solver\_diagonal\_maximum\_value (maximum diagonal term total matrix, only for pardiso solver)
- -post\_solver\_diagonal\_maximum\_node (node number at which the maximum value is found)
- -post\_solver\_diagonal\_ratio (ratio maximum/minimum diagonal terms total matrix, only for pardiso solver)
- -post\_solver\_iterations (total number of iterations of iterative linear equation solver, only for bicg solver)

If you set *switch* to **-no** then the information will not be determined (this saves a little bit of computer time). Default, if **post** global is not specified, *switch* to **-yes**.

# 6.939 post\_group\_volume\_summed volume\_group\_0 volume\_group\_1

This record will be filled with the total volume of the elements in each group. So *volume group*  $\theta$  is the summed volume of the elements in element group 0, etc.

**6.940** post\_integrate index data\_item\_name data\_item\_index data\_item\_number

Here you can specify results that should be integrated over time. The integrated results will be placed in the **post integrate result** record with the same index.

An example looks like:

. . .

```
groundflow_pressure
groundflow_velocity
end_initia
...
post_node 1 -average -geometry_line 4
...
post_integrate 3 -post_node_result 1 -gvely
...
```

Here the **post\_node** record first takes care that the average groundflow y-velocity at nodes on a line are determined, among other dof's. The **post\_integrate** record integrates that average groundflow y-velocity over time. In this way the total groundflow debit volume over a line is registered.

#### 6.941 post integrate result index result

See post integrate.

#### **6.942** post line index $x_0 y_0 z_0 x_1 y_1 z_1$

This record specifies a line in space for which the average or sum of the dof values will be calculated. The values are placed in a record **post\_line\_dof** with the same *index*. Internally in TOCHNOG, **post\_point** records are used to evaluate the dof's on the line.

In 1D only  $x_0$  and  $x_1$  should be specified, etc.. In the example below, the average of the *x*-velocity between the points (3,1) and (3,7) will be printed

...
number\_of\_space\_dimensions 2
materi\_velocity
...
end\_data
...
post\_line 1 3. 1. 3. 7.
...
print\_filter 0 -post\_line\_dof 1 -velx
...
control\_timestep 1 1. 100.
control\_print 1 -post\_line\_dof

The coordinates are defined in the initial mesh. See also: **post** line **n** and **post** line **operat**.

# 6.943 post line operat index operat

If *operat* is set to **-average** then the average is calculated for the **post\_line** record with the same index. If *operat* is set to **-sum** then the sum is calculated for the **post\_line** record with the same index.

If this **post** line operat is not specified, then *operat* is set to -average.

**6.944** post line dof  $index dof_0 dof_1 \dots$ 

Average dofvalues at a selected line. See **post** line.

6.945 post line dof calcul...

See post calcul.

**6.946** post line n index n

Use n **post\_point** records to evaluate the dof's along the line. Default n is 5. See **post\_line**.

## 6.947 **post\_node** index data\_item operat geometry\_entity\_name geometry\_entity\_index

If *operat* is set to **-sum**, results for the nodal *data\_item* are summed. For example, you can take for *data\_item* the **-node\_rhside** and for *operat* you take **-sum** so that the total external force on a boundary is determined.

If operat is set to -average, results for the nodal data item are averaged.

If  $data_item$  is set to -**node\_rhside** and *operat* is set to -**moment**, the moment is determined around the origin (x=0,y=0) in a 2D calculation; this is done by calculating in the nodes the tangential force extracted from -**node\_rhside** and multiplying it with the radial distance from the origin.

This operation is done for nodes which are placed on the geometrical entity geometry\_entity\_name geometry\_entity\_index. Instead of a geometrical entity you can also use -all to tell that all nodes should be used. Instead of a geometrical entity you can also use -ra ... -ra to tell that the nodes of the range should be used.

The result of this **post\_node** record is put into the **post\_node\_result** record (with the same *index*).

#### 6.948 post node factor index factor

You can multiply the result of **-post\_node** with *factor*. Default, if **post\_node\_factor** is not specified, we take *factor* equal to 1.

#### 6.949 post node result index result\_0 result\_1 ...

See post node.

# 6.950 post\_node\_rhside fixed value\_0 value\_1 ...

This record will be filled with the sum of **node\_rhside** for those dof's which are prescribed (eg with a **bounda\_dof**). For example, in a calculation with only velocities (displacements) as primary dof's, this record contains the sum of the reaction force at the nodes in which the velocity is prescribed. Values are only filled for principal dof's (materi velocity, groundflow pressure, condif temperature, ...).

#### 6.951 post node rhside free value\_0 value\_1 ...

Same as **post\_node\_rhside\_fixed**, now for free values however. For example, in a calculation with only velocities (displacements) as primary dof's, this record contains the sum of the unbalance force at the nodes in which the velocity is not prescribed.

#### 6.952 post node rhside ratio ratio

This record gives during a calculation a measure for the inaccuracy of the calculation. For each primary dof the ratio between the size of the right-hand-side (nodal forces or nodal fluxes ...) in fixed nodes and free nodes.

If the size for the fixed nodes is below 1.e-10 the *ratio* is directly filled the result for free nodes. See also **post node rhside ratio dof type**.

# **6.953** post node rhside ratio dof type $dof_type_0 \dots$

With this option you can specify a list of doftypes which should be used in the calculation of the **post\_node\_rhside\_ratio** result. For example, if both **groundflow\_pressure** and **condif\_temperature** are initialised, then you can use only the groundflow pressure in the accuracy ratio determination by specifying **post\_node\_rhside\_ratio\_dof\_type**-groundflow pressure.

If **post\_node\_rhside\_ratio\_dof\_type** is not specified and **materi\_velocity** is initialised then automatically **post\_node\_rhside\_ratio\_dof\_type** -**materi\_velocity** will be used.

# 6.954 post node rhside ratio method method

By setting *method* to **-post\_node\_rhside\_free** the *ratio* is directly filled with **post\_node\_rhside\_free**. Default, when this **post\_node\_rhside\_ratio\_method** record is not specified, the default definition as specified in **post\_node\_rhside\_ratio** is used,

#### **6.955** post point index x y z

This record specifies a point in space for which dof values will be calculated. The values are placed in a record **post\_point\_dof** with the same *index*. The values are obtained by determining in which element the point is located and then using the element's interpolation functions. In 1D only x should be specified, etc.. The coordinates are defined in the initial mesh. This option is only available for isoparametric elements.

#### 6.956 post point element group index element\_group

Limit the search for the element in which the post point with the same *index* is located to the specified *element group*.

# **6.957** post point dof index $dof_0 dof_1 \dots$

Unknown values at a selected point. See **post** point.

# 6.958 post point dof calcul...

See post calcul.

#### 6.959 post point eps iso index eps

Tolerance with which a **post\_point** is accepted to be part of an element. The default value is 1.e-3. You can increase the default value if a **post\_point** is exactly on or over the border of the mesh, so that the **post\_point** may be not found; typically try 0.1 or so.

#### 6.960 post point move index switch

If *switch* is set to **-yes** the **post\_point** with the same index will be moved along with the velocity field. Thus, with this option you can follow with a **post\_point** the dof's of a material particle.

Please realise that this option should only be used if **materi\_velocity** is initialised in the initialisation part.

# **6.961** post quadrilateral *index* $x_0 y_0 z_0 x_1 y_1 z_1 x_2 y_2 z_2$ $x_3 y_3 z_3$

This record specifies a quadrilateral in space for which the average of the dof values will be calculated. The values are placed in a record **post\_quadrilateral\_dof** with the same *index*. Internally in TOCHNOG, **post\_point** records are used to evaluate the dof's on the quadrilateral. In 2D only  $x_0 y_0$ ,  $x_1 y_1$ , etc. should be specified. The coordinates are defined in the initial mesh. See also: **post\_quadrilateral\_n**.

#### **6.962** post quadrilateral dof *index* $dof_0 dof_1 \dots$

Average dof values at a selected quadrilateral. See **post** quadrilateral.

# 6.963 post quadrilateral dof calcul...

See post calcul.

### 6.964 post quadrilateral element group index element\_group

Select the specific element group from which the dof values should be taken.

# 6.965 post quadrilateral n index n

Use n **post\_point** records in each direction to evaluate the dof's along the quadrilateral. Default n is 5. See **post\_quadrilateral**.

# 6.966 post strain volume absolute index volume\_increase\_absolute

This record will hold after the calculation the absolute volume increase summed over the elements that are selected in the **strain\_volume\_element**, **strain\_volume\_element\_group** and **strain\_volume\_geometry** records (with the same index).

The actual volume increase which you will find in this **post\_strain\_volume\_absolute** record will depend on the relative volume strain or absolute volume increase that you specified, but also on stiffnesses of neighboring zones, boundary conditions, etc.

You can use this **post\_strain\_volume\_absolute** result to decide to manually change the specified relative volume strain or absolute volume increase and rerun the calculation.

## 6.967 post strain volume initial index volume\_initial

Initial volume of selected elements.

#### 6.968 post strain volume relative index volume strain relative

Relative volume strain percentage. Otherwise the same as **post** strain volume absolute.

#### 6.969 print apply switch

If *switch* is set to **-no**, then all **control\_print\_\*** records will not be applied. Default, if **print\_apply** is not specified, *switch* is set to **-yes**.

#### 6.970 print arithmetic switch

If *switch* is set to **-yes**, all evaluated arithmetics will be printed. See the start of the data part for an explanation about arithmetics. The printing will be done to the file **tochnog arithmetic.txt**.

#### 6.971 print control switch

If *switch* is set to **-yes**, the control index being evaluated will be printed. Handy for keeping track on what the program is doing.

### 6.972 print data name switch

If *switch* is set to **-yes**, all possible data names will be printed. The printing will be done to the file **tochnog data name.txt**.

This is convenient to search in the **tochnog\_data\_name.txt** file fast for options. For example under linux to search all options which have the word **group** in it do **grep group tochnog data name.txt**.

The possible data names may include also internal names that Tochnog uses during the calculation; so for each name you can check this users manual if it is a name that you can use as input in the input file, or not.

# 6.973 print\_database\_calculation switch

If *switch* is set to **-yes**, the database will be written after successful completion of a calculation to the file *name.dbs*, where *name* is the name of the input file. If *switch* is set to **-no**, the database will not be written.

Default, *switch* is set to **-yes**.

# 6.974 print define switch

If *switch* is set to **-yes**, all evaluated defines will be printed. See the start of the data part for an explanation about defines. The printing will be done to the file **tochnog define.txt**.

#### 6.975 print element geometry present switch

See element \_geometry \_present. See also print \_element \_geometry \_present \_node \_type. Default *switch* is set to -no.

# 6.976 print\_element\_geometry\_present\_node type node\_type

If *node\_type* is set to **-node\_start\_refined** the values of **-node\_start\_refined** are used. If *node\_type* is set to **-node** the values of **-node** are used. If *node\_type* is set to **-plus\_displacement** the values of **-node** plus nodal displacements. Default *node\_type* is set to **-node** start refined.

#### 6.977 print failure switch

If *switch* is set to **-yes** then failure of elements due to one of the failure criteria (**group\_materi\_failure\_rupture**, etc.) will be reported.

# **6.978 print\_filter** *index data\_item\_name data\_item\_index number\_0 number\_1* ...

The data selected in the records control print, control print dof, control print dof rhside and control print element will be filtered at output. Thus only a limited amount of data will actually be printed. Here data item name is the name of the data item to be filtered, e.g. data\_item\_name is -node dof. data\_item\_index is the index of the data item name record which passes the filter. If, for example, data item index is 3 then only index 3 passes the filter. If data item index is -all then all indices pass the filter. If, for example, data\_item\_index is -geometry line 3 (valid if data\_item\_name is -node or another nodal item) then only records with coordinates located on line 3 pass the filter. If, for example, data item index is -geometry line 3 (valid if data item name is -element or another element item) then only element with at least one coordinate located on line 3 pass the filter. If, for example, data item index is -ra ... -ra then indices in this range pass the filter. If, for example, data\_item\_index is -macro 4 and data\_item\_name is data valid at a node (or element), then only nodes (or elements) generated by the macro number 4 pass the filter (see control mesh macro \* for macro's). If, for example, data item index is -macro -none and *data* item name is data valid at a node (or element) then only nodes (or elements) not generated by any macro pass the filter (see control mesh macro \* for macro's).

For example, if  $number_0$  is 3 then the fourth value of a record passes the filter. If  $num-ber_0$  is -all the whole record passes the filter. If, for example,  $number_0$  is -velx while data item name is -node dof then only x-velocities pass the filter.

Some examples are

print\_filter 1 -node\_dof -all -temp -sigxx (temperatures and xx-stresses)
print\_filter 2 -node -geometry line 3 0 (x-coordinates on line 3)

With control \_print \_filter you can select if the records control \_print, control \_print \_dof or control \_print \_dof \_rhside (with the same index) should use specific filters (specify the indices of the filter for *print\_filter\_index*), should use all filters (specify -all for *print\_filter\_index*), or should use no filter at all (specify -none for *print\_filter\_index*). Default, if control \_print\_filter is not specified, all filters will be used for a print option.

Example:

print \_ filter 1 -node \_ dof ...
print \_ filter 2 -node \_ dof \_ all ...
print \_ filter 3 ...
...
control \_ print \_ dof 10 ...
control \_ print \_ filter 10 1 2 (only use filter 1 and 2)
...
control \_ timestep 20 ...
control \_ print 20 ...
control \_ print \_ filter 20 - all (use all filters)

All used filters are placed in-line for a data item; thus only data which passes all used filters for that data item will be printed.

## 6.979 print gid calculation switch

If you set *switch* to **-yes** the gid files will be printed at the end of the calculation. If you set *switch* to **-no** the gid files will not be printed at the end of the calculation. Default, if *switch* is not specified, it is set to **-yes**.

6.980 print frd freecad switch

See **control\_print\_frd\_freecad**, but now for all frd printing however. If both **control\_print\_frd\_freecad** and **print\_frd\_freecad** are present, the **control\_print\_frd\_freecad** dictates what happens for the specific control index.

If both print\_frd\_freecad and control\_print\_frd\_freecad are not specified the *switch* is set to -no.

#### 6.981 print frd prepomax switch

See control\_print\_frd\_prepomax, but now for all frd printing however. If both control\_print\_frd\_prepomax and print\_frd\_prepomax are present, the control\_print\_frd\_prepomax dictates what happens for the specific control index.

If both print frd prepomax and control print frd prepomax are not specified the *switch* is set to -no.

## 6.982 print gid contact spring2 number\_of\_nodes

Set *number\_of\_nodes* to 2 if you want to draw **contact\_spring2** with two nodes, and to 1 if you want to draw **contact\_spring2** with one node. Default, if **print\_gid\_contact\_spring2** is not specified, then 1 is used for *number\_of\_nodes*.

# 6.983 print gid coord switch

If *switch* is set to **-yes** the coordinates of nodes is plotted in gid.

## 6.984 print gid define switch

If *switch* is set to **-yes** defined names will be used i.s.o. geometry names (when a geometry is defined as a name with **start\_define** ... **end\_define**). If *switch* is set to **-no** that will not be done. Default, if *switch* is not specified, *switch* is set to **-yes**.

## 6.985 print gid group switch

If *switch* is set to **-yes** the element groups are plotted in gid as result field; in gid you can do a **contour\_fill** to visualize this result field. If *switch* is set to **-no** the element groups are not plotted in gid as result. Default *switch* is set to **-yes**.

#### 6.986 print gid mesh activate gravity switch

See also mesh activate gravity time.

# 6.987 print gid node method method

See print gid node method.

#### 6.988 print gid spring2 number\_of\_nodes

Set *number\_of\_nodes* to 2 if you want to draw **spring2** with two nodes, and to 1 if you want to draw **spring2** with one node. Default, if **print\_gid\_spring2** is not specified, then 1 is used for *number\_of\_nodes*.

# **6.989** print gid smooth dof $dof_0 dof_1 \dots$

This option allows you to smooth results in gid files. With  $dof_0 dof_1 \dots$  you specify the dof's to be smoothed. As a special option you can specify **-all** so that all dof's will be smoothed.

The smoothing is done a number of times, with increasing smooth results. You can specify this number of times with the optional **print\_gid\_smooth\_n**; if you don't specify this optional number of smoothings it is done 10 times.

## 6.990 print gid smooth n index number\_of\_smoothings

See print gid smooth dof.

## 6.991 print group data dataitem\_name\_0 dataitem\_name\_1 ...

Print in the gid files **group\_\*** data items for isoparametric finite elements. As a typical example use **-group\_materi\_elasti\_young**; then you get in the gid plot what the young model distribution is for isoparametric finite elements in the mesh.

For different values of group data in integration points, the value of the last integration point will be used.

For elements which do not have a specific group data item a value 0 will be plotted. Tochnog sets the gid group data information in the timesteps, so only after timesteps have been taken you will see meaningful results for the group data in gid plots.

The values will also be placed in the **element** print group data records.

### 6.992 print gmsh calculation switch

If you set *switch* to **-yes** the gmsh files will be printed at the end of the calculation. If you set *switch* to **-no** the gmsh files will not be printed at the end of the calculation. Default, if *switch* is not specified, it is set to **-no**.

#### 6.993 print gmsh dummy switch

See **control\_print\_gmsh\_dummy**. This **print\_gmsh\_dummy** holds for all gmsh printing, unless it is overruled by a **control print gmsh dummy**.

#### 6.994 print gmsh node method method

See print gmsh node method.

#### **6.995** print mesh dof $dof_0 dof_1 \dots$

This option allows you to print results for dof's (temperatures, groundflow pressures, ...) in a first calculation and use these results later in a second calculation as boundary conditions. This comes handy when you need to run the second calculation multiple times, and the results for the printed dof's can be taken from the first calculation. In this way, the computing time of the second calculation can be smaller, and also a different FE mesh can be used in the first calculation and the second calculation for the different doffields.

In the first calculation you can print the dof's with the command **print\_mesh\_dof**; the results will be printed in the file **print\_mesh\_dof.txt**. The *dof\_0 dof\_1* ... of **print\_mesh\_dof** specify the dof's which will be printed. In the first calculation printing of the dof's to the file **print\_mesh\_dof.txt** will actually be done for when *switch* is set to **-yes** in **control print mesh\_dof**.

For the second calculation rename the file **print\_mesh\_dof.txt** into **bounda\_mesh\_dof.txt**. You can specify which of the dof's in the file **bounda\_mesh\_dof.txt** will actually be used a prescribed value ('boundary condition') with the *dof\_0 dof\_1*... of **bounda\_print\_mesh\_dof**. You can restrict the nodes to which this will be done by **bounda\_print\_mesh\_dof\_geometry** (please realise using a geometry point with a very large tolerance in combination with **geometry\_element\_group** you can effectively select the geometry formed by an element group).

The FE meshes as used in the first calculation and in the second calculation need not be the same, and are also allowed to vary in time (in building processes, excavations, etc.). Nodes from the second mesh will be located in the first mesh, and doffields will be interpolated from the first mesh to the second mesh. In case a node from the second mesh is not inside any isoparametric element of the first mesh, the value for the dof's as specified in the optional **bounda\_print\_mesh\_dof\_values** will be used. In **bounda\_print\_mesh\_dof\_values** you need to specify values for each and every dofthat was specified with **print\_mesh\_dof** in the first calculation. If the node of the second mesh cannot be found in the first mesh and also **bounda\_print\_mesh\_dof\_values** is not specified then the dof's will be taken from the closest node of the first mesh.

Results for the second mesh will be linearly interpolated in time from results of the first mesh.

Example first calculation in which only a temperature field is calculated:

```
echo -yes
number_of_space_dimensions 2
condif_temperature
end_initia
...
print_mesh_dof -temp
...
control_timestep 10 ...
control_print_mesh_dof 10 -yes (print in print_mesh_dof.txt)
...
```

Example second calculation in which the temperature field calculated in the first calculation is imposed:

```
echo -yes

number_of_space_dimensions 2

condif_temperature

materi_velocity

materi_displacement

materi_stress

end_initia

...

bounda_print_mesh_dof -temp

bounda_print_mesh_dof_values 20. (read from bounda_mesh_dof.txt)

...
```

6.996 print node geometry present switch

See **node\_geometry\_present**. See also **print\_node\_geometry\_present\_node\_type**. Default *switch* is set to **-no**.

# 6.997 print\_node\_geometry\_present node type node\_type

If *node\_type* is set to **-node\_start\_refined** the values of **-node\_start\_refined** are used. If *node\_type* is set to **-node** the values of **-node** are used. If *node\_type* is set to **-node** the values of **-node** are used. If *node\_type* is set to **-plus\_displacement** the values of **-node** plus nodal displacements. Default *node\_type* is set to **-node** start refined.

#### 6.998 print precision number\_of\_values

With *number\_of\_values* you can set for all printing how many values will be used at printing. For example, setting *number\_of\_values* to 4 the internal tochnog double **98.123456789** will be printed as **98.12** when using **control print**, **control print** gid etc.

# 6.999 print tecplot calculation switch

If you set *switch* to **-yes** the tecplot files will be printed at the end of the calculation. If you set *switch* to **-no** the tecplot files will not be printed at the end of the calculation. Default, if *switch* is not specified, it is set to **-yes**.

#### 6.1000 print vtk calculation switch

If you set *switch* to **-yes** the vtk files will be printed at the end of the calculation. If you set *switch* to **-no** the vtk files will not be printed at the end of the calculation. Default, if *switch* is not specified, it is set to **-yes**.

# 6.1001 print\_vtk\_coord switch

If switch is set to -yes the coordinates of nodes is plotted in vtk.

## 6.1002 print vtk group switch

If *switch* is set to **-yes** the element groups are plotted in vtk as result field. If *switch* is set to **-no** the element groups are not plotted in vtk as result. Default *switch* is set to **-yes**.

# 6.1003 print vtk node method method

See print vtk node method.

#### 6.1004 print where switch

If *switch* is set to **-yes**, information will be printed about the taks that tochnog is performing (evaluation boundary conditions, loop over elements, etc.).

This is convenient for very large calculations, to see what is being done and how the calculation proceeds.

#### 6.1005 processors nproc

With this record you can set the number of shared memory CPUs you want to use (*nproc*). If your TOCHNOG implementation does not allow for more processors, this record is ignored. In fact, not the number of processors but the number of threads is set (that is, if you use 2 threads while your system only supports 1 processor than those threads are split over that single processor).

Error messages may become confusing when you use more than one processor.

Default n proc is 1.

#### 6.1006 processors maximum switch

If *switch* is set to **-yes**, the **processors** record will be set to the maximum number as allowed by your computer.

Default *switch* is set to **-yes**. This **processors\_maximum** record will not be used if the **processors** record is specified.

## 6.1007 processors partition npartition

The element loop is parallised as follows. The master process gives away small amounts of the total number of elements to child processes, and if a child process is ready it gets a new small amount of the master process. In fact, a child process gets each time an amount of  $\frac{nelement}{npartion*processors}$  where *nelement* is the number of elements, *npartition* is specified in **processors** \_ **partition**, and *processors* is specified in **processors**. Default, if **processors partition** is not specified, we set *npartition* to 1.

6.1008 repeat save result index result\_0 result\_1 ...

See **control\_repeat\_save**. The index is the number of repetition (index 0 is repeat 0, index 1 is repeat 1, etc.)

# **6.1009** repeat save calculate result average 0 variance 0 average 1 variance 1 . . .

See control repeat save calculate.

#### **6.1010** safety slip circle grid middle index $x_first y_first x_last y_last$

This record specifies a grid with middles of a circle for safety factor calculations. With  $x_{first} y_{first}$  you specify the first middle. With  $x_{last} y_{last}$  you specify the last middle. With safety\_slip\_circle\_grid\_middle\_n you specify the number of middles that should be evaluated in the safety calculation; all middles together form a equidistant grid between  $x_{first} y_{first}$  and  $x_{last} y_{last}$ .

As a special option you can only specify  $x_{first} y_{first}$  and not specify safety  $slip_circle_grid_middle_n$ ; then only one middle  $x_{first} y_{first}$  will be evaluated for the circle in the safety calculation.

See also control safety slip.

# 6.1011 safety slip circle grid middle n index n

See safety\_slip\_circle\_grid\_middle.

# 6.1012 safety slip circle grid radius index r\_first r\_last

This record specifies the radius of a circle for safety factor calculations.

With  $r\_first$  you specify the first radius. With  $r\_last$  you specify the last radius. With **safety\_slip\_circle\_grid\_radius\_n** you specify the number of radius that should be evaluated in the safety calculation; all radius to be evaluated will be put equidistant between r first and r last.

As a special option you can only specify  $r_{first}$  and not specify safety  $slip_circle_grid_radius_n$ ; then only one radius  $r_{first}$  will be evaluated for the circle in the safety calculation.

# 6.1013 safety slip circle grid radius n index n

See safety slip circle grid radius.

#### 6.1014 safety slip circle grid result index x y r safety\_factor

This record will after the calculation be filled with the middle, radius and safety factor for the critical surface (for the slip circles with the same index).

#### 6.1015 safety slip circle grid segment n index n

With this record you can specify how many segments in the circle will be used in the integration of the safety factor. A high number of segments gives more accuracy but is time consuming. A low number of segments is less accurate but fast. Default, if **safety slip circle grid segment n** is not specified, then 90 segments will be used.

# **6.1016** safety slip circle line middle $index x_first y_first x_last y_last$

This record specifies a line with middles of a circle for safety factor calculations. With  $x_{first} y_{first}$  you specify the first middle. With  $x_{last} y_{last}$  you specify the last middle. With safety\_slip\_circle\_line\_middle\_n you specify the number of middles that should be evaluated in the safety calculation; all middles together form a equidistant line between  $x_{first} y_{first}$  and  $x_{last} y_{last}$ .

As a special option you can only specify  $x_{first} y_{first}$  and not specify safety  $slip_circle_line_middle_n$ ; then only one middle  $x_{first} y_{first}$  will be evaluated for the circle in the safety calculation.

See also control\_safety\_slip.

#### 6.1017 safety slip circle line middle n index n

See safety slip circle line middle.

# 6.1018 safety slip circle line radius index r\_first r\_last

This record specifies the radius of a circle for safety factor calculations.

With  $r\_first$  you specify the first radius. With  $r\_last$  you specify the last radius. With **safety\_slip\_circle\_line\_radius\_n** you specify the number of radius that should be evaluated in the safety calculation; all radius to be evaluated will be put equidistant between  $r\_first$  and  $r\_last$ .

As a special option you can only specify  $r_{first}$  and not specify safety  $slip_circle_line_radius_n$ ; then only one radius  $r_{first}$  will be evaluated for the circle in the safety calculation.

# 6.1019 safety slip circle line radius n index n

See safety\_slip\_circle\_line\_radius.

#### 6.1020 safety slip circle line result index x y r safety\_factor

This record will after the calculation be filled with the middle, radius and safety factor for the critical surface (for the slip circles with the same index).

# 6.1021 safety slip circle line segment n index n

With this record you can specify how many segments in the circle will be used in the integration of the safety factor. A high number of segments gives more accuracy but is time consuming. A low number of segments is less accurate but fast. Default, if **safety slip circle line segment n** is not specified, then 90 segments will be used.

# 6.1022 **safety\_slip\_combined\_linear** *index* $x_first, 0 y_first, 0 x_first, 1 y_first, 1 \dots x_last, 0 y_last, 0 x_last, 1 y_last, 1 \dots$

This record specifies combined linear lines along which the safety factor should be calculated.

All data with *first* specifies the first combined linear line. The  $x\_first, 0 y\_first, 0$  specifies the first point of the first line piece of the first combined linear line, the  $x\_first, 1 y\_first, 1$  specifies the second point of the first line piece of the first combined linear line The  $x\_first, 2$   $y\_first, 2$  specifies the first point of the second line piece of the first combined linear line, the  $x\_first, 3 y\_first, 3$  specifies the second point of the second point of the second line piece of the first combined linear line, the  $x\_first, 3 y\_first, 3$  specifies the second point of the second line piece of the first combined linear line the second line piece of the first combined linear line, the  $x\_first, 3 y\_first, 3$  specifies the second point of the second line piece of the first combined linear line.

All data with *last* specifies the last combined linear line. The  $x\_last, 0$   $y\_last, 0$  specifies the first point of the first line piece of the last combined linear line, the  $x\_first, 1$   $y\_first, 1$  specifies the second point of the first line piece of the last combined linear line The  $x\_last, 2$   $y\_last, 2$  specifies the first point of the second line piece of the last combined linear line, the  $x\_first, 3$   $y\_first, 3$  specifies the second point of the second point of the second line piece of the last combined linear line, the  $x\_first, 3$   $y\_first, 3$  specifies the second point of the second line piece of the last combined linear line the second linear line etc. This last combined linear line should have an equal number of points as the first combined linear line.

With **safety\_slip\_combined\_linear\_n** you specify the number of combined linear lines that should be evaluated in the safety calculation; all combined linear lines to be evaluated will be put equidistant between the first combined linear line and the second combined linear line.

As a special option you can only specify data for the first combined linear line, and specify not data for the last combined linear line and not **safety\_slip\_combined\_linear\_n**; then only one combined linear line will be used.

See also control safety slip.

#### 6.1023 safety slip combined linear n index n

See safety slip combined linear.

# 6.1024 safety slip combined linear result index $x_0 y_0 x_1 y_1$ ...safety factor

This record will after the calculation be filled with the combined linear line for the critical surface (for the combined linear lines circles with the same index).

## 6.1025 safety slip combined linear segment n index n

With this record you can specify how many segments in a line piece of a combined linear line will be used in the integration of the safety factor. A high number of segments gives more accuracy but is time consuming. A low number of segments is less accurate but fast. Default, if **safety\_slip\_combined\_linear\_segment\_n** is not specified, then 10 segments will be used.

6.1026 safety\_slip\_ellipsoide index middle\_x\_first middle\_y\_first middle\_z\_first base1\_x\_first base1\_y\_first base1\_z\_first base2\_x\_first base2\_y\_first base2\_z\_first a\_first b\_first c\_first middle\_x\_last middle\_y\_last middle\_z\_last base1\_x\_last base1\_y\_last base1\_z\_last base2\_x\_last base2\_y\_last base2\_z\_last a\_last b\_last c\_last

This record specifies a 3D ellipsoide for which the safety factor should be calculated. The ellipsoide equation is  $\frac{x^2}{a^2} + \frac{y^2}{b^2} + \frac{z^2}{c^2} = 1$ , where x, y and z are local coordinates in the ellipsoide. The ellipsoide is specified by 12 parameters in tochnog.

All parameters with first specifies the first ellipsoide. The middle\_x\_first middle\_y\_first middle\_z\_first specifies the ellipsoide middle (for which the local coordinates are 0; x = 0, y = 0, z = 0). The base1\_x\_first base1\_y\_first base1\_z\_first specifies the direction of the local x axes in space. The base2\_x\_first base2\_y\_first base2\_z\_first specifies the direction of the local y axes in space. Tochnog determines automatically the direction of the local z axes in space. The a b c specifies the radii in respective the local x, y and z direction.

All parameters with *last* specifies the last ellipsoide.

With **safety\_slip\_ellipsoide\_n** you specify the number of variations that should be used for each of the specified ellipsoids parameters. All parameters will be interpolated between the values specified for the first and last ellipsoide. In case you want to keep a parameter fixed, thus it should not be varied, simply specify an equal value for the parameter in the first and last ellipsoide. As a special option you can only specify parameters for the first ellipsoide, and specify not parameters for the last ellipsoide.

See also control safety slip.

#### 6.1027 safety slip ellipsoide method index method

The normal on the ellipsoide surface is uniquely defined, so that the normal stresses are uniquely defined. The slip direction in the surface is not uniquely defined however. Below several possibilities are listed.

If *method* is set to **-safety\_slip\_ellipsoide**, then the ellipsoide local x direction will be used as slip direction (to be more precise, the projections on the ellipsoide surface will be used everywhere).

If *method* is set to **-materi\_displacement** or **-materi\_velocity\_integrated**, then the last calculated displacements will be used as slip direction (to be more precise, the projections on the ellipsoide surface will be used everywhere).

If *method* is set to **-materi\_velocity**, then the last calculated velocities will be used as slip direction (to be more precise, the projections on the ellipsoide surface will be used everywhere).

Default, if safety slip ellipsoide method is not specified, method is set to -safety slip ellipsoide.

# 6.1028 safety slip ellipsoide n index n

See safety slip ellipsoide.

# 6.1029 safety\_slip\_ellipsoide\_result index middle\_x middle\_y middle\_z base1\_x base1\_y base1\_z base2\_x base2\_y base2\_z a b c safety\_factor

This record will after the calculation be filled with the ellipsoide for the critical surface and the safety factor.

## 6.1030 safety slip ellipsoide segment n index n

With this record you can specify how many segments in an ellipsoide will be used in the integration of the safety factor. The ellipsoide is internally in tochnog integrated in a local  $\phi$  and  $\theta$  direction, over **safety\_slip\_combined\_linear\_segment\_n** segments each. A high number of segments gives more accuracy but is time consuming. A low number of

segments is less accurate but fast. Default, if **safety\_slip\_ellipsoide\_n** is not specified, then 90 segments will be used.

### 6.1031 safety slip grd index switch

If *switch* is set to **-yes**, Tochnog will read a slip surface from the file *index.***grd**. The file is in .grd format, as used by the **surfer** program from **Golden Software**. Thus the format is:

DSAA nx ny xmin xmax ymin ymax zmin zmax ... (for first y specify z values for all x) ... (for second y specify z values for all x) ...

This safety slip grd is only available in 3D.

# 6.1032 safety\_slip\_grd\_method index method

With this record you can specify with which method the slip direction is chosen (this is the direction in which the slip shear force will be determined, to calculate the safety factor).

If *method* is set to -safety\_slip\_grd\_direction the direction specified in -safety\_slip\_grd\_direction will be used. If *method* is set to -materi\_velocity the last calculated -materi\_velocity directions will be used. If *method* is set to -materi\_displacement the last calculated materi\_displacement directions will be used. If *method* is set to -materi\_velocity\_integrated the last calculated -materi\_velocity\_integrated directions will be used. If somewhere the direction is not specified by the above, because the used direction is a null vector, then Tochnog will ask you to specify safety\_slip\_grd\_method\_direction additionally, and then that direction will be used there.

Default, if safety\_slip\_grd\_method is not specified, method is set to -safety\_slip\_grd\_direction.

# 6.1033 safety slip grd method direction index dir\_x dir\_y dir\_z

See safety slip grd method.

#### 6.1034 safety slip grd segment n index n

With this record you can specify how many segments in each part of the surface of the grd file will be used in the integration of the safety factor. In total the surface has  $nx^*ny$  parts; each of these parts will be integrated with  $n^*n$  segments. Default, if **safety\_slip\_grd\_segment\_n** is not specified, then n will be set to 10.

# **6.1035** safety slip multi linear index $x_{first,0}$ y first, 0 x first, 1 y first, 1 ... x last, 0 y last, 0 x last, 1 y last, 1 ...

This record specifies multi linear lines along which the safety factor should be calculated.

All data with *first* specifies the first multi linear line. The  $x_first$ ,  $\theta y_first$ ,  $\theta$  specifies the first point of the first line piece of the first multi linear line, the  $x_first$ ,  $1 y_first$ , 1 specifies the second point of the first line piece of the first multi linear line which is also the first point of the second line piece of the first multi linear line, etc.

All data with *last* specifies the last multi linear line. The  $x\_last, 0 y\_last, 0$  specifies the first point of the first line piece of the last multi linear line, the  $x\_first, 1 y\_first, 1$  specifies the second point of the first line piece of the last multi linear line which is also the first point of the second line piece of the last multi linear line, etc. This last multi linear line should have an equal number of points as the first multi linear line.

With **safety\_slip\_multi\_linear\_n** you specify the number of multi linear lines that should be evaluated in the safety calculation; all multi linear lines to be evaluated will be put equidistant between the first multi linear line and the second multi linear line.

As a special option you can only specify data for the first multi linear line, and specify not data for the last multi linear line and not **safety\_slip\_multi\_linear\_n**; then only one multi linear line will be used.

See also control safety slip.

### 6.1036 safety slip multi linear n index n

See safety slip multi linear.

**6.1037** safety slip multi linear result index  $x_0 y_0 x_1 y_1 \dots safety_factor$ 

This record will after the calculation be filled with the multi linear line for the critical surface (for the multi linear lines circles with the same index).

#### 6.1038 safety slip multi linear segment n index n

With this record you can specify how many segments in a line piece of a multi linear line will be used in the integration of the safety factor. A high number of segments gives more accuracy but is time consuming. A low number of segments is less accurate but fast. Default, if **safety\_slip\_multi\_linear\_segment\_n** is not specified, then 10 segments will be used.

# 6.1039 safety slip set index index\_0 index\_1 index\_1 ...

This records defines the indices of safety geometries belong to a set. For all safety geometries of a set, the minimal safety factor will be determined.

As a special option you can also define a range.

# 6.1040 safety slip set result index index safety\_factor

This record will be filled after the calculation with the minimal safety factor of the geometries in the set.

#### 6.1041 slide axisymmetric index switch

If *switch* is set to **-yes** the slide defined by the **slide geometry** is axisymmetric.

#### 6.1042 slide geometry index geometry entity geometry entity index

This record generates slide friction forces when a material slides over the geometry specified by geometry\_entity\_entity\_index.

This option comes handy when it is a priori known at which nodes sliding will occur, which is typically the case in an Eulerian calculation.

Also slide plasti friction should be specified.

See also **node** slide.

# 6.1043 slide plasti friction index phi c

This record specifies friction for the **slide** geometry option. The maximum friction force between the material and the side surface equals  $c + F_n * tan(phi)$  where c is the cohesion, *phi* is the friction angle in radians and  $F_n$  is the normal force.

The plastic yield function will become available in the records  $\mathbf{node}$  slide f.

# **6.1044** slide plasti tension *index sig\_t*

This record specifies maximum tensile force for the **slide\_geometry** option.

The plastic yield function will become available in the records  $\mathbf{node\_slide\_f}$ .

#### 6.1045 slide user index switch

If switch is set to -yes the user supplied routine for slide friction is called.

See also the file **user.cpp** in the distribution.

# 6.1046 slide damping index damping\_n damping\_t

This specifies the normal damping and tangential damping for sliding. See also **control slide damping apply**.

#### **6.1047** slide stiffness index stiffness\_n stiffness\_t

This specifies the normal stiffness and tangential stiffness for sliding.

The slide directions will become available in the records **node\_slide\_direction**. The slide forces will become available in the records **node\_slide\_force**.

See also control slide stiffness apply.

#### 6.1048 solver solver type

You can set here the solver type to one of solvers as specified in **control\_solver**. The solver set here holds for the entire calculation (as opposed to the **control\_solver** which only holds for the corresponding time steps). In fact, each **control\_solver** will be overwritten by a specified **solver**.

When using the bicg solver, consider also setting **solver\_matrix\_symmetric** to **-yes**, in order to speed up the speed of the bicg solver.

#### 6.1049 solver bicg error error

With *error* you set the termination error ratio between the initial and final error in the bicg iterations.

See also control solver bicg error.

#### 6.1050 solver bicg restart nrestart

With *nrestart* you set the number of restarts in the bicg iterations.

See also control solver bicg restart.

#### 6.1051 solver bicg stop switch

If *switch* is set to **-yes**, the calculation is stopped if the bicg solver does not converge. If *switch* is set to **-no**, the calculation is not stopped if the bicg solver does not converge.

See also control solver bicg stop.

#### 6.1052 solver matrix save switch

If *switch* is set to **-yes**, the solver saves and applies the decomposed matrix, but not in case Tochnog thinks for some reason that the matrix needs to be decomposed at each timestep. This can save CPU time, since further decompositions of the matrix are not required anymore (only backsubstitution to find the solution vector).

If *switch* is set to **-no**, the solver does not save the decomposed matrix.

If *switch* is set to **-always**, the solver saves and applies the decomposed matrix, even in case Tochnog thinks for some reason that the matrix needs to be decomposed at each timestep.

This option is only available in combination with the pardiso solver. See also control solver matrix save.

#### 6.1053 solver matrix symmetric switch

If *switch* is set to **-yes** then, if needed, matrices are symmetrized so that less memory will be needed and a symmetrical equation solver can be used.

#### 6.1054 solver pardiso ordering ordering

See also control solver pardiso ordering.

#### 6.1055 solver pardiso out of core switch

If *switch* is set to **-yes** the pardiso solver is called with a 'out of core' option. See for further the pardiso solver in the the intel mkl library.

#### 6.1056 solver pardiso processors nproc

Set the number of processors to be used by the pardiso solver. Only 1 or the maximum number of the computer is allowed, nothing in between. The pardiso solver in the intel mkl library can have problems with parallelisation; so if you use this option check your results.

Default *nproc* is set to 1 (no parallelisation).

## 6.1057 solver pardiso processors maximum switch

If *switch* is set to **-yes** the maximum number of processors will be used for the pardiso solver. Else only 1 processor will be used for the pardiso solver. See also **solver pardiso processors**.

# 6.1058 strain\_settlement\_parameters index time\_global, start time\_plus reference creep strain rate reference time power n lateral factor

With this option data items you can specify an extra vertical settlement creep strain. Think of geotechnics soil dumping as a typical example where after dumping some extra vertical straining shows up over time.

The vertical settlement creep strain of a soil particle is assumed to be:

$$\dot{\epsilon_{zz}} = \frac{\dot{\epsilon}_r}{1 + \left(\frac{t - t_{plus}}{t_r}\right)^n}$$

The user supplied parameters are:  $\dot{\epsilon}_r$  as reference creep rate,  $t_{plus}$ ,  $t_r$  as reference time and n as power constant. Creep strain starts when the global time  $t_{global}$  reaches time\_global, start. So  $time_{age,start}$  can be used to set where in the creep strain curve the material will start with creeping. The time t in this equation is the time elapse after the material has become active (so the time after dumping the material, which typically is different for each finite element).

The horizontal creep strains  $\dot{\epsilon_{xx}} = f\dot{\epsilon_{zz}}$  and  $\dot{\epsilon_{yy}} = f\dot{\epsilon_{zz}}$  are assumed to be a lateral factor f times the vertical creep strain.

This strain \_settlement \_parameters should be combined with the mesh \_gravity \_activate \_ time option as follows:

```
mesh_activate_gravity_time 10 ...
mesh_activate_gravity_time_strain_settlement 10 -yes
strain_settlement_parameters 20 ...
...
```

The mesh\_activate\_gravity\_time\_strain\_settlement indicates that the mesh activation should not be used by itself, but is only used to determine element activation times needed for the strain settlement parameters option.

See also strain settlement element group.

6.1059 strain\_settlement\_diagram index dof\_0 value\_0 dof\_1 value\_1

With the records strain\_settlement\_diagram, strain\_settlement\_diagram\_dof and strain\_settlement\_diagram\_number you can specify parameters of strain\_settlement\_parameters to be dependend on dof values (strains, stresses, ...) or calculated dof values (total pressures, ....).

In strain settlement diagram dof you can set a name of dof label (eg -sigyy) or of post calcul label (eg -to pres).

In strain\_settlement\_diagram\_number you can set the number of the parameter of strain\_settlement\_parameters to be made dependend (eg 1 for *time\_plus*, 2 for *reference creep strain rate* etc.). You are not allowed to specify number 0 (so for *time\_global,start*).

In strain\_settlement\_diagram you can set the dependency as sets dof-value. If the dof value is outside the diagram the parameter of the strain settlement will not change.

# 6.1060 strain settlement diagram dof dof

See strain settlement diagram.

#### 6.1061 strain settlement diagram number number

See strain settlement diagram.

# 6.1062 strain\_settlement\_element\_group index element\_group\_0 element\_group\_1 ...

This record specifies the element groups for which the **strain\_settlement\_parameters** with the same parameters will be used. As a special option you can use **-all**, such that all elements groups will be used.

# **6.1063 strain\_volume\_absolute\_time** *index time\_0 volume\_increase\_absolute\_0 time\_1 volume\_increase\_absolute\_1* ...

See strain volume element.

#### 6.1064 strain volume element index element\_0 element\_1 ...

With the **strain\_volume\_\*** data items you can specify an extra volumetric strain component which Tochnog should add to specified elements, element groups or a geometry. Think of geotechnics grouting as a typical example.

If you specify strain\_volume\_\* then  $\frac{1}{3}$  of the specified volumetric strain is imposed as xx-strain, yy-strain and zz-strain. The actual straining of elements in the calculation will depend on boundary conditions, external forces, etc. For example, if you have a calculation where all displacements are suppressed by boundary conditions, the actual deformation of elements is null, and the specified strain\_volume\_\* only leads to stresses. For example, if you have a 3d calculation where elements are free to deform the actual deformation of elements will become equal to the specified strain\_volume\_\*. For example, if you have a 2d plain strain calculation where elements are free to deform the actual deformation of elements will be smaller than specified strain\_volume\_\* (since there is no deformation possible in the third direction).

Use **strain\_volume\_element** to specify element numbers for which the volume strain should be applied. Use **strain\_volume\_element\_group** to specify element group numbers for which the volume strain should be applied. Use **strain\_volume\_geometry** to specify a geometry for which the volume strain should be applied.

You can either specify relative volume strains (relative to the initial volume) or absolute volume changes. Use strain\_volume\_relative\_time to specify a time versus relative volume strain diagram. These relative volume strains should be specified as percentage (thus, 100 would be a volume strain equal to the initial volume, so 100 percent extra volume). Use strain\_volume\_absolute\_time to specify a time versus absolute volume increase diagram. These absolute volume increases should be specified as real volume (thus  $m^3$  if you use would meter m as length unit in your input file). Exactly one of strain\_volume\_relative\_time or strain\_volume\_absolute\_time should be given in the input file, not both. At times outside strain\_volume\_relative\_time, or strain\_volume\_absolute\_volume increase, are assumed to be zero.

If none of strain\_volume\_element, strain\_volume\_element\_group or strain\_volume\_geometry is given then the strain\_volume\_relative\_time or strain\_volume\_absolute\_time will be applied to all isoparametric elements.

The volumetric strain option presently is available only for small deformation analysis. The volumetric strain can be be applied to isoparametric elements only. The volumetric strain is not available for membrane elements.

See also post strain volume absolute and post strain volume relative.

6.1065 strain\_volume\_element\_group index element\_group\_0 element\_group\_1

See strain volume element.

6.1066 strain\_volume\_geometry index geometry\_item\_name geometry item index

See strain volume element.

**6.1067 strain\_volume\_relative\_time** index time\_0 relative\_volume\_strain\_0 time\_1 relative\_volume\_strain\_1 ...

See strain volume element.

# 6.1068 support edge normal index stiffness normal stiffness tangential

Distributed support of an edge (winkler foundation). The *stiffness\_normal* specifies the normal stiffness of the support per unit length in 2D, and per unit area in 3D. The *stiffness\_tangential* specifies the tangential stiffness. This option is meant for 2D and 3D calculations.

Also the record **support** edge normal geometry should be specified.

Attention: this option is only available for linear and quadratic isoparametric elements.

See also node support edge normal force.

# 6.1069 support\_edge\_normal\_damping index damping\_normal damping\_tangential

Distributed damping at an edge. The *damping\_normal* specifies the normal damping of the support per unit length in 2D, and per unit area in 3D. The *damping\_tangential* specifies the tangential damping. This option is meant for 2D and 3D calculations.

If you want to use **support\_edge\_normal\_damping** to absorb stress wave at the boundaries of the mesh (think of vibration or earthquake analysis), there are typical values to be used for the normal and tangential damping.

For absorbing boundaries the damping\_normal typically should be set to  $C_n \rho V_n$ . The parameter  $C_n$  typically is chosen as 1. The  $\rho$  is the material density. The pressure wave velocity is  $V_n = \sqrt{\frac{E_{ocd}}{\rho}}$  with oedometric stiffness  $E_{oed} = \frac{(1-\nu)E}{(1+\nu)(1-2\nu)}$  where E is the Young's modulus, and  $\nu$  is Poisson's ratio. For absorbing boundaries the damping\_tangential typically should be set to  $C_t \rho V_t$ . The parameter  $C_n$  typically is chosen as 0.25. The shear wave velocity is  $V_t = \sqrt{\frac{G}{\rho}}$  with shear modulus  $G = \frac{E}{2(1+\nu)}$ .

Also the records  $support\_edge\_normal$  and  $support\_edge\_normal\_geometry$  should be specified.

<u>Attention</u>: this option is only available for linear and quadratic isoparametric elements.

See also **control\_support\_edge\_normal\_stiffness\_freeze** and **node\_support\_edge\_normal\_force**. See also **support\_edge\_normal\_damping** for automatic specification of damping properties.

# 6.1070 support edge normal damping automatic index switch

If you set *switch* to **-yes** this record generates damping just like the **support\_edge\_normal\_damping** record. Now however you do not need to specify the damping properties yourself; they are calculated by Tochnog using the Young value E and the Poisson ratio  $\nu$  from the isoparametric element attached to the support.

# 6.1071 support\_edge\_normal\_damping\_automatic\_apparent index switch

If you set *switch* to **-yes** this record generates damping just like the **support\_edge\_normal\_damping** record. Now however you do not need to specify the damping properties yourself; they are calculated by Tochnog using the apparent young and poisson value from the isoparametric element attached to the support.

#### 6.1072 support edge normal density index density\_normal density\_tangential

Distributed density at an edge. The *density\_normal* specifies the normal density of the support per unit length in 2D, and per unit area in 3D. The *density\_tangential* specifies the tangential density. This option is meant for 2D and 3D calculations.

Attention: this option is only available for linear and quadratic isoparametric elements.

# 6.1073 support\_edge\_normal\_element\_node index element\_0 element 1 ...

Selects the element and local nodes for which the **support\_edge\_normal** record with the same *index* should be applied.

#### 6.1074 support edge normal element group index element group

Restricts the element group to which the **support\_edge\_normal** record with the same *index* should be applied.

# 6.1075 support\_edge\_normal\_element\_side index element\_0 element\_1 ...side

Selects the elements and local side number for which the **support\_edge\_normal** record with the same *index* should be applied.

# **6.1076** support edge normal factor index $a_0 a_1 \ldots a_n$

The same as **force\_edge\_normal\_factor**, now however for the support stiffnesses (and not the force).

# 6.1077 support edge normal force initial index a\_0 a\_1

This record allows you to specify an initial normal force in the support, linear varying in depth direction. The initial normal force actually is  $a_0 + a_1 * y$  in 2D, or  $a_0 + a_1 * z$  in 3D.

# 6.1078 support\_edge\_normal\_geometry index geometry\_entity\_name geometry\_entity\_index

Selects the area for which the **support\_edge\_normal** record with the same *index* should be applied. For example, **-geometry\_line 1** can be used in 2D, indicating that the nodes on line 1 get the distributed support.

6.1079 support edge normal node index node\_0 node\_1 node\_2...

Selects the nodes for which the **support\_edge\_normal** record with the same *index* should be applied. The *node*  $\theta$  etc. specify global node numbers.

# 6.1080 support\_edge\_normal\_plasti\_compression index normal\_force\_minimum tangential\_force\_factor

With *normal\_force\_minimum* you can limit the amount of compression force that a support can take. Any compression force lower than this *normal\_force\_minimum* will actually be set to *normal\_force\_minimum*. Typically you want to specify a negative value for *index normal\_force\_minimum*.

With *tangential\_force\_factor* you can model frictional slip in the tangential direction. The tangential force is limited to *tangential\_force\_factor* times the normal force. Larger tangential forces are not allowed.

This **support\_edge\_normal\_plasti\_compression** will only be used if the normal force does not exceed the maximum tension force as specified in **support\_edge\_normal\_plasti\_tension** or **support\_edge\_normal\_plasti\_tension** double.

All forces are per unit length in 2D, and per unit area in 3D.

#### 6.1081 support edge normal plasti friction index cohesion friction\_coefficient

With this record you can limit the amount of friction force that a support can take. The maximum allowed friction force is the *cohesion* plus the *friction\_coefficient* multiplied with the absolute value of the normal force.

All forces are per unit length in 2D, and per unit area in 3D.

# 6.1082 support edge normal plasti tension index switch

If *switch* is set to **-yes** and the normal force in the support is tension, then all forces are set to 0. This models gap building between the support and the element edge.

#### 6.1083 support edge normal plasti tension double index normal\_force\_maximum

With *normal\_force\_maximum* you can limit the amount of tension force that a support can take. As opposed to **support\_edge\_normal\_plasti\_tension**, you can specify a non-zero value with this option. If a normal force higher than this *normal\_force\_maximum* occurs it will be set to *normal\_force\_maximum*, and tangential shear forces will be set to zero. Typically you want to specify zero or a positive value for *index normal\_force\_maximum*, although a negative value is also allowed.

Not both of support\_edge\_normal\_plasti\_tension and support\_edge\_normal\_plasti\_tension\_doub can be specified.

All forces are per unit length in 2D, and per unit area in 3D.

#### 

In case of plasticity in a support you can require that Tochnog includes a part of the original elastic stiffness in the element stiffness matrix to get more stable iterations. The part of the original stiffness included needs to be specified with *factor*, between 0 and 1. The stiffness is only included in the matrix, and not in the right-hand-side; so it will only influence convergence behaviour, but not the final results if a sufficient amount of steps is taken. Default, if **support\_edge\_normal\_plasti\_residual\_stiffness** is not specified, *factor* is set to 0.

#### 6.1085 support edge normal time index time load time load ...

This record specifies a diagram with a multiplication factor for the support edge force. Linear interpolation is used to extend the *time load* values to the intervals between these pairs. Outside the specified time range a factor 0 is used.

If this record is not specified, a factor of 1 is applied at all times.

6.1086 target item index data item name data item index number

See also: target value.

#### 6.1087 target value index value tolerance

This allows for testing the results of the calculation. Typically, *data\_item\_name* is **node\_dof** but also other data items can be tested. The record with index *data\_item\_index* will be tested. If *data\_item\_name* is **-node\_dof** then *number* can be **-velx**, **-temp**, etc.

(see **dof\_label**); else, for example, *number* is 3 states that the fourth value needs to be checked. The result should not differ more from *value* than *tolerance*.

For a calculation with no problems, the tochnog.log file contains a line stating that the calculation did start followed by a line stating that the calculation did end. If this is not precisely the case, some problem did occur or the targeted results differ too much. In the example below it is checked that the pressure in node 6 does not differ more than  $1.10^{-5}$ .

target\_item 0 -node\_dof 6 -pres target\_value 0 1.2 1.e-5

The checked value, 1.2 in this case, has been found from a previous computation that is regarded as reliable. The present computed value is compared with the earlier one. If they agree within the specified tolerance, 1.e-5 in this case, then Tochnog is silent. If they do not, then Tochnog writes an error message into the file "tochnog.log".

6.1088 time calculation elapsed time in seconds

Elapsed computer time up to moment of printing (wall clock time).

# 6.1089 time current current\_time

Current time in calculation.

# 6.1090 timestep predict velocity switch

Normally tochnog will use as prediction for velocities in a timestep the previous calculated velocities from the previous timestep.

However, if there is no inertia, and **convection\_apply** is **-no** tochnog will use as prediction for velocities in a timestep a zero velocity.

You can require that tochnog does the normal prediction from the previous timestep however by setting *switch* to **-yes**; you typically want to do that in eulerian calculations.

#### 6.1091 timestep iterations automatic apply switch

If *switch* is set to **-no** any **control\_timestep\_iterations\_automatic** records will be neglected.

#### 6.1092 tochnog version index day month year

This record contains the build day, the build month and the build year.

#### 6.1093 truss rope apply switch

If *switch* is set to **-no**, any truss rope data in the input file will be ignored. This is done for all timesteps.

This option is convenient for testing your input file just linear, without the need to outcomment each and every part with truss rope data. See also **control truss rope apply**.

#### 6.1094 volume factor $a_0 a_1 \ldots a_n$

This data item defines a polynomial in space in 1D or 2D. The polynomial specifies the cross-sectional area (in 1D) or the thickness (2D) as function of the global x coordinate (1D) or the global x,y coordinates (2D). For example, in a 1D solid calculation it can be used to specify varying cross-sectional areas of bars, or in a 1D flow calculation it can be used to specify the cross- sectional area of a channel.

In 1D the polynomial is  $a_0 + a_1 x$  (specify 2 values). In 2D the polynomial is  $a_0 + a_1 x + a_2 y$  (specify 3 values).

If this record is not specified, the cross-sectional area is 1 (1D) or the thickness is 1 (2D).

See also volume element factor.

# 6.1095 volume factor $\mathbf{x} x_0 fac_{01} x_1 fac_{12} \dots x_n$

This specifies an in x-direction changing volume factor for elements. Left from  $x_0$  the factor is 1. From  $x_0$  to  $x_1$  the factor is  $fac_{01}$ . Etcetera. And right from  $x_n$  the factor is 1 again.

#### **6.1096 zip** *switch*

If *switch* is set to **-yes** all **\*flavia\***, **\*msh**, **vtk**, **\*.plt** and **\*dbs** files are zipped with the **gzip** program. The **gzip** program should be installed on your computer.

This comes convenient in large calculation with lots of output, where you want to use results later and save disk space during the calculation.

If also **control\_zip** is specified for a certain control index that overrules this **zip** for that control index.

# 6.1097 end\_data (last record of data part)

# 7 Runtime file

. . .

You can use a runtime file to give to Tochnog data on the fly (while it is running). The runtime file will be read at the start of each time step. The runtime file needs to have the same name as the input file, with the extension **run** instead of **dat** however. Suppose the name of the normal input file is **beam.dat**, then the name of the runtime file is **beam.run**. The runtime file always needs to be ended with two **end data** statements.

As a typical example you can use this runtime file when you are doing a long calculation and you decide while the calculation is running that you want extra output. Suppose the normal input file **tochnog.dat** contains:

control\_timestep 100 ...

Then you can decide to get some extra GID plotting files, while Tochnog is already running, by using the runtime file **tochnog.run** with:

control\_print\_gid 100 -yes end data end data

When you want to de-activate the printing of GID files again then set the runtime file to:

control\_print\_gid 100 -no end data end data

As a special option, you can use **exit\_tochnog -yes** in the runtime file; then Tochnog will exit the calculation after printing the complete database and GID files.

After the runtime file is read, it will be automatically deleted by Tochnog.

# 8 Interaction analyzes and advanced analyzes

## 8.1 Fluid-structure interaction

If a solid construction interacts with a fluid, both the solid and fluid can be modeled with the materi equation. Interaction forces between solid and fluid will automatically be generated. If required, a temperature field may be imposed. An example of a input file is given below

```
. . .
materi velocity
materi stress
condif temperature
end initia
. . .
element group -ra -from 0 -to 100 -ra 1
element group -ra -from 101 -to 200 -ra 2
. . .
type 1 -materi -condif
group materi elasti young 1...
group materi memory-updated
group condif conductivity 3 ...
. . .
type 2 -materi
group materi elasti compressibility 2...
group materi viscosity 2 ...
group materi memory-updated linear
group condif conductivity 2 ...
. . .
```

Elements 0-100 are solids (with temperature) and elements 101-200 are fluids (with temperature).

#### 8.2 Consolidation analysis: ground water flow in deforming solid

The ground water flow equation can be combined with the materi equations. The solid will deform due to the ground water flow pressure gradient and ground water flow pressure will change due to solid volume changes. An example of a input file is given below

materi\_velocity materi\_stress

```
groundflow_pressure
end_initia
....
groundflow_consolidation_apply -yes ....
groundflow_density ....
groundflow_phreatic_level ...
...
group_type 0 -materi -groundflow
group_materi_elasti_young 0 ....
group_materi_memory -updated
group_groundflow_capacity 0 ....
...
```

The stresses as initialized by **materi\_stress** are effective stresses. Internally the program calculates with total stresses (effective stress + total pressure) in the material equilibrium equation. You can obtain the total stresses for postprocessing by means of the **post\_calcul** option.

To account for the gravitational stresses, use a density  $\rho_{sat}$  in the **group\_materi\_density** record. Here  $\rho_{sat}$  is the saturated density of the groundwater-soil mixture (mass of soil + water per unit volume of the soil-water mixture). Also specify the gravitation in the **force\_gravity** record and, if required, also the **force\_gravity\_time** record to apply the gravitation slowly.

#### 8.3 Heat transport in ground water flow

Heat transport in a ground water flow can be analyzed by combining the convection and diffusion of heat equation with the ground water flow equation. Now the velocity in the convection and diffusion of heat equation is taken from the groundflow velocity field ( $\beta_i = v_i^{\text{g}}$ ) if **groundflow velocity** is initialized. An example of a input file is given below

groundflow\_pressure groundflow\_velocity condif\_temperature end\_initia ... type 0 -groundflow -condif group\_groundflow\_compressibility 0 ... group\_condif\_conductivity 0 ...

If both materi\_velocity and groundflow\_velocity are initialized,  $\beta_i = v_i + v_i^{g}$ .

#### 8.4 Heat transport in materials

Heat transport in a material can be analyzed by combining the convection and diffusion of heat equation with the materi equations. In this way thermal stresses or heat induced convection can be analyzed. Now the velocity in the convection and diffusion of heat equation is taken from the velocity field ( $\beta_i = v_i$ ). An example of a input file is given below

```
materi_velocity
materi_stress
condif_temperature
end_initia
....
type 0 -materi -condif
group_materi_elasti_young 0 ....
group_materi_expansion 0 ....
group_materi_memory 0 -updated
group_condif_conductivity 0 ...
```

# 8.5 Restart a calculation

You can use a dbs file to restart a calculation. In fact, a dbs file is an input file itself. It contains the record **icontrol** which contains the last control index actually performed with the previous calculation. You can add more **control** \* records and start the file again; it will then continue with these new **control** \* records.

You cannot use dbs files with contain control repeat for restarting a calculation.

# 9 Final topics (input trouble, save memory /CPU time, ...)

# 9.1 Environment symbols

Records with a length of 1, and no index, you can also set via an environment symbol. You need to use capital characters in doing so. Typical examples are

- PROCESSORS 4
- PRINT GID CALCULATION -no
- PRINT GMSH CALCULATION -yes
- PRINT NODE GEOMETRY PRESENT -yes

In windows set environment symbols in your advanced system settings. In a linux bash shell set environment symbols in your .bashrc file (eg **export PROCESSORS=4**).

# 9.2 Checking your geometry \* records

Set print\_node\_geometry\_present -yes and set print\_element\_geometry\_present -yes. Then look with gmsh if the geometries are like you want.

# 9.3 Continuing an analysis

- Copy the database from the previous calculation to a new file, e.g. **new.dat**.
- Run a new calculation with **new.dat**.

This can also be done with a database that is written as indermediate database in a previous calculation, for example directly after gravity. See also **icontrol**.

#### 9.4 Use -node as geometry entity.

As a special option you can use a node as a geometrical entity. For example the following imposes a boundary condition on the temperature of node 6:

#### bounda dof 10 -node 6 -temp

Notice that -node 6 is used in the format of a geometry entity.

#### 9.5 Use -geometry list as geometry entity.

As a special option you can use a list as a geometrical entity. For example the following imposes a boundary condition on the nodes of geometry list 6:

geometry\_list 6 54 43 26 38 62 bounda dof 10 -geometry list 6 -temp

# 9.6 List input files with options

You can search for input files in your distribution which contain multiple words. For example change to the test/other directory. Suppose you want to see in which files you can see transient consolidation in a deforming soil.

In linux use the following command to list input file: grep -il materi\_velocity \*.dat | xargs grep -il groundflow\_capacity | xargs grep -il groundflow\_consolidation.

In MS Windows use: windows explorer - Search - Advanced options - File contents and search for materi velocity AND groundflow capacity AND groundflow consolidation.

#### 9.7 Geometrically linear material

Either do this:

- Initialise -materi velocity and -materi displacement
- Use -total linear for the material.

or do this:

- Initialise -materi velocity and -materi velocity integrated
- Use -fixed in space for mesh
- Use -updated linear for the material.

#### 9.8 Dynamic calculations

Dynamic calculations are triggered by setting **inertia\_apply** -yes. Take care that you have specified all required data, like material density, etc.

In case you want to reduce numerical damping, you can use the following piece of input file, notice the **materi dynamic**.

materi \_displacement materi \_velocity end \_initia ... inertia \_apply -yes materi \_dynamic -yes ... control\_timestep .....

For heavy non-linear calculations this may converge worse than the default euler backward time integration.

To get damping similar to rayleigh damping in structural dynamics use:

group\_materi\_damping ... (similar to rayleigh damping mass term, use rayleigh alpha \* material density ) group\_materi\_viscosity ... (similar to rayleigh damping stiffness term, use rayleigh beta \* material young ) ...

See also **materi\_dynamic** to influence numerical damping in dynamic calculations with solid materials.

#### 9.9 Input file syntax

. . .

• If you don't understand the syntax of an option, please look in the tochnog/test/other directory for example files. Under linux search for the command, eg grep control\_print\_filter \*.dat to get example files with control\_print\_filter.

#### 9.10 Check large calculations

- Set both solver -none and linear calculation apply -yes; run and check in gid the boundary conditions, forces, change of element groups, etc. In a complex model you can check geometries that you use by imposing an artificial boundary on them, eg **bounda\_dof ...** -temp with value 1, and look in gid if you see that boundary condition showing up at the correct nodes.
- Only set linear calculation apply -yes ; run and check linear solution fields.
- Do not set anything special ; run and check solution fields.

# 9.11 Diverging calculations

- Try the linear elements bar2, -quad4, -tria3, -hex8 and -tet4 in stead of quadratic elements.
- Try solver matrix save -no (always setup new system matrix)
- Try group \_\_materi \_ plasti \_\_mohr \_ coul \_ direct i.s.o. group \_\_materi \_ plasti \_\_mohr \_ coul
- Try small fixed timesteps (do not use automatic time stepping).
- Try more iterations with **control timestep iterations**.
- Try a lower interface stiffness.
- Try higher water capacity in calculation with consolidation (so water less stiff, anyway not too stiff relative to soil)).
- Set group interface materi residual stiffness to 1.

# 9.12 Saving CPU time

- Check that the computer doesn't swap to disk (use top in linux, use task manager in windows). In case of swapping lower the memory usage (see the section 'Saving computer memory').
- For large calculations with many **post\_\*** commands: use **post\_apply -no** and use **control\_post\_apply index -yes** only at the moment that you actually need post results

#### 9.13 Saving computer memory

Try the following steps, in order of priority:

• solver matrix symmetric -yes.

- solver -matrix iterative bicg.
- Use bounda alternate.
- If possible, don't use extreme large indices (since memory is allocated for all indices).

#### 9.14 Inaccurate results

- Set the interface stiffness to about 10 times the neighbouring element young divided by the neighbour length.
- If a structure is submerged in water, eg a one-side submerged dam, you need to impose the correct pressure condition; but you also need to impose the water loading by a **force\_edge\_water**.

#### 9.15 Element sides

This sections defines node numbers for element sides 0, 1, ... respectively.

For a **bar2** element the sides have the nodes numbers 0 and 1.

For a tria3 element the sides have the nodes numbers 0,1 and 1,2 and 2,0.

For a tria6 element the sides have the nodes numbers 0,1,2 and 0,3,5 and 2,4,5.

For quad elements the sides are in the order below, upper, left, right; see the pictures in **elements**.

For hex elements the sides are in the order below, upper, front, back, right, left; see the pictures in **elements**.

For a **tet4** element the sides have the nodes numbers 0,1,2 and 0,1,3 and 1,2,3 and 0,2,3.

For a tet10 element the sides have the nodes numbers 0,1,2,3,4,5 and 0,1,2,6,7,9 and 2,4,5,7,8,9 and 0,3,5,6,8,9.

For a **prism6** element the sides have the nodes numbers 1,2,3 and 4,5,6 and 1,2,4,5 and 0,2,3,5 and 0,1,3,4.

For a **prism15** element the sides have the nodes numbers 0,9,1,11,10,2 and 3,12,4,14,13,5 and 0,9,1,6,7,3,12,4 and 1,10,2,7,8,4,13,5 and 0,11,2,6,8,3,14,5.

For a **prism18** element the sides have the nodes numbers 0,1,2,3,4,5 and 12,13,14,15,16,17 and 0,1,2,6,7,8,12,13,14 and 2,4,5,8,10,11,14,16,17 and 0,3,5,6,9,11,12,15,17.

#### 9.16 Badly shaped elements

Each element should have at maximum one common side with a neigbouring element. For example two neigbouring quad4 elements have only one common side in a proper element mesh; if the neighbouring quad4 elements have two sides in common, the elements are badly shaped.

Some tochnog options will not work correctly if the mesh contains badly shaped elements.

#### 9.17 Youtube

Tutorial movies can be found on <a href="https://www.youtube.com/channel/UC7qvITX-SLwA4RuqMPYBmDQ">https://www.youtube.com/channel/UC7qvITX-SLwA4RuqMPYBmDQ</a>

### 9.18 External programs.

- http://www.gidhome.com Commercial pre- and postprocessor; easy. It can write for Tochnog input directly. It can read results both from control \_print\_gid and from control \_print\_vtk. See the gid directory in your distribution.
- http://mecway.com Commercial finite element program; easy. It can write for Tochnog input abaqus. It can read results from control print gmsh.
- http://www.gnuplot.info/ Free x-y plotter; easy. It can read results from control print history and control print data versus data.
- http://gmsh.info Free pre- and postprocessor; complex. It can write for Tochnog input abaqus. It can read results from control print gmsh.
- http://www.paraview.org Free postprocessor; complex. It can read results from control print vtk.
- http://www.freecadweb.org/ Free CAD program; complex. It can write for Tochnog input abaqus and read results from control print frd.
- http://lace.fs.uni-mb.si/wordpress/borovinsek/?page\_id=41 Free prepomax pre- and postprocessor for MS Windows; easy. It can write for Tochnog input\_abaqus and read results from control print frd. Isoparametric elements only.
- http://www.bconverged.com/download.php Free postprocessor CGX for MS Windows; easy. It can read results from control print frd.
- https://www.mikepoweredbydhi.com/products/feflow Dedicated ground water analysis program. Tochnog can read the mesh and hydraulic pressure heads with input feflow \*.
- https://ngsolve.org/ Free preprocessor in 3D. Tochnog can read the nodes and elements.

### 9.19 Forces are setup in the element loop in timesteps

The records **force\_edge**, **force\_edge\_normal**, **force\_edge\_projected**, **force\_volume**, **condif\_heat\_edge\_normal**, **condif\_convection\_edge\_normal** and **condif\_radiation\_edge\_normal** are evaluated inside the element loop. Hence, the resulting nodal forces only get their values after a timestep is performed (since the element loop is performed in time steps).

# 9.20 Running in a Microsoft windows bash shell

When you want to run the linux executable in a Microsoft windows bash shell you need to set the environment symbol KMP\_AFFINITY to disabled. So do **export KMP\_AFFINITY=disabled** in the windows bash shell.

# 10 User supplied subroutines

Several skeleton user supplied subroutines are available in the file user.cpp. As a special option you can use an ABAQUS umat.f (ABAQUS is a trademark of Dassault Systemes; see HTTP://www.abaqus.com for the ABAQUS homepage). See also group\_materi\_umat in this manual. We do not support any compilation, linking or run-time problems with user supplied routines.

We only have available user.cpp and umat.f on linux 64 bit.

We use ourselves the linux version specified in **linux\_version.txt** file in the user supplied distribution. If you want to use the user supplied routines it is convenient to also use the same operating system in order to prevent compiling and linking problems; for this distribution the needed libraries are supplied.

If you insist on using another distribution than we do, you should install yourself the needed libraries (with commands like: sudo apt-get install gcc-7 g++-7). For the distribution of **linux version.txt** the needed libraries are supplied.

Do the following:

- Download the tochnog linux 64 bit user supplied.tar.gz distribution.
- $\bullet\,$  In the user supplied distribution read  ${\bf README}\,\,$   ${\bf UMAT}\,\,$   ${\bf USER.txt}\,$
- In the user supplied distribution look in the **makefile** how to compile.

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